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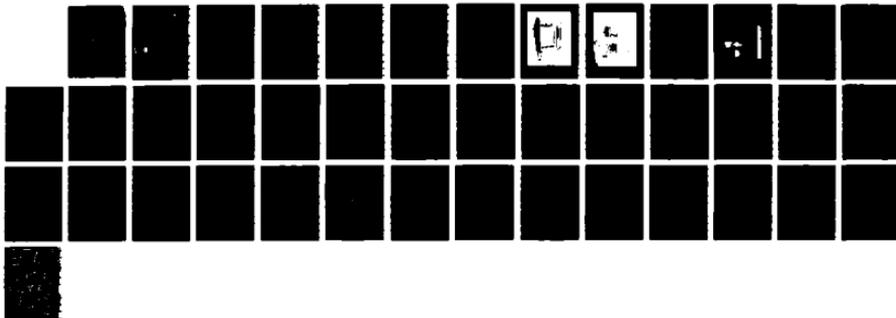
EPIC-2 PREDICTED SHOCK ENVIRONMENTS IN ROLLED  
HOMOGENEOUS ARMOR FOR NONPE. (U) ARMY BALLISTIC  
RESEARCH LAB ABERDEEN PROVING GROUND MD E F QUIGLEY  
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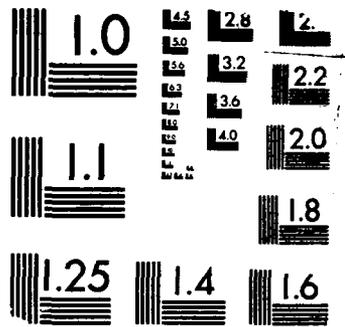
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TECHNICAL REPORT BRL-TR-2886

EPIC-2 PREDICTED SHOCK  
ENVIRONMENTS IN ROLLED  
HOMOGENEOUS ARMOR FOR  
NONPERFORATING BALLISTIC  
IMPACT

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ENNIS F. QUIGLEY

JANUARY 1988

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US ARMY BALLISTIC RESEARCH LABORATORY  
ABERDEEN PROVING GROUND, MARYLAND

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<p>A series of controlled experiments were conducted to characterize the ballistic shock environment produced in rolled homogeneous armor by nonperforating ballistic impact. Displacements, velocities, accelerations and strains were measured at various locations on the back of 38 and 70 mm thick, 914 mm x 914 mm plates impacted by ball bearings and small caliber projectiles (30 caliber to 20 mm). Projectile impact velocities ranged from 335 m/s to 1508 m/s. Peak displacement measurements obtained for projectile impact agreed well with simple momentum predictions. However, velocity and acceleration measurements were inconsistent and a number of accelerometers were destroyed. Because of these measurement difficulties, the EPIC-2 hydro-code was used to calculate the plate velocity and acceleration distributions for the impact of the 20 mm projectile at impact velocities of 366, 1012 and 1508 m/s. Plate displacements, velocities, accelerations and strains were calculated for a 300 μs time period and comparison of measured and calculated displacements and strains for the three impact velocities showed good to excellent agreement. Maximum calculated accelerations below the impact point on the back surface of the plates ranged from 500,000 g's to 4,000,000 g's.</p>			
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## I. INTRODUCTION

A series of controlled experiments[1,2] were conducted by the US Army Combat Systems Test Activity, Aberdeen Proving Ground, MD to characterize the ballistic shock environment produced in rolled homogeneous armor (RHA) by nonperforating ballistic impact. Displacements, velocities, accelerations and strains were measured at various locations on the back of 914 mm x 914 mm x 38 mm and 914 mm x 914 mm x 70 mm RHA plates impacted by small caliber projectiles (30 caliber to 20 mm). Projectile impact velocities ranged from 335 m/s to 1508 m/s. Although peak displacement measurements obtained agreed well with simple momentum predictions, velocity and acceleration measurements were inconsistent and a number of accelerometers were destroyed. Because of these measurement difficulties, the EPIC-2 hydro-code[3] was used to calculate the plate velocity and acceleration distributions for the impact of the 20 mm steel projectile at impact velocities of 366, 1012 and 1508 m/s. This report presents the results of the EPIC-2 calculations.

## II. BALLISTIC IMPACT EXPERIMENT

Figure 1 is a photograph of one of the test plates bolted to the support structure. Eighteen bolts were used to attach the plate to the structure. The plate was mechanically and electrically isolated from the structure by two dissimilar materials (aluminum and fiber glass) placed between the plate and the bolting bars. The ball-peen and sledge hammers in the photograph were used to check out the instrumentation. Also seen in the photograph is a 50 mm diameter ball bearing used in one of the experiments.

Displacements were measured using commercial, non-contacting eddy-current probes and a locally fabricated, capacitive displacement transducer. Velocity transducers supplied by the Naval Ship Research and Development Center were used to measure the plate velocities. Three different models of piezoresistive accelerometers were used to measure accelerations. Conventional foil-type and semi-conductor type strain gages were used to measure strain. The acceleration, velocity and strain transducers were attached to the plate while the displacement transducers were mounted on a transducer support frame directly behind the plate. Figure 2 is a photograph showing the transducers on the back of the plate and Figure 3 is a diagram of the locations of those transducers to the left of the impact point as seen from the back of the plate.

Figure 4 shows the 20 mm projectile before and after impact for the three impact velocities. The single projectile on the left is the "before". There was some plastic deformation but no material loss for the 366 m/s velocity. For the 1012 m/s velocity there was considerable plastic deformation and material loss. At 1508 m/s, the experimentalist could find nothing remaining of the projectile.

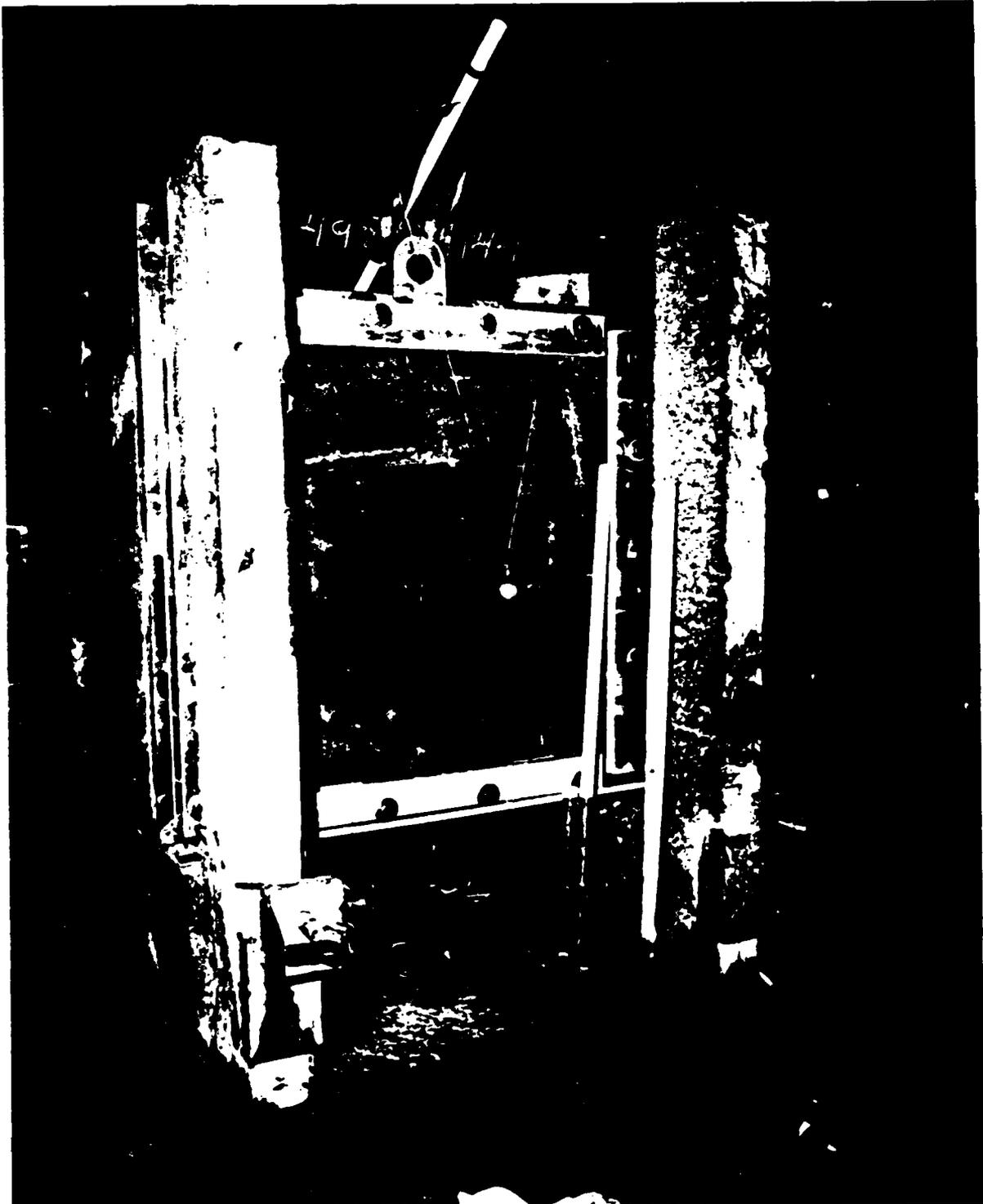


Figure 1. Photograph of RHA plate bolted to support structure.

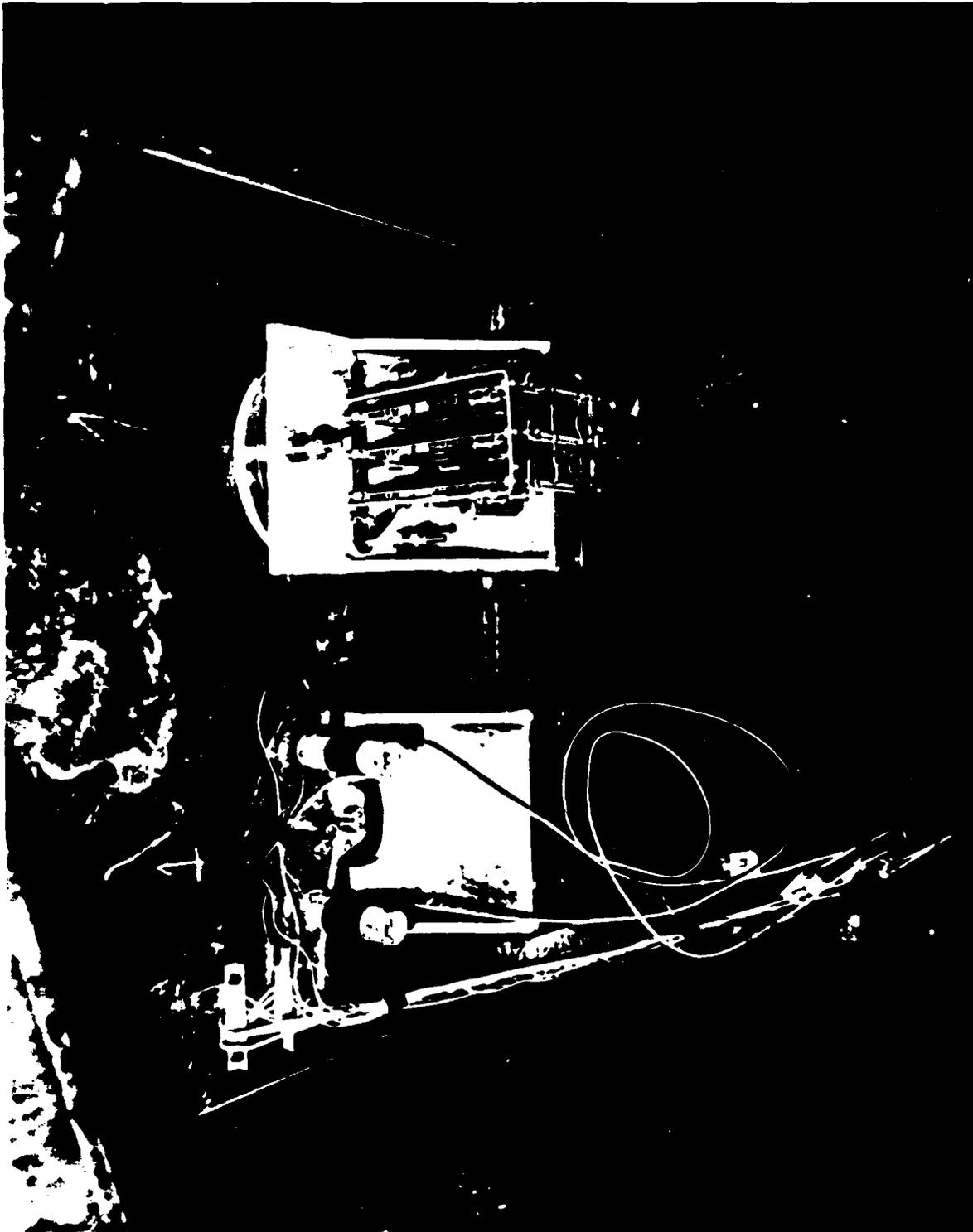


Figure 2. Photograph of transducers on back of plate.

