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THE EFFECT OF LOADING ON THE  
 LASER ABLATION OF GRAPHITE/  
 EPOXY COMPOSITE MATERIAL

THESIS

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 Captain, USAF

AFIT/GAE/ENY/89D-12

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**AIR FORCE INSTITUTE OF TECHNOLOGY**

Wright-Patterson Air Force Base, Ohio

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OF GRAPHITE/EPOXY COMPOSITE MATERIAL

THESIS

Presented to the Faculty of the School of Engineering  
of the Air Force Institute of Technology  
Air University  
in Partial Fulfillment of the  
Requirements for the Degree of  
Master of Science in Aeronautical Engineering

Joseph L. Hamrick, II, BS  
Captain, USAF

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Preface

This research was initiated to fill a gap in existing knowledge about the mechanical processes which cause the removal of solid matter from fiber-reinforced composite materials during ablation. The effect these processes have upon the energy required to remove the mass is of interest to anyone trying to predict the effect of thermal radiation on load-bearing composite structures.

Dr. George Sendecky of the Wright Research and Development Center's Flight Dynamics Laboratory suggested the problem, and has provided valuable assistance to me in this research. Mr. F. E. Barnett of WRDC Materials Laboratory provided important financial and facility support to the work. 1st Lt Dave Felker and his coworkers in WRDC/FIBCA were essential in the fabrication of the experimental specimens.

I'm especially indebted to Capt Jerry R. Couick and the Rockwell Power Services contractors at the Air Force Weapons Laboratory for their test facility support.

Thanks also to my thesis committee; Dr. Peter J. Torvik, Lt Col Ronald L. Bagley and Dr. Shanker Mall. And also to Jay Anderson and Dan Rioux for much needed technical support. The technicians in WRDC/FIBTA were also very helpful.

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Most importantly, all praise and honor to my Saviour Jesus Christ for (among many things) the fact that I was selected for this program and given the strength to complete it.

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### Notation

A	Coefficient of elastic line solution in section AB
B	" " " " " " "
C'	" " " " " " BC
D'	" " " " " " "
$\vec{q}$	→ heat flux $\left(\frac{W}{m^2}\right)$
$\hat{n}$	→ outward normal unit vector
dA	→ element of surface area ( $m^2$ )
g	→ heat generation ( $W/m^3$ )
dV	→ element of volume ( $m^3$ )
$\rho$	→ mass density ( $kg/m^3$ )
$\rho_t$	→ radius of curvature due to temperature gradient (cm)
$C_p$	→ specific heat at constant pressure ( $J/kg \cdot K$ )
T	→ temperature (degrees Kelvin)
t	→ time (seconds)
$K_{ij}, \bar{K}$	→ conductivity tensor
I, J, K	→ row, column and layer of volume element and/or temperature node
L, M, N	→ number of rows, columns and layers used to divide region V
$Q^*$	→ effective enthalpy of ablation $\left(\frac{kJ}{gm}\right)$

- $I$  → laser irradiance ( $\text{kW/cm}^2$ )
- $\eta_d$  → efficiency of discrete mass removal
- $m$  → mass (gm)
- $\alpha$  → layup angle
- $\sigma_x$  → axial stress (psi, MPa)
- $M$  → bending moment (in·lb, N·m)
- $d$  → diameter of graphite fiber (cm)
- $E_i$  → Young's moduli (psi, MPa)
- $\epsilon^o$  → strain of laminate midsurface
- $Q_{ij}, \bar{Q}_{ij}$  → anisotropic stiffnesses (psi, MPa)
- $z$  → depth below laminate midsurface (cm)
- $N_x$  → axial load per unit width (lb/in, N/m)
- $\epsilon_{ult}$  → strain at failure for a fiber
- $\delta$  → energy (kJ)
- $K_I$  → mode I stress intensity factor ( $\text{lb}\cdot\text{in}^{3/2}$ ,  $\text{kg}/\text{mm}^{3/2}$ )
- $V_i$  → volume fraction of component i
- $\chi_i$  → mass fraction of component i

Abstract

The purpose of this investigation was to quantify the effect initial structural loading of a graphite/epoxy member has upon the mass removal rate during laser ablation. The effective heat of ablation ( $Q_{\text{eff}}^*$  = energy absorbed/mass removed) was used as a measure of this efficiency. A simple physical model of the important factors affecting the graphite/epoxy was developed, and predictions were made of the effect of loading on  $Q_{\text{eff}}^*$ .

A three-dimensional finite difference heat transfer code was written to predict the temperature distribution in the composite. The orthotropic nature of the thermal conductivity tensor in the plies was modeled, to accurately model the heat flow from the irradiated region.

The effect of thermal and mechanical loads upon the stress distribution in a single fiber was calculated, and a linear decrease in  $Q_{\text{eff}}^*$  with increasing stress was predicted. The coefficient of increasing ablation efficiency, ( $\eta_d$ ), was postulated to increase linearly with axial stress  $\sigma_x$ . This was based upon the hypothesis that fracture of individual fibers will be a process linearly dependent upon applied stress, and will remove a fraction of the composite's mass without requiring the absorption of laser energy.

Uniaxial tensile coupons were fabricated from Hercules AS4/3502 graphite/epoxy prepreg, in balanced, symmetric laminates laid up in a pattern of  $(0/+60/-60)_{ns}$ . These specimens were placed under tensile loads between 0% and 50% of the laminate's fracture strength. While under fixed grip loading conditions, they were irradiated with a 10.6  $\mu\text{m}$  device at irradiances between 5 and 26  $\text{kW}/\text{cm}^2$ .

Linear regression of data taken at  $\sim 15\text{kW}/\text{cm}^2$  calculated  $\eta_d = 0.013 \text{ kJ/gm} + 1.74(\text{kJ/gm})/\text{GPa} \cdot \sigma_x$ . At 26  $\text{kW}/\text{cm}^2$ ,  $\eta_d = 0.087 \text{ kJ/gm} + 3.36(\text{kJ/gm})/\text{GPa} \cdot \sigma_x$ . Therefore, it may be concluded that axial fiber stress does linearly decrease the 10.6  $\mu\text{m}$  laser energy needed to ablate AS4/3502 graphite/epoxy. The total effect seen in this study was less than 20 percent of the unloaded value.

It is recommended that further analytic modeling of the interaction between applied load and fiber fracture be undertaken to allow prediction of  $\eta_d$  values for different materials without the collection of experimental data.

THE EFFECT OF LOADING ON THE LASER ABLATION  
OF GRAPHITE/EPOXY COMPOSITE MATERIAL

I. Introduction

The laser-induced ablation of fiber-reinforced composite materials has been studied extensively. However, the majority of experiments performed have been on samples of material without any externally applied loads (5,19). Since most structures of interest do bear loads, the effect of these loads on the phenomenon of ablation is important in understanding the interaction between laser radiation and structures.

Although load enhancement of mass removal rates has been observed experimentally (17), little, if any, analytical modeling of laser ablation phenomena has included appropriate mechanical processes to explain this observation (19).

Also, global structural instability has typically been modeled as simple degradation of material strength properties at elevated temperatures (19). This approach is clearly not applicable to simple mass removal, which takes place under no load in materials needing no structural strength (2).

Examination of solid particles ejected from ablating graphite/epoxy composite during the Very High Irradiance (HI-I) and Screening Test Series (STS-1) tests (2,18) indicated that 'rafts' of solid laminae, on the order of 1000-5000 microns were ejected from ablating samples. This was under conditions of no pre-loading. It is conjectured that if the state of the ablation surface is such that discrete volumes of material can be broken off and ejected without being vaporized by laser radiation, any increased stress due to applied loads will increase the rate of material fracture. This would have the effect of allowing more mass to be removed discretely, and lower the apparent heat of ablation ( $Q_{eff}^*$ ).

## II. Theoretical Analysis

### Temperature Distribution

The prediction of stress states in the sample requires a knowledge of the temperature distribution created in the sample due to laser heating as well as the loading distribution. Various one- (1D) and two-dimensional (2D) finite difference schemes have been used in the past (5,13-16). However, the detailed knowledge of temperature gradients needed to predict the fracture mechanics of the fibers suggests a three-dimensional (3D) model is needed. The anisotropic nature of the composite material also contributes to this necessity.

The conservation of energy on a infinitesimal control volume leads to the differential equation of heat conduction in a solid (10:5). Left in terms of a finite control volume it is

$$-\int_{\Delta} \vec{q} \cdot \hat{n} \, dA + \int_V g \, dV = \int_V \rho C_p \frac{\partial T}{\partial t} \, dV \quad (1)$$

where

$$-\int_{\Delta} \vec{q} \cdot \hat{n} \, dA = \text{heat flux through boundaries of } V \quad (2)$$

$$\int_V g \, dV = \text{heat generation in } V \quad (3)$$

$$\int \rho C_p \frac{\partial T}{\partial t} dV = \text{rate of energy storage within volume } V \quad (4)$$

The heat flux term,  $\vec{q}$ , for an isotropic solid, is proportional to the temperature gradient,  $\vec{q} = -k\nabla T$ . For the anisotropic case,  $k$  is not a constant, but a second order tensor (10:612)

$$q_i = -K_{ij} \frac{\partial T}{\partial x_j} \quad (5)$$

A discretization of the volume of interest into small elements (rectangular prisms were used for simplicity in this case, although other schemes are possible) yields an explicit value for the temperature rise,  $dT$ , at each element during a small interval of time,  $dt$ . This rise is added to the existing temperature of the element, and then the process is repeated for the next time interval. Phase changes, ablation and temperature dependent properties may be incorporated for each element.

Neglecting heat generation and substituting Equation (5) into Equation (2), and approximating derivatives with first order terms, yields the following equations for the energy flux into an element of volume  $V_{IJK}$ .

$$q_{x_{I,J,K}} = K_{11} \left[ \frac{T_{I,J,K} - T_{I+1,J,K}}{\Delta x} \right] \Delta y \Delta z + K_{12} \left[ T_{I,J,K} - T_{I,J+1,K} \right] \Delta z \quad (6a)$$

$$q_{y_{I,J,K}} = K_{12} (T_{I,J,K} - T_{I+1,J,K}) \Delta z + K_{22} \frac{(T_{IJK} - T_{I,J+1,K})}{\Delta y} \Delta x \Delta z \quad (6b)$$

$$q_{z_{I,J,K}} = K_{33} \frac{(T_{I,J,K} - T_{I,J,K+1})}{\Delta z} \Delta x \Delta y \quad (6c)$$

where the zero elements of  $K$  have been left out.

To compute the temperatures in a region divided into  $L$  rows,  $M$  columns and  $N$  layers, an  $(L+2) \times (M+2) \times (N+2)$  array of temperature nodes was created. Nodes on the extremities were set identically equal to their closest neighbors on the inside. This ensured zero flux (due to zero temperature gradient) on all boundaries, creating insulated boundary conditions.

The only energy input into the volume came from absorbed laser flux, which was added to each face exposed to the irradiance.

Like 1D and 2D explicit schemes the size of time interval,  $dt$ , is constrained to be less than a critical value for stability. As suggested by Torvik (13:13), a conservatively stable time increment was chosen by using the smallest  $dt$  necessary. That is, the smallest distance increment of the three possible was used to calculate  $dt$ .

Existing data on high temperature thermal properties of AS4/3502 are extremely scarce. Data for similar materials were collected and applied (3,5,8,19). Thermal property

data on AS4/3501-6 composite is much more prevalent, and has been used interchangeably with that available on AS4/3502 in this work. The computer code applies these properties as a function of temperature through the point where data are available. For temperatures above those where data exist, the last recorded value of the property is maintained.

In a one dimensional fiber reinforced epoxy laminae, the principal directions of thermal conductivity are obviously along the fiber, normal to the fiber in the plane of the laminae, and normal to the plane of the laminae (Figure 1). The conductivity,  $K_{11}$ , in the direction along the fiber,  $\hat{e}_1$ , is dominated by the conductivity of the fiber itself. Data for this conductivity are approximated by the High Temperature Materials Information Analysis Center (HTMIAC) (8). The data is calculated by a parallel conductor analysis using  $13 \text{ W/m}^{\circ}\text{K}$  for fiber conductivity, and an isotropic conductivity value for the matrix resin.

It is assumed that in the cured laminate, fiber to matrix ratios will be similar in the laminae plane,  $\hat{e}_2$ , and normal to them,  $\hat{e}_3$ . Therefore, laminate conductivities  $K_{22}$  and  $K_{33}$  are assumed to be equal.

In the layers laid at  $\pm \alpha$  (60 degrees in this case), the principal directions no longer coincide with the global axes, and the conductivity tensor is no longer diagonal.

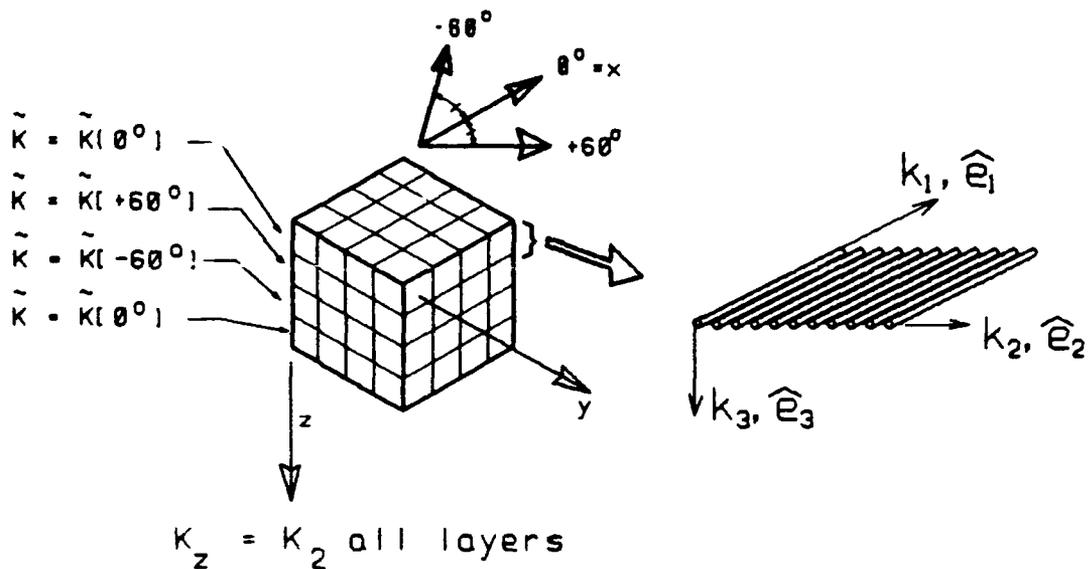


Figure 1. Thermal Conductivity in Orthotropic Laminae  
 The conductivity values in the rotated layers are found by the second order tensor transformation (11).

$$K'_{ij} = a_{ik} a_{jl} K_{kl} \quad (7)$$

where

$K'_{ij}$  is the tensor in the rotated axes  $\hat{e}'_1, \hat{e}'_2, \hat{e}'_3$

$K_{kl}$  is the tensor in the principal axes  $\hat{e}_1, \hat{e}_2, \hat{e}_3$

$a_{ik}$  is the direction cosine between the  $\hat{e}_k$  and  $\hat{e}'_i$  axes

Letting  $A_{ik}$  be a matrix of the direction cosines  $a_{ik}$ ,

Equation (7) can be written:

$$K'_{ij} = A_{ik} K_{kl} A^T_{jl} \quad (8)$$

For the rotation of the sample in the laminate described, this leads to two non-diagonal values:

$$K_{12} = K_{21} = (K_{11} - K_{22}) \cos\alpha \sin\alpha \quad (9)$$

The only conductivity terms calculated in the computer code used for this project are these and terms on the diagonal. It would be a simple matter to calculate and include the remaining terms if necessary, to provide for totally anisotropic constructions.

The material properties of each element in the region are calculated for the current temperature. The flux components are then estimated from the current temperature distribution, using Equations (6). Equation (1) is then solved for the incremental temperature change,  $\Delta T$ . This increase in temperature is then added to the cell's current value, and the region is checked for the onset of phase change.

This phase change is ablation in this case, but it might also be melting or sublimation in the case of other materials. When the phase change temperature is exceeded, the amount of energy absorbed is subtracted from the cell's total phase-change energy, and the temperature is set back to the phase change temperature. If the cell's energy of phase change is exceeded during this check in later iterations, the cell is 'removed' from the calculations by

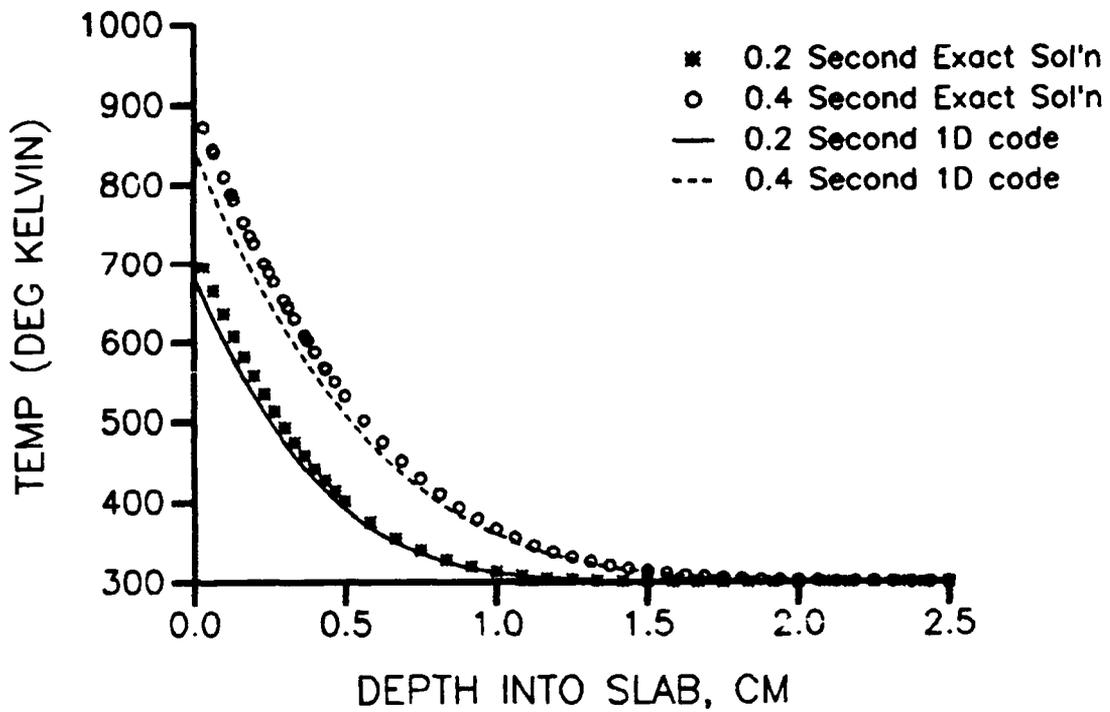
having its conductivities multiplied by zero in the following iterations.

The current code using this algorithm is listed in Appendix A. It remains predominately stable by the criterion that temperatures monotonically decrease from the irradiated regions toward insulated boundaries. This is the case for all but one or two cells, which remain 5 or 6 degrees below the initial temperature imposed upon the region; but no adjacent cells suffer this anomaly, and the remainder of the region's temperature values increase as expected even at long run times (high iteration count).

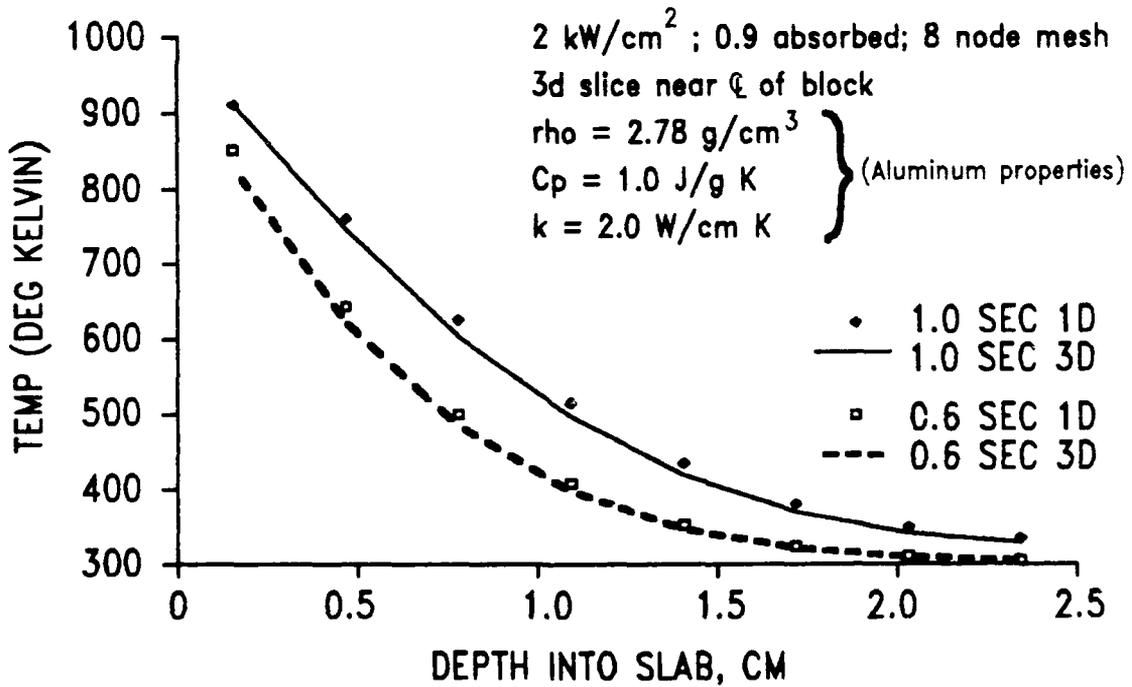
No further investigation was made into this difficulty since the code agrees within 5 percent with simple 1D and closed-form analytic predictions of the temperature profile through the thickness (Figure 2). It also predicts experimental results (Figure 3) and the reverse temperature gradient was found to decrease with increasing mesh refinement.

#### Stress Distribution in a Single Fiber

Once the temperature distribution has been calculated, the mechanical response of the fibers will be determined. It is expected that fracture of the fibers will occur at stresses below their ultimate strength due to growth of cracks. Initial cracks in the fiber exist in some distribution. Thermal expansion stresses may cause fracture of some fraction of the fibers by forcing unstable



(a) Semi-Infinite Solution vs. 98 Node 1D Code



(b) 1D Code vs. 3D Code

Figure 2. Comparison of Various Finite Difference Temperature Predictions

growth of these cracks. This is assumed because of the evidence of discrete fiber removal from various composite materials with no applied loads (2).

To calculate the stress growing the cracks,  $\sigma_x$ , (Figure 4) the fibers will be modeled as single beams laying on an elastic foundation (Figure 5). This is to include the influence of the resin layer beneath the fibers.

#### Modeling of Single Graphite Fiber

Elementary beam theory provides the following equations for stresses in a prismatic bar (4:214):

$$\sigma_x = \frac{M \cdot z}{I} \quad (10)$$

To find this moment, a single graphite fiber will be modeled as a round Bernoulli-Euler beam with two distinct regions. One region extends from the centerline to the edge of the laser beam. The next region extends beyond the edge of the laser beam into the unheated remainder. This region is many orders of magnitude longer than the fiber diameter,  $d$ , so that the fiber may be considered to extend semi-infinitely (Figure 6).