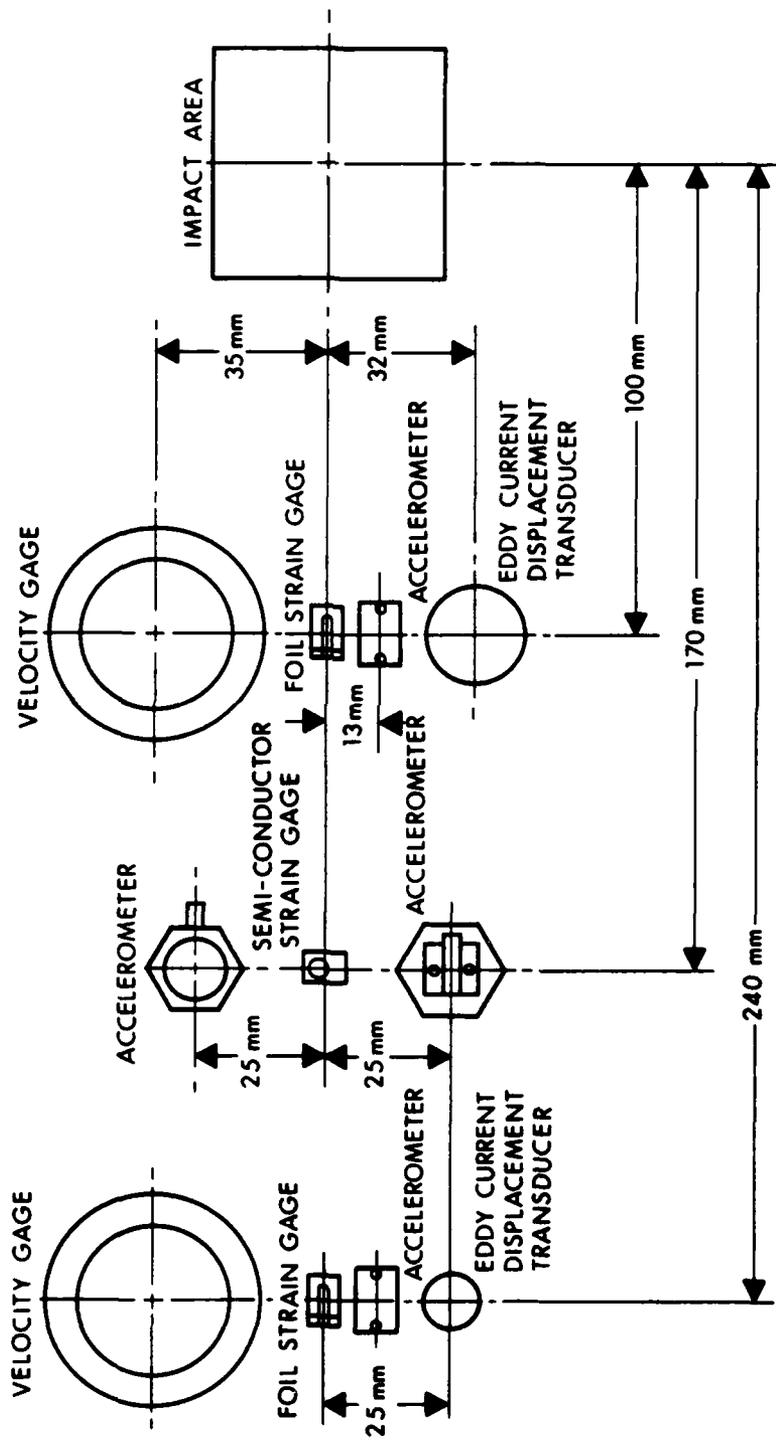


Figure 3. Location of transducers to the left of the impact area.



366 m/s



1012 m/s

1508 m/s



Figure 4. Photograph of projectile before and after impact.

### III. THE EPIC-2 CODE CALCULATIONS

The major characteristics of the EPIC-2 code are listed in Table 1. The code performs elastic-plastic impact computations in two dimensions for axisymmetric and plane strain problems with either free or fixed boundaries. It is based on a Lagrangian finite element and lumped mass formulation in which the equations of motion are integrated directly. Nonlinear material strength and compressibility effects are included to account for elastic-plastic flow and wave propagation. The code has material descriptions for strain hardening, strain rate effects, thermal softening and failure. It uses a constant strain, triangular finite element which is well suited to represent the severe distortions occurring during high velocity impact.

The finite element models of the projectile and the 38 mm thick plate are shown in Figure 5. The projectile consisted of 84 elements and 57 nodes. The horizontal distance between the nodes was 2.54 mm and the vertical distance between the nodes ranged from 1.9 mm to 4.45 mm. The bolted, square plates were modeled as circular ones, 914 mm in diameter with free boundaries. Only the first 100 mm of the plate's 457 mm radius is shown. The 38 mm thick plate consisted of 671 nodes and 1200 elements and the 70 mm plate consisted of 1037 nodes and 1920 elements. Both the horizontal and vertical distances between the nodes were 7.62 mm. The material properties of the projectile and the plates used in the calculations are listed in Table 2.

The use of an axisymmetric code to model the experiments was justified since the impact point in the experiments was the center of a square plate. The plate's response for a brief period of time after impact would be axisymmetric and independent of the plate's geometry and edge boundary conditions. The times calculated for the dilatation and shear waves to travel the shortest round trip distance between the impact point and the plate's edges were 157  $\mu$ s and 285  $\mu$ s, respectively. Plate displacements, velocities, accelerations and strains were calculated for the first 300  $\mu$ s after impact. Comparison of calculated axial displacements at a point 240 mm out from the impact point for free and fixed edges showed identical plate responses for times less than 215  $\mu$ s. Comparison of calculated radial strains at the same location showed identical response for times less than 125  $\mu$ s and minor differences in the response for times greater than 125  $\mu$ s.

Because EPIC-2 calculates strain rates and not strains per se, the radial strains were calculated from the EPIC-2 calculated normal stresses using

$$\epsilon_{rr} = E^{-1}[\sigma_{rr} - \nu\{\sigma_{\theta\theta} + \sigma_{zz}\}]$$

where  $\epsilon_{rr}$  is the radial strain,  $\sigma_{rr}$  is the radial stress,  $\sigma_{\theta\theta}$  is the tangential stress and  $\sigma_{zz}$  is the axial stress. The values of Young's Modulus,  $E$ , and Poisson's ratio,  $\nu$ , used in the calculations were  $2.041 \times 10^7$  Ncm<sup>-2</sup> and 0.275, respectively.

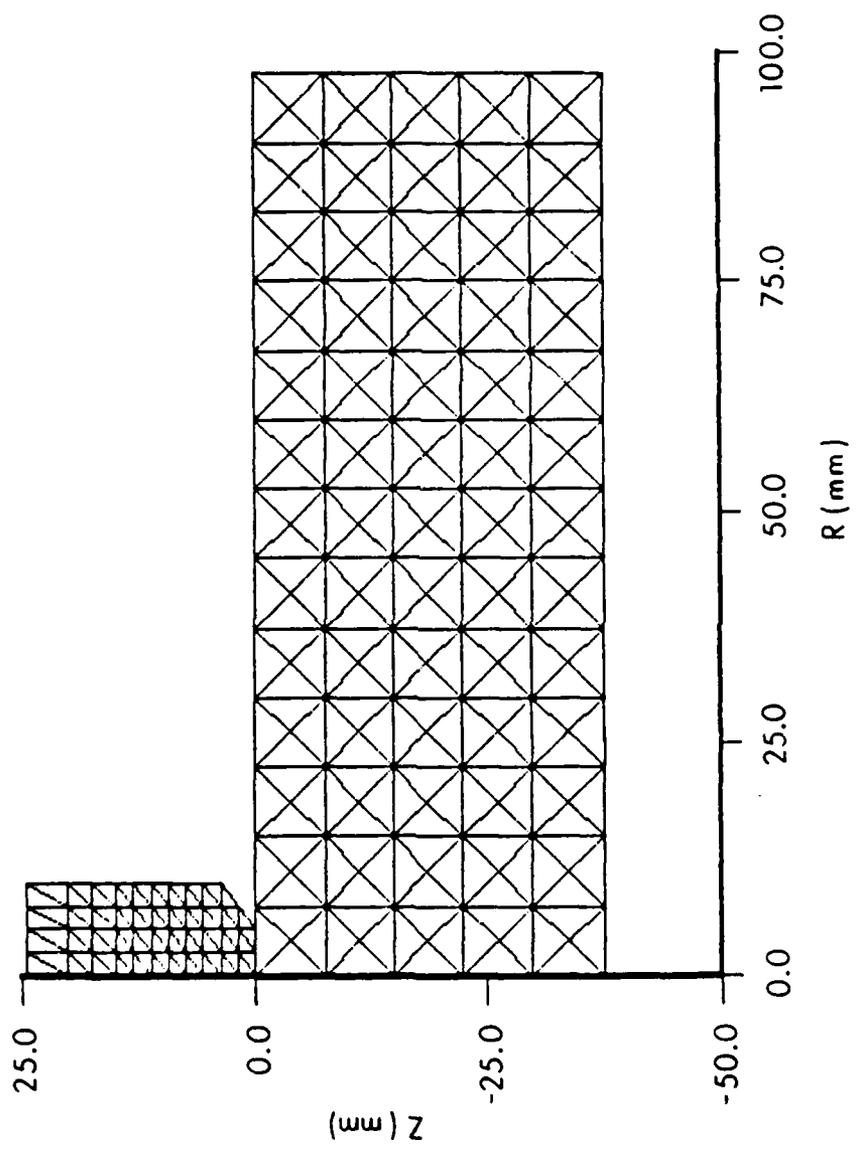


Figure 5. Finite element model of projectile and plate.

**Table 1. EPIC-2 Code Characteristics**

<b>DISCRETIZATION:</b>	<b>FINITE ELEMENT METHOD</b> <ul style="list-style-type: none"> <li>• 2D constant strain triangles</li> <li>• Lumped mass formulation</li> </ul>
<b>MESH DESCRIPTION:</b>	<b>LAGRANGIAN</b>
<b>MATERIAL MODEL:</b>	<b>CONSTITUTIVE MODEL</b> <ul style="list-style-type: none"> <li>• Incremental elastic-plastic</li> <li>• Von Mises yield criterion</li> <li>• Compressibility effects</li> <li>• Strain rate effects</li> <li>• Strain hardening</li> <li>• Thermal softening</li> </ul> <b>EQUATION OF STATE</b> <ul style="list-style-type: none"> <li>• Mie-Gruneisen</li> <li>• Ideal gas</li> </ul>
<b>FAILURE CRITERIA:</b>	<b>VOLUMETRIC STRAIN</b> <b>EFFECTIVE PLASTIC STRAIN</b>
<b>POST-FAILURE MODELS:</b>	<b>PRESSURE CUTOFF</b> <b>SHEAR AND TENSION FAILURE</b> <b>TOTAL FAILURE</b>

#### **IV. COMPARISON OF CALCULATED AND MEASURED RESPONSES**

Figures 6 through 19 are plots of the calculated and measured plate response histories at the 100 mm, 170 mm and 240 mm transducer locations (see Figure 3) for the first 300  $\mu$ s. Figures 6 through 10 are the responses for the 38 mm thick plate for the impact velocity of 366 m/s. Figures 11 through 15 are for the same plate thickness for the 1012 m/s impact velocity. Figures 16 through 19 are for the 70 mm thick plate for the impact velocity of 1508 m/s. Comparisons of the calculated and the measured plate's responses were made for the first 250  $\mu$ s.

Table 2. Material Properties

Property	Projectile	Plate
Density ( $\text{Ns}^2\text{cm}^{-2}$ )	$7.840 \times 10^{-5}$	$7.904 \times 10^{-5}$
Specific heat ( $\text{cm}^2\text{s}^{-2}\text{K}^{-1}$ )	$4.770 \times 10^6$	$4.523 \times 10^6$
Shear modulus ( $\text{Ncm}^{-2}$ )	$7.752 \times 10^6$	$8.019 \times 10^6$
Viscosity ( $\text{Nscm}^{-2}$ )	0.0	0.0
Yield stress ( $\text{Ncm}^{-2}$ )	$1.103 \times 10^5$	$9.162 \times 10^4$
Ultimate stress ( $\text{Ncm}^{-2}$ )	$1.103 \times 10^5$	$1.032 \times 10^5$
Ultimate strain	0.18	0.165
Maximum negative pressure ( $\text{Ncm}^{-2}$ )	$6.897 \times 10^6$	$6.897 \times 10^6$
Strength equation		
Strain rate coefficient	0.0	0
1st pressure coefficient	0.0	0.0
2nd pressure coefficient	0.0	0.0
1st temperature coefficient	1.0	1.0
2nd temperature coefficient	0.0	0.0
Equation of state		
1st pressure coefficient ( $\text{Ncm}^{-2}$ )	$1.640 \times 10^7$	$1.590 \times 10^7$
2nd pressure coefficient ( $\text{Ncm}^{-2}$ )	$2.944 \times 10^7$	$5.170 \times 10^7$
3rd pressure coefficient ( $\text{Ncm}^{-2}$ )	$5.000 \times 10^7$	$5.216 \times 10^8$
Gruneisen coefficient	1.16	1.69
Artificial viscosity equation		
Linear coefficient	0.2	0.2
Quadratic coefficient	4.0	4.0
Shear and tensile failure		
Equivalent strain	$9.99 \times 10^2$	$9.99 \times 10^2$
Volumetric strain	$9.99 \times 10^2$	$9.99 \times 10^2$
Total failure		
Equivalent strain	1.8	$9.99 \times 10^2$
Initial temperature (K)	$2.94 \times 10^2$	$2.94 \times 10^2$

Zero times of the measured axial displacements were not at the time of impact and did not correspond to the zero time of the calculated displacements. In order to compare the data, zero time for the measured data was adjusted to give the best agreement with the calculated data. The agreement in Figure 6 is good whereas the agreement in Figures 7, 11, 12 and 16 is very good. The sign of the calculated displacements was inverted to agree with the positive displacement direction of the measured data.

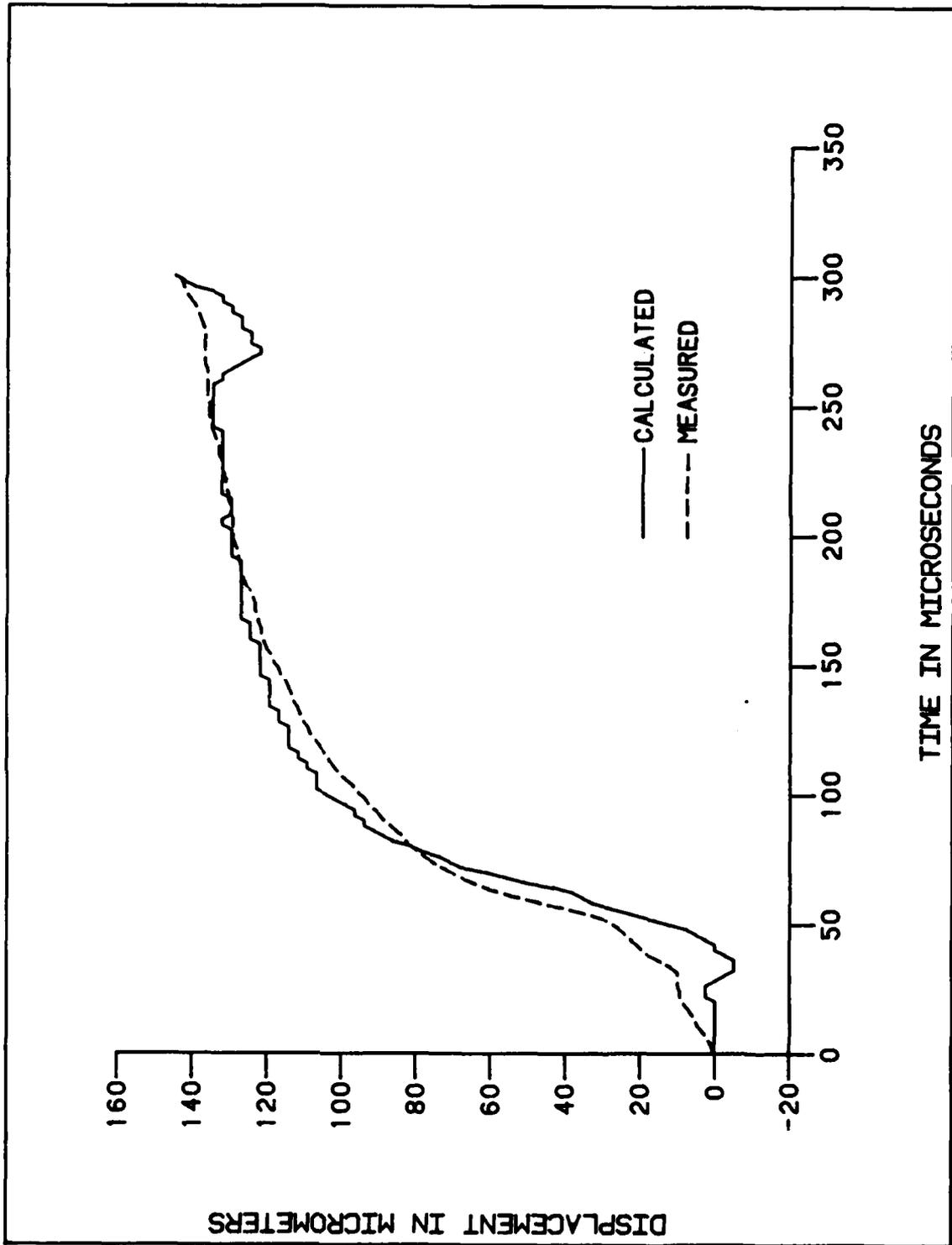


Figure 6. Axial displacement histories of 38 mm thick plate at 100 mm location for projectile impact velocity of 366 m/s.

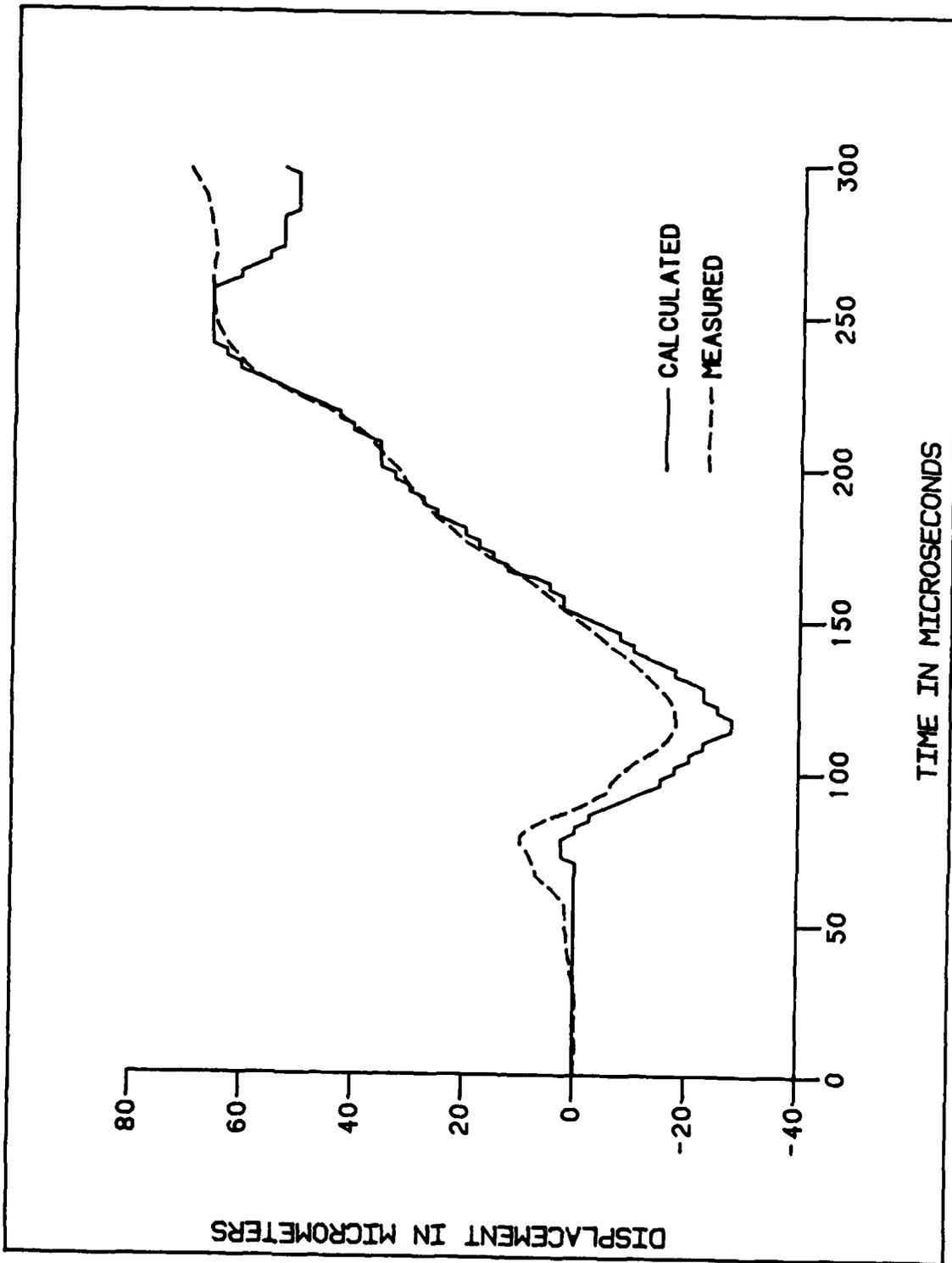


Figure 7. Axial displacement histories of 38 mm thick plate at 240 mm location for projectile impact velocity of 366 m/s.