The axial stress in a fiber due to an axial load is simply the load divided by the fiber cross-sectional area,

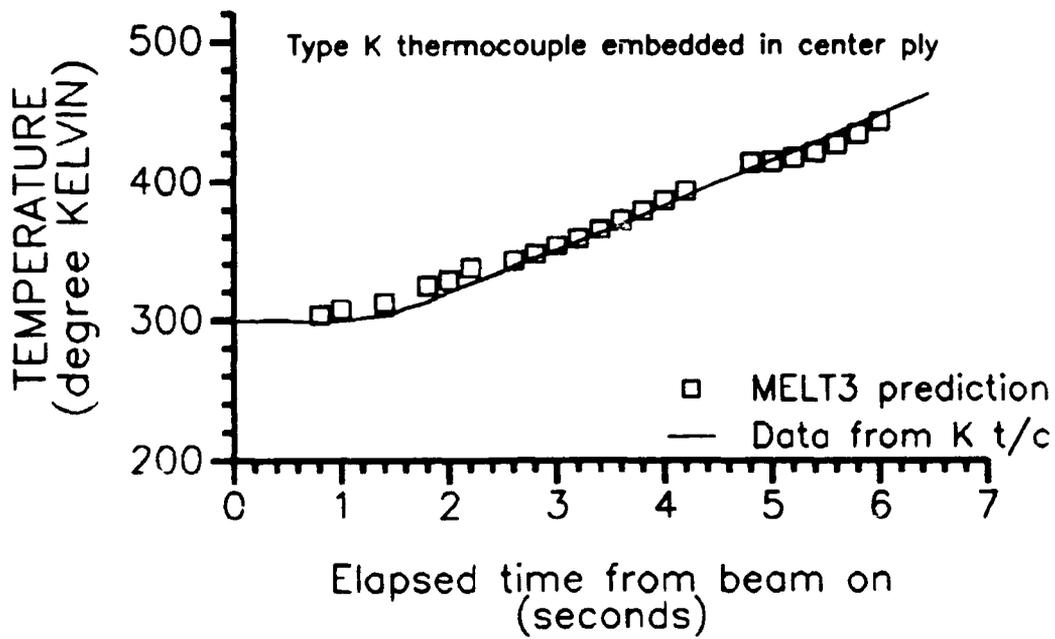


Figure 3. Comparison of MELT3D Predictions with Experimental Data

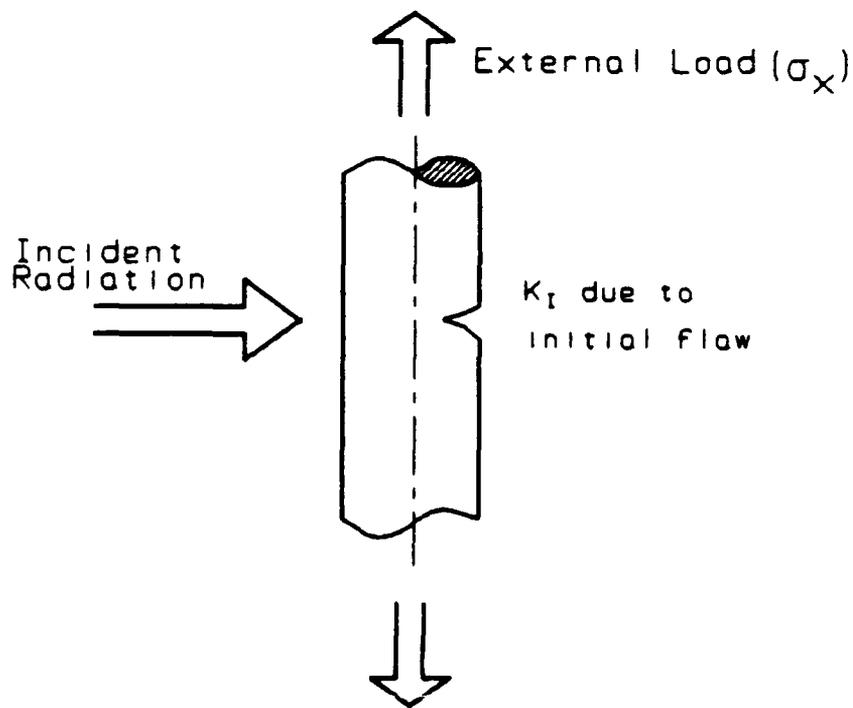


Figure 4. Single Graphite Fiber

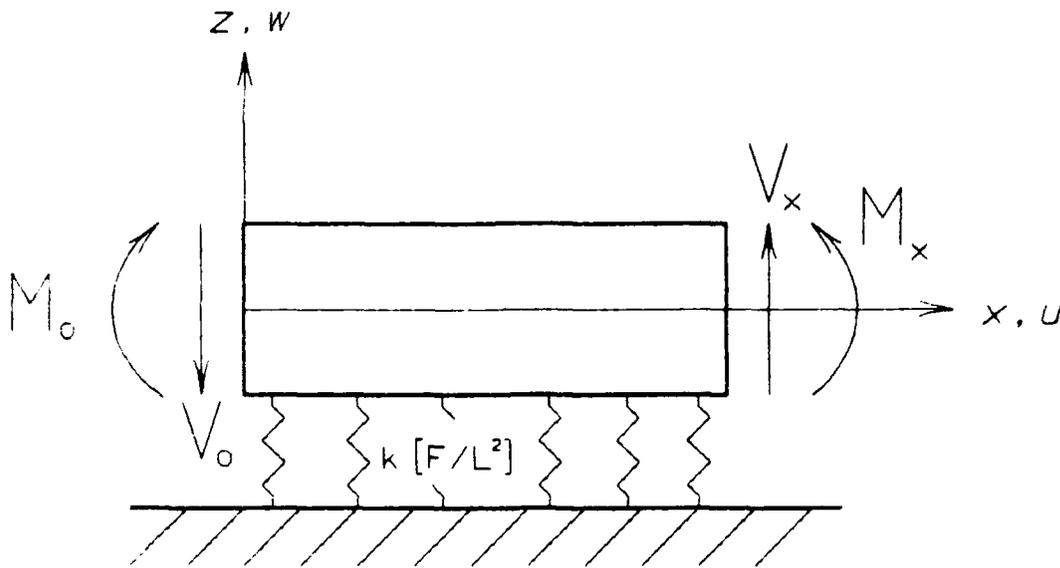


Figure 5. Beam Element on Elastic Foundation

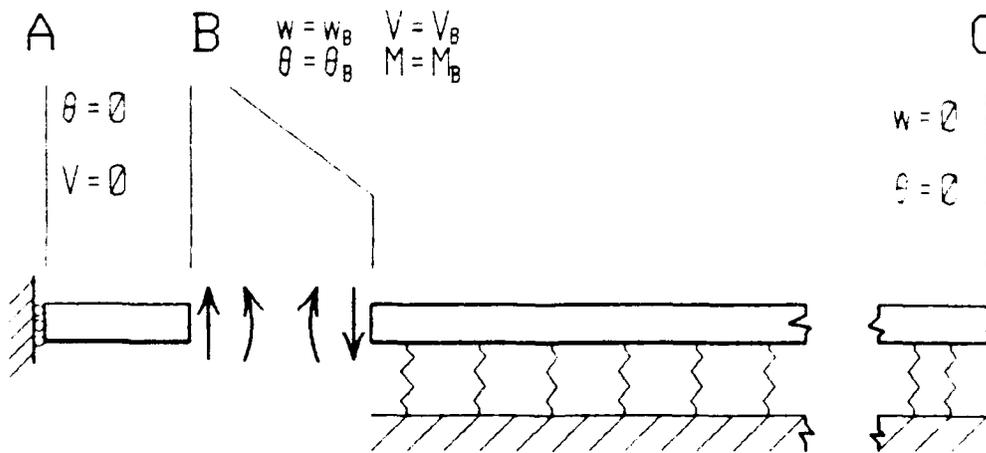


Figure 6. End Conditions for Beams AB and BC

if shearing on the matrix/fiber surface interface is ignored. While this shear is the only physical mechanism for transferring load to the fiber, the remainder of the analysis will be concerned with the heated region after the matrix may be considered to have been removed by pyrolysis. This means all the load will be transferred to a fiber by interaction with the matrix in the region away from the actual laser spot, so simple uniaxial stress is assumed.

The actual value of this stress in a single fiber is found by calculating the total strain the laminate will experience, and imposing this strain on a single fiber

$$\sigma_x = E_{\text{fiber}} \epsilon^0 \quad (11)$$

where  $\epsilon^0$  is the strain of the mid-surface (which is the strain everywhere in the cross section, because balanced, symmetric laminates do not couple axial strains and midplane bending).

Although the stresses in the direction of the fiber,  $\sigma_x$ , will actually be the sum of axial and bending moment components, for the conditions being explored in this effort, the stresses due to bending are insignificant compared to stresses due to axial loads. See Appendix B for details of this analysis.

### Stress Distribution in the Laminate (9)

In a given ply of the laminate, the plane stress

$(\sigma_z = \tau_{xz} = \tau_{yz} = 0)$  constitutive law can be written as

$$\begin{Bmatrix} \sigma_1 \\ \sigma_2 \\ \tau_{12} \end{Bmatrix} = \begin{bmatrix} Q_{11} & Q_{12} & 0 \\ Q_{12} & Q_{22} & 0 \\ 0 & 0 & Q_{66} \end{bmatrix} \begin{Bmatrix} \epsilon_1 \\ \epsilon_2 \\ \gamma_{12} \end{Bmatrix}$$

where the reduced stiffnesses  $Q_{ij}$  are

$$Q_{11} = \frac{E_1}{1 - \nu_{12}\nu_{21}}$$

$$Q_{12} = \frac{\nu_{12}E_2}{1 - \nu_{12}\nu_{21}} = \frac{\nu_{21}E_1}{1 - \nu_{12}\nu_{21}}$$

$$Q_{22} = \frac{E_2}{1 - \nu_{12}\nu_{21}}$$

$$Q_{66} = G_{12}$$

and for a lamina at an angle  $\theta$  with respect to the structural axes, the stress-strain relations in terms of "transformed reduced stiffnesses"  $\bar{Q}_{ij}$  are

$$\begin{Bmatrix} \sigma_x \\ \sigma_y \\ \tau_{xy} \end{Bmatrix} = \begin{bmatrix} \bar{Q}_{11} & \bar{Q}_{12} & \bar{Q}_{16} \\ \bar{Q}_{12} & \bar{Q}_{22} & \bar{Q}_{26} \\ \bar{Q}_{16} & \bar{Q}_{26} & \bar{Q}_{66} \end{bmatrix} \begin{Bmatrix} \epsilon_x \\ \epsilon_y \\ \gamma_{xy} \end{Bmatrix} \quad (12)$$

in which

(13)

$$\bar{Q}_{11} = Q_{11} \cos^4 \theta + 2(Q_{12} + 2Q_{\sigma\sigma}) \sin^2 \theta \cos^2 \theta + Q_{22} \sin^4 \theta$$

$$\bar{Q}_{12} = (Q_{11} + Q_{22} - 4Q_{\sigma\sigma}) \sin^2 \theta \cos^2 \theta + Q_{12} (\sin^4 \theta + \cos^4 \theta)$$

$$\bar{Q}_{22} = Q_{11} \cos^4 \theta + 2(Q_{12} + 2Q_{\sigma\sigma}) \sin^2 \theta \cos^2 \theta + Q_{22} \sin^4 \theta$$

$$\bar{Q}_{1\sigma} = (Q_{11} + Q_{12} - 2Q_{\sigma\sigma}) \sin \theta \cos^3 \theta + (Q_{12} - Q_{22} + 2Q_{\sigma\sigma}) \sin^3 \theta \cos \theta$$

$$\bar{Q}_{2\sigma} = (Q_{11} + Q_{12} - 2Q_{\sigma\sigma}) \sin^3 \theta \cos \theta + (Q_{12} - Q_{22} + 2Q_{\sigma\sigma}) \sin \theta \cos^3 \theta$$

$$\bar{Q}_{\sigma\sigma} = (Q_{11} + Q_{22} - 2Q_{12} - 2Q_{\sigma\sigma}) \sin^2 \theta \cos^2 \theta + Q_{\sigma\sigma} (\sin^4 \theta + \cos^4 \theta)$$

The strains anywhere through the thickness of the laminate may be expressed in terms of the mid-surface's

strain  $\{\epsilon_x^0, \epsilon_y^0, \gamma_{xy}^0\}^T$  and curvature  $\{\kappa_x, \kappa_y, \kappa_{xy}\}^T$  such

that for any given lamina

$$\begin{Bmatrix} \sigma_x \\ \sigma_y \\ \tau_{xy} \end{Bmatrix}_k = \begin{bmatrix} \bar{Q}_{11} & \bar{Q}_{12} & \bar{Q}_{1\sigma} \\ \bar{Q}_{12} & \bar{Q}_{22} & \bar{Q}_{2\sigma} \\ \bar{Q}_{1\sigma} & \bar{Q}_{2\sigma} & \bar{Q}_{\sigma\sigma} \end{bmatrix}_k \left\{ \begin{Bmatrix} \epsilon_x^0 \\ \epsilon_y^0 \\ \gamma_{xy}^0 \end{Bmatrix} + z \begin{Bmatrix} \kappa_x \\ \kappa_y \\ \kappa_{xy} \end{Bmatrix} \right\}$$

Integrating these stresses over the thickness produces the following expressions for force per unit width and moment per unit width

$$\begin{Bmatrix} N_x \\ N_y \\ N_{xy} \end{Bmatrix} = \int_{-t/2}^{t/2} \begin{Bmatrix} \sigma_x \\ \sigma_y \\ \tau_{xy} \end{Bmatrix} dz = \sum_{k=1}^N \int_{z_{k-1}}^{z_k} \begin{Bmatrix} \sigma_x \\ \sigma_y \\ \tau_{xy} \end{Bmatrix}_k dz$$

$$\begin{Bmatrix} M_x \\ M_y \\ M_{xy} \end{Bmatrix} = \int_{-t/2}^{t/2} \begin{Bmatrix} \sigma_x \\ \sigma_y \\ \tau_{xy} \end{Bmatrix} z dz = \sum_{k=1}^N \int_{z_{k-1}}^{z_k} \begin{Bmatrix} \sigma_x \\ \sigma_y \\ \tau_{xy} \end{Bmatrix}_k z dz$$

where  $z_k$  and  $z_{k-1}$  are defined in Figure 7. This can be expressed as

$$\begin{Bmatrix} N_x \\ N_y \\ N_{xy} \end{Bmatrix}_k = \begin{bmatrix} A_{11} & A_{12} & A_{1\sigma} \\ A_{12} & A_{22} & A_{2\sigma} \\ A_{1\sigma} & A_{2\sigma} & A_{\sigma\sigma} \end{bmatrix} \begin{Bmatrix} \epsilon_x^0 \\ \epsilon_y^0 \\ \gamma_{xy}^0 \end{Bmatrix} + \begin{bmatrix} B_{11} & B_{12} & B_{1\sigma} \\ B_{12} & B_{22} & B_{2\sigma} \\ B_{1\sigma} & B_{2\sigma} & B_{\sigma\sigma} \end{bmatrix} \begin{Bmatrix} \kappa_x \\ \kappa_y \\ \kappa_{xy} \end{Bmatrix}$$

$$\begin{Bmatrix} M_x \\ M_y \\ M_{xy} \end{Bmatrix}_k = \begin{bmatrix} B_{11} & B_{12} & B_{1\sigma} \\ B_{12} & B_{22} & B_{2\sigma} \\ B_{1\sigma} & B_{2\sigma} & B_{\sigma\sigma} \end{bmatrix} \begin{Bmatrix} \epsilon_x^0 \\ \epsilon_y^0 \\ \gamma_{xy}^0 \end{Bmatrix} + \begin{bmatrix} D_{11} & D_{12} & D_{1\sigma} \\ D_{12} & D_{22} & D_{2\sigma} \\ D_{1\sigma} & D_{2\sigma} & D_{\sigma\sigma} \end{bmatrix} \begin{Bmatrix} \kappa_x \\ \kappa_y \\ \kappa_{xy} \end{Bmatrix}$$

where

$$A_{ij} = \sum_{k=1}^N (\bar{Q}_{ij})_k (z_k - z_{k-1})$$

$$B_{ij} = \frac{1}{2} \sum_{k=1}^N (\bar{Q}_{ij})_k (z_k^2 - z_{k-1}^2)$$

$$D_{ij} = \frac{1}{3} \sum_{k=1}^N (\bar{Q}_{ij})_k (z_k^3 - z_{k-1}^3)$$

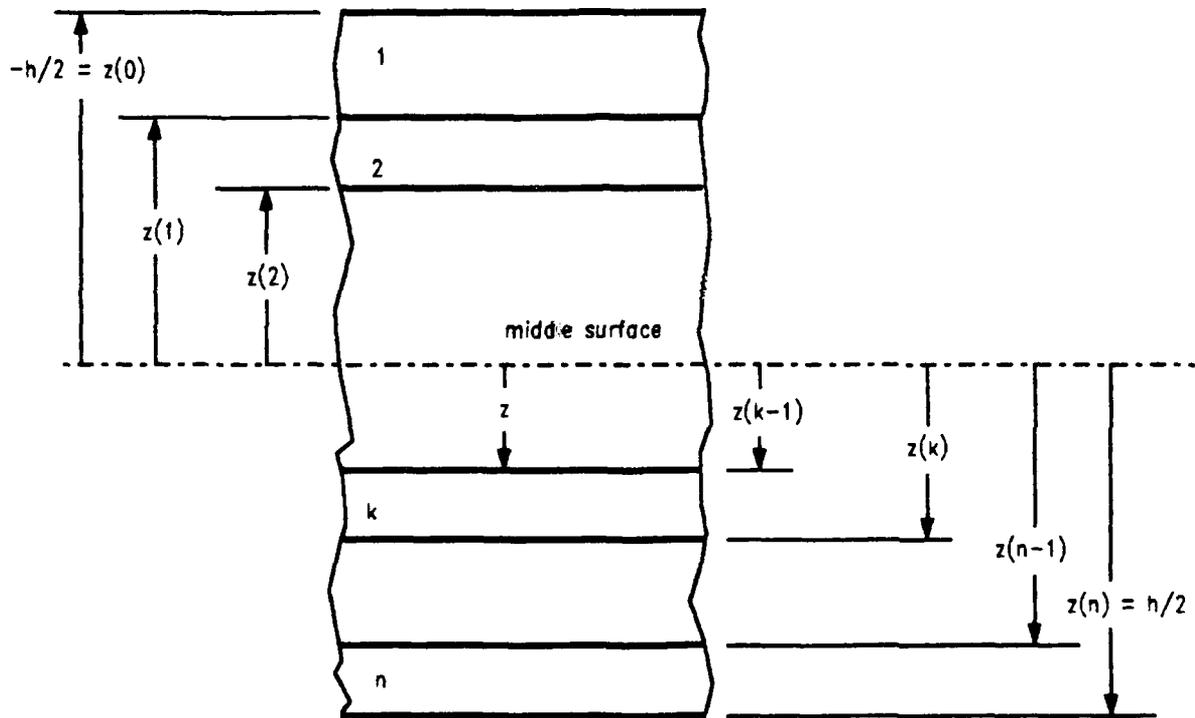


Figure 7. Geometry of an n-layered laminate (9:154)

In (9) it is shown that for a laminate cured at a temperature other than the service temperature, 'thermal forces and moments' may be defined by

$$\begin{Bmatrix} N_x^T \\ N_y^T \\ N_{xy}^T \end{Bmatrix}_k = \int \begin{bmatrix} \bar{Q}_{11} & \bar{Q}_{12} & \bar{Q}_{1\theta} \\ \bar{Q}_{12} & \bar{Q}_{22} & \bar{Q}_{2\theta} \\ \bar{Q}_{1\theta} & \bar{Q}_{2\theta} & \bar{Q}_{\theta\theta} \end{bmatrix}_k \begin{Bmatrix} \alpha_x^0 \\ \alpha_y^0 \\ \alpha_{xy}^0 \end{Bmatrix}_k \Delta T dz$$

$$\begin{Bmatrix} M_x^T \\ M_y^T \\ M_{xy}^T \end{Bmatrix}_k = \int \begin{bmatrix} \bar{Q}_{11} & \bar{Q}_{12} & \bar{Q}_{1\theta} \\ \bar{Q}_{12} & \bar{Q}_{22} & \bar{Q}_{2\theta} \\ \bar{Q}_{1\theta} & \bar{Q}_{2\theta} & \bar{Q}_{\theta\theta} \end{bmatrix}_k \begin{Bmatrix} \alpha_x^0 \\ \alpha_y^0 \\ \alpha_{xy}^0 \end{Bmatrix}_k \Delta T z dz$$

where the thermal expansion coefficients for an arbitrary lamina at angle  $\theta$ ,  $\{\alpha_x, \alpha_y, \alpha_{xy}\}^T$ , are given by vector transformation of the two coefficients in the lamina's coordinates,  $\{\alpha_1, \alpha_2, 0\}^T$ .

Finally, by defining effective forces and moments

$$\begin{Bmatrix} \bar{N}_x \\ \bar{N}_y \\ \bar{N}_{xy} \end{Bmatrix} = \begin{Bmatrix} N_x + N_x^T \\ N_y + N_y^T \\ N_{xy} + N_{xy}^T \end{Bmatrix} = \begin{bmatrix} A_{11} & A_{12} & A_{1\theta} \\ A_{12} & A_{22} & A_{2\theta} \\ A_{1\theta} & A_{2\theta} & A_{\theta\theta} \end{bmatrix} \begin{Bmatrix} \epsilon_x^0 \\ \epsilon_y^0 \\ \gamma_{xy}^0 \end{Bmatrix} + \begin{bmatrix} B_{11} & B_{12} & B_{1\theta} \\ B_{12} & B_{22} & B_{2\theta} \\ B_{1\theta} & B_{2\theta} & B_{\theta\theta} \end{bmatrix} \begin{Bmatrix} \alpha_x \\ \alpha_y \\ \alpha_{xy} \end{Bmatrix}$$

$$\begin{Bmatrix} \bar{M}_x \\ \bar{M}_y \\ \bar{M}_{xy} \end{Bmatrix} = \begin{Bmatrix} M_x + M_x^T \\ M_y + M_y^T \\ M_{xy} + M_{xy}^T \end{Bmatrix} = \begin{bmatrix} B_{11} & B_{12} & B_{1\theta} \\ B_{12} & B_{22} & B_{2\theta} \\ B_{1\theta} & B_{2\theta} & B_{\theta\theta} \end{bmatrix} \begin{Bmatrix} \epsilon_x^0 \\ \epsilon_y^0 \\ \gamma_{xy}^0 \end{Bmatrix} + \begin{bmatrix} D_{11} & D_{12} & D_{1\theta} \\ D_{12} & D_{22} & D_{2\theta} \\ D_{1\theta} & D_{2\theta} & D_{\theta\theta} \end{bmatrix} \begin{Bmatrix} \alpha_x \\ \alpha_y \\ \alpha_{xy} \end{Bmatrix}$$

or

$$\left\{ \begin{array}{c} \bar{N} \\ \bar{M} \end{array} \right\} = \left[ \begin{array}{c|c} \frac{A}{B} & \frac{B}{D} \end{array} \right] \left\{ \begin{array}{c} \epsilon^o \\ \kappa \end{array} \right\} \quad (17)$$

we can invert the stiffness matrices A, B and D and express the strains and curvatures as

$$\left\{ \begin{array}{c} \epsilon^o \\ \kappa \end{array} \right\} = \left[ \begin{array}{c|c} \frac{A'}{H'} & \frac{B'}{D'} \end{array} \right] \left\{ \begin{array}{c} \bar{N} \\ \bar{M} \end{array} \right\}.$$

With the layup used, effective engineering properties can be defined

$$E_x = \frac{A_{11}A_{22} - A_{12}^2}{h A_{22}}$$

where h is total laminate thickness such that

$$\epsilon_x^o = \sigma_x / E_x \quad (19)$$

With the properties of the laminae used here being

$$E_1 = 20.8 \times 10^6 \text{ psi (143.0 GPa)}$$

$$E_2 = 1.38 \times 10^6 \text{ psi (9.51 GPa)}$$

$$\nu_{12} = 0.31$$

$$G = 0.85 \times 10^6 \text{ psi (5.86 GPa)}$$

$$\epsilon_{1_{ult}} = 1.42\%$$

it follows that the effective modulus of the total laminate is  $E_x = 8.1 \times 10^6$  psi (55.8 GPa). Therefore,  $\epsilon_x^0 = \sigma_x/E_x$  is the mid-surface strain which will be assumed for each fiber in the laminate. The stress in any lamina will be assumed to be this strain multiplied by the lamina's stiffness in the x direction. This is a simplifying assumption, since once ablation occurs symmetry and balance are destroyed, creating a laminate with different stiffness coefficients. These differences will be assumed to have negligible effect upon the axial stresses in the test samples.

#### Interaction of Thermal and Mechanical Loads

The matrix epoxy absorbs little of the total energy of ablation, and practically no evidence of discrete removal of the matrix exists (2). Therefore, the discrete mass removal model will only consider fracture and discrete removal of the fibers. This leads to a formulation of the form

$$\mathcal{E}_{total} = V_m \mathcal{E}_m + V_f \mathcal{E}_f \quad (20)$$

where  $V_m$  = volume fraction of matrix

$V_f$  = volume fraction of fibers

But  $\mathcal{E}_f$  is only the energy absorbed by that fraction of the fibers which ablate,  $\eta_a$ . A fraction  $\eta_d (= 1-\eta_a)$  will be removed intact, absorbing essentially no energy.

Although  $Q_{eff}^*$  is an empirical measure of all the energy delivered to a material divided by the mass removed during irradiation, it is a reasonable approximation to the actual ablation energy.

From Figure A2,  $C_p$  is less than 2000 J/kg·K. From 300 °K to ablation at 3800 °K, the energy absorbed will be:

$$\begin{aligned} \dot{E} &= C_p \Delta T \\ &= 2000 \frac{\text{J}}{\text{kg K}} \cdot 3500 \text{ }^\circ\text{K} \\ &= 7 \frac{\text{kJ}}{\text{gm}} \\ &\approx 20\% \cdot Q_{eff}^* \end{aligned}$$

The total energies of temperature rise to the ablation point plus the ablation phase change energy will both be considered in  $Q_{eff}^*$ .

So we have

$$Q_{eff}^* = \frac{\text{Projected} - \text{Reflected} - \text{Radiated} - \text{Conducted} = \text{Energy of Temperature Rise} + \text{Energy of Phase Change}}{\text{Reflected} + \text{Radiated} + \text{Conducted} + \text{Energy of Temperature Rise} + \text{Energy of Phase Change}} = \frac{\text{Material Lost as Solid} + \text{Material Lost as Vapor}}{\text{Material Lost as Solid} + \text{Material Lost as Vapor}}$$

The true Q would be the ratio of the last two terms.

Looking more closely at Equation (20), the energy absorbed will be examined on a layer by layer basis. Each layer is assumed to be either entirely fiber or entirely

matrix, with thicknesses divided in the same ratio as the composite's fiber-to-matrix distribution. The energy absorbed during the ablation will be

$$Q_{\text{abs}} = \eta_a Q_f^* \Delta m_f = \eta_a Q_f^* \rho_f A_{\text{laser}} \Delta h_f$$

where

$Q_f^*$  = effective enthalpy of ablation of fibers alone

$\Delta h_f$  = thickness of a fiber layer

$A_{\text{laser}} = A_l$  = laser spot area

$\Delta h_f$  = thickness of a fiber layer

and the other symbols have their previous definitions. In the next layer of matrix, the energy absorbed is simply

$$Q_{\text{abs}} = Q_m^* \Delta m_m = Q_m^* \rho_m A_{\text{laser}} \Delta h_m$$

The effective  $Q^*$  calculated during the ablation of all these layers would be

$$Q_{\text{eff}}^* = \frac{\sum Q_{\text{abs}}}{\sum \Delta m}$$

$$= \frac{N_f \left[ \eta_a Q_f^* \rho_f A_l \Delta h_f \right] + N_m \left[ Q_m^* \rho_m A_l \Delta h_m \right]}{N_f \rho_f A_l \Delta h_f + N_m \rho_m A_l \Delta h_m} \quad (21)$$

where  $N_f$  = total number of fiber layers

$N_m$  = total number of matrix layers

$N_f \Delta h_f = h_f$ , total fiber thickness

$N_m \Delta h_m = h_m$ , total matrix thickness

and note that the total thickness ablated is

$$h = h_f + h_m$$

where

$$h_m = h V_m$$

$$h_f = h V_f \quad (22)$$

Substituting Equations (22) back into Equation (21) results in

$$Q_{off}^* = \frac{\eta_d \rho_f V_f Q_f^* + \rho_m V_m Q_m^*}{\rho_f V_f + \rho_m V_m} = \frac{(1-\eta_d) \rho_f V_f Q_f^* + \rho_m V_m Q_m^*}{\rho_f V_f + \rho_m V_m} \quad (23)$$

$$= \frac{[1-\eta_d] \rho_f V_f Q_f^* + \rho_m V_m Q_m^*}{\rho} \quad \text{where } \rho \text{ is total composite density}$$

$$= [1 - \eta_d] x_f Q_f^* + x_m Q_m^* \quad (24)$$

where  $x_f = \rho_f V_f / \rho$  = mass fraction of fibers

$x_m = \rho_m V_m / \rho$  = mass fraction of matrix

It can be seen when  $\eta_d$  is zero,  $Q_{off}^*$  becomes simply  $Q_{off}^* = Q_f^* \chi_f + Q_m^* \chi_m$ , which is the limiting case that would have been arrived at by simple intuition.