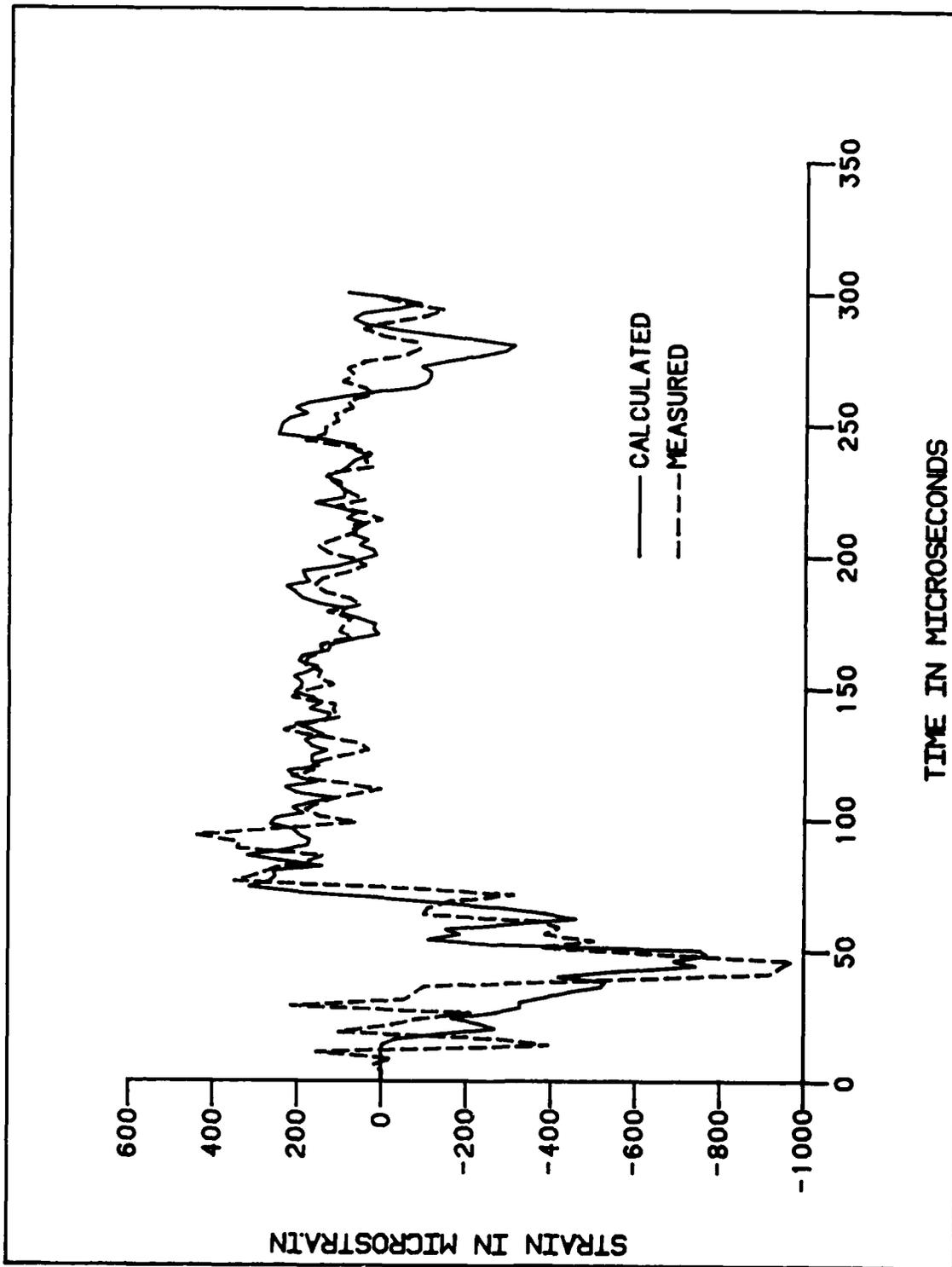


Figure 8. Radial strain histories of 38 mm thick plate at 100 mm location for projectile impact velocity of 366 m/s.

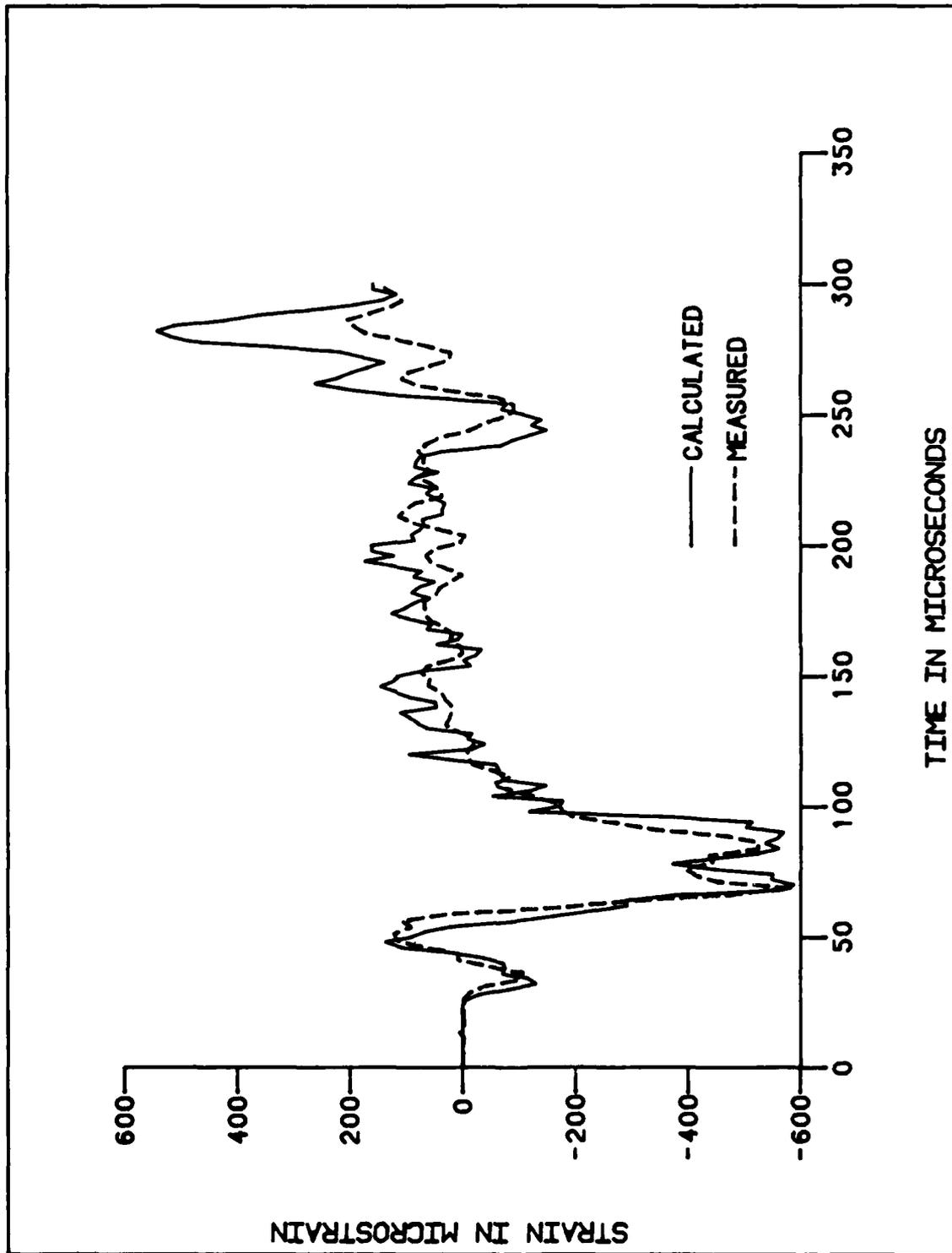


Figure 9. Radial strain histories of 38 mm thick plate at 170 mm location for projectile impact velocity of 366 m/s.

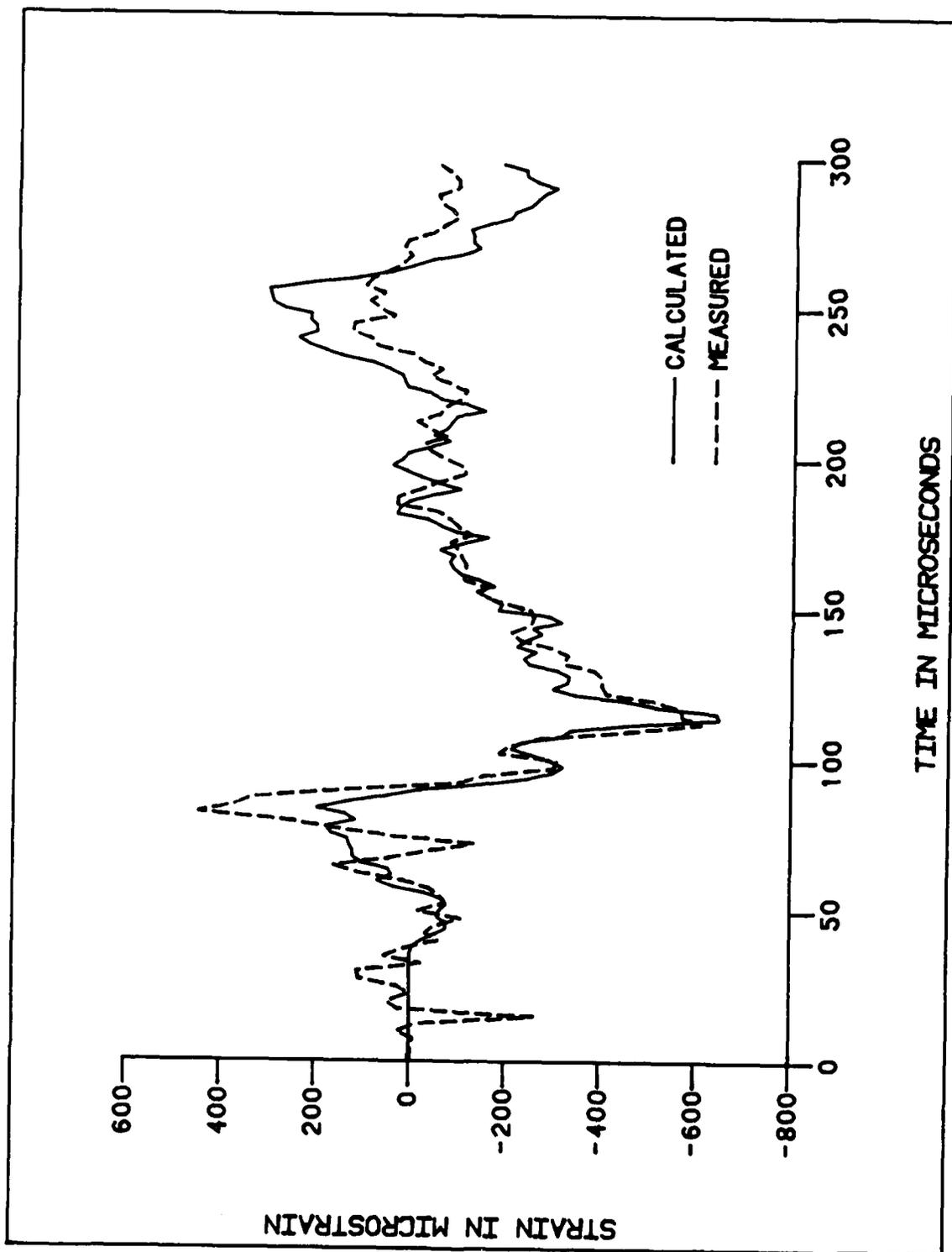


Figure 10. Radial strain histories of 38 mm thick plate at 240 mm location for projectile impact velocity of 366 m/s.