The form of relationship between applied stress  $\sigma_x$  and discrete removal efficiency  $\eta_d$  is suggested by the fracture mechanics of a single fiber. The mass of discrete fibers removed will be

$$\eta_d M_f = \rho_f \frac{\pi d^2}{4} \bar{L} N_d \quad (25)$$

where

$N_d$  = number of discrete fibers removed

$\bar{L}$  = average length of a removed fiber

The brittle fracture of a given fiber will be governed by the applied stress intensity factor (SIF)  $K_I$ . When this applied SIF exceeds the critical SIF  $K_{IC}$  crack growth will occur. If additional energy is available to allow  $K_I$  to remain above  $K_{IC}$ , unstable crack growth will fracture the fiber (1). The applied SIF is calculated from

$$K_I = \frac{\sigma_x}{f \sqrt{\pi a}} \quad (26)$$

where  $f$  is a function of the crack geometry. For a given constant distribution of initial cracks of length  $a$ ,  $K_I$  is

simply proportional to applied stress  $\sigma_x$ . If damage is assumed to be proportional to  $K_x$ , this suggests a linear variation of  $\eta_d$  with the applied stress  $\sigma_x$ . To account for the possibility of some threshold stress,  $\sigma_{x0}$ , below which no discrete mass removal by fracture occurs, a variation will be assumed of the form

$$\eta_d = \eta_{d0} + \eta_{dx} \cdot \sigma_x \quad (27)$$

Available data on this phenomenon is too sparse to be conclusive, but it does support the assumption of a monotonically increasing function of  $\sigma_x$  (17). A small amount of additional data collected in preparation for the primary set of experiments also lends credibility to the form of Equation (27). See Appendix D for details.

Substituting Equation (27) into Equation (24) yields

$$Q_{eff}^* = [1 - \eta_{d0} - \eta_{dx} \sigma_x] x_f Q_f^* + x_m Q_m^* \quad (28)$$

The values of  $Q_f^*$  and  $Q_m^*$  could be deduced from experiments conducted at  $\sigma_x = 0$ , or assumed to be (5)

$$\begin{aligned} Q_f^* &\approx 43 \text{ kJ/gm} \\ Q_m^* &\approx 4 \text{ kJ/gm} \end{aligned} \quad (29)$$

Using the mass fractions measured for the particular specimens used in this thesis, and the values in Equation (29), the predicted effect of loading on  $Q_{eff}^*$ , will have a form like that shown in Figure 8.

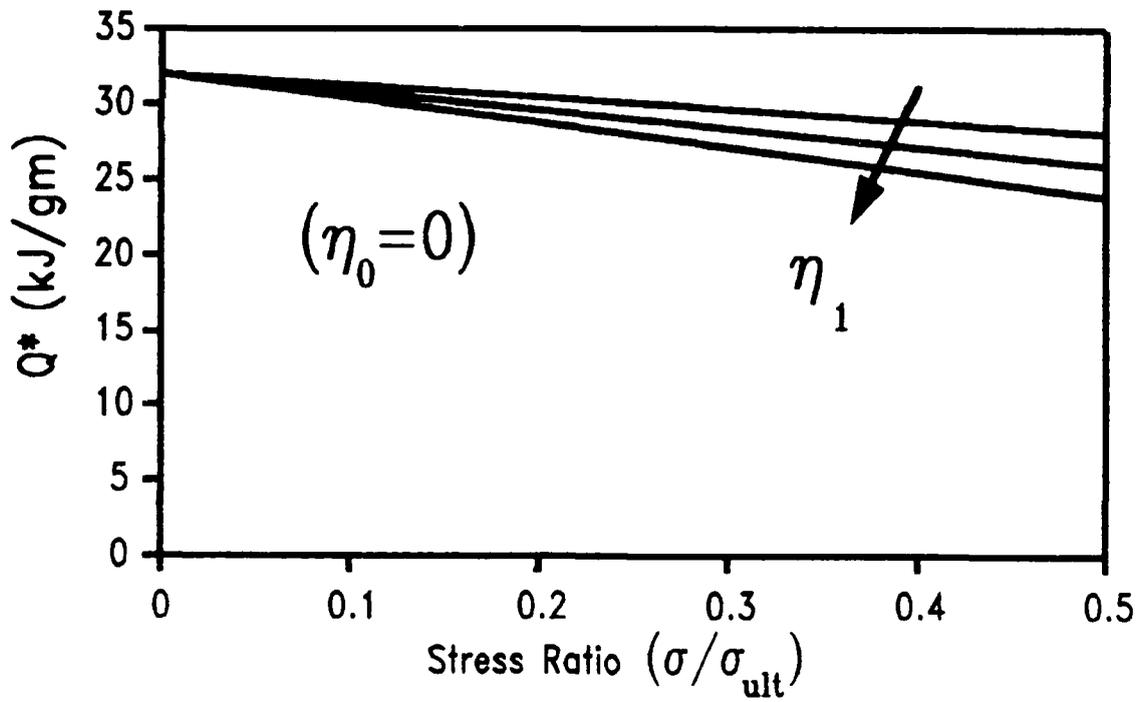


Figure 8. Predicted  $Q^*$  Behaviour Under Loading

### III. Experimental Apparatus

#### Test Specimens

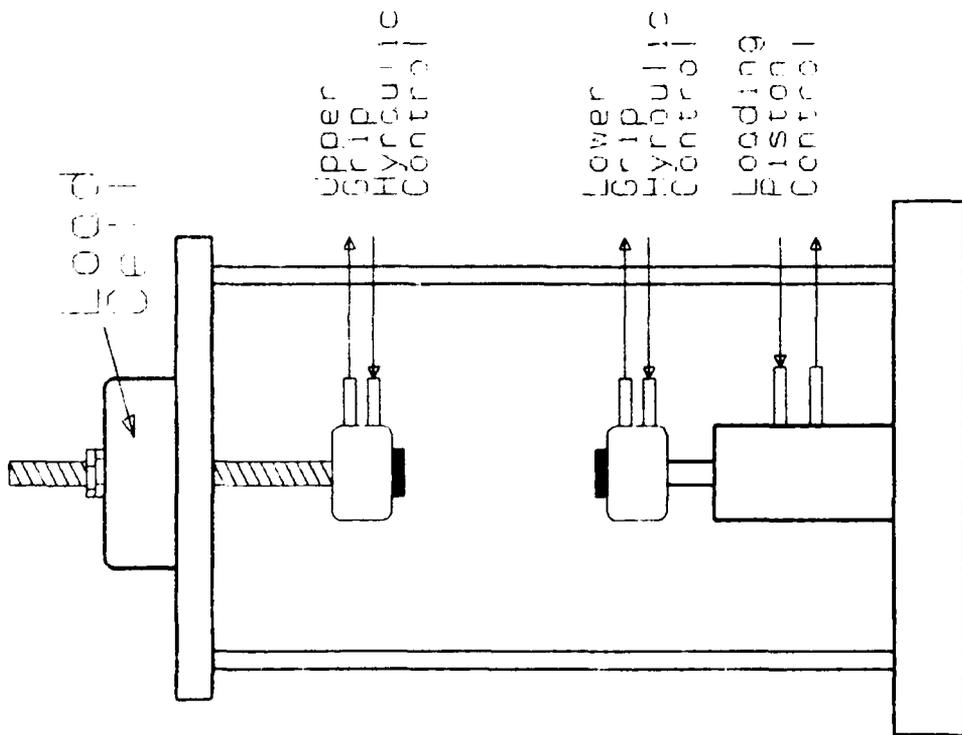
Fabrication of the test specimens is detailed in Appendix C. A listing of the specimens fabricated and pertinent pre-test data is given in Table I. All specimens were laid up as balanced, symmetric laminates. The thinner (0.244" = 0.620 cm) samples were (0, ± 60)<sub>60</sub>. The thicker samples (0.367" = 0.932 cm) were (0, ± 60)<sub>120</sub>.

#### Specimen Instrumentation

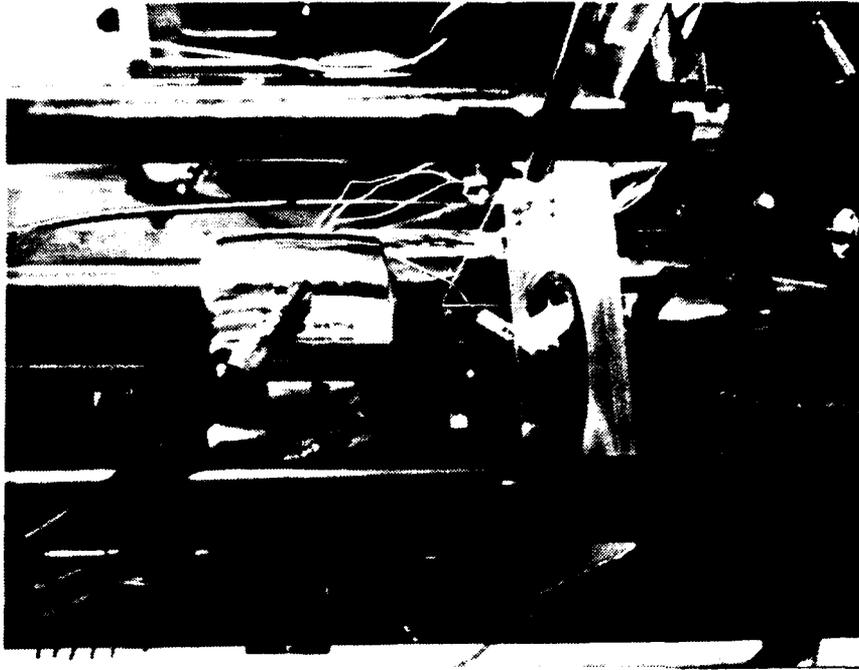
Type K (chromel/alumel) thermocouples and/or strain gauges were attached to some specimens (Figure Clb). All samples were weighed before being tested to determine their mass.

#### Tensile Loading Machine

The device used to load the samples in tension was a hydraulically actuated frame built by the Air Force Weapons Laboratory's Directed Energy Weapon Effects Branch (WL/TALE). A schematic diagram is shown in Figure 9a, and a photograph of the grips holding a sample follows in Figure 9b. Loads of up to 30000 pounds of force (133400 N) were applied in this program. The force was measured



(a)



(b)

Figure 9. Tensile Loading Machine

by a Strainert Universal Flat Load Cell, Model FL50U-3DPKD, SN 06669-1 (50000 lb capacity) located above the top hydraulic grip of the machine.

TABLE I  
Test Specimen Specifications

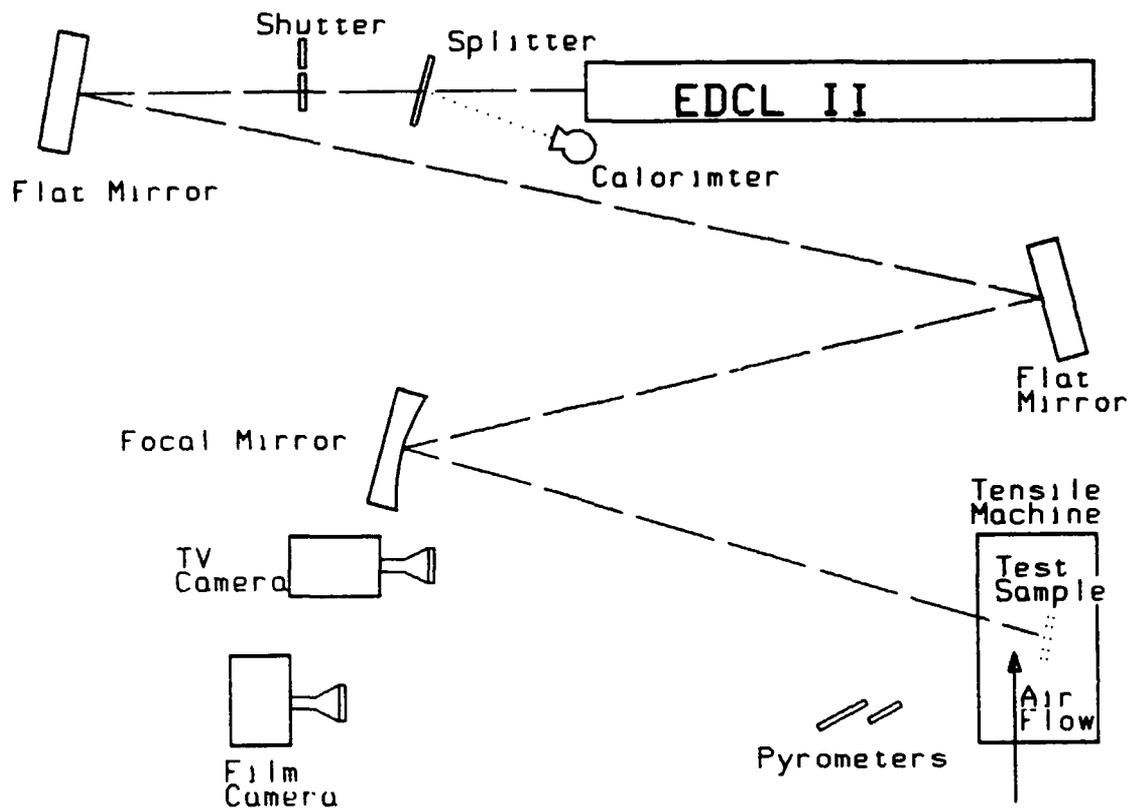
Shot#	SampleID#	Beam Diameter		Spot Area (cm <sup>2</sup> )	Average Panel Properties		
		Minor (cm)	Major (cm)		Density (gm/cm <sup>3</sup> )	Resin (Wt%)	Voids (Wt%)
1	CH13989-1#5	2.7	2.9	6.15	NA	NA	NA
2	CH13989-1#3	2.7	2.9	6.15	NA	NA	NA
3	CH13989-4#1	2.5	2.7	5.30	NA	NA	NA
4	CH13989-4#2	2.5	2.8	5.50	NA	NA	NA
5	CH13989-4#3	2.6	2.8	5.72	NA	NA	NA
6	CH13989-4#5	2.6	2.8	5.72	NA	NA	NA
7	CH13989-5#1	2.6	2.8	5.72	NA	NA	NA
8	JH13189-1#3	2.6	2.8	5.72	1.59	27.75	1.32
9	JH13189-1#4	2.6	2.8	5.72	1.59	27.75	1.32
10	JH13189-2#3	1.65	1.8	2.33	1.59	28.13	0.92
11	JH13189-2#4	1.65	1.8	2.33	1.59	28.13	0.92
12	JH13189-2#5	1.65	1.8	2.33	1.59	28.13	0.92
13	JH13189-2#6	1.65	1.8	2.33	1.59	28.13	0.92
14	JH13189-2#7	1.65	1.8	2.33	1.59	28.13	0.92
15	JH13189-2#8	1.65	1.8	2.33	1.59	28.13	0.92
16	JH13189-1#3	1.65	1.8	2.33	1.59	27.75	1.32
17	CH13989-2#1	1.2	1.3	1.23	NA	NA	NA
18	CH13989-2#2	1.2	1.3	1.23	NA	NA	NA
19	JH13189-2#9	1.2	1.3	1.23	1.59	28.13	0.92
20	CH13989-3#1	1.2	1.3	1.23	NA	NA	NA
21	JH13189-1#4	1.2	1.3	1.23	1.59	27.75	1.32
22	CH13989-3#2	1.2	1.3	1.23	NA	NA	NA
23	CH13989-3#3	1.2	1.3	1.23	NA	NA	NA
24	CH13989-3#4	1.2	1.3	1.23	NA	NA	NA
25	CH13989-3#6	1.2	1.3	1.23	NA	NA	NA
26	JH13189-2#10	1.2	1.3	1.23	1.59	28.13	0.92
27	JH13189-1#6	1.2	1.3	1.23	1.59	27.75	1.32
28	CH13989-2#7	1.5	1.7	2.00	NA	NA	NA
29	JH13189-1#7	1.5	1.7	2.00	1.59	27.75	1.32
30	JH13189-1#5	1.5	1.7	2.00	1.59	27.75	1.32
31	CH13989-3#8	1.5	1.7	2.00	NA	NA	NA
32	CH13989-3#9	1.5	1.7	2.00	NA	NA	NA
33	CH13989-3#7	1.5	1.7	2.00	NA	NA	NA
34	CH13989-3#10	1.5	1.7	2.00	NA	NA	NA
35	JH13189-2#11	1.5	1.7	2.00	1.59	28.13	0.92
36	JH13189-2#12	1.5	1.7	2.00	1.59	28.13	0.92

## Continuous Wave (CW) 10.6 $\mu\text{m}$ Laser and Associated Diagnostics

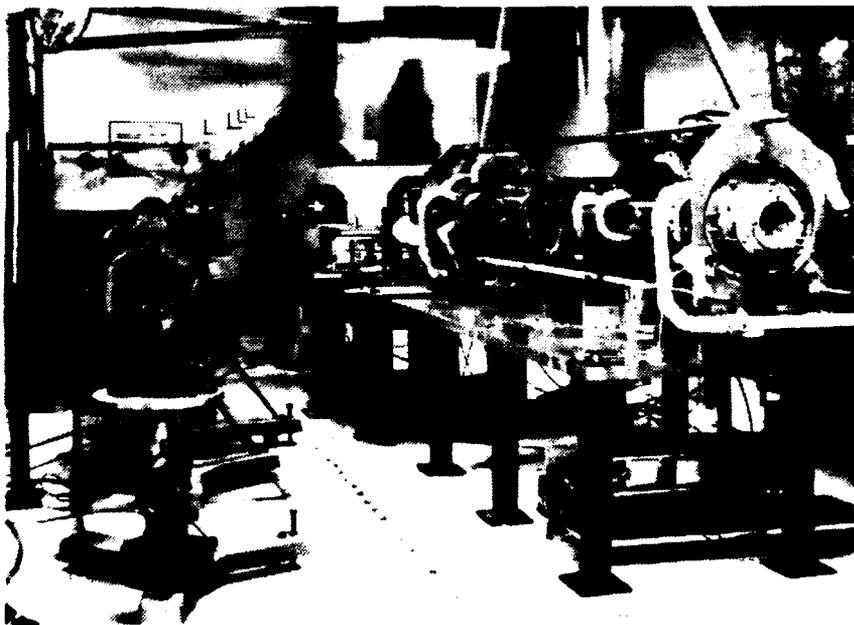
The Electric Discharge Coaxial Laser II (EDCL-II) is a 10.6  $\mu\text{m}$  infrared carbon dioxide laser capable of 40 kW of output power. As shown in Figure 10, in the test program described here a fraction of this power was removed by a sodium chloride beam splitter to be measured by a Coherent Model 213 power meter (SN B0H375). Diamond-lathed copper mirrors are used to take the remaining power and focus it on the target plane in the shape and intensity desired.

At the target, infrared emission from the target is detected by a Thermogage Model 8000-1A germanium pyrometer (SN 3247). This data provided information on the surface temperature of the ablating samples.

16mm motion pictures are also recorded at 500 frames per second (fps), using a Redlake Hycam II Model 41-0064 (SN333). Video tape of the ablating samples was recorded with a Sony DXC-M3 Color Video Camera and a Panasonic AG6200 VHS tape recorder. Placement of the camera and recorder made recording their serial numbers impossible.



(a)



(b)

Figure 10. Laser Optical Train

#### IV. Experimental Procedure

##### Specimen Preparation

The sample to be tested is placed in the upper hydraulic grip and positioned to be perpendicular to both the axis of loading and the incident laser beam. The lower grip is placed around the bottom of the sample and any instrumentation wires are routed away toward their respective recording devices (Figure 9b). 1600 psi (11 MPa) hydraulic pressure was supplied to the grips to secure the specimen. Data recording devices are checked to see that proper readings are being received from each of the instruments.

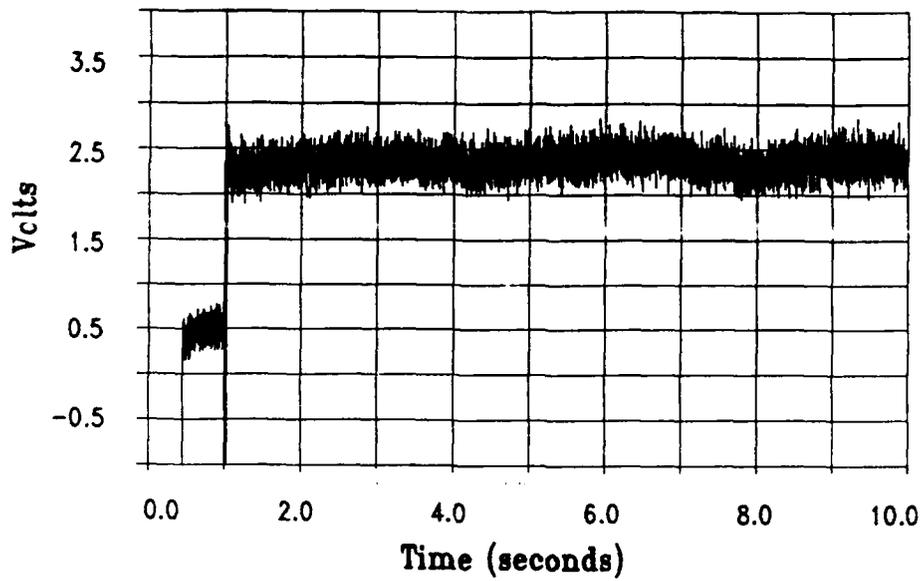
##### Laser Optical Train

The optical train is adjusted to produce a desired spot size upon the target. Accounting for the expected device output power, this adjustment controls the irradiance on target. Aberrations in the irradiance profile which might be encountered include ellipticity of an otherwise circular beam, irradiance fluctuations at various locations within the spot area, and power fluctuations with time during the test.

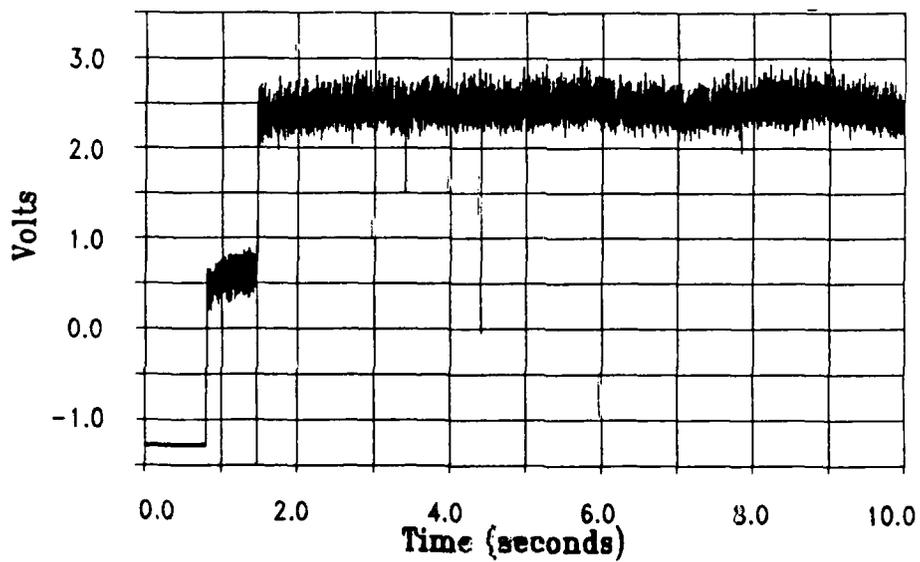
The first difficulty, a non-circular beam, occurs when magnifying mirrors reflect the beam at a finite angle with respect to their centerline. Since it is impossible to reflect the laser energy perpendicular to the mirror and still usefully propagate it, some ellipticity must remain in the beam, unless two or more mirrors can be used to offset each others effects. Major and minor diameters are recorded in Table I.

The next problem is affected by the ellipticity, by alignment of the laser resonator cavity, and by inherent design of the EDCL-II. Both ellipticity and spacial power variations can be quantified by measuring the burn profile in a soft material like plexiglass before testing the sample, or to a less accurate degree, by the shape of the sample's burn craters.

The last problem, temporal fluctuations in power, can only be measured during the test by a fast response power meter measuring scattered radiation from some part of the optical train. If the variations seen are significant, then they must be accounted for after the fact in the analysis of the test data. As shown in a typical plot of this reading (Figure 11), there were no significant fluctuations during these tests.



(a) AFIT Shot #4 EDCL II P(t)



(b) AFIT Shot #5 EDCL II P(t)

Figure 11. Typical Power vs. Time Curves

### Data Collection During Irradiation

Once all instruments are connected and operating properly, recording of the various signals is begun, the load on the tensile machine is raised to the level desired, and the laser is turned on.

After 1 to 2 seconds are allowed for the laser power output to stabilize, opaque shutters keeping the beam from going to the target are removed. The time allowed for the shutter to eclipse the beam is kept well below 5 percent of the total irradiation time to ensure the variation in intensity profile is insignificant to the sample's total interaction with the radiation.

After a predetermined amount of time, the shutter is returned to its initial position, removing the radiation from the target. The laser is then shut down.

### Post-test Data Collection

Any significant fibers broken off during the test which can be seen downwind of the purging air flow are collected. The sample is then weighed to determine its mass loss. It is then placed in a plastic bag and kept from further handling until photomicrographs can be taken.

After the photography of the sample's ablation surface, the depth of the crater ablated in the sample is measured with a depth gauge or other micrometer.

## V. Experimental Data

Table II and Figures 12, 13, and 14 summarize the results obtained on the EDCL-II device. Sample calculations showing equations used to reduce the data are shown in the following paragraphs.

Before irradiation tests occurred, two samples 'A' and 'B' were pulled in tension to failure in the tensile machine. Sample A's load/strain curve is shown in Figure 12. Sample B is similar. Compare the effective modulus  $E_x = 8.1 \text{ msi}$  (55.9 GPa) to that obtained on similar samples tested on an AFIT MTS tensile testing machine (also ~8.1 msi (55.9 GPa) from Figure C5). This is in excellent agreement with the effective modulus calculated in Chapter II.