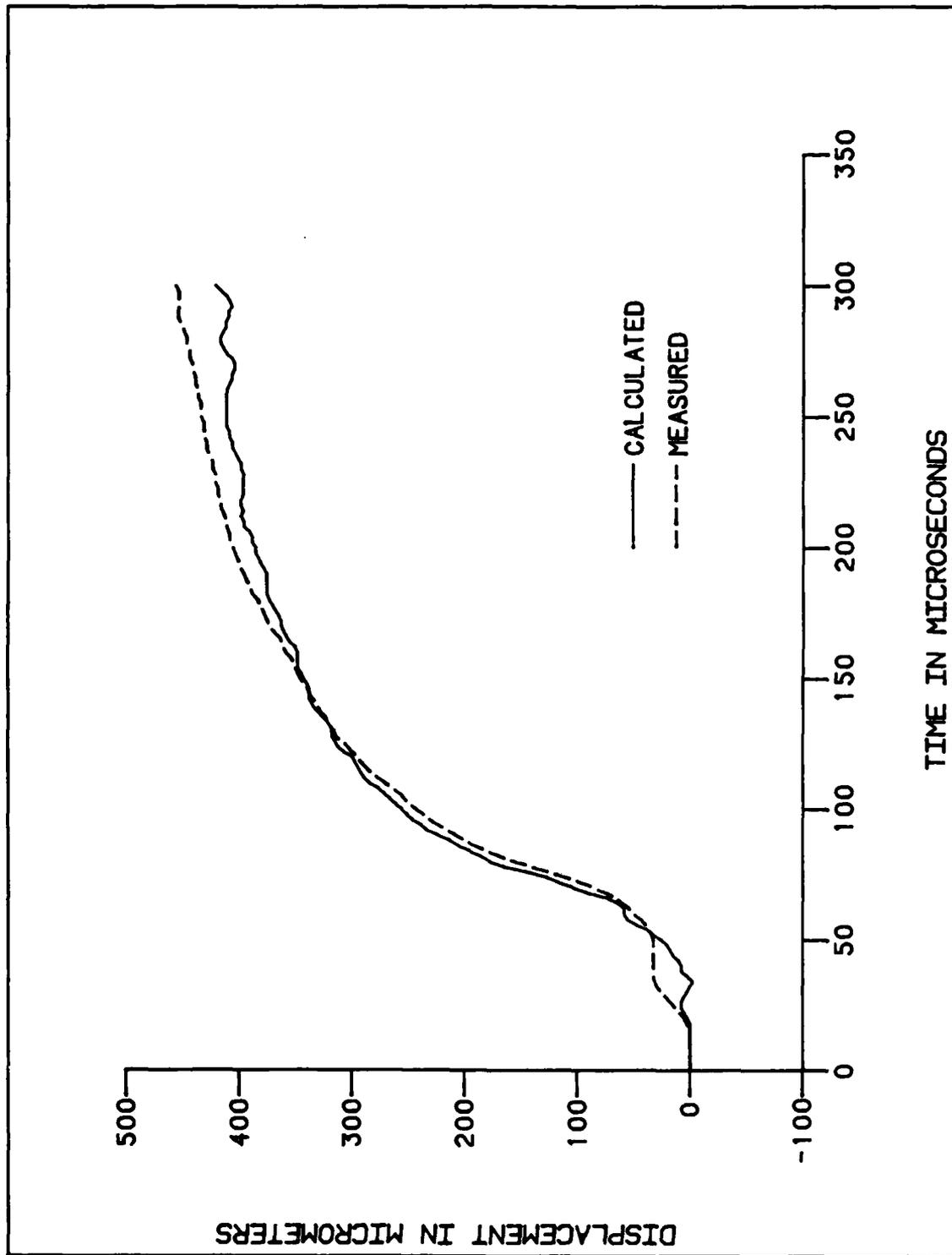


Figure 11. Axial displacement histories of 38 mm thick plate at 100 mm location for projectile impact velocity of 1012 m/s.

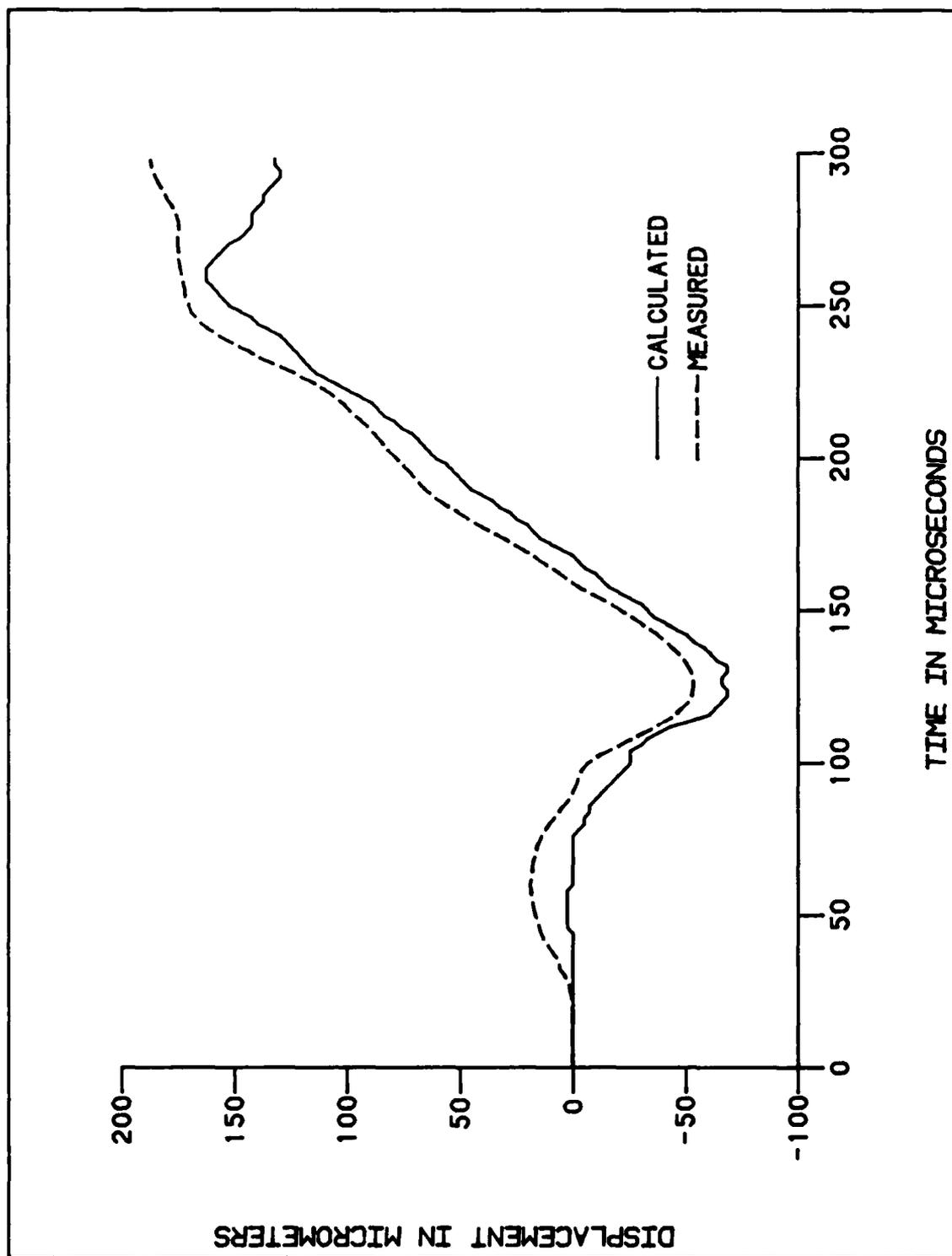


Figure 12. Axial displacement histories of 38 mm thick plate at 240 mm location for projectile impact velocity of 1012 m/s.

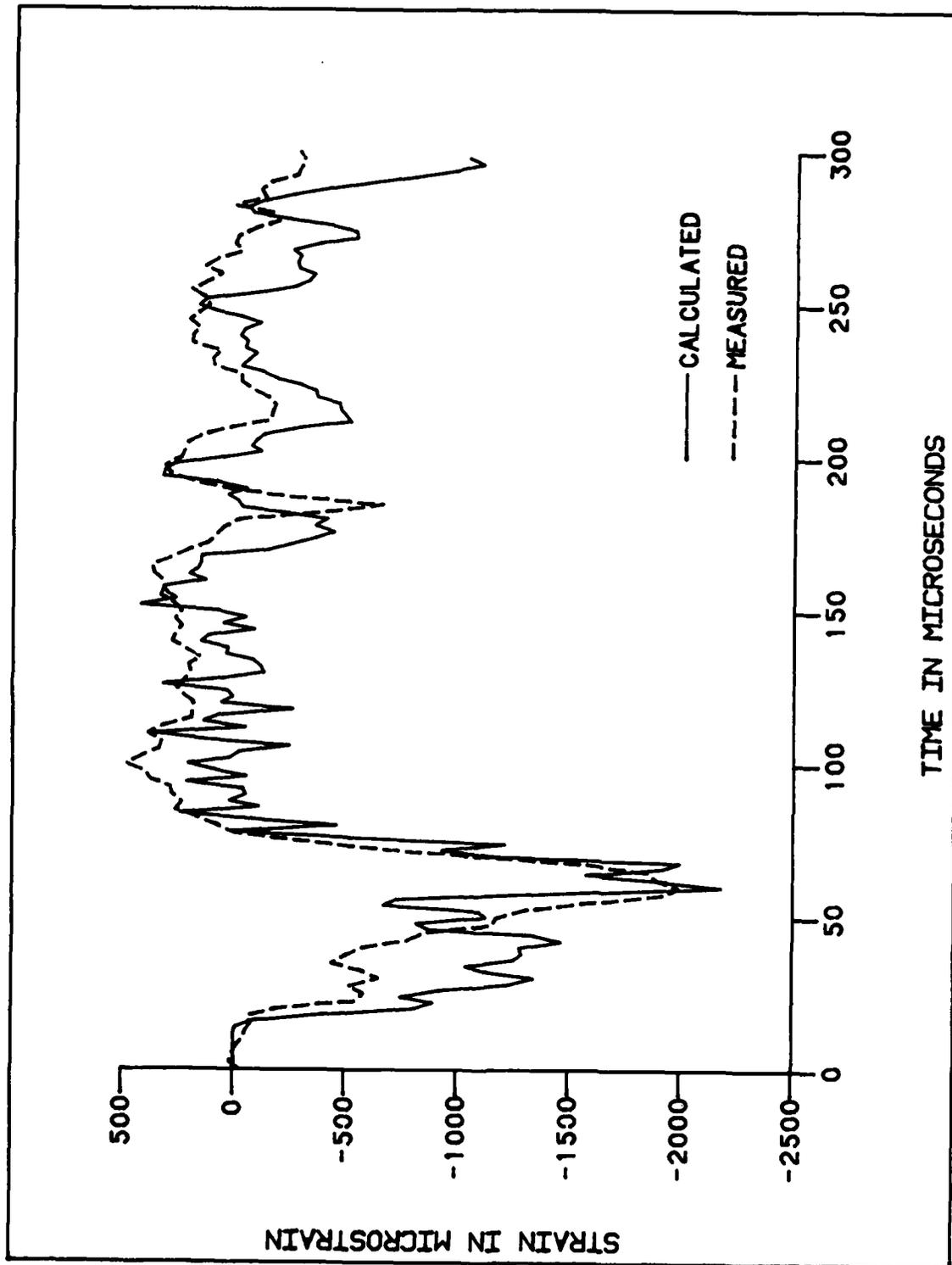


Figure 13. Radial strain histories of 38 mm thick plate at 100 mm location for projectile impact velocity of 1012m/s.