The strength of the laminate was calculated by finding the stress which places the ultimate strain,  $\epsilon_{ult}$ , on the laminate

$$\begin{aligned}\sigma_{ult} &= E_x \epsilon_{ult} = 8.1 \times 10^6 \text{ psi} * 0.0142 \\ &= 115020 \text{ psi} = 793 \text{ MPa}\end{aligned}$$

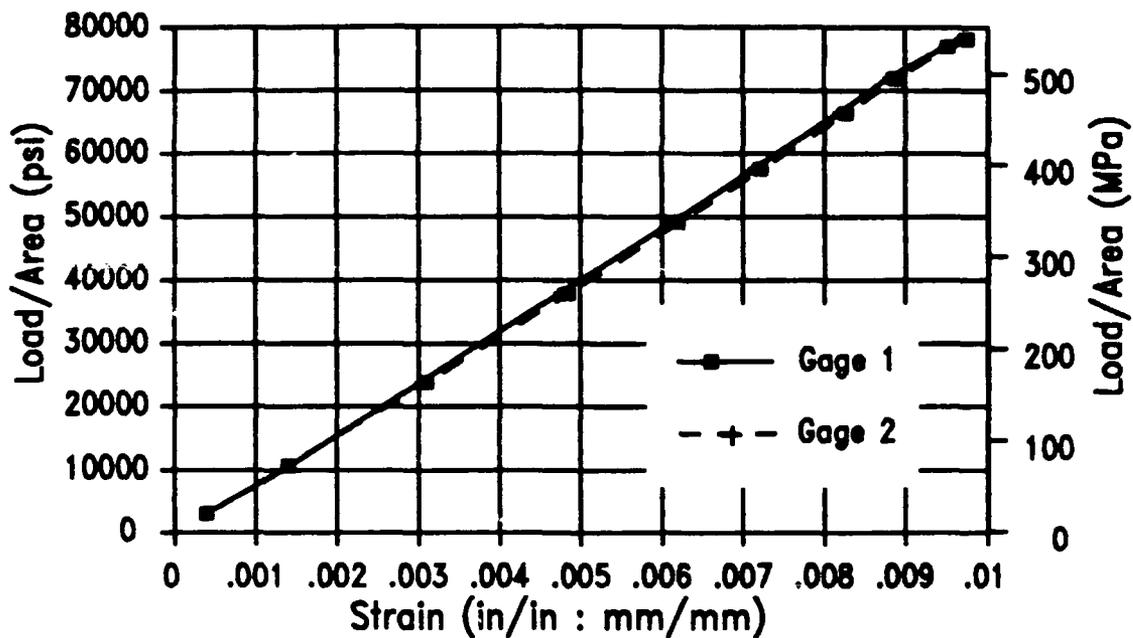


Figure 12. Stress vs. Strain Curve for Test Sample A

The load which applies this stress to the laminate is the 'ultimate load' used to normalize the data in the Stress Ratio column of Table II. For 0.247" (0.627 cm) thick samples, it is approximately 28400lb, (126300 N). For 0.370" (0.939 cm) thick samples, 42500lb, (189000 N).

It was found that the mass losses achieved on the samples used in tests 1 to 9 were too small to allow accurate measurements. The analytic balances available could weigh samples less than 160 grams to within 0.1 milligram, but heavier samples had to be weighed on a device with only 0.1 gram resolution. The 2" (5.08 cm)

**TABLE II.**  
**Laser Interaction Parameters and Results**

Shot #	POWER on TARGET (kW)	IRRADIANCE $\left(\frac{\text{kW}}{\text{cm}^2}\right)$	TENSILE LOAD		STRESS RATIO	LASER TIME (sec)	MASS LOSS (gm)	$Q_{\text{mass}}^*$ $\left(\frac{\text{kJ}}{\text{gm}}\right)$
			(lbm)	(kN)				
10	31.77	13.6	0	0	0.00	2.02	1.95	32.99
11	30.64	13.1	9400	41.8	0.22	2.02	1.98	31.12
12	30.73	13.2	14800	65.8	0.35	1.77	1.82	29.95
13	32.72	14.0	19000	84.5	0.44	1.43	1.56	30.07
14*	31.68	13.6	18700	83.1	0.44	1.61	1.71	29.90
15	31.60	13.5	14400	64.1	0.34	1.76	1.81	30.65
16	31.60	13.5	9500	42.3	0.33	1.51	1.50	31.89
17	31.77	25.9	0	0	0.00	2.02	2.28	28.13
18	32.72	26.7	0	0	0.00	1.27	1.45	28.64
19	31.25	25.5	5300	23.6	0.12	1.02	1.07	29.84
20	31.51	25.7	4500	20.0	0.11	1.01	1.09	29.30
21	31.68	25.9	5000	22.2	0.17	0.63	0.63	31.83
22	31.68	25.9	9700	43.2	0.23	1.03	1.17	27.99
23	31.51	25.7	9800	43.6	0.23	1.11	1.42	24.61
24	31.25	25.5	14600	64.9	0.34	1.24	1.45	26.72
25*	32.38	26.4	19000	84.5	0.44	1.02	1.45	22.76
26	32.03	26.1	15000	66.7	0.35	1.02	1.18	27.59
27	31.60	25.8	4700	20.9	0.16	0.78	0.80	30.96
28	31.68	15.8	0	0	0.00	2.01	1.95	32.64
29	31.25	15.6	0	0	0.00	1.27	1.20	33.15
30	31.51	15.7	2800	12.5	0.10	1.01	0.94	33.68
31	30.99	15.5	2900	12.9	0.07	1.76	1.80	30.16
32	31.25	15.6	5300	23.6	0.12	1.51	1.48	31.92
33	32.29	16.1	7600	33.8	0.18	1.52	1.50	32.74
34	31.77	15.9	9700	43.2	0.23	1.28	1.23	32.93
35	31.60	15.8	11900	52.9	0.28	1.01	0.95	33.56
36	31.77	15.9	14500	64.5	0.34	1.01	1.06	30.09

\* Samples #14 and #25 broke in half during testing

wide samples used for the first 9 tests were all greater than 160 grams. The 0.1 gram resolution was more than 10% of the achieved mass losses. This poor accuracy makes the data unusable, and the values won't be reported here.

The first column in Table II, Power on Target (kW) was found by taking power out of the EDCL-II device, measured as described in Chapter III, and plotted on a laser printer. The laser plot was then redigitized to record the final value representative of the total laser power. This number was multiplied by 0.868. This number was supplied by the WL/TALE personnel as the factor of mirror losses.

$$P_{\text{target}} = P_{\text{device}} * 0.868.$$

This number has an uncertainty of  $\pm 15\%$ , a typical value accepted by the laser effects research community for ballistic calorimeters. The next column, Irradiance ( $\text{kW}/\text{cm}^2$ ) is obtained by measuring the major and minor axes of the burn spot, with an uncertainty of  $\pm 0.05$  cm, calculating the burn area, and dividing this area into the Power on Target.

$$I = \frac{P_{\text{target}}}{\pi/4 (d_{\text{minor}})(d_{\text{major}})}$$

The tensile load from the load cell is also plotted on a laser printer and redigitized. Definite bounds on the uncertainty associated with this number are unavailable. The stress ratio was found by taking the load and dividing by the calculated fracture load for the sample.

Laser time on target, obtained from an infrared detector viewing a mirror down the beam train from the laser shutter, was also plotted and redigitized. This uncertainty is no larger than the resolution of placing the digitizer cursor on the plot. The uncertainty in time values will therefore be ignored.

Mass loss values came from weighing the samples on a analytic balance before and after testing. Uncertainty here is  $\pm 0.2$  milligrams. This is less than 0.06 percent for all samples reported and will also be ignored.

The effective enthalpy of ablation becomes

$$Q_{\text{eff}}^* = \frac{\text{Energy}}{\text{mass loss}} = \frac{P_{\text{target}} \cdot t}{\Delta m}$$

$$= \frac{P_{\text{target}} (\pm 15\%) \cdot t (\pm 0\%)}{\Delta m (\pm 0\%)}$$

$$Q_{\text{eff}}^* = 15\% + 0\% + 0\% = \pm 15\%$$

Although this uncertainty is larger than the ordinate change for the 15 kW/cm<sup>2</sup> data, this will not invalidate the

conclusions stated in Chapter VI. For the data taken at 26 kW/cm<sup>2</sup>, changes larger than 15 percent are seen.

As stated in Chapter II, the effective enthalpy of ablation,  $Q_{\text{eff}}^*$ , is based upon the energy delivered to the target. Energy lost to the surroundings will not contribute to material ablation, making the value, of  $Q_{\text{eff}}^*$  higher than the actual enthalpy value of the ablating material. The most significant loss mechanism for material ablating at 3800°K will be reradiation of energy away from the target and conduction away from the ablation surface. This first value may be approximated by calculating the energy flux of a body at temperature T with an emissivity  $\epsilon$  (assumed to be equal to the absorptivity  $\alpha$ )

$$q = \epsilon \sigma T^4 = \alpha \sigma T^4$$

$$= 0.9 \left[ 5.67 \times 10^{-8} \frac{\text{W}}{\text{m}^2 \text{ } ^\circ\text{K}^4} \right] \left[ 3800 \text{ } ^\circ\text{K} \right]^4$$

where  $\sigma$  is the Stefan-Boltzman constant

$$= 10.6 \times 10^6 \text{ W/m}^2$$

$$= 1.06 \text{ kW/cm}^2 \approx 8\% \cdot \left( 13 \text{ kW/cm}^2 \right)$$

$$\approx 4\% \cdot \left( 26 \text{ kW/cm}^2 \right)$$

The second loss, conduction away from the ablation surface, will be estimated as conduction into the depth of the sample and radially from the ply being ablated. From Figure B6, the average thermal gradient through the thickness is approximately  $1.5 \times 10^5$  °K/cm at the ablation surface. From Figure A2,  $K_{ss} \approx 1$  W/m·K.

$$\begin{aligned}
 q_{\text{cond}} &= K \nabla T \\
 &= 1 \frac{\text{W}}{\text{m} \cdot \text{K}} \cdot 1.5 \times 10^5 \frac{\text{K}}{\text{cm}} \\
 &= 1.5 \text{ kW/cm}^2 \\
 &\approx 12\% \cdot \left[ 13 \text{ kW/cm}^2 \right] \\
 &\approx 6\% \cdot \left[ 26 \text{ kW/cm}^2 \right]
 \end{aligned}$$

Radial conduction will also remove energy from the ablation surface. A conservative bound on this loss is estimated by taking the maximum gradient ( $1.5 \times 10^5$  °K/cm), maximum conductivity (10 W/m·°K from Figure A2) and calculating the power lost in the ply being ablated. For a 1cm diameter spot and 0.0132 cm thick plys

$$\begin{aligned}
 P_{\text{radial}} &= 10 \frac{\text{W}}{\text{m} \cdot \text{K}} \cdot 1.5 \times 10^5 \frac{\text{K}}{\text{cm}} \cdot \pi \cdot 1 \text{ cm} \cdot 0.0132 \text{ cm} \\
 &= 0.622 \text{ kW} \\
 &\approx 2\% \cdot P_{\text{laser}}
 \end{aligned}$$

Therefore, the reported values of  $Q_{\text{eff}}^*$  are too high by a figure on the order of 22 percent, compared to values computed by accounting for energy losses.

Figure 13 is a plot of the data for 26 kW/cm<sup>2</sup>. Figure 14 shows 13 kW/cm<sup>2</sup> and 16 kW/cm<sup>2</sup> data on the same plot. This was done because of the close agreement of the unloaded  $Q^*$  values at both intensities. The magnitude of the correlation coefficient of Figure 14, being higher than that of the 26 kW/cm<sup>2</sup> data alone (0.627 vs. 0.599), justifies this method of presentation.

The lines fit to Figures 13 and 14 are least squares linear regressions of  $Q_{\text{eff}}^*$  on stress ratio yielding lines of the form

$$Q_{\text{eff}}^* = A + B \cdot \frac{\sigma}{\sigma_{\text{ult}}}$$

Equations (28) and (29) are solved for  $\eta_0$  and  $\eta_1$  using the values for A and B above. It was calculated that

$$\begin{aligned} \eta_d &= \eta_0 + \eta_1 \cdot \frac{\sigma}{\sigma_{\text{ult}}} \\ &= 0.013 \frac{\text{kJ}}{\text{gm}} + 1.74 \frac{\text{kJ/gm}}{\text{GPa}} \sigma \end{aligned}$$

for data taken at 15kW/cm<sup>2</sup>.

$$\eta_d = 0.087 \frac{\text{kJ}}{\text{gm}} + 3.36 \frac{\text{kJ/gm}}{\text{GPa}} \sigma$$

for data taken at 26 kW/cm<sup>2</sup>.

The small value of  $\eta_0$  in both cases raises the question of its significance. To discern this, the

average value of  $Q_{\bullet ff}^*$  taken at zero load was subtracted from each data set. A straight line was fit through the origin, and the sums of the residuals squared associated with each fit were compared to those from the two parameter slope, B.

For 15 kW/cm<sup>2</sup>

$$Q_{\bullet ff}^* = 32.96 \frac{\text{kJ}}{\text{gm}} - 5.80 \frac{\text{kJ}}{\text{gm}} \cdot \left( \frac{\sigma}{\sigma_{ult}} \right)$$

$$\text{residuals squared} = 18.2 \left( \frac{\text{kJ}}{\text{gm}} \right)^2$$

for one parameter fit

$$\left( Q_{\bullet ff}^* - 32.93 \frac{\text{kJ}}{\text{gm}} \right) = -5.71 \frac{\text{kJ}}{\text{gm}} \cdot \left( \frac{\sigma}{\sigma_{ult}} \right)$$

$$\text{residuals squared} = 23.0 \left( \frac{\text{kJ}}{\text{gm}} \right)^2$$

For 26 kW/cm<sup>2</sup>

$$Q_{\bullet ff}^* = 30.23 \frac{\text{kJ}}{\text{gm}} - 11.2 \frac{\text{kJ}}{\text{gm}} \cdot \left( \frac{\sigma}{\sigma_{ult}} \right)$$

$$\text{residuals squared} = 44.7 \left( \frac{\text{kJ}}{\text{gm}} \right)^2$$

and for one parameter fit

$$\left( Q_{\bullet ff}^* - 32.00 \frac{\text{kJ}}{\text{gm}} \right) = -17.30 \frac{\text{kJ}}{\text{gm}} \cdot \left( \frac{\sigma}{\sigma_{ult}} \right)$$

$$\text{residuals squared} = 55.7 \left( \frac{\text{kJ}}{\text{gm}} \right)^2$$

Fitting a one parameter fit to the data to account for an offset,  $\eta_o$ , does not give any better representation of the data collected here.

Figures 15a through 15g show representative photographs of the test samples. In addition to ply

separation and loose fibers hanging from the ablation crater surfaces, global structural damage is obvious at the higher load levels. This type of damage obviously contributed (in an unknown degree) to the mass loss enhancement. Macroscopic fiber removal occurred during many of the tests, as shown by fiber bundles seen on the floor of the test area. A typical bundle would be on the order of 2mm wide by 5mm to 7mm long. No attempt was made to systematically collect these bundles, but many were found downstream of the nitrogen sample purge.

Figures 16a and 16b show scenes from the video tape taken during the irradiation of sample #5. The darker bands and regions are areas of lower temperature. The orientation of these bands, which coincide with ply orientation, is taken as evidence of discrete ply removal. A solid ply breaking off and being carried away by the purge gas would expose a ply beneath it at lower temperature. This was seen in MELT3D.FOR predictions, and has been observed in other laser effects tests. If all plies were ablated entirely, without discrete removal, new plies will not be exposed until the layer beneath had heated up to the ablation temperature, and no darker bands would be visible on film or video.

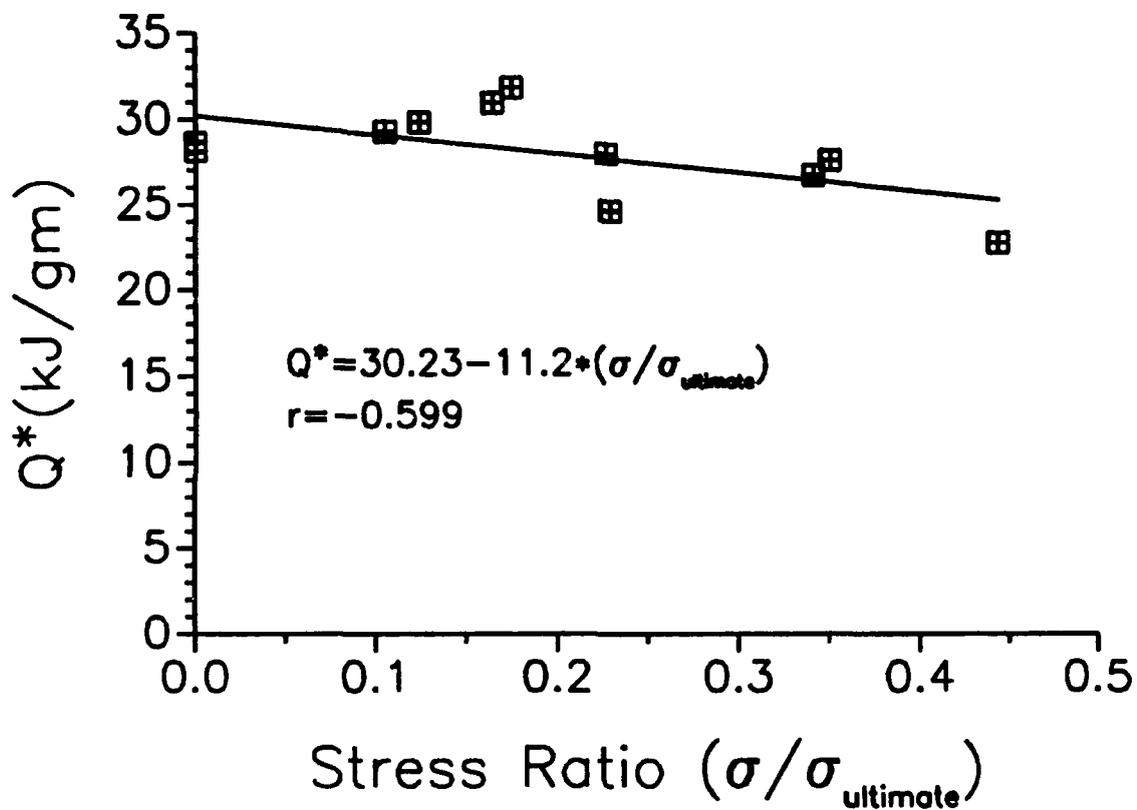


Figure 13.  $Q^*$  versus Stress Ratio for 26 kW/cm<sup>2</sup>

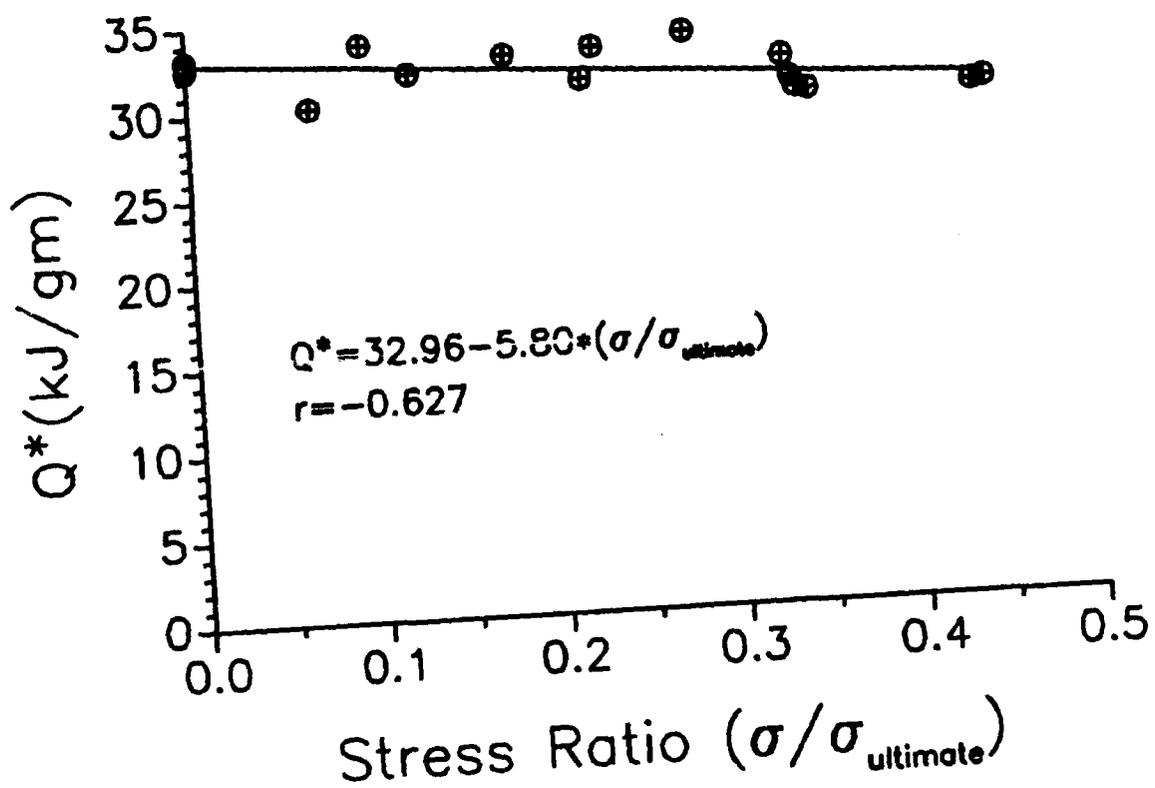


Figure 14.  $Q^*$  versus Stress Ratio for 13 & 16 kW/cm<sup>2</sup>



(a) Shot #10;  $I = 13 \text{ kW/cm}^2$ ; Stress Ratio = 0.0

Figure 15. Representative Test Sample Photographs

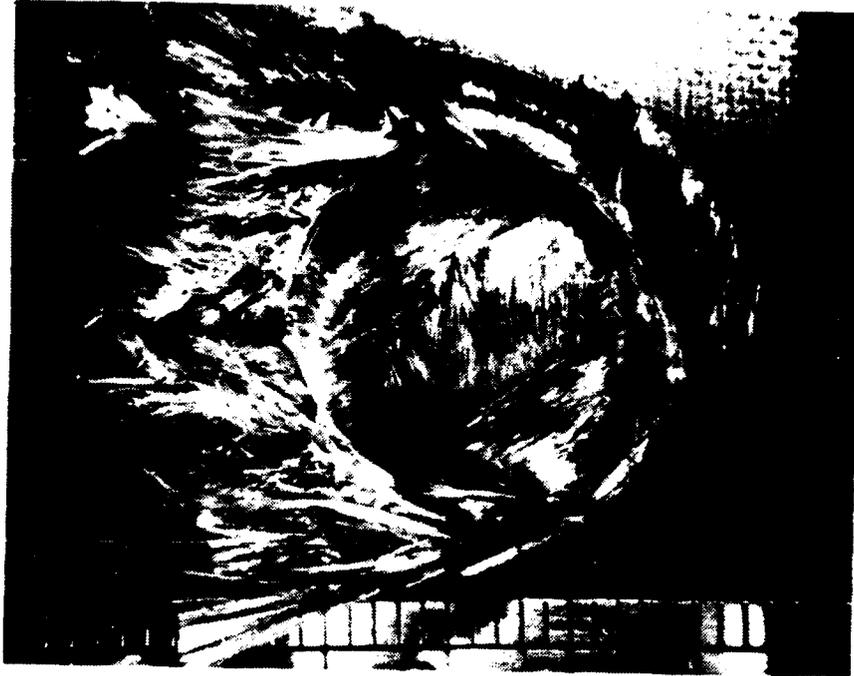


(b) Shot #11;  $I = 13 \text{ kW/cm}^2$ ; Stress Ratio = 0.22



(c) Shot #12;  $I = 13 \text{ kW/cm}^2$ ; Stress Ratio = 0.35

Figure 15. Representative Test Sample Photographs (Cont'd)

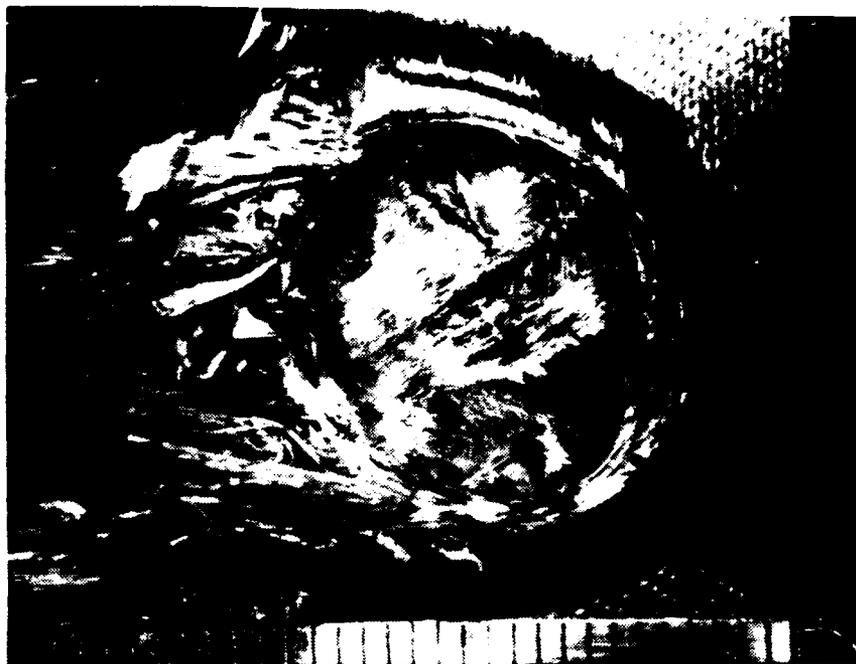


(d) Shot #13;  $I = 13 \text{ kW/cm}$  ; Stress Ratio = 0.44

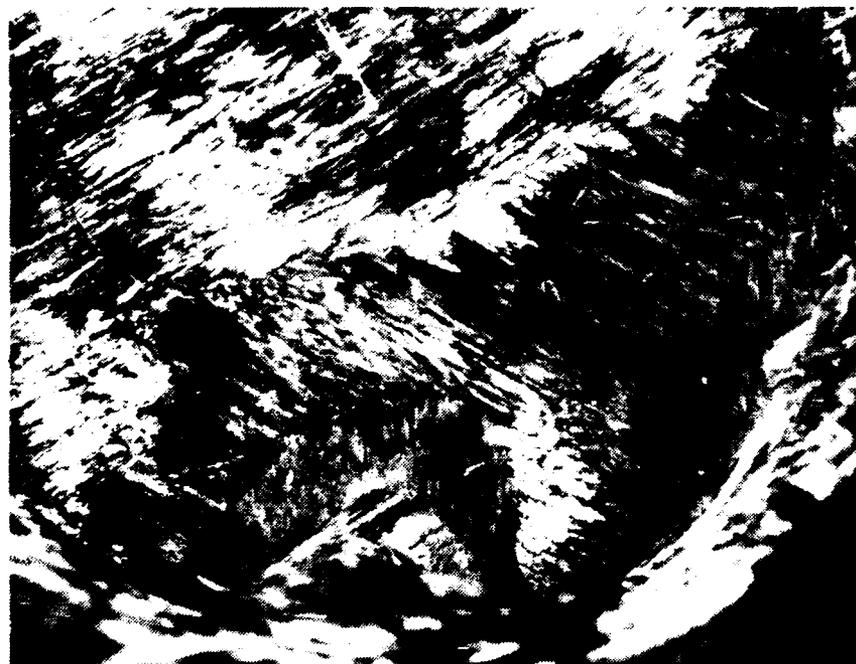


(e) Shot #13; Close Up

Figure 15. Representative Test Sample Photographs (Cont'd)

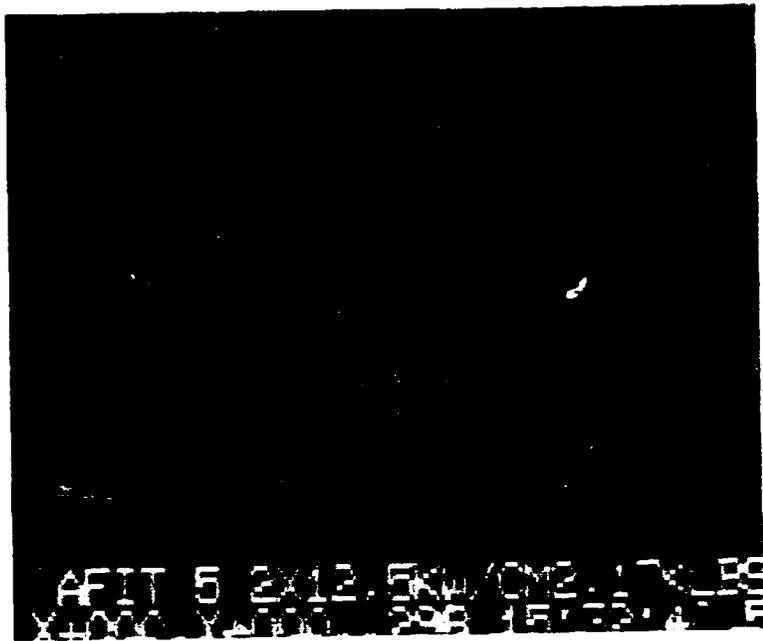


(f) Shot #15;  $I = 13 \text{ kW/cm}^2$ ; Stress Ratio = 0.34

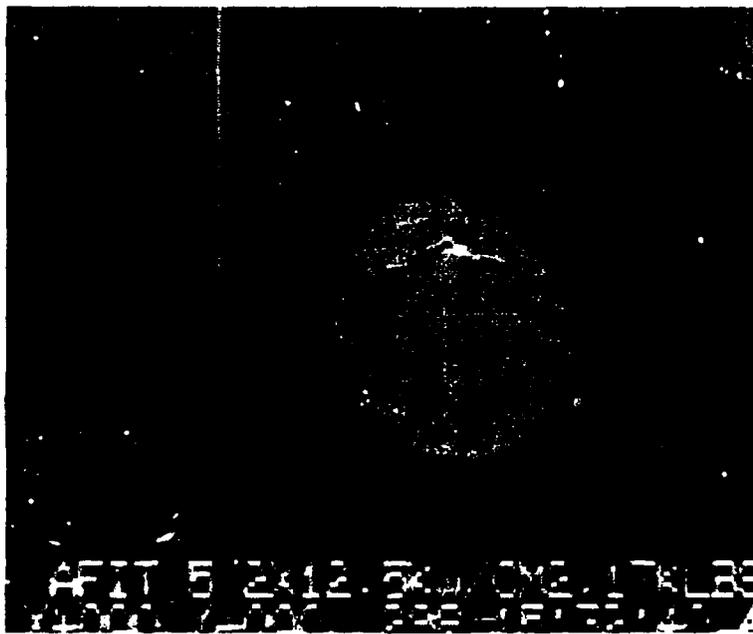


(g) Shot #15; Close Up

Figure 15. Representative Test Sample Photographs (Concl'd)



(a)



(b)

Figure 16. Video Camera Images of Shot #5

## VI. Conclusions and Recommendations

As Figures 13 and 14 show, there is a slight trend toward decreasing  $Q^*$  at higher stresses. The data indicate that at stress ratios explored, only a 9% decrease in  $Q^*$  was seen at 13-16 kW/cm<sup>2</sup> and 19.8% decrease at 26 kW/cm<sup>2</sup>.