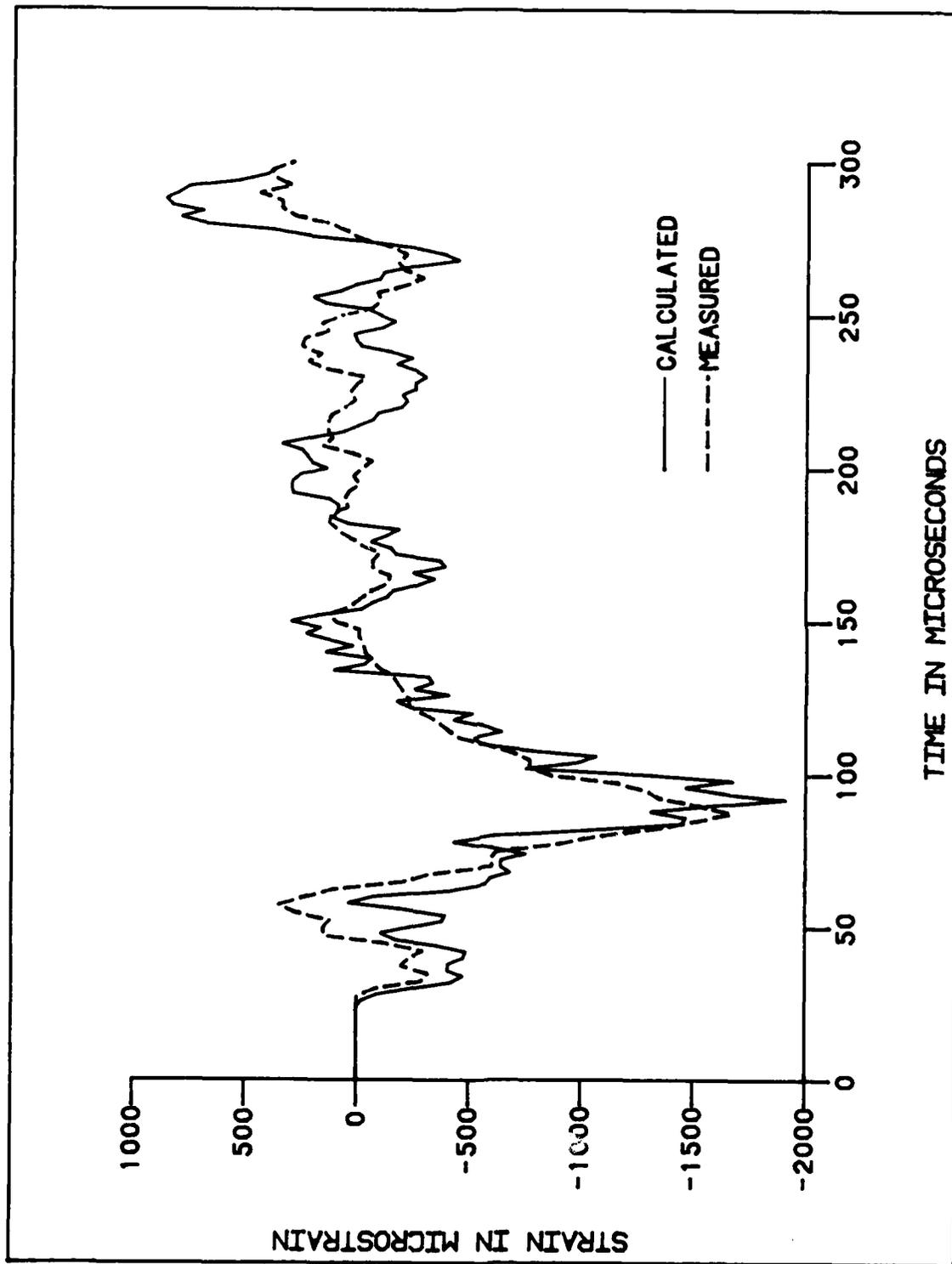


Figure 14. Radial strain histories of 38 mm thick plate at 170 mm location for projectile impact velocity of 1012 m/s.

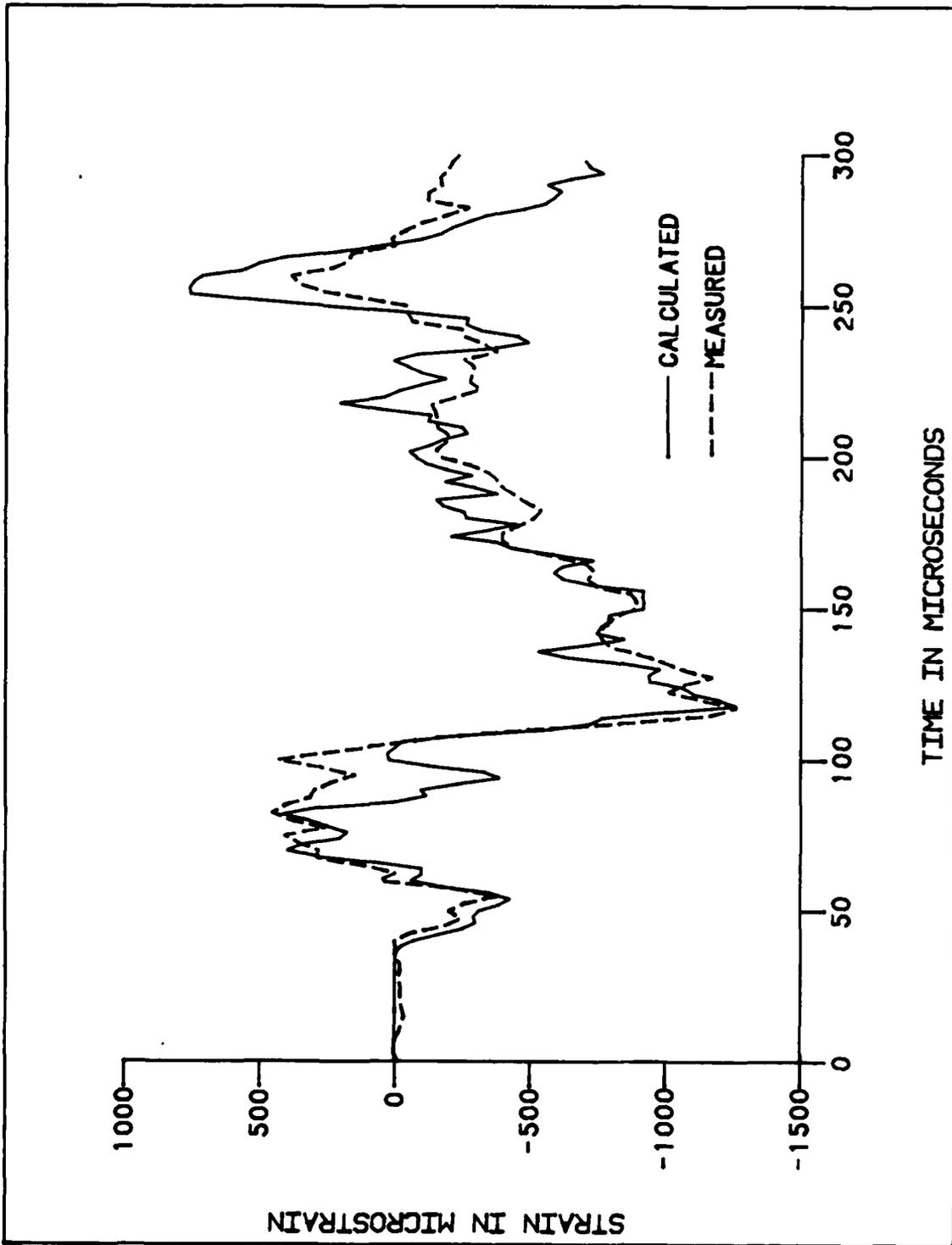


Figure 15. Radial strain histories of 38 mm thick plate at 240 mm location for projectile impact velocity of 1012 m/s.

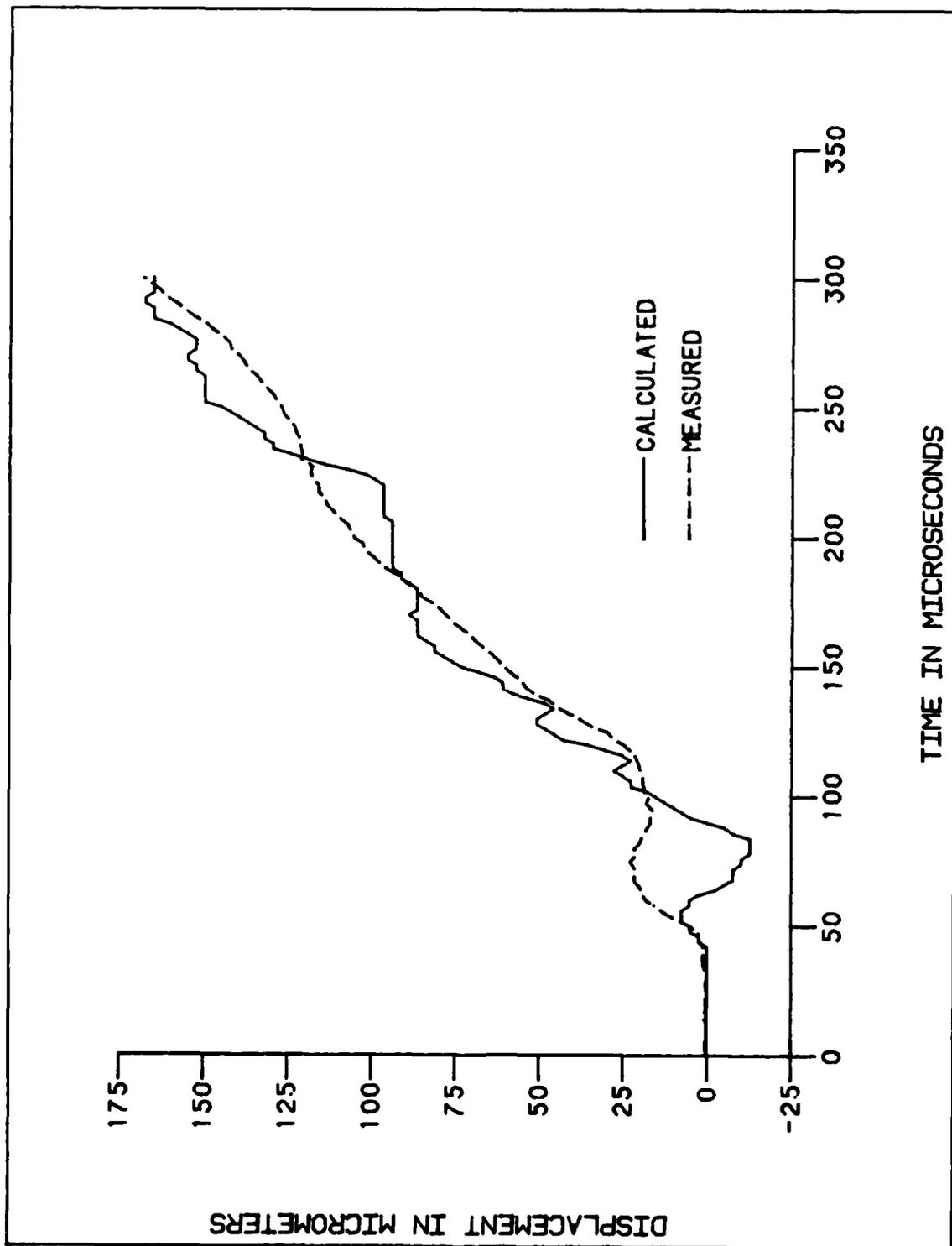


Figure 16. Axial displacement histories of 70 mm thick plate at 240 mm location for projectile impact velocity of 1508 m/s.