This shows that the initial postulate is essentially correct. That is, discrete fiber removal occurs during ablation. The removal by a fracture mechanism which appears to be a linear function of applied stress (or some linear mechanism) also appears to be verified by the experimental data collected.

Therefore it is concluded that graphite/epoxy will ablate at rates that increase linearly with applied axial stress. This will continue until the applied load is so high that the tensile specimen will fracture due to the creation of an initial flaw by burning off one or two plies on the surface (12). One sample did this during the tests at LHMEI-1 (Appendix D) and two samples broke midway during irradiation (likely at the point when the cross section became too small to carry the applied load) in the EDCL-II test series.

No specific experimental efforts in this area are recommended for the immediate future. For the stress ratios seen here, ablation efficiency increase is small

enough that it may be neglected for many engineering purposes, and the unloaded ablation enthalpy values may be used.

More sophisticated analytic modeling might be undertaken to explore what factors might be affecting the particular size of broken fibers. The work might yield some insight to the rise in  $Q_{\text{eff}}^*$  seen at  $16 \text{ kW/cm}^2$  between  $\sigma/\sigma_{\text{ult}} = 0.12$  and  $0.28$ .

It might also prove useful to explore the change in stiffness of a laminate as a cylindrical hole is ablated from the sample. Integrating the mass loss as a function of instantaneous stress (instead of assuming constant stress) is a likely next step.

If any further experiments are conducted, collection of the ablated particles must be attempted. Differing ply layups may also be interesting to explore. More care needs to be taken when designing test samples; as mentioned in Chapter IV, analytic balances capable of weighing larger masses are more coarse. It is imperative to build samples light enough to have their mass loss weighed to high precision. A conflicting desire, unfortunately, is the recommendation to test samples much wider than the laser beam diameter, to eliminate effects of the laminate's edges.

More sophisticated modeling of the boundary conditions on a fiber might be undertaken; specifically the effect, if

any, of layers above the plane of the ablation surface upon the fracture characteristics of the fiber. Also, a less empirical model of  $\eta_d$  might be found by measuring the distribution of initial cracks in the virgin fiber, and using that distribution to account for different lengths of removed fibers.

Appendix A: MELT3D.FOR, Three Dimensional Heat  
Conduction and Ablation Program

This appendix shows the flow of logic for this program in Figure A1. A listing of the code follows. Graphs showing the temperature dependence of thermal properties computed in this program are given in Figure A2. A listing of the subroutines called by MELT3D.FOR concludes the appendix.

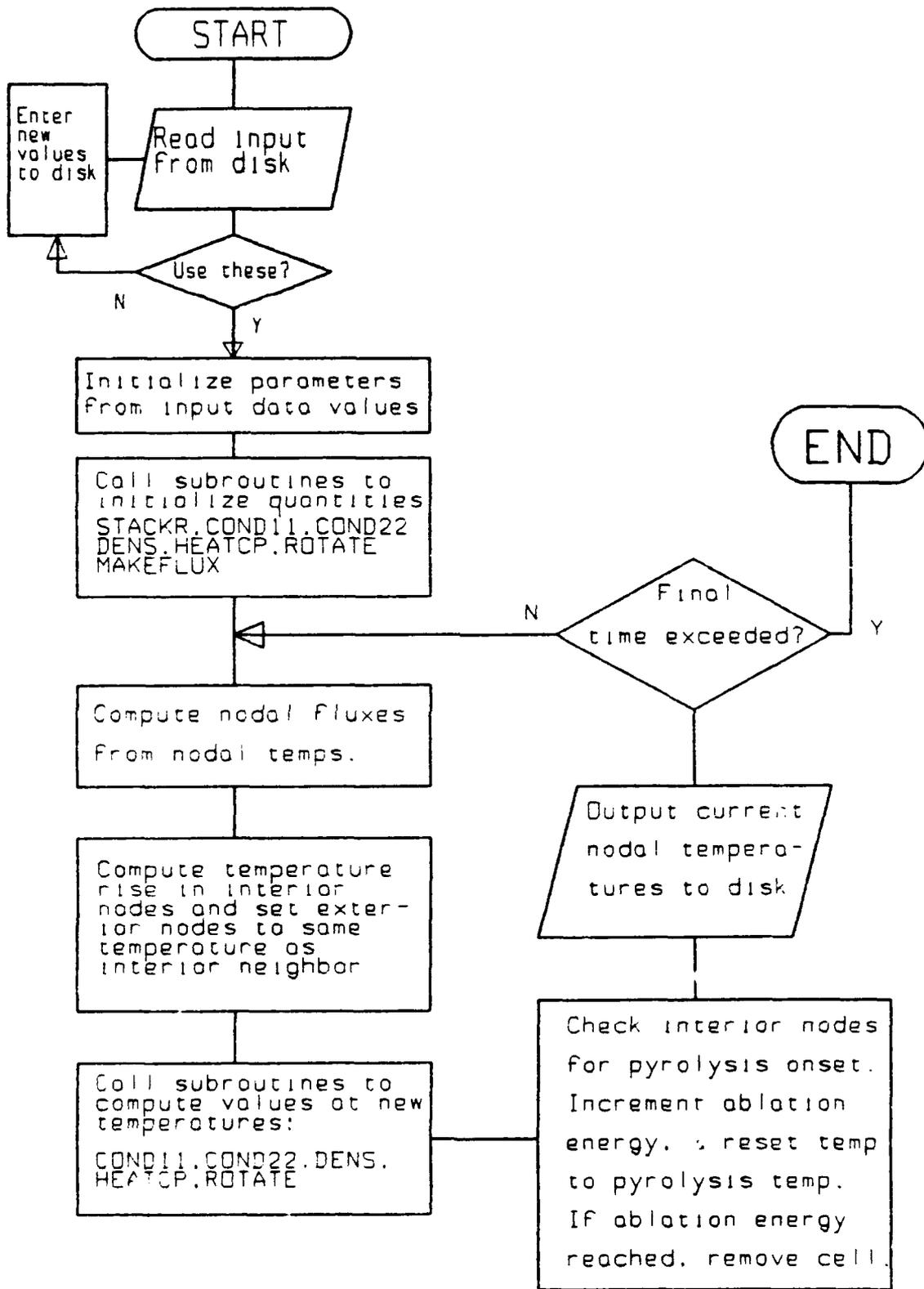


Figure A1. Flowchart for MELT3D.FOR



```

C      QY(I,J,K) = FLUX IN POSITIVE Y DIRECTION THRU THE LEFT SIDE FACE
C      OF CUBE I,J,K (Watts)
C      QZ(I,J,K) = FLUX IN POSITIVE Z DIRECTION (DOWN) THRU TOP FACE OF
C      CUBE I,J,K (Watts)
C      FLUX(I,J,K) = LASER FLUX INTO TOP FACE OF CUBES EXPOSED TO THE
C      BEAM (Watts; [= IRRAD(I,J,K)*DX*DY])
C      RK11(I,J,K,) = CONDUCTIVITY TENSOR ELEMENT 1,1 AT I,J,K,(W/m*K)
C      RK22(I,J,K,) = CONDUCTIVITY TENSOR ELEMENT 2,2 AT I,J,K,(W/m*K)
C      RK12(I,J,K,) = CONDUCTIVITY TENSOR ELEMENT 1,2 AT I,J,K,(W/m*K)
C      RHO(I,J,K) = DENSITY OF BULK MATERIAL, kg/m**3, IN CELL I,J,K
C      SPECHEAT(I,J,K) = HEAT CAPACITY, W*s/kg*K, IN CELL I,J,K
C
C      XMAX = LENGTH OF REGION, cm
C      YMAX = WIDTH OF REGION, cm
C      ZMAX = THICKNESS OF REGION, cm
C      NUMX = NUMBER OF CELLS IN X DIRECTION
C      NUMY = "      "      "      "      "      Y      "
C      NUMZ = "      "      "      "      "      Z      DIRECTION, = (# PLYS )
C
C      DX,DY,DZ = LENGTHS OF INCREMENTAL DISTANCES, meters
C      VOL = DX*DY*DZ , I.E. VOLUME OF INCREMENTAL CELL
C
C      COND??(TEMP(I,J,K),RK?) = SUBROUTINES TO
C      COMPUTE TEMPERATURE VARIATION IN PRINCIPLE CONDUCTIVITIES
C      RK1 AND RK2
C      DENS(TEMP,RHO)          = SUBROUTINE TO COMPUTE RHO(TEMP)
C      HEATCP(TEMP,SPECHEAT) = "      "      "      "      SPECHEAT(TEMP)
C      MAKEFLUX = SUBROUTINE TO COMPUTE IRRAD VALUES AND EXPOSED CELLS
C
C*****
C*****
C      INTEGER FLAG(10,10,10)
C      CHARACTER*70 LABTXT
C      CHARACTER*10 OUTFILE
C      REAL IRRAD
C
C      DIMENSION TEMP(10,10,10), FLUX(10,10,10), RK11(10,10,10)
C      DIMENSION RK12(10,10,10),RK22(10,10,10),NSTACK(100)
C      DIMENSION RHO(10,10,10), SPECHEAT(10,10,10)
C      DIMENSION SLUSH(10,10,10), QX(10,10,10)
C      DIMENSION QY(10,10,10),QZ(10,10,10),RK33(10,10,10)
C
C      COMMON /DATA1/ IRRAD,ABSORB,XMAX,YMAX,ZMAX,NUMX,NUMY,NUMZ,
C      &RK1MAX,RK2MAX,RHOMIN,CPMIN,HABLAT,TIMEMX,TINITL,SAFE
C      COMMON /DATA2/ RADIUS, XFLUX, YFLUX, TPYRO ,ALPHA,ITYPE,IPRN,
C      &IXOUT,IYOUT,IZOUT,OUTFILE,LABTXT
C
C*****
C*****
C      ENTER DATA DIALOG TO READ LAST RUNS DATA INPUT, AND ALLOW
C      ANY CHANGES DESIRED

```

```

C
WRITE(*,*) 'ENTER 0 TO CONTINUE WITH FILE DATA, 1 TO ENTER CORREC
&T DATA INTERACTIVELY'
READ(*,'(I1)') LOOP

C
CALL DATALK(LOOP)
CALL FLXTLK(LOOP)
C*****
C OPEN FILE TO OUTPUT DATA TO, BLANK FILE NAME FORCES DOS TO
C PROMPT FOR IT, UNIX NEEDS EXPLICIT NAME
C*****
C
OPEN (UNIT=12,FILE = OUTFILE ,STATUS='NEW')
WRITE(12,'(A70)') LABTXT

C
C*****
C*****
C
C START COMPUTING INITIALLY NEEDED QUANTITIES
C CHANGE XMAX, IN CENTIMETERS, TO METERS
C*****
C*****
C
DX = XMAX / (100.0*FLOAT(NUMX) )
DY = YMAX / (100.0*FLOAT(NUMY) )
DZ = ZMAX / (100.0*FLOAT(NUMZ) )
VOL = DX*DY*DZ
PI = 4.000*ATAN(1.000)
ALPHA = ALPHA*PI/180.00
NGONE = 0
T = 0.00

C
C*****
C NOTE, CASE SPECIFIC STATEMENTS FOLLOW:.....
C THE MAXIMUM CONDUCTIVITY FOR STABILITY PURPOSES IS BEING
C COMPUTED FOR THE SPECIFIC CASE OF K22 = K33, AND ASSUMING THE
C MAXIMUM SUM OF K(11) IS ALWAYS LESS THAN THE TRACE OF THE
C CONDUCTIVITY MATRIX. THEREFOR , THE MAXIMUM IS BEING SET TO
C THE TRACE OF THE MATRIX, AND K22=K33.
C*****
C
CONMAX = (RK1MAX + 2.0*RK2MAX)/3.0
DT = (AMIN1(DX,DY,DZ)**2)*RHOMIN*CPMIN/(2.00*SAFE*CONMAX)
NTIME = IFIX (TIMEMX/ DT )
C*****
C THE FOLLOWING SUBROUTINE WILL INITIALIZE THE STACKING SEQUENCE OF
C THE PLYS, TO DICTATE THE ROTATION OF EACH PLY IN THE GLOBAL COORD
C SYSTEM
C*****
C
CALL STACKR(NSTACK)
C

```

```

C*****
C   INITIALIZE THE TEMPERATURE AND PROPERTIES ARRAYS
C   CHANGE HABLAT,kJ/gm, TO J/kg BY MULTIPLYING BY 10**6
C*****
C
C   DO 99, I = 1, NUMX+2
C
C   DO 99, J = 1, NUMY+2
C   DO 99, K = 1, NUMZ+2
C       TEMP(I,J,K) = TINITL
99  CONTINUE
C
C   DO 101, I = 2, NUMX+1
C   DO 102, J = 2, NUMY+1
C   DO 103, K = 2, NUMZ+1
C
C       CALL COND11( TEMP(I,J,K), RK1)
C       CALL COND22( TEMP(I,J,K), RK2,IMETAL)
C       CALL DENS  ( TEMP(I,J,K), RHO(I,J,K))
C       CALL HEATCP( TEMP(I,J,K), SPECHEAT(I,J,K))
C   SLUSH(I,J,K) = HABLAT*1.0E+06*RHO(I,J,K)*VOL
C   FLAG(I,J,K) = 1
C
C*****
C   NOW ROTATE THE CONDUCTIVITY TENSOR TO PLACE EACH LAYERS PROPER
C   TENSORIAL QUANTITIES INTO ITS ARRAY INITIALLY
C
C       CALL ROTATE( RK11(I,J,K), RK22(I,J,K), RK12(I,J,K), RK33(I,J,K),
C & NSTACK(K), RK1,RK2, ALPHA, IMETAL)
C
103  CONTINUE
102  CONTINUE
101  CONTINUE
C
C*****
C*****
C   NEXT, INITIALIZE THE FLUX DISTRIBUTION ON THE SAMPLE; THIS
C   IS AGAIN CASE SPECIFIC, AND IS DETERMINED BY THE SUBROUTINE
C   LINKED AS 'MAKEFLUX', WHICH GETS ITS VALUES FROM THE COMMON
C   BLOCKS
C*****
C*****
C
C       CALL MAKEFLUX (FLUX)
C
C*****
C*****
C   NOW ITS TIME TO ACTUALLY INCREMENT THE TIME AND LOOP THRU
C   THE REGION, SUMMING HEAT BALANCES ON EACH CUBE DX*DY*DZ
C

```

```

C      NOTE!!!!!!!
C      IF ANY INDEX IS 1 or ?MAX, THE TEMP THERE IS SET EQUAL TO
C      INDEX=2 OR ?MAX-1, SO THAT NO FLUX, I.E. INSULATED BOUNDARY,
C      IS ENFORCED.
C
C      NOTE ALSO, THE FLUXES ARE SET TO ZERO IF THE NODE CALCULATED
C      HAS ABLATED OR MELTED AWAY, OR IF THE CELL INTO WHICH IT'S
C      HEAT WOULD PASS IS MISSING!!!!!!
C*****
C*****
C
C      DO 200, N = 1, NTIME
C      T = FLOAT(N) * DT
C*****
C      DO 201, I = 1, NUMX+1
C      DO 202, J = 1, NUMY+1
C      DO 203, K = 1, NUMZ+1
C
C          QX(I,J,K) = FLOAT(FLAG(I,J,K)*FLAG(I+1,J,K))*((RK11(I,J,K)*
&      (TEMP(I,J,K) - TEMP(I+1,J,K))*DY*DZ/DX ) + ( RK12(I,J,K)*
&      (TEMP(I,J,K) - TEMP(I,J+1,K)) * DZ ))
C
C          QY(I,J,K) = FLOAT(FLAG(I,J,K)*FLAG(I,J+1,K))*((RK12(I,J,K)*
&      (TEMP(I,J,K) - TEMP(I+1,J,K)) * DZ ) + ( RK22(I,J,K) *
&      (TEMP(I,J,K) - TEMP(I,J+1,K)) * DZ*DX/DY))
C
C          QZ(I,J,K) = FLOAT(FLAG(I,J,K)*FLAG(I,J,K+1))* ( RK33(I,J,K)*
&      (TEMP(I,J,K) - TEMP(I,J,K+1)) * DY*DX/DZ )
C
203    CONTINUE
202    CONTINUE
201    CONTINUE
C
C*****
C*****
C*****
C      NOW LOOP OVER THE INTERIOR OF THE ENTIRE MATERIAL CUBE, WHICH
C      EXCLUDES THE PERIPHERIAL NODES, WHICH ONLY EXIST TO ENFORCE THE
C      BCs.
C      THE INTERIOR, HOWEVER, SIMPLY IS THE CONDUCTION PROBLEM,
C      UNLESS CELLS ARE REMOVED AND THEN LASER FLUX IS APPLIED TO THE
C      CELL WHICH USED TO BE BENEATH IT.
C*****
C*****
C
C      DO 210, I = 2, NUMX+1
C      DO 211, J = 2, NUMY+1
C      DO 212, K = 2, NUMZ+1
C
C          EIN = FLUX(I,J,K)*DX*DY + QX(I-1,J,K) - QX(I,J,K) + QY(I,J-1,K)
&      -QY(I,J,K) + QZ(I,J,K-1) - QZ(I,J,K)

```

```

C
  DELTAT = EIN*DT/ (RHO(I,J,K) * SPECHEAT(I,J,K)* VOL)
  TEMP(I,J,K) = TEMP(I,J,K) + DELTAT
C
212  CONTINUE
211  CONTINUE
210  CONTINUE
C
C*****
C*****
C  NOW GO AROUND EACH BOUNDARIES AND SET THE TEMPERATURE ON EACH
C  EDGE NODE EQUAL TO THE TEMPERATURE JUST INSIDE FROM IT, SO
C
C  THAT ON SUCH BOUNDARIES, THERE WILL BE NO FLUX THROUGH
C  TO THE INTERIOR
C*****
C
  DO 220, I = 1, NUMX+2
  DO 221, J = 1, NUMY+2
    TEMP(I,J,1) = TEMP(I,J,2)
    TEMP(I,J,NUMZ+2) = TEMP(I,J,NUMZ+1)
221  CONTINUE
220  CONTINUE
C
  DO 230, I = 1, NUMX+2
  DO 231, K = 1, NUMZ+2
    TEMP(I,1,K) = TEMP(I,2,K)
    TEMP(I,NUMY+2,K) = TEMP(I,NUMY+1,K)
231  CONTINUE
230  CONTINUE
C
  DO 240, J = 1, NUMY+2
  DO 241, K = 1, NUMZ+2
    TEMP(1,J,K) = TEMP(2,J,K)
    TEMP(NUMX+2,J,K) = TEMP(NUMX+1,J,K)
241  CONTINUE
240  CONTINUE
C*****
C*****
C
C  NOW THAT THE NEW TEMPERATURES HAVE BEEN COMPUTED, RECALCULATE
C  THE PROPERTIES IN THE REGION
C  AGAIN, IF IMETAL IS 1, SKIP ANISOTROPIC TENSOR, SET
C  RK11=RK1=RK22=RK33 AND RK12=0
C*****
C*****
  DO 301, I = 2, NUMX+1
  DO 302, J = 2, NUMY+1
  DO 303, K = 2, NUMZ+1
C
  CALL COND11( TEMP(I,J,K), RK1)
  CALL COND22( TEMP(I,J,K), RK2, IMETAL)
  CALL DENS ( TEMP(I,J,K), RHO(I,J,K))

```

```

      CALL HEATCP( TEMP(I,J,K), SPECHEAT(I,J,K))
C
      CALL ROTATE( RK11(I,J,K), RK22(I,J,K), RK12(I,J,K), RK33(I,J,K),
& NSTACK(K), RK1,RK2, ALPHA, IMETAL)
C
C*****
C   IF THIS NODE IS NOT AT THE PYROLYSIS TEMPERATURE, GO ON TO THE
C   NEXT NODE.  IF IT IS AT PYROLYSIS TEMPERATURE, INCREMENT THE
C   'COUNTER' KEEPING TRACK OF THE ABLATION ENERGY BEING DUMPED INTO
C   THE CELL, AND RESET THE TEMPERATURE OF THE CELL BACK DOWN TO THE
C   PYROLYSIS TEMPERATURE, TPYRO.
C*****
C*****
C
      IF (TEMP(I,J,K) .LE. TPYRO) GOTO 303
C
      SLUSH(I,J,K) = SLUSH(I,J,K) - RHO(I,J,K)*SPECHEAT(I,J,K)*
& (TEMP(I,J,K)-TPYRO)*VOL
      TEMP(I,J,K) = TPYRO
C*****
C   IF THE 'SLUSH' ENERGY, I.E. ENERGY TO ABLATE THE WHOLE CELL
C   HASN'T BEEN REACHED, GO ON TO NEXT CELL.  IF IT HAS, REMOVE IT
C   PLACE IT'S FLUX ON THE NEXT CELL
C*****
C
      IF (SLUSH(I,J,K) .GT. 0.00) GOTO 303
      FLAG(I,J,K) = 0
      FLUX(I,J,K+1) = FLUX(I,J,K)
      FLUX(I,J,K) = 0.000
      TEMP(I,J,K) = 0.000
C*****
C   INCREMENT COUNTER TO SEE HOW MANY CELLS ARE REMOVED
C*****
      NGONE = NGONE + 1
C*****
C   LINES TO OUTPUT TO FILE AND TO THE SCREEN
C
C
      WRITE(12,800) 'CR',I,J,K,T
      WRITE(*,800) 'CR',I,J,K,T
C
C
303  CONTINUE
302  CONTINUE
301  CONTINUE
C*****
C
      IF ( MOD (N,IPRN) .EQ. 0) THEN

      WRITE(12,(''TI'',F10.5)) T

```

```

DO 310, I = 2, NUMX+1, IXOUT
DO 310, J = 2, NUMY+1, IYOUT
DO 310, K = 2, NUMZ+1, IZOUT
C
Y = 100.0*DY* FLOAT(J-2)
X = 100.0*DX* FLOAT(I-2)
Z = 100.0*DZ* FLOAT(K-2)
WRITE(12, '(1X,3F10.4,F8.0)') X,Y,Z,TEMP(I,J,K)
C
310 CONTINUE
END IF
C
C
IF ( MOD (N,ITYPE) .EQ. 0) THEN
WRITE(*,803) T
WRITE(*,801) (100.0*FLOAT(J-2)*DY, J=NUMY+1,2,-IYOUT)

DO 410, K = 2, NUMZ+1, IZOUT
WRITE(*,*) 'PLY LAYER # ',K,' DEPTH= ', 100.0*FLOAT(K-2)*DZ,'CM'
DO 411, I = 2, NUMX, IXOUT
X = 100.0*FLOAT(I-2)*DX
WRITE(*,802) X,(TEMP(I,J,K), J = NUMY+1,2,-IYOUT)
411 CONTINUE
410 CONTINUE
END IF
C*****
C
C
200 CONTINUE
C
CALL DENS (TINITL, RHOMAX)
DELTAM = RHOMAX * VOL * NGONE * 1.0E03
WRITE(*,*) 'RHOMAX,VOL,NGONE = ', RHOMAX,VOL,NGONE
WRITE(*,*) 'MASS LOSS IS ', DELTAM
C
WRITE(12,*) 'NGONE = ',NGONE
WRITE(12,*) 'MASS REMOVED = ', DELTAM
800 FORMAT (1X,A2,3I4,F10.5,'CELL IJK REMOVED AT TIME T')
801 FORMAT(1X,'X / Y>',100F7.2)
802 FORMAT(1X,F5.1,100F7.1)
803 FORMAT('1','TIME IS ',F10.6,' SECONDS')

STOP
END