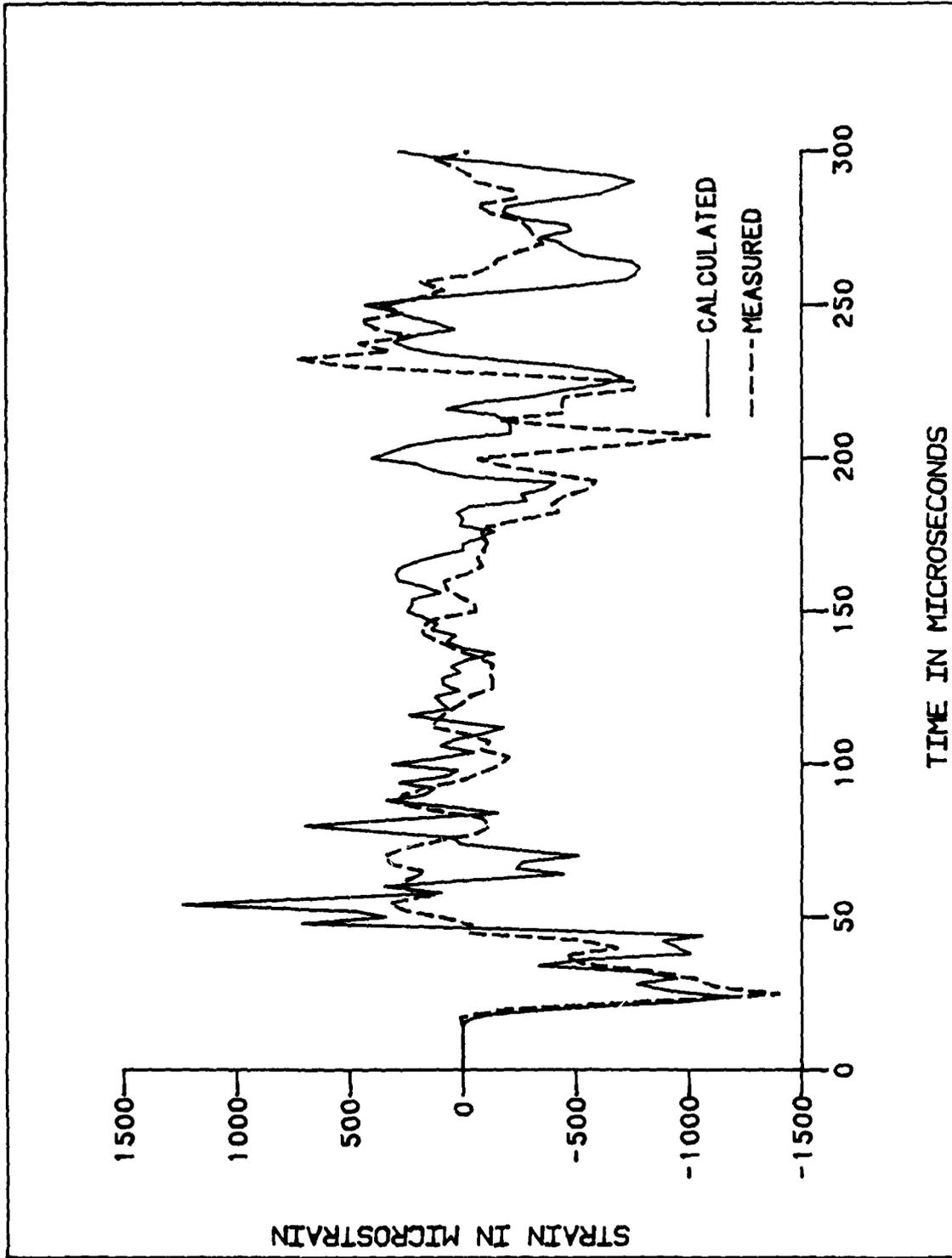


Figure 17. Radial strain histories of 70 mm thick plate at 100 mm location for projectile impact velocity of 1508 m/s.

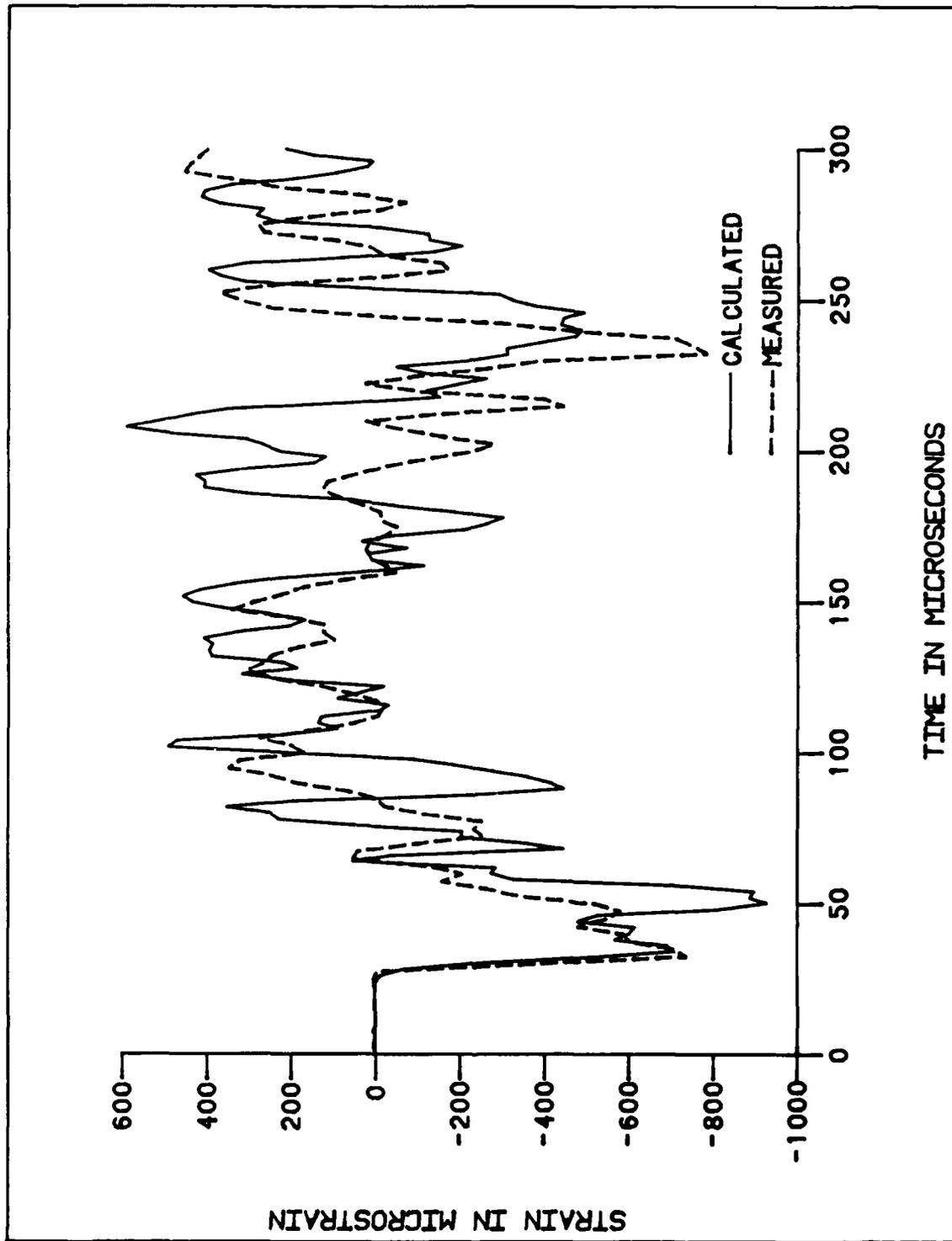


Figure 18. Radial strain histories of 70 mm thick plate at 170 mm location for projectile impact velocity of 1508 m/s.

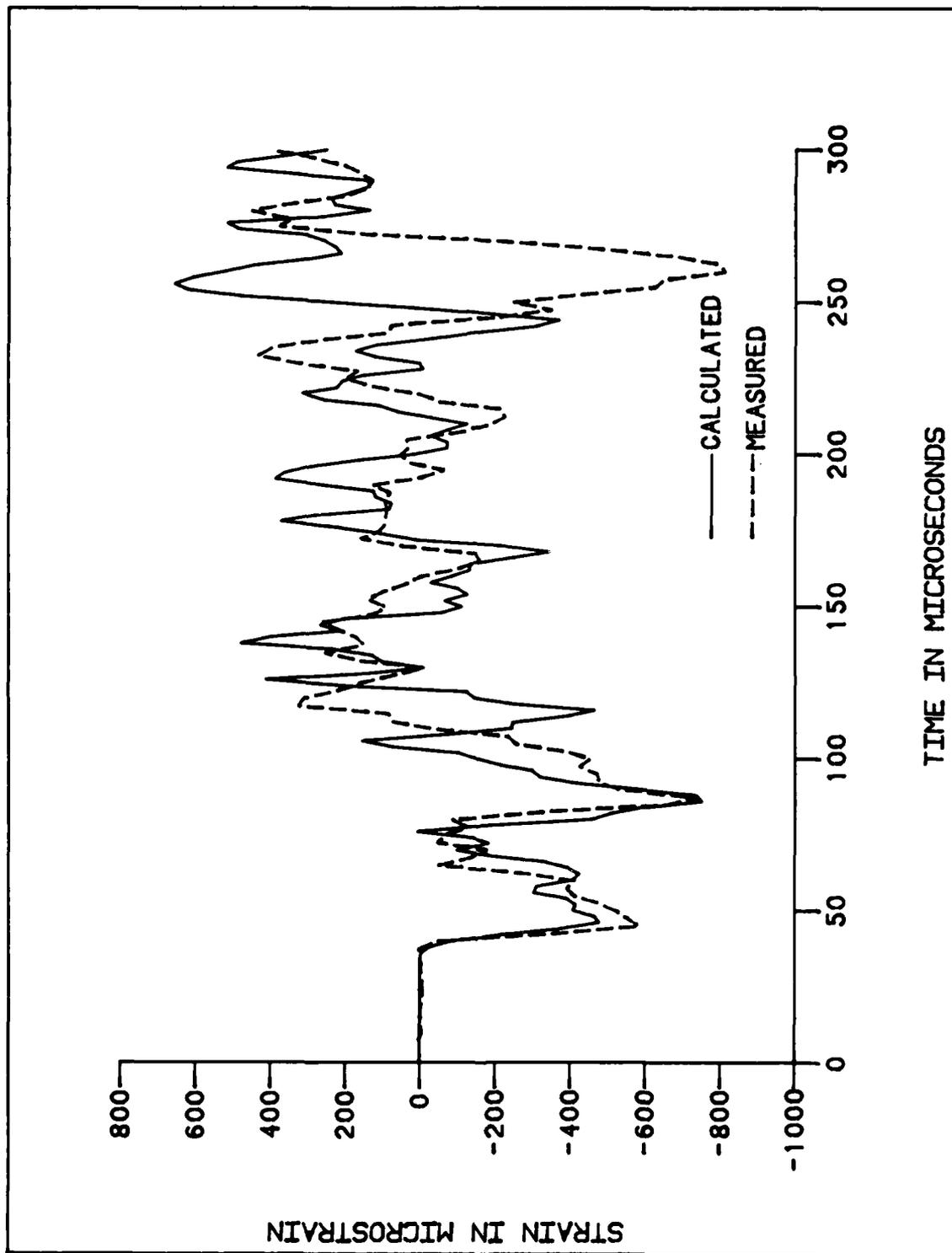


Figure 19. Radial strain histories of 70 mm thick plate at 240 mm location for projectile impact velocity of 1508 m/s.