```

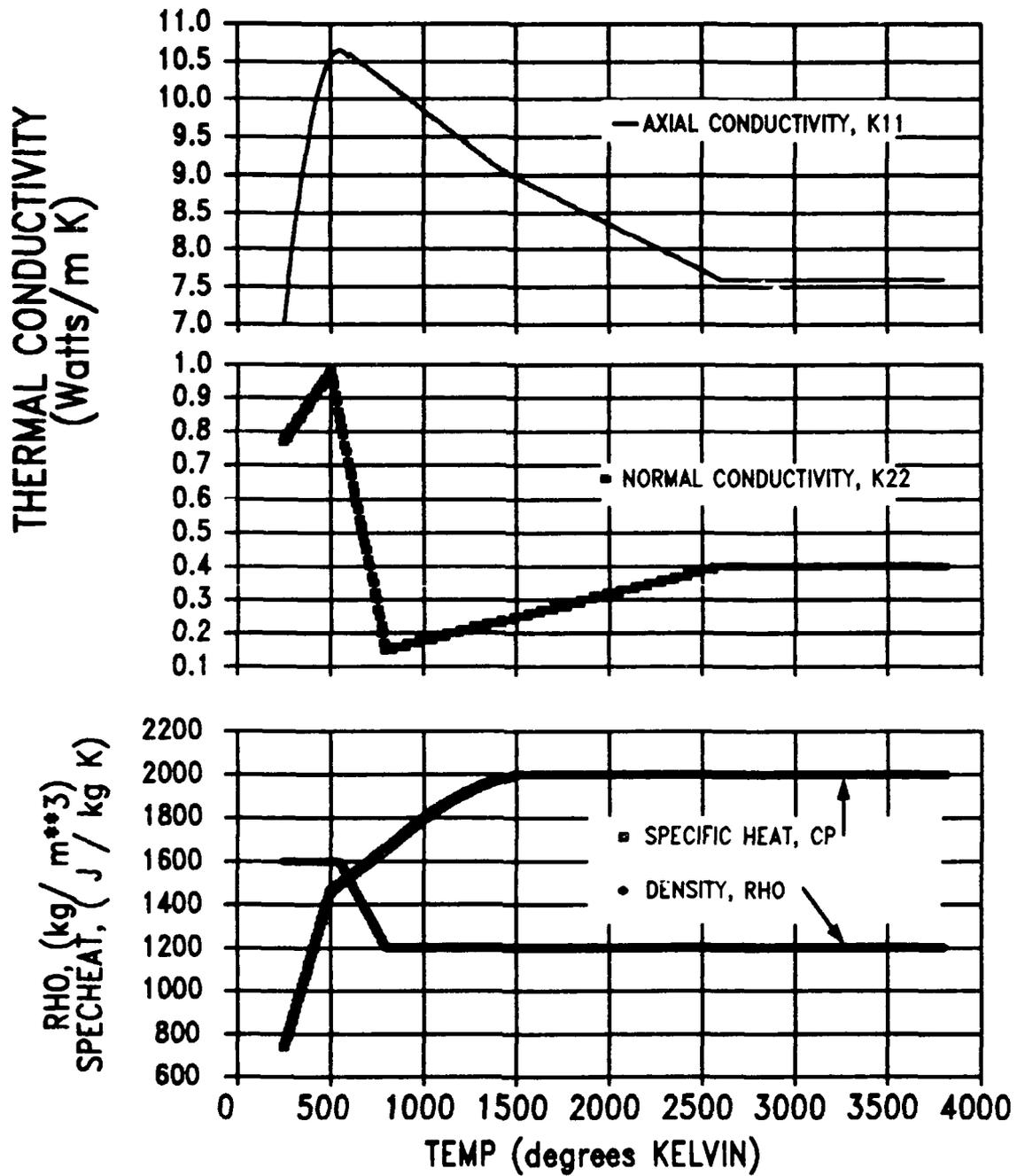


Figure A2. Values of Thermal Properties Used in MELT3D.FOR

```

SUBROUTINE COND11(T, RK1)
C*****
C   SUBROUTINE TO COMPUTE PRINCIPLE (AXIAL) CONDUCTIVITY, K11, AS A
C   FUNCTION OF TEMPERATURE, T, IN DEGREES KELVIN, BETWEEN 250K AND
C   3800K. CONDUCTIVITY, K11 IS RETURNED IN WATTS/M*K
C*****
C*****
      IF (T .GE. 2600) THEN
         RK1 = 7.6
      ELSE IF (T .GE. 1400) THEN
         RK1 = 10.85 - 1.25E-03 * T
      ELSE IF (T .GE. 600) THEN
         RK1 = 11.725 - 1.875E-03 * T
      ELSE IF (T .GE. 250) THEN
         RK1 = -1.46101 + T*(.04390889-3.982522E-05*T)
      ELSE IF (T .LT. 250) THEN
         RK1 = 7.0
         IERR = 1
      END IF
      RETURN
      END
C*****
SUBROUTINE COND22 (T, RK2, METAL)
C*****
C   SUBROUTINE TO COMPUTE PRINCIPLE (NORMAL) CONDUCTIVITY, K22, AS A
C   FUNCTION OF TEMPERATURE, T, IN DEGREES KELVIN, BETWEEN 250K AND
C   3800K. CONDUCTIVITY, K22 IS RETURNED IN WATTS/M*K
C*****
      IF (T .GE. 2600) THEN
         RK2 = 0.4
      ELSE IF (T .GE. 800) THEN
         RK2 = 3.88888889E-02 + 1.38888889E-04 * T
      ELSE IF (T .GE. 500) THEN
         RK2 = 2.36333333 - 2.7666667E-03 * T
      ELSE IF (T .GE. 250) THEN
         RK2 = 0.56 + 8.4E-04*T
      ELSE IF (T .LT. 250) THEN
         RK2 = 0.77
         IERR = 1
      END IF
      METAL = 0
      RETURN
      END
C*****
SUBROUTINE DENS(T,P)
C*****
C   SUBROUTINE TO COMPUTE DENSITY, P, AS A FUNCTION OF TEMPERATURE, T,
C   IN DEGREES KELVIN, BETWEEN 250K AND 3800K. DENSITY, P, IS
C   RETURNED IN kg/m**3
C*****
      IF (T .LT. 550) THEN

```

```

        P = 1600.0
ELSE IF (T .GE. 800) THEN

        P = 1200.0
ELSE IF (T .GT. 550) THEN
        P = 2480.0 - 1.6 * T
END IF
RETURN
END
C*****
SUBROUTINE HEATCP(T,CP)
C*****
C SUBROUTINE TO COMPUTE HEAT CAPACITY, CP, AS A FUNCTION OF
C TEMPERATURE,T,IN DEGREES KELVIN, BETWEEN 250K AND 3800K.
C HEAT CAPACITY, CP IS RETURNED IN JOULES / kg K
C*****
IF (T .GE. 1500) THEN
        CP = 2000.0
ELSE IF (T .GE. 850) THEN
        CP = 741.825 + T*(1.484103 - 4.316507E-04*T)
ELSE IF (T .GE. 500) THEN
        CP = 1131.4286 + 0.657142857 * T
ELSE IF (T .LT. 500) THEN
        CP = 20.0 + 2.88 * T
END IF
RETURN
END
C*****
C*****
SUBROUTINE STACKR(NSTACK)
DIMENSION NSTACK(100)
COMMON /DATA1/ IRRAD,ABSORB,XMAX,YMAX,ZMAX,NUMX,NUMY,NUMZ,
&RK1MAX,RK2MAX,RHOMIN,CPMIN,HABLAT,TIMEMX,TINITL,SAFE
C*****
C FOR A (0, +/- ALPHA)NS LAYUP, 0 MEANS 0, +1 MEANS +ALPHA,
C -1 MEANS - ALPHA
C*****
NPLYS = NUMZ
DO 10, I = 2, (NPLYS/2)+1
        IF (MOD(I-2,3) .EQ. 0) THEN
                NSTACK(I) = 0
        ELSE IF (MOD(I-2,3) .EQ. 1 ) THEN
                NSTACK(I) = 1
        ELSE IF (MOD(I-2,3) .EQ. 2) THEN
                NSTACK(I) = -1
        END IF
10 CONTINUE
C
DO 20, I = NPLYS/2 + 2, NPLYS+1
        J = (I - (NPLYS/2 +1) ) * 2 - 1
        NSTACK(I) = NSTACK(I-J)
20 CONTINUE
RETURN

```

```

END
C*****
SUBROUTINE FLXTLK (LOOP)
C
CHARACTER*70 LABTXT
CHARACTER*10 OUTFILE
COMMON /DATA1/ IRRAD, ABSORB, XMAX, YMAX, ZMAX, NUMX, NUMY, NUMZ,
&RK1MAX, RK2MAX, RHOMIN, CPMIN, HABLAT, TIMEMX, TINITL, SAFE
C
COMMON /DATA2/ RADIUS, XFLUX, YFLUX, TPYRO, ALPHA, ITYPE, IPRN,
&IXOUT, IYOUT, IZOUT, OUTFILE, LABTXT
C
OPEN (UNIT=11, FILE='3DINPUT2.DAT', STATUS='OLD')
READ (11, *) RADIUS, XFLUX, YFLUX, TPYRO, ALPHA, ITYPE, IPRN, IXOUT,
&IYOUT, IZOUT
READ (11, '(A10,A70)') OUTFILE, LABTXT
C
DX = XMAX / (100.0*FLOAT(NUMX) )
DY = YMAX / (100.0*FLOAT(NUMY) )
DZ = ZMAX / (100.0*FLOAT(NUMZ) )
CONMAX = (RK1MAX + 2.0*RK2MAX)/3.0
DT = (AMIN1(DX,DY,DZ)**2)*RHOMIN*CPMIN/(2.00*SAFE*CONMAX)
NTIME = IFIX (TIMEMX/ DT )
C
IF (LOOP .EQ. 0) THEN
    GO TO 999
END .F
C
017 CONTINUE
WRITE(*,*) ' '
WRITE(*,*) 'THE TIME STEP IS ',DT,' SECONDS'
WRITE(*,*) 'BROKEN INTO ',NTIME,' INCREMENTS'
WRITE(*,*) ' '
WRITE(*,*) ' '
WRITE(*,*) '1 RADIUS OF CIRCULAR BEAM IS ',RADIUS,' CENTIMETERS'
WRITE(*,*) '2 X COORD OF BEAM CENTER IS ',XFLUX,' CENTIMETERS'
WRITE(*,*) '3 Y COORD OF BEAM CENTER IS ',YFLUX,' CENTIMETERS'
WRITE(*,*) '4 ABLATION OR MELT TEMP IS ',TPYRO,' DEG KELVIN'
WRITE(*,*) '5 + AND - WRAP ANGLE OF LAYUP IS ',ALPHA,' DEGREES'
WRITE(*,*) '6 INTEGER FREQUENCY TO TYPE RESULTS TO CRT IS',ITYPE
WRITE(*,*) '7 INTEGER FREQUENCY TO SEND RESULTS TO DISK IS',IPRN
WRITE(*,*) '8 DISK FILE TO OUTPUT TO IS(lowercase) ',OUTFILE
WRITE(*,*) '9 LABEL TO INSERT AT TOP OF DISKFILE IS'
WRITE(*, '(A70)') LABTXT
WRITE(*,*) '10 FREQUENCY TO SEND X DATA TO DISK', IXOUT
WRITE(*,*) '11 FREQUENCY TO SEND Y DATA TO DISK', IYOUT
WRITE(*,*) '12 FREQUENCY TO SEND Z DATA TO DISK', IZOUT
WRITE(*,*) ' '
WRITE(*,*) 'Enter an integer # of variable to change, or 0 to
&continue'
WRITE(*,*) ' '
WRITE(*,*) ' '

```

```

      READ(*,*) NSWER
C
C      IF NSWER ISN'T A PERMISSIBLE # (1-9), PROMPT AGAIN
C
      IF ((NSWER .LT. 1 .OR. NSWER.GT.12).AND. NSWER.NE.0) THEN
          WRITE(*,*) 'REPLY MUST BE 0 OR 1 thru 12, PLEASE TRY AGAIN'

          GOTO 017
      END IF

C
      GOTO (1,2,3,4,5,6,7,8,9,10,11,12), NSWER
C*****
C      IF ANSWER IS 0, JUMP OUT OF LOOP AND WRITE NEW VARIABLES TO DATA
C      FILE, BY JUMPING TO LINE #100
C*****
      GOTO 100
001  WRITE(*,*)'NEW RADIUS, CENTIMETERS, IS?'
      READ(*,*) RADIUS
      GOTO 017
002  WRITE(*,*)'NEW X COORD OF CENTERLINE, IN CENTIMETERS, IS?'
      READ(*,*) XFLUX
      GOTO 017
003  WRITE(*,*)'NEW Y COORD OF CENTERLINE, IN CENTIMETERS, IS?'
      READ(*,*) YFLUX
      GOTO 017
004  WRITE(*,*)'NEW PYRO OR MELT TEMP, IN KELVIN, IS?'
      READ(*,*) TPYRO
      GOTO 017
005  WRITE(*,*)'NEW WRAP ANGLE, DEGREES, IS?'
      READ(*,*) ALPHA
      GOTO 017
006  WRITE(*,*)'INTEGER FOR SCREEN OUTPUT ?'
      READ(*,*) ITYPE
      GOTO 017
007  WRITE(*,*)'INTEGER FOR DISKFILE OUTPUT ?'
      READ(*,*) IPRN
      GOTO 017
008  WRITE(*,*)'NAME FOR DISK FILE ?'
      READ(*, '(A10)') OUTFILE
      GOTO 017
009  WRITE(*,*)' LABEL FOR DESCRIBING DISK FILE ?'
      READ(*, '(A70)') LABTXT
      GOTO 017
010  WRITE(*,*)'INTEGER FOR X DISKFILE OUTPUT ?'
      READ(*,*) IXOUT
      GOTO 017
011  WRITE(*,*)'INTEGER FOR Y DISKFILE OUTPUT ?'
      READ(*,*) IYOUT
      GOTO 017
012  WRITE(*,*)'INTEGER FOR Z DISKFILE OUTPUT ?'
      READ(*,*) IZOUT
      GOTO 017
C

```

```

100  CONTINUE
      REWIND 11
      WRITE(11,*) RADIUS, XFLUX, YFLUX, TPYRO, ALPHA, ITYPE, IPRN, IXOUT
      &, IYOUT, IZOUT
      WRITE(11, '(A10,A70)') OUTFILE, LABTXT
999  CONTINUE
      CLOSE (11)
      RETURN
      END

```

```

C*****
C*****

```

```

      SUBROUTINE DATALK (LOOP)
      REAL IRRAD
      COMMON /DATA1/ IRRAD, ABSORB, XMAX, YMAX, ZMAX, NUMX, NUMY, NUMZ,
      &RK1MAX, RK2MAX, RHOMIN, CPMIN, HABLAT, TIMEMX, TINITL, SAFE
C
      OPEN (UNIT=10, FILE='3DINPUT.DAT', STATUS='OLD')
      READ (10,*) IRRAD, ABSORB, XMAX, YMAX, ZMAX, NUMX, NUMY, NUMZ
      READ (10,*) RK1MAX, RK2MAX, RHOMIN, CPMIN, HABLAT, TIMEMX, TINITL, SAFE
C
      IF (LOOP .EQ. 0) THEN
        GO TO 999
      END IF

```

```

C
017  CONTINUE
      DX = XMAX / (100.0*FLOAT(NUMX) )
      DY = YMAX / (100.0*FLOAT(NUMY) )
      DZ = ZMAX / (100.0*FLOAT(NUMZ) )
      CONMAX = (RK1MAX + 2.0*RK2MAX) / 3.0
      DELTAT = (AMIN1(DX,DY,DZ)**2)*RHOMIN*CPMIN/(2.0*SAFE*CONMAX)
      WRITE(*,*) ' '
      WRITE(*,*) ' '
      WRITE(*,*) ' '
      WRITE(*,*) '1 IRRADIANCE INCIDENT IS      ', IRRAD, '      W/cm**2'
      WRITE(*,*) '2 SURFACE ABSORBTIVITY IS      ', ABSORB
      WRITE(*,*) '3 LENGTH(X) OF SPECIMEN IS      ', XMAX, ' CENTIMETERS'
      WRITE(*,*) '4 YMAX (Y) OF SPECIMEN IS      ', YMAX, ' CENTIMETERS'
      WRITE(*,*) '5 THICKNESS(Z) SPECIMEN IS      ', ZMAX, ' .CENTIMETERS'
      WRITE(*,*) '6 NUMBER OF X NODES IS          ', NUMX
      WRITE(*,*) '7 NUMBER OF Y NODES IS          ', NUMY
      WRITE(*,*) '8 NUMBER OF Z NODES IS          ', NUMZ
      WRITE(*,*) '9 MAX AXIAL CONDUCTIVITY IS      ', RK1MAX, '      W/mK'
      WRITE(*,*) '10 MAX NORMAL CONDUCTIVITY IS    ', RK2MAX, '      W/mK'
      WRITE(*,*) '11 MINIMUM DENSITY IS           ', RHOMIN, '      kg/m**3'
      WRITE(*,*) '12 MINIMUM Cp IS                ', CPMIN, '      J/ kg K'
      WRITE(*,*) '13 THERMOCHEM HEAT OF ABLATION   ', HABLAT, '      kJ/gm'
      WRITE(*,*) '14 MAXIMUM TIME DURATION IS      ', TIMEMX, '      SECONDS'
      WRITE(*,*) '15 INITIAL SAMPLE TEMPERATURE   ', TINITL, '      KELVIN'
      WRITE(*,*) '16 SAFETY FACTOR FOR TIME INCREMENT', SAFE
      WRITE(*,*) 'DT will be      ', DELTAT, ' Seconds'
      WRITE(*,*) 'Enter an integer # of variable to change, or 0 to
      &continue'
      WRITE(*,*) ' '

```

```

WRITE(*,*) ' '
READ(*,*) ANSWER
C
C   IF ANSWER ISN'T A PERMISSIBLE # (1-16), PROMPT AGAIN
C
   IF ((ANSWER .LT. 1 .OR. ANSWER.GT.16).AND. ANSWER.NE.0) THEN
       WRITE(*,*) 'REPLY MUST BE 0 OR 1 thru 16, PLEASE TRY AGAIN'
       GOTO 017
   END IF

   GOTO (1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16), ANSWER
C*****
C   IF ANSWER IS 0, JUMP OUT OF LOOP AND WRITE NEW VARIABLES TO DATA
C   FILE, BY JUMPING TO LINE #100
C*****
   GOTO 100
001  WRITE(*,*)'NEW IRRAD, W/cm**2, IS?'
      READ(*,*) IRRAD
      GOTO 017
002  WRITE(*,*)'NEW SURFACE ABSORBTIVITY (0.0 to 1.0)?'
      READ(*,*) ABSORB
      GOTO 017
003  WRITE(*,*)'NEW LENGTH, CM, IS?'
      READ(*,*) XMAX
      GOTO 017
004  WRITE(*,*)'NEW WIDTH, CM, IS?'
      READ(*,*) YMAX
      GOTO 017
005  WRITE(*,*)'NEW THICKNESS, CM, IS?'
      READ(*,*) ZMAX
      GOTO 017
006  WRITE(*,*)'NEW NUMBER OF X NODES, IS?'
      READ(*,*) NUMX
      GOTO 017
007  WRITE(*,*)'NEW NUMBER OF Y NODES, IS?'
      READ(*,*) NUMY
      GOTO 017
008  WRITE(*,*)'NEW NUMBER OF Z NODES, IS?'
      READ(*,*) NUMZ
      GOTO 017
009  WRITE(*,*)'NEW MAXIMUM AXIAL CONDUCTIVITY, W/ m K, IS?'
      READ(*,*) RK1MAX
      GOTO 017
010  WRITE(*,*)'NEW MAXIMUM NORMAL CONDUCTIVITY, W/ m K, IS?'
      READ(*,*) RK2MAX
      GOTO 017
011  WRITE(*,*)'NEW MINIMUM DENSITY, kg/m**3, IS?'
      READ(*,*) RHOMIN
      GOTO 017
012  WRITE(*,*)'NEW MINIMUM SPECIFIC HEAT(Cp), J/kg K, IS?'
      READ(*,*) CPMIN
      GOTO 017
013  WRITE(*,*)'NEW THERMOCHEM HEAT OF ABLATION, kJ / gm, IS?'

```

```

      READ(*,*) HABLAT
      GOTO 017
014  WRITE(*,*) 'NEW MAXIMUM TIME,SECONDS, IS?'
      READ(*,*) TIMEMX
      GOTO 017
015  WRITE(*,*) 'NEW INITIAL TEMPERATURE, deg KELVIN, IS?'
      READ(*,*) TINITL
      GOTO 017
016  WRITE(*,*) 'NEW SAFETY FACTOR IS?'
      READ(*,*) SAFE
      GOTO 017

100  CONTINUE
      REWIND 10
      WRITE(10,*) IRRAD, ABSORB, XMAX, YMAX, ZMAX, NUMX, NUMY, NUMZ
      WRITE(10,*) RK1MAX, RK2MAX, RHOMIN, CPMIN, HABLAT, TIMEMX, TINITL, SAFE
999  CONTINUE
      CLOSE (10)
      RETURN
      END

```

C\*\*\*\*\*

      SUBROUTINE MAKEFLUX(FLUX)

C\*\*\*\*\*

C      INITIALIZES A CIRCULAR BEAM OF UNIFORM INTENSITY, IRRAD\*ABSORB  
C      CENTERED ON THE FACE OF REGION

C\*\*\*\*\*

      CHARACTER\*70 LABTXT

      CHARACTER\*10 OUTFILE

      REAL IRRAD

      DIMENSION FLUX(10,10,10)

      COMMON /DATA1/ IRRAD, ABSORB, XMAX, YMAX, ZMAX, NUMX, NUMY, NUMZ,

      &RK1MAX, RK2MAX, RHOMIN, CPMIN, HABLAT, TIMEMX, TINITL, SAFE

      COMMON /DATA2/ RADIUS, XFLUX, YFLUX, TPYRO, ALPHA, ITYPE, IPRN,

      &IXOUT, IYOUT, IZOUT, OUTFILE, LABTXT

C

      DO 100, I = 1, NUMX+2

      DO 101, J = 1, NUMY+2

      DO 102, K = 1, NUMZ+2

          FLUX(I,J,K) = 0.0

102

      CONTINUE

101

      CONTINUE

100

      CONTINUE

C

      NXCL = NINT ( FLOAT(NUMX) / (XMAX/XFLUX) )

      NYCL = NINT ( FLOAT(NUMY) / (YMAX/YFLUX) )

      DX = XMAX / FLOAT((NUMX) )

      DY = YMAX / FLOAT((NUMY) )

C

      DO 200, I = 1, NUMX+2

      DO 201, J = 1, NUMY+2

C

      XSQUAR=( DX \* FLOAT( IABS(NXCL-I) ) )\*\*2.0

      YSQUAR=( DY \* FLOAT( IABS(NYCL-J) ) )\*\*2.0

```

PNTRAD=XSQUAR+YSQUAR
DIFF=PNTRAD - (RADIUS*RADIUS + (0.5*(DX*DX + DY*DY) ))
C
  IF (DIFF .LT. 0.0) THEN
C*****
C    CHANGE FLUX FROM WATTS/CM**2 TO WATTS/M**2 BY MULTIPLYING BY
C    10000.0
C*****
      FLUX(I,J,2)= IRRAD*ABSORB*10000.0
    END IF
C
201  CONTINUE
200  CONTINUE
C
  DO 300, I = 1, MUMX
    WRITE(*,900) (FLUX (I,J,2), J=1,NUMY)
300  CONTINUE
C
900  FORMAT(1X,100F4.0)
      RETURN
      END
C*****
      SUBROUTINE ROTATE(R11,R22,R12,R33, NSTAK, RK1, RK2, ALPHA,
&IMETAL)
C*****
C    SUBROUTINE TO ROTATE CONDUCTIVITIES TO MAKE TENSOR OUT OF THEM
C    IF IMETAL IS 1, SKIP ANISOTROPIC TENSOR, SET RK11=RK1=RK22=RK33
C    AND RK12=0
C*****
      IF (IMETAL .EQ. 1) THEN
        R11 = RK1
        R22 = RK1
        R33 = RK1
        R12 = 0.000
        GOTO 104
      END IF
C
      IF (NSTAK .EQ. 0) THEN
        R11 = RK1
        R22 = RK2
        R12 = 0.000
        R33 = RK2
C
      ELSE IF (NSTAK .EQ. 1) THEN
        R33 = RK2
        R11 = RK1*COS(ALPHA)*COS(ALPHA) +
&          RK2*SIN(ALPHA)*SIN(ALPHA)
&        R22 = RK2*COS(ALPHA)*COS(ALPHA) +
&          RK1*SIN(ALPHA)*SIN(ALPHA)
        R12 = COS(ALPHA)*SIN(ALPHA)*(RK1-RK2)
C
      ELSE IF (NSTAK .EQ. -1) THEN

```

```

        R33 = RK2
        R11 = RK1*COS(ALPHA)*COS(ALPHA) +
&          RK2*SIN(ALPHA)*SIN(ALPHA)
        R22 = RK2*COS(ALPHA)*COS(ALPHA) +
&          RK1*SIN(ALPHA)*SIN(ALPHA)
        R12 = COS(ALPHA)*SIN(-ALPHA)*(RK1-RK2)
    END IF
C
C*****
C  JUMP DOWN TO HERE IF ISOTROPIC MATERIAL
C*****
104  CONTINUE
      RETURN
      END

```

Appendix B: Formulation of Beam Equations on an Elastic Foundation

An elastic beam in flexure on a foundation with a finite modulus is modeled as a simple Bernoulli-Euler beam element subject to a distributed loading. This loading will be broken into two distinct parts:  $p(x)$ , the normal transverse loading component, and  $q(x)$ , the reaction of the elastic foundation. If the foundation is assumed to be linearly elastic,  $q(x) = k \cdot w(x)$ . The differential equation of the beam in Figure B1 becomes

$$EI w'''' + kw = - p(x) \quad (B1)$$

The homogenous solution to this equation (7:4) is

$$w(x) = e^{\beta x} (A' \cos \beta x + B' \sin \beta x) + e^{-\beta x} (C' \cos \beta x + D' \sin \beta x) \quad (B2)$$

where  $\beta^4 = k/4EI$ .

Without any loading  $p(x)$ , the portion of the beam from B to C will obey the homogenous solution. Since the fibers are very long compared to their diameter, the fiber may be considered to be semi-infinite in length.

The requirement for the displacement and slope of the fiber to vanish at large  $x$  ( $w = dw/dx = 0; x \rightarrow \infty$ ) dictates that the coefficients of  $e^{+\beta x}$  must vanish...

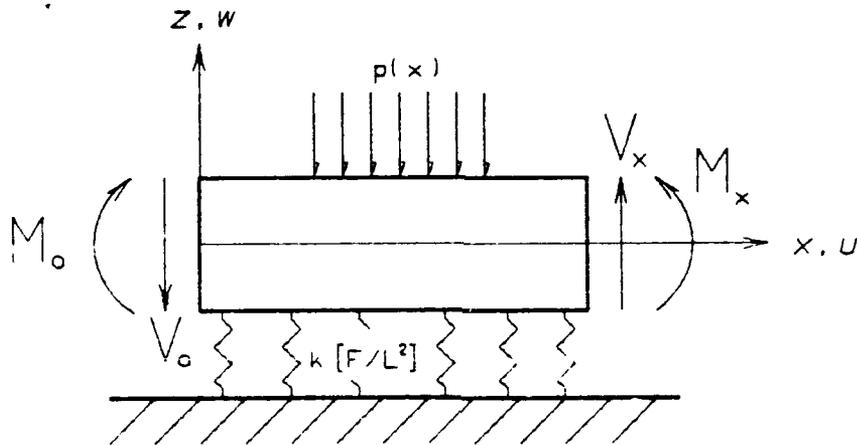


Figure B1. Beam Element on Elastic Foundation

$$w_{BC}(x) = e^{-\beta x} [C' \cos \beta x + D' \sin \beta x] \quad (B3a)$$

$$w'_{BC}(x) = \beta e^{-\beta x} [(-C' + D') \cos \beta x + (-C' - D') \sin \beta x] \quad (B3b)$$

$$w''_{BC}(x) = 2\beta^2 e^{-\beta x} [-D' \cos \beta x + C' \sin \beta x] \quad (B3c)$$

$$w'''_{BC}(x) = 2\beta^3 e^{-\beta x} [(C' + D') \cos \beta x + (-C' + D') \sin \beta x] \quad (B3d)$$

The thermal gradient across the portion of the beam from A to B would induce, in a beam element entirely free to rotate, a curvature of  $1/\rho_t$  where  $\rho_t$  is the radius of curvature (Figure B2).

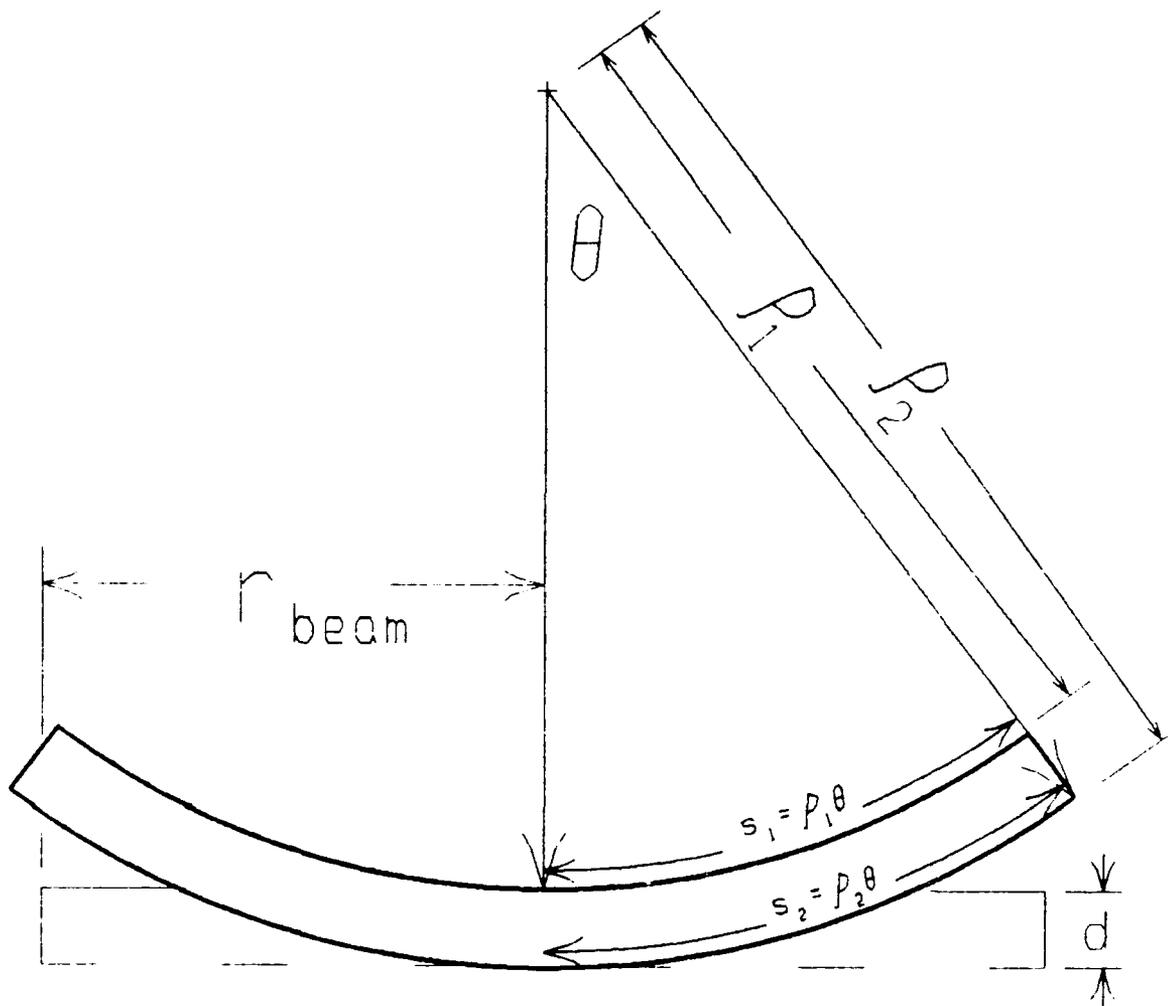


Figure B2. Thermal Curvature of Free Beam with a Temperature Gradient

$$s_1 = r + \Delta u_U = \rho_1 \theta$$

$$s_2 = r + \Delta u_L = \rho_2 \theta$$

where  $\Delta u_U$  is the displacement of the top edge of the fiber where  $\Delta u_U$  is the displacement of the lower edge of the fiber

$$\theta = \frac{r + \Delta u_U}{\rho_1} = \frac{r + \Delta u_L}{\rho_1 + d} = \frac{r + \Delta u_L}{d + \frac{r + \Delta u_U}{\theta}}$$

$$\theta = \frac{\Delta u_L - \Delta u_U}{d} = \frac{r\alpha_1 T_L - r\alpha_1 T_U}{d}$$

$$\rho_1 = \rho_t = \frac{r + \Delta u_U}{\theta}$$

$$\rho_t = \frac{d \left( 1 + \alpha_1 T_U \right)}{\alpha_1 \Delta T} \quad (B4)$$

However, the section AB is not free to rotate, but instead has a curvature (assumed constant) equal to the sum of thermal curvature,  $1/\rho_t$  plus the curvature due to the bending moment  $M_B$  applied at the ends of section AB (4:406). If the matrix resin (elastic foundation) is removed by pyrolysis at  $700^\circ\text{K}$  and the fiber remains intact, at temperatures above  $700^\circ\text{K}$  the heated section AB may be considered to be constrained as shown in Figure B3, with curvature given by

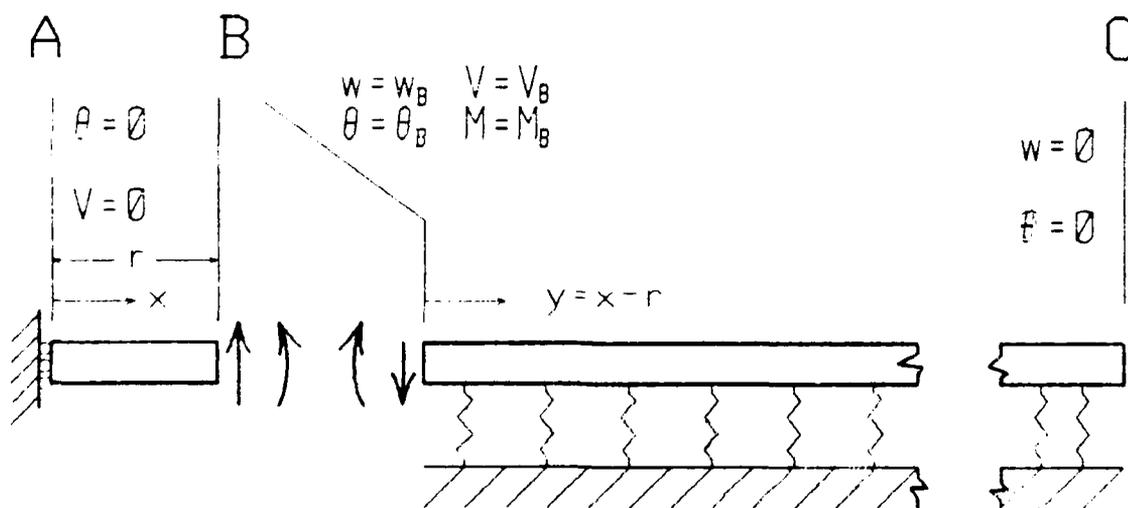


Figure B3. End Conditions for Beams AB and BC

$$\begin{aligned}
 w''_{AB}(x) &= w''_{\text{homogeneous}} + w''_{\text{thermal}} \\
 &= \frac{M_B}{EI} + \frac{1}{\rho_t}
 \end{aligned}$$

Two integrations with respect to  $x$  yields

$$w_{AB}(x) = \left[ \frac{1}{\rho_t} + \frac{M_B}{EI} \right] * \frac{x^2}{2} + Ax + B \quad (B5)$$

Symmetry of the fiber about  $x=0$  requires that  $w'(0) = 0$  and  $V(0) = EIw''''(0) = 0$ . Continuity at point B ( $x=r$ ) is required of the deflection, slope and moment. The substitution  $y = x-r$  will be made in Equation (B3) so that section BC may be thought of as a semi-infinite beam starting at  $y=0$ . This yields the following system to be solved...

$$w_{AB}(x) = \left[ \frac{1}{\rho} + \frac{M_B}{EI} \right] * \frac{x^2}{2} + Ax + B \quad (B6)$$

$$w_{BC}(y) = e^{-\beta y} (C' \cos \beta y + D' \sin \beta y) \quad (B7)$$

subject to

$$w'_{AB}(0) = 0 \quad (B8a)$$

$$w''_{AB}(0) = 0 \quad (b)$$

$$w_{AB}(x=r) = w_{BC}(y=0) \quad (c)$$

$$w'_{AB}(x=r) = w'_{BC}(y=0) \quad (d)$$

$$M_B = EI w''_{BC}(y=0) \quad (e)$$

where  $M_B$ ,  $A$ ,  $B$ ,  $C'$ ,  $D'$  are to be determined from Equations (B8).

$$(B8a), (B6) \rightarrow \left[ \frac{1}{\rho_t} + \frac{M_B}{EI} \right] \cdot 0 + A = 0 \rightarrow A = 0$$

$$(B8c) \rightarrow \left[ \frac{1}{\rho_t} + \frac{M_B}{EI} \right] * \frac{r^2}{2} + B = C' \quad (B9)$$

$$(B8d) \rightarrow \left[ \frac{1}{\rho_t} + \frac{M_B}{EI} \right] * r = \beta(-C' + D') \quad (B10)$$

$$(B8e) \rightarrow M_B = EI (-2\beta^2 D') \rightarrow D' = \frac{-M_B}{2\beta^2 EI} \quad (B11)$$

Since  $w''_{AB} \equiv 0$ , Equation (B8b) is solved identically. However, equilibrium of forces in the  $z$  direction requires that  $V(x=r) = 0 = V(y=0)$ . This leads to

$$0 = 2\beta^3 (C' + D') \rightarrow C' = -D' \quad (B12)$$

(B10), (B11), (B12)  $\rightarrow$

$$\begin{aligned} \left[ \frac{1}{\rho_t} + \frac{M_B}{EI} \right] r &= 2\beta D' \\ &= 2\beta \left[ \frac{-M_B}{2\beta^2 EI} \right] \end{aligned}$$

$$\frac{r}{\rho_i} = \frac{-M_B}{\beta EI} - \frac{M_B r}{EI}$$

$$M_B = \frac{-rEI}{\left(\frac{1}{\beta} + r\right)\rho_i} \quad (B13)$$

(B11), (B13) →

$$D' = \frac{-M_B}{2\beta^2 EI}$$

$$= \frac{rEI}{\rho_i \left(\frac{1}{\beta} + r\right) 2\beta^2 EI}$$

$$= \frac{r}{2\rho_i \beta^2 \left(\frac{1}{\beta} + r\right)} \quad (B14)$$

(B12), (B14) →

$$C' = -D'$$

$$= \frac{-r}{2\rho_i \beta^2 \left(\frac{1}{\beta} + r\right)} \quad (B15)$$

(B9), (B13), (B15) →

$$B = C' - \frac{r^2}{2} \left( \frac{1}{\rho_i} + \frac{M_B}{EI} \right)$$

$$\begin{aligned}
B &= \frac{-r}{2\rho_t \beta^2 \left(\frac{1}{\beta} + r\right)} - \frac{r^2}{2\rho_t} - \frac{M_B r^2}{23I} \\
&= \frac{r}{2\rho_t} \left[ \frac{r^2}{\gamma} - r - \frac{1}{\beta^2 \gamma} \right] \qquad (B16)
\end{aligned}$$

where  $\gamma \equiv \frac{1}{\beta} + r$

Note two interesting limits on these constants;

$$\lim_{\beta \rightarrow \infty} M_B = \frac{-EI}{\rho_t}$$

which is the solution for a double-fixed beam subjected to a temperature gradient (4:406). This agrees with the intuitive observation that a beam section BC on an infinitely stiff foundation will act like a fixed wall to the section AB.

$$\text{Also, } \lim_{\beta \rightarrow \infty} \gamma = r, \text{ which implies } \lim_{\beta \rightarrow \infty} B = \frac{r}{2\rho_t} \left[ r - r - 0 \right] = 0.$$

This also agrees with the solution for a double fixed beam.

The reduction in bending moment in the section AB can be seen most easily by defining a quantity

$$M^* \equiv \frac{M_B}{-EI/\rho_t} = \frac{r}{\gamma} \quad (B17)$$

$M^*=1$  implies the bending moment in AB is identical to that of a fixed wall beam. The effect of laser beam radius  $r$  and foundation stiffness  $\beta$  is shown in Figure B4, which clearly shows that for spot sizes and foundation stiffnesses seen in the experimental setup,  $M^* \approx 1$ .

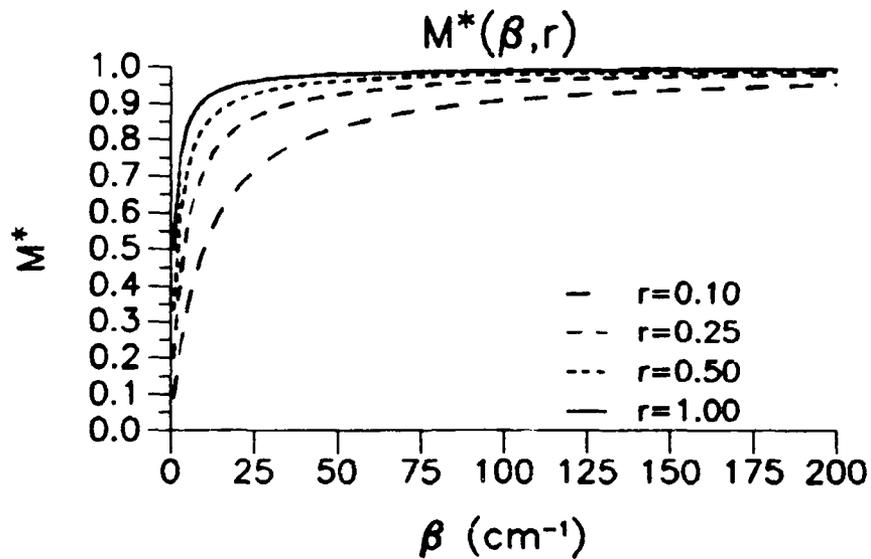


Figure B4.  $M^*$  versus  $\beta$  and  $r$

To estimate  $\beta$ , first estimate the foundation modulus  $k$ , which is the ratio of beam loading per unit length to the deflection (7). As shown in Figure B5, for a length of beam  $L$  with force  $p$  on it, the pressure on the foundation of modulus  $E_m$  will be  $\sigma = \frac{p}{dL}$ . This will also be the stress in the foundation which causes strain  $\epsilon = \delta/h$ , where  $h$  is the thickness of the foundation.

$$\epsilon = \frac{\sigma}{E_m} = \frac{\delta}{h} = \frac{p}{dL E_m}$$

Since  $k = \frac{p}{L \epsilon}$ , we get

$$\begin{aligned} k &= \frac{p}{L} \frac{d L E_m}{ph} \\ &= \frac{d E_m}{h} \end{aligned}$$

Assuming a foundation of epoxy resin as thick as the distance between fibers (assumed equal to a fiber diameter)

$$k \approx \frac{d E_m}{h} = \frac{8 \times 10^{-4} \text{ cm} \cdot 2.4 \times 10^7 \text{ N cm}^{-2}}{8 \times 10^{-4} \text{ cm}}$$

Therefore,

$$\begin{aligned} \beta^4 &= \frac{k}{4 E I} = \frac{k}{4 E \frac{\pi}{64} d^4} \\ &= \frac{2.4 \times 10^7 \text{ N cm}^{-2}}{4 \times 2.35 \times 10^7 \text{ N cm}^{-2} \frac{\pi}{64} (8 \times 10^{-4} \text{ cm})^4} \\ \beta &\approx 1890 \text{ cm}^{-1} \gg 200 \text{ cm}^{-1} \end{aligned}$$

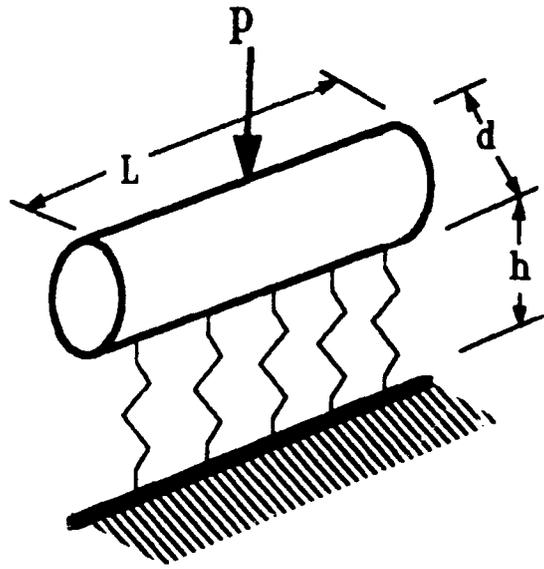


Figure B5. Estimation of Foundation Modulus

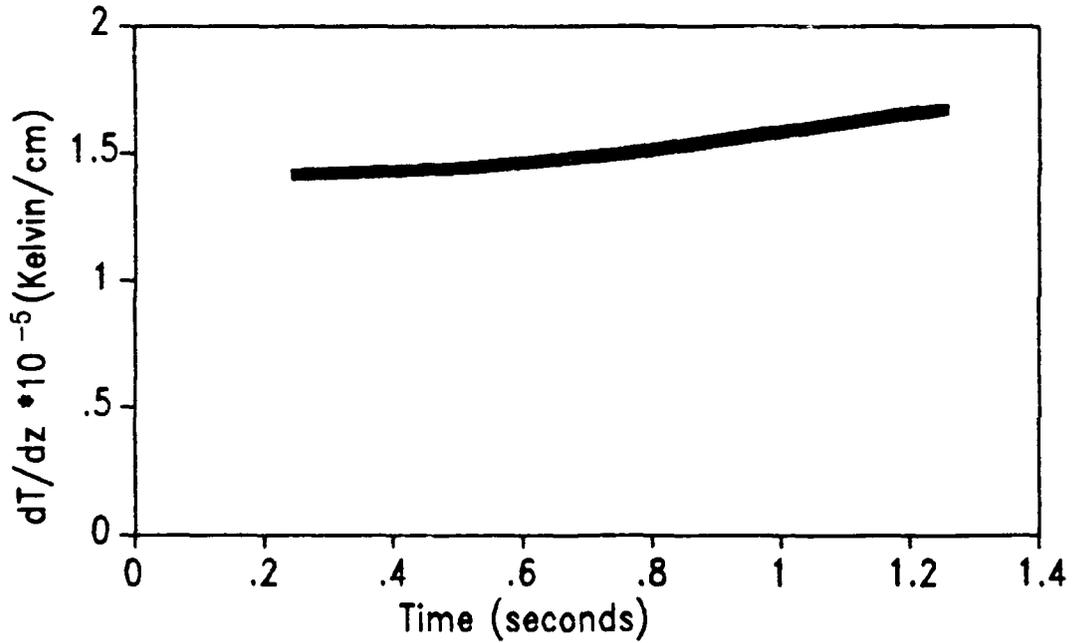


Figure B6. Temperature Gradient in Graphite/Epoxy at  $10\text{kW/cm}^2$  via MELT3D.FOR

For a laser beam radius of  $r=1.0$  cm and  $\beta \gg 200$  cm<sup>-1</sup>,  $M_{\text{B}} \approx EI/\rho_t$ .

The maximum axial stress from  $M_{\text{B}}$  will be

$$\begin{aligned}\sigma_{x_M} &= \frac{M y}{I} = \frac{M(d/2)}{I} \\ \therefore \sigma_{x_M} &= \frac{E I}{\rho_t} \frac{(d/2)}{I}\end{aligned}\tag{B18}$$

From MELT3D.FOR, the magnitude of the thermal gradient at 10kW/cm<sup>2</sup> is < 200,000 °K/cm (Figure B6). Across the first surface fiber, the temperature difference is  $\Delta T \approx 150^\circ\text{K}$ .

From Chapter III,

$$\begin{aligned}\rho_t &= \frac{d(1 + \alpha T_U)}{\alpha \Delta T} \\ &= \frac{8 \times 10^{-4} \text{ cm} (1 - 3.3 \times 10^{-7} \text{ }^\circ\text{K}^{-1} \cdot 3800^\circ\text{K})}{-3.3 \times 10^{-7} \cdot 150^\circ\text{K}} \\ &= -16.1 \text{ cm}\end{aligned}\tag{B19}$$

Therefore from Equation (B18) and Equation (B19)

$$\begin{aligned}\sigma_{x_M} &\approx \frac{E (d/2)}{\rho_t} \\ &\approx \frac{2.4 \times 10^7 \text{ Ncm}^{-2} \cdot \frac{8 \times 10^{-4} \text{ cm}}{2}}{-16.1 \text{ cm}} \\ &\approx -600 \text{ N cm}^{-2} = -6 \text{ MPa}\end{aligned}$$

But  $\sigma_{ult} = 3795 \text{ MPa}$ , so

$$\left| \frac{\sigma_{xM}}{\sigma_{ult}} \right| = \frac{6}{3795} = 0.16\%$$

But  $\sigma_{x_{axial}} = E_1 \epsilon_1$ . When loaded to 2/3 fracture load,  $\epsilon_1 = \frac{2\epsilon_{ult}}{3}$ .

$$\sigma_{x_{axial}} = \frac{2 \epsilon_{ult}}{3} E_1$$

$$\sigma_{x_{axial}} = \frac{2 (0.0148)}{3} 235 \times 10^9 \text{ Pa}$$

$$= 2.31 \text{ GPa}$$

$$\frac{\sigma_{x_{axial}}}{\sigma_{ult}} = \frac{2.31 \text{ GPa}}{3.795 \text{ GPa}} = 61\%$$

$$\frac{\sigma_{xM}}{\sigma_{x_{axial}}} = \frac{6 \text{ MPa}}{2310 \text{ MPa}} = 0.26\%$$

Therefore, bending stresses are appropriately negligible.

## Appendix C: Fabrication of AS4/3502 Graphite/Epoxy

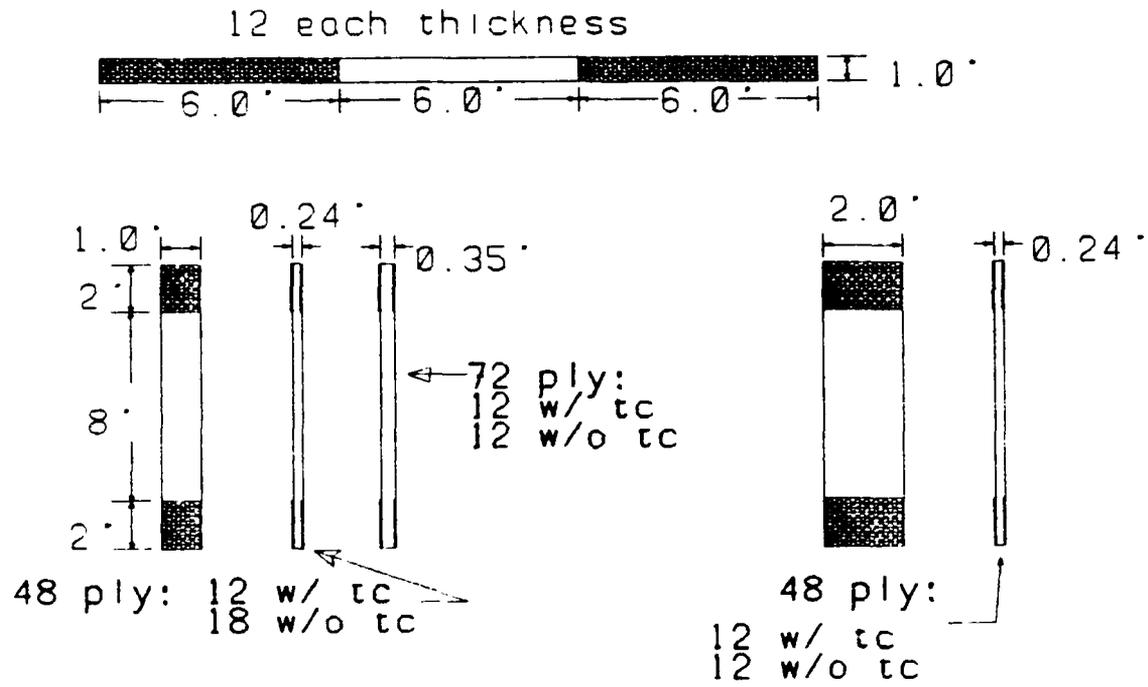
### Test Specimens

The graphite/epoxy (GE) specimens tested in this program were laid up, cured and cut to size by the Structures Survivability/Supportability Group of the Flight Dynamics Laboratory, Wright Research and Development Center (WRDC/FIBCA). Specimen specifications described in Figure C1 were given to the thesis sponsor, and fabricated as described in this appendix.

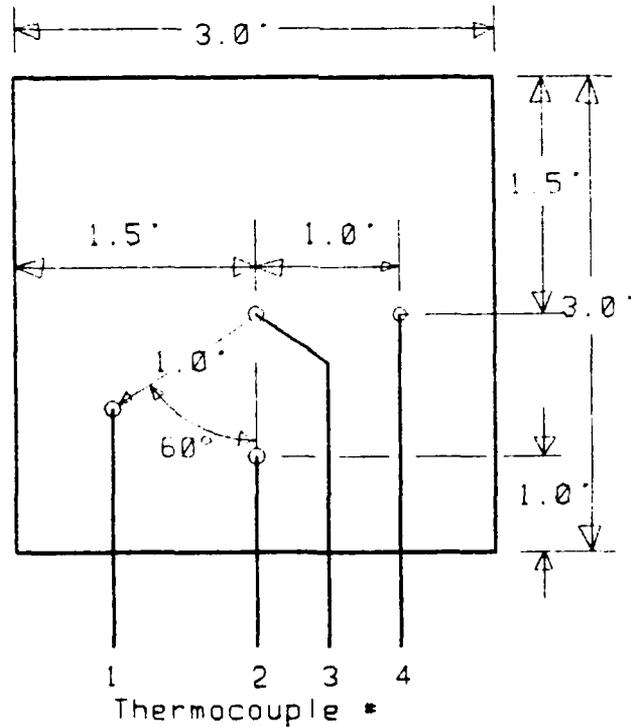
Layup of the panels was performed by a vacuum bag process as shown in Figure C2.

After the panels are laid up and sealed under vacuum (Fig C2a) they are cured in an autoclave in a manner similar to that suggested in Reference 6:20 (Figure C3).

1. Pull 20 in. (508mm) minimum vacuum on part.
2. Place in autoclave.
3. Raise temperature to 275°F(135°C) at 3-5°F/minute (2-3°C/minute).
4. Hold for 15 minutes at 275°F(135°C) under vacuum pressure only.
5. Pressurize autoclave to 85psi(586 kPa).
6. Hold at 275°F(135°C), 85psi(586 kPa), and vacuum on the part for 45 minutes.
7. Raise temperature to 350°F(177°C) at 3-5°F/minute (2-3°C/minute).
8. Hold for 2 hours.
9. Cool part to 150°F(66°C) in not less than 45 minutes, maintain pressure and vacuum.
10. Remove from autoclave and unbag.



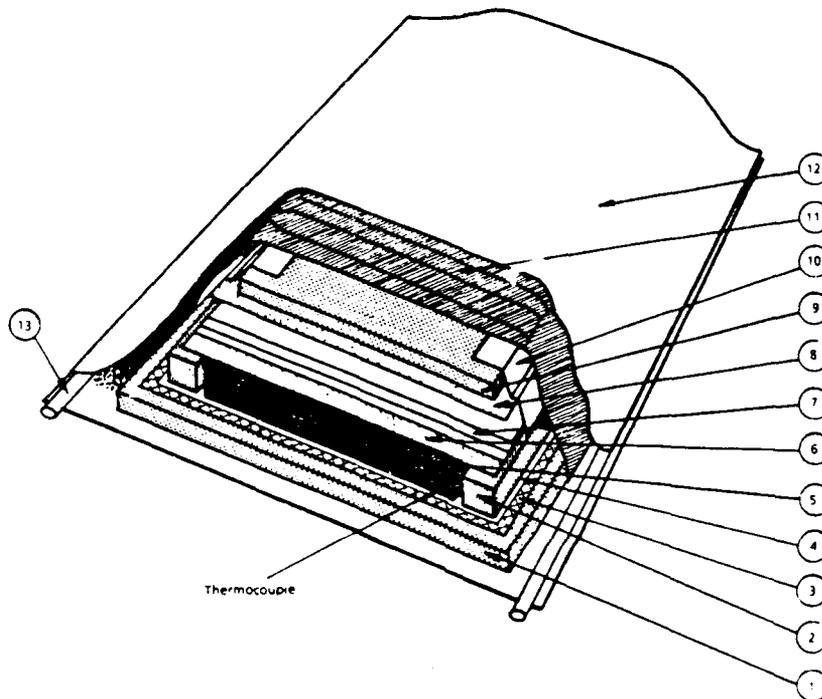
(a) Tensile Specimens



(b) Unloaded Specimens

Figure C1. Test Specimen Specifications

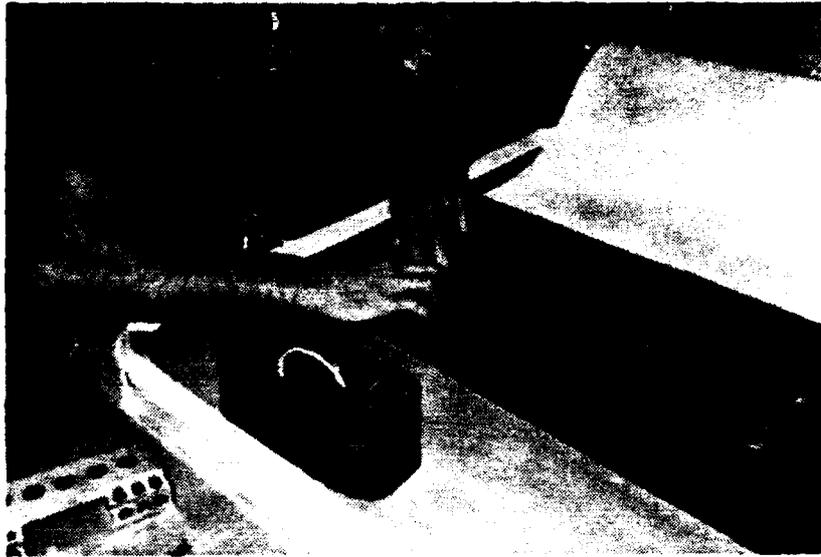
**TYPICAL ACCEPTANCE PANEL  
FABRICATION SEQUENCE**



1. Base plate: Aluminum 1/4 to 1/2 in. thick
2. Cork dam: Cork 1/8 x 1 in. with pressure-sensitive adhesive backing (Corprene) or equivalent
3. Release film: Teflon, nonperforated 0.001-0.004 in. thick
4. Release fabric: Fabric enfab TX 10-40 release (porous) or equivalent
5. Prepreg layup
6. Release fabric: Fabric enfab TX 10-40 release (porous)
7. Resin bleeders: Cloth, fiberglass No. 120 (as per calculation on following page)
8. Release film: Teflon, nonperforated 0.001-0.004 in. thick
9. Caul plate: Aluminum, 0.030 in. minimum
10. Tape: Pressure-sensitive, green polyester silicone
11. Air bleeder: Style 1581 glass or equivalent
12. Vacuum bag: Film capron 80, hi-temp nylon, 0.002 in. thick
13. High temperature sealant: Schnee Morehead

Mold release: Frekote-33 or equivalent

(a) Typical Vacuum Bag Layup (6:23)  
Figure C2. Fabrication of Test Samples



(b) Layup Process



(c) Layup Process (Continued)  
Figure C2. (Continued)



(d) Layup Process (Concluded)  
Figure C2. (Concluded)

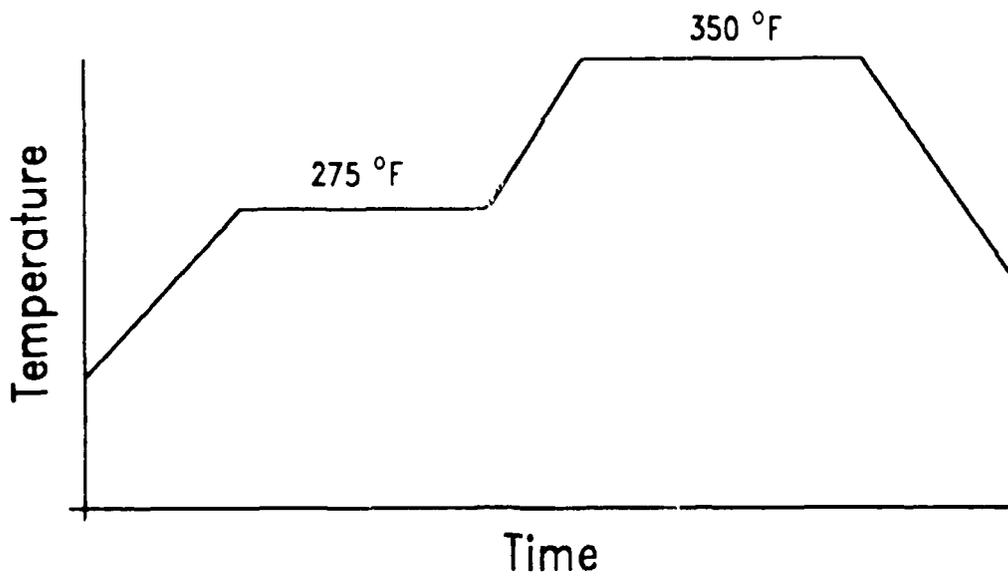


Figure C3. Hercules Suggested Cure Cycle for  
AS4/3502 (6:20)

The cured panels were examined by computerized axial tomography (CAT) scan to reveal any areas of incomplete interply bonding. Panels fabricated with thermocouple wire embedded in them were x-rayed to discern any shifting of the thermocouple positions that may have occurred. No flaws were visible to either of these techniques.

Evidence that the structural integrity of the specimens was as high as practically achievable was also found in post mortem inspection of a failed specimen. One sample was tested in tension to failure (Figure C4). At 87% of the predicted strength, failure occurred at the grip fixture (note step in output created by delay in switching the amplifier gain as strain gauge output changed scales).

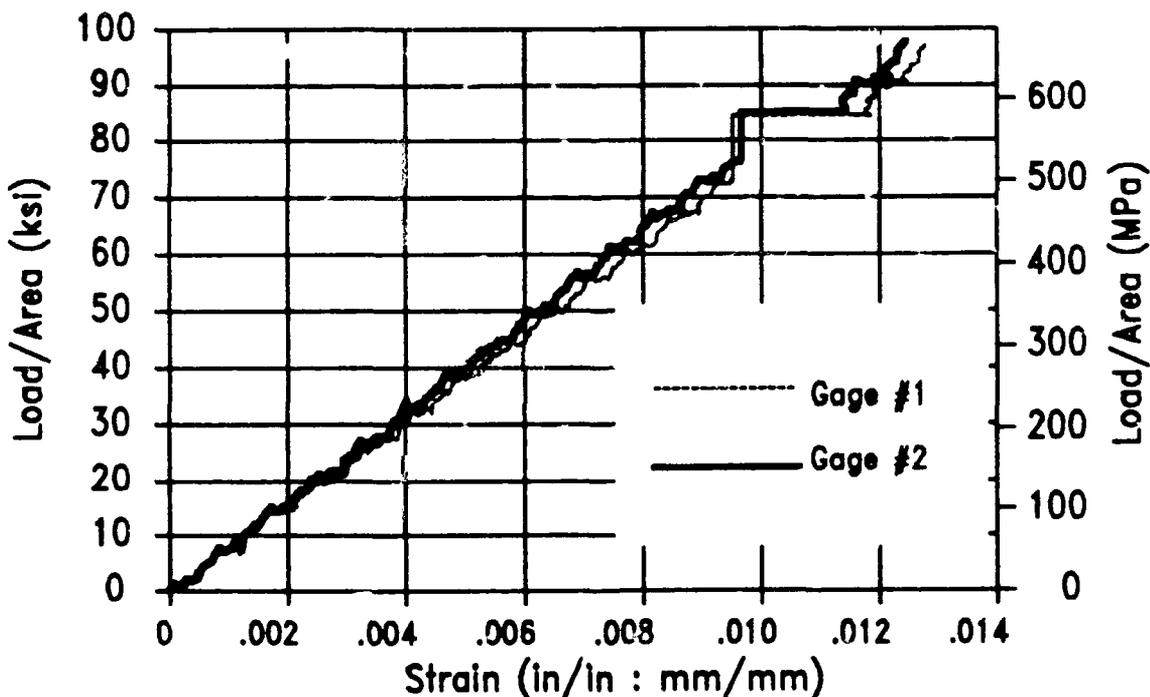


Figure C4. Load versus Strain Gage Output for Sample JH13189-4 #3

Failure was limited to the immediate area of the fracture. Virtually no delamination was observed except in the midplane of the specimen, where the failure remained between the intially separated ply. The strong bonding of fibers to matrix which prevented this delamination from running into other plys, and the nearly intact fracture surface show that the lamination process in this specimen was very sucessful (Figures C5-C8).

Lastly, fiberglass end tabs were attached to specimens intended for tensile testing, and panels were cut to size using a diamond saw.

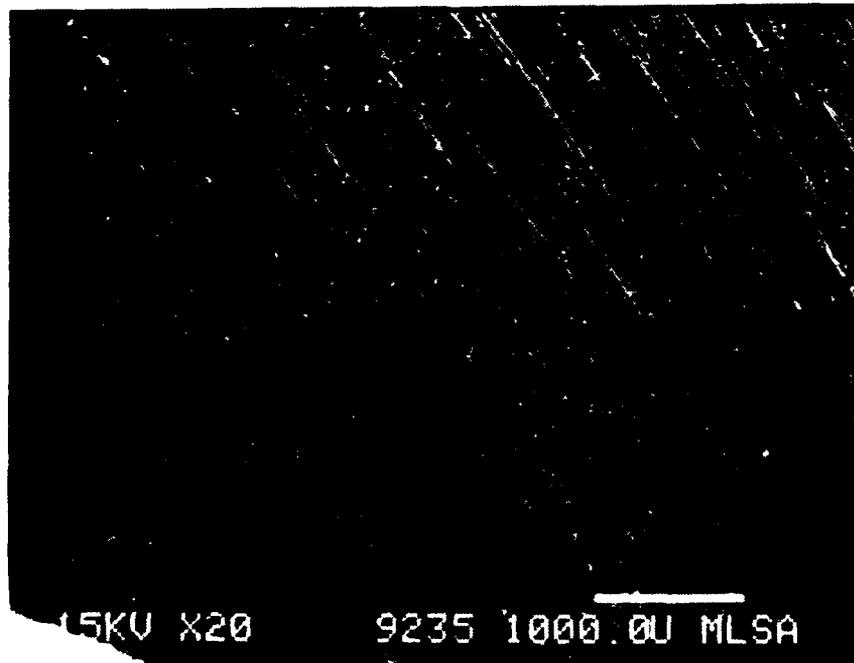


Figure C5. Sample JH13189-4 #3, Delamination Surface (20X)

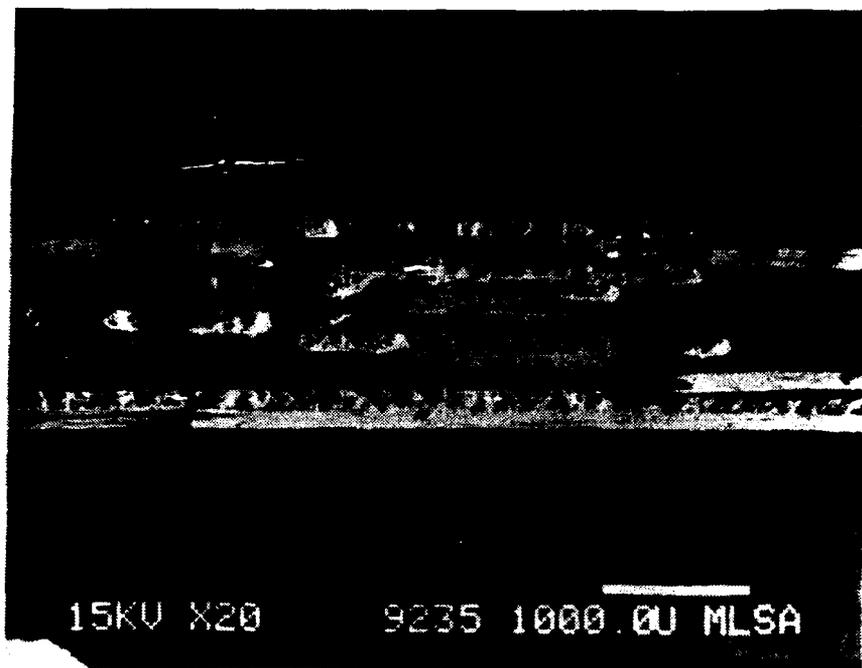


Figure C6. Sample JH13189-4 #3, Fracture Surface (20X)

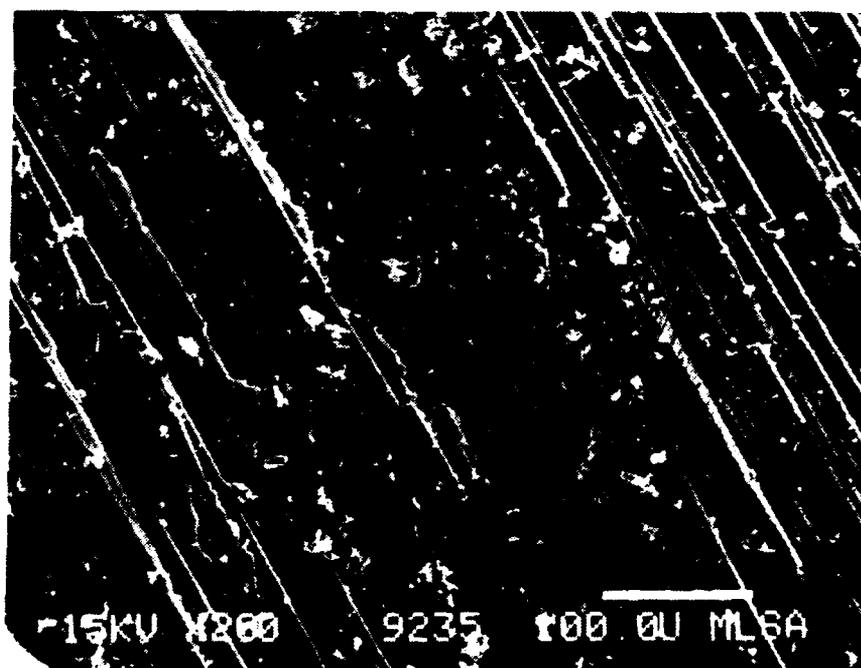


Figure C7. Sample JH13189-4 #3, Delamination Surface (200X)

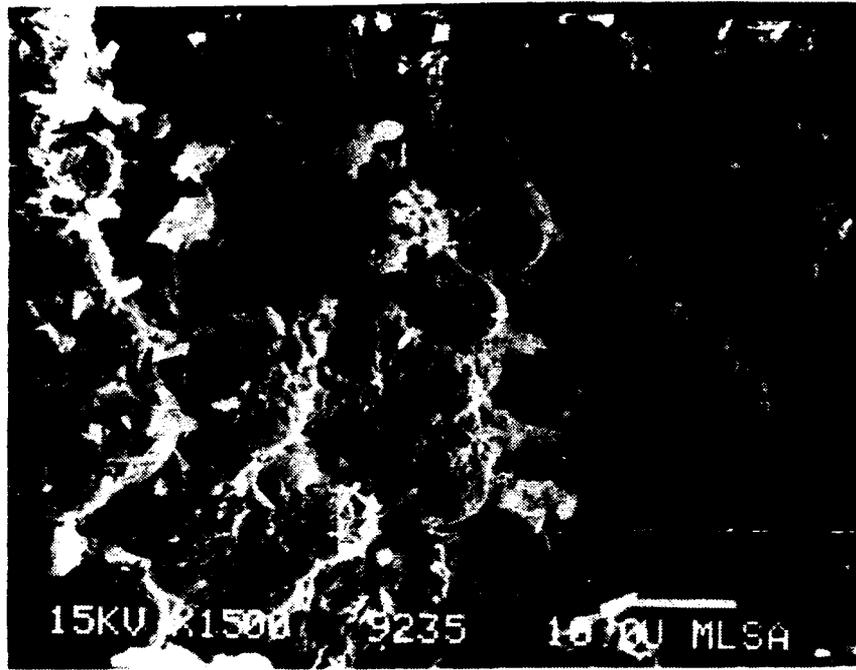


Figure C8. Sample JH13189-4 #3, Fracture Surface (1500X)

Appendix D: Test Results from LHMEL-1

To verify the plausibility of exploring a decrease in effective ablation enthalpy for graphite/epoxy, initial tests were conducted using the 15 kW Laser Hardened Materials Evaluation Laser-1 (LHMEL-1). These were cursory tests with very limited objectives. The only results intended were to see if graphite/epoxy did indeed ablate faster when loaded in tension than when unloaded. These data are shown in Figure D1. The number labeling the data points are intensities at which each was taken. Some data on temperature rise in the center of sample #CH13989-4#12 was also collected (Figure 3). This sample was 2.0 inches (5.08 cm) wide and 0.244 inches (0.620 cm) thick the thermocouple plotted in Figure 3 was in the center of the sample. It was exposed to  $1700 \text{ kW/cm}^2$  (11 kW power over  $6.53 \text{ cm}^2$  area) for 6.0 seconds. These parameters were input to MELT3D.FOR to generate the data plotted in the figure.

Table III.  
LHMEL-1 Test Results

Power (kW)	Area ( $\text{cm}^2$ )	Intensity ( $\text{kW/cm}^2$ )	Time (sec)	Mass Loss (gm)	Q*eff (kJ/gm)	LOAD (lbf)	Stress (N)	Stress Ratio
11.5	1.54	7.5	2.00	0.65	35.3	12000	53379	0.33
11.5	1.54	7.5	1.19	0.40	34.2	0	0	0.00
11.5	2.18	5.3	2.18	0.75	33.4	12000	53379	0.50
11.5	2.18	5.3	3.00	0.90	38.3	6000	26689	0.25
11.5	2.18	5.3	2.20	0.60	42.1	9000	40034	0.37
11.5	2.18	5.3	6.50	1.88	39.7	0	0	0.00

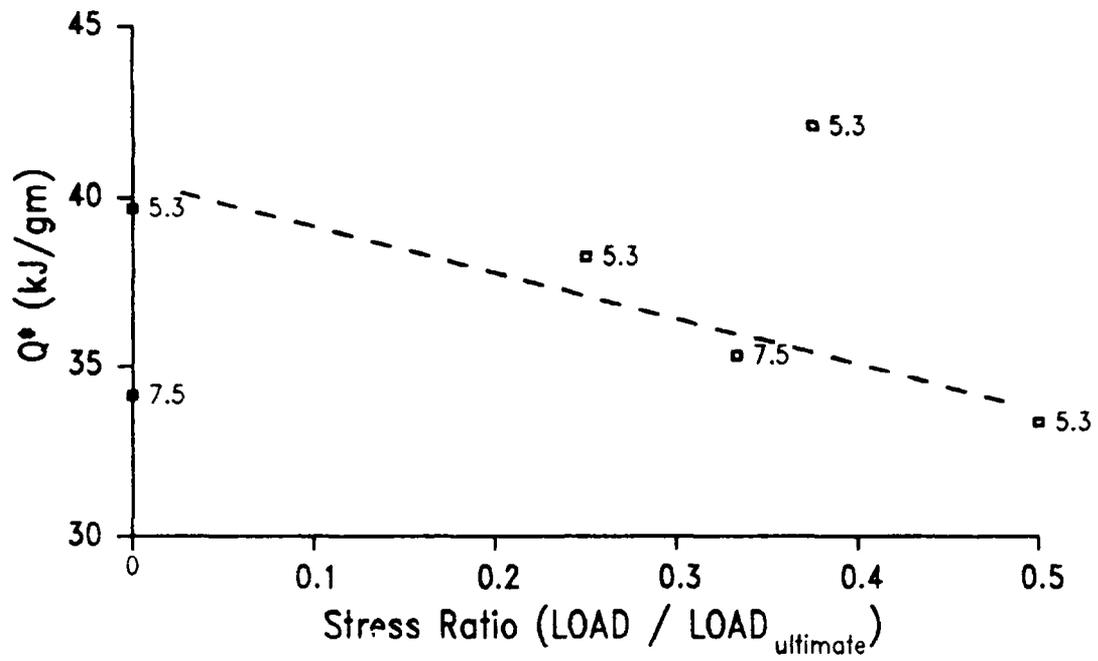


Figure D1. LHME1-1 Preliminary Results

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19. Abstract

An investigation was conducted to quantify the effect structural loading of a graphite/epoxy member has upon the mass removal rate during laser ablation. An effective heat of ablation ( $Q^*_{\text{eff}}$  = energy absorbed / mass removed) was used as a measure of this efficiency. A simple physical model of the important factors affecting the graphite/epoxy was developed, and predictions were made of the effect of loading on  $Q^*_{\text{eff}}$ .

A three-dimensional finite difference heat transfer code was written to predict the temperature distribution in the composite while modeling the orthotropic nature of the thermal conductivity tensor.

The effect of thermal and mechanical loads upon the stress distribution in a single fiber was calculated. A hypothesis was formed that fracture of individual fibers, a process linearly dependent upon applied stress, will remove a fraction of the composite independent of the absorption of laser energy. A coefficient of increasing ablation efficiency,  $\eta_d$ , was postulated to increase linearly with axial stress, leading to a linear decrease in  $Q^*_{\text{eff}}$  with increasing stress.

Uniaxial tensile coupons were fabricated from AS4/3502 graphite/epoxy prepreg, in balanced, symmetric layups ( $[0/\pm 60]_nS$ ). These were placed under tensile loads between 0% and 50% of the laminate's ultimate strength. While loaded, they were irradiated with the EDCL-II 10.6  $\mu\text{m}$  device at irradiances between 5 and 26  $\text{kW}/\text{cm}^2$ .

Linear regression of data taken at 15  $\text{kW}/\text{cm}^2$  showed  $\eta_d = 0.013 \text{ kJ/gm} + 1.74 \text{ (kJ/gm)}/\text{GPa} \times \sigma$ . At 26  $\text{kW}/\text{cm}^2$ ,  $\eta_d = 0.087 \text{ kJ/gm} + 3.36 \text{ (kJ/gm)}/\text{GPa} \times \sigma$ .

It is concluded that axial fiber stress does linearly decrease the 10.6  $\mu\text{m}$  laser energy needed to ablate AS4/3502 graphite/epoxy. A decrease of less than 20 percent was seen in the tests reported here.

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