As in the case of the measured displacements, the zero times for the measured radial strains were not at the time of impact. However, the zero time for the three strain measurements of each experiment was the same. For comparison purposes, the zero time of the measured data was determined by matching up the arrival of the dilation wave associated with the measured strain with that of the calculated strain at the 170 mm location and applying the same zero time adjusted to the other strain measurements. This procedure was followed for each impact velocity. The 170 mm location was chosen because the strain gage there was a semi-conductor type. The gages at the 100 and 240 mm locations were foil type and were subjected to possible electronic cross-talking. The agreement between the calculated and measured radial strains ranged from very good to excellent.

## V. PREDICTED SHOCK ENVIRONMENTS

The axial acceleration histories calculated at the 0 mm and 240 mm locations on the back surface of the plates are shown in Figures 20 through 25. Figure 20 is the acceleration history at the 0 mm position for the 366 m/s impact velocity. The plot shows a frequency of approximately 170 kHz and a maximum peak acceleration of 500 kG's. Figure 21 is the acceleration history at the 240 mm position. The maximum peak acceleration is approximately 40 kG's, down by a factor of 10 compared to the maximum peak acceleration at the 0 mm location. Figure 22 is the acceleration history at the 0 mm position for the 1012 m/s velocity. Again the acceleration frequency is approximately 170 kHz and the peak acceleration is in excess of 3 MG's. Figure 23 is the acceleration history at the 240 mm position. Here the peak acceleration is approximately 150 kG's, down by a factor of 20. Figure 24 is the acceleration history at the 0 mm position for the 1508 m/s velocity. As before, the frequency is approximately 170 kHz. The peak acceleration is in excess of 2 MG's. This is lower than the peak acceleration for the 1012 m/s velocity because the higher velocity plate is 32 mm thicker. Figure 25 is the acceleration history at the 240 mm position. Here the peak acceleration is approximately 100 kG's, down by a factor of 20.

The maximum peak acceleration occurs during the first 50  $\mu$ s at the 0 mm location for all three impact velocities. The calculated projectile impact duration times were 43  $\mu$ s, 48  $\mu$ s and 65  $\mu$ s for the impact velocities of 366 m/s, 1012 m/s and 1508 m/s, respectively.

## VI. CONCLUSIONS

1. The EPIC-2 hydro-code has calculated the responses of RHA plates to nonperforating ballistic impact which are in very good agreement with experimental results.
2. The predicted shock environment in the vicinity of the impact is characterized by a high frequency acceleration history (kHz range) with large peak accelerations (kG's to MG's range). However, it appears that the magnitude of the peak acceleration decreases rapidly as the disturbance propagates radially outward from the point of impact.

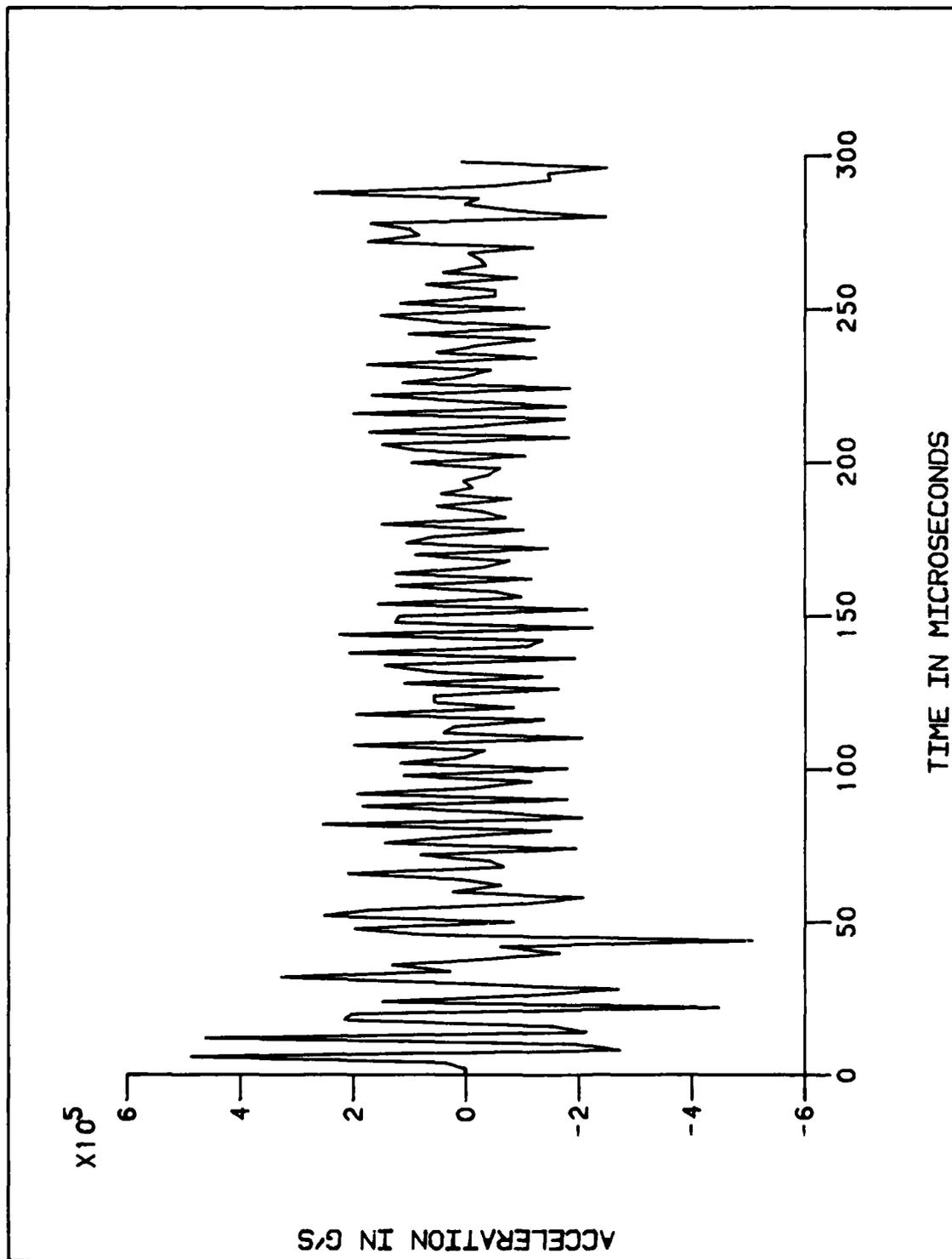


Figure 20. Axial acceleration history of 38 mm thick plate at 0 mm location for projectile impact velocity of 366 m/s.

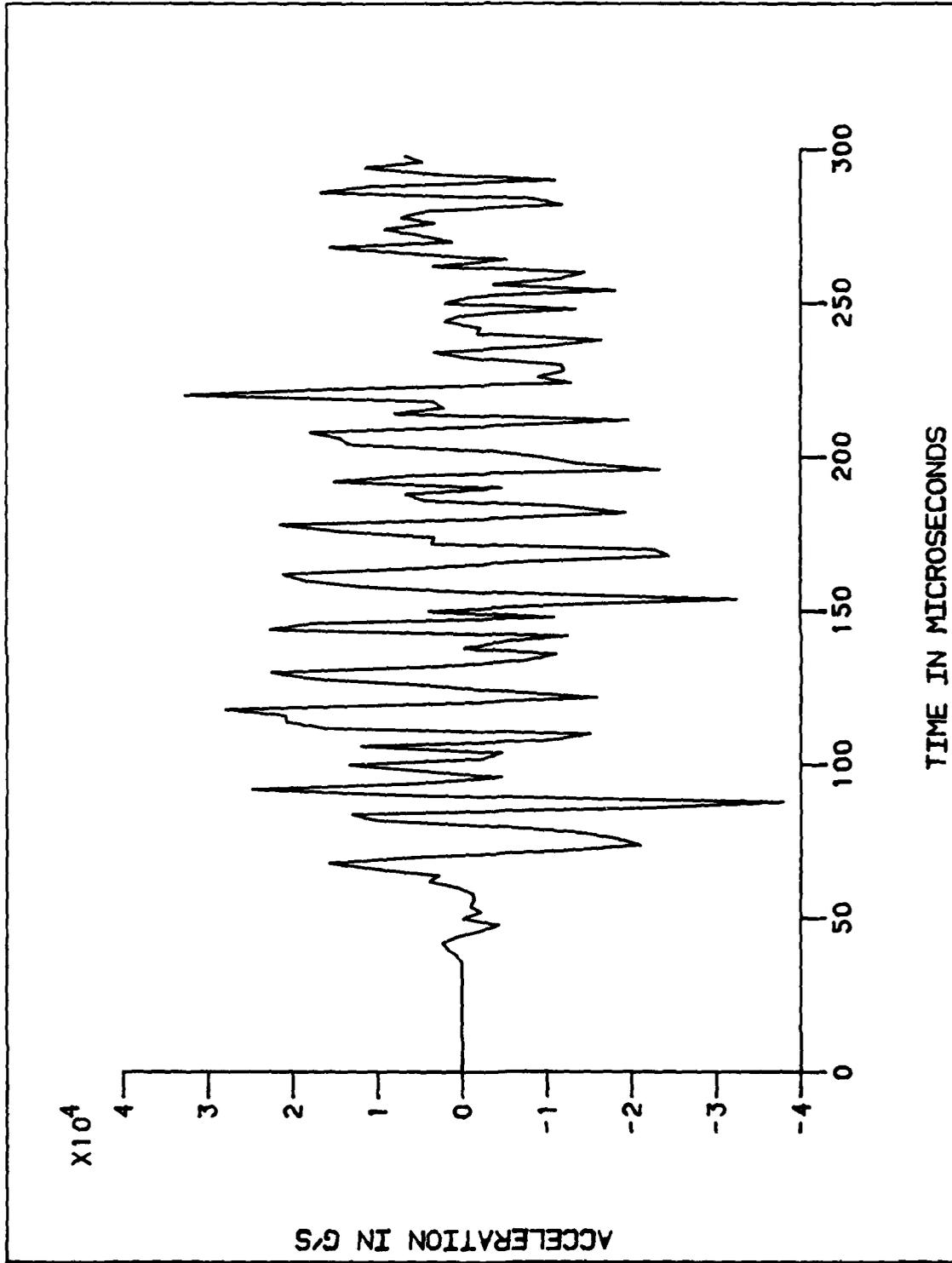


Figure 21. Axial acceleration history of 38 mm thick plate at 240 mm location for projectile impact velocity of 366 m/s.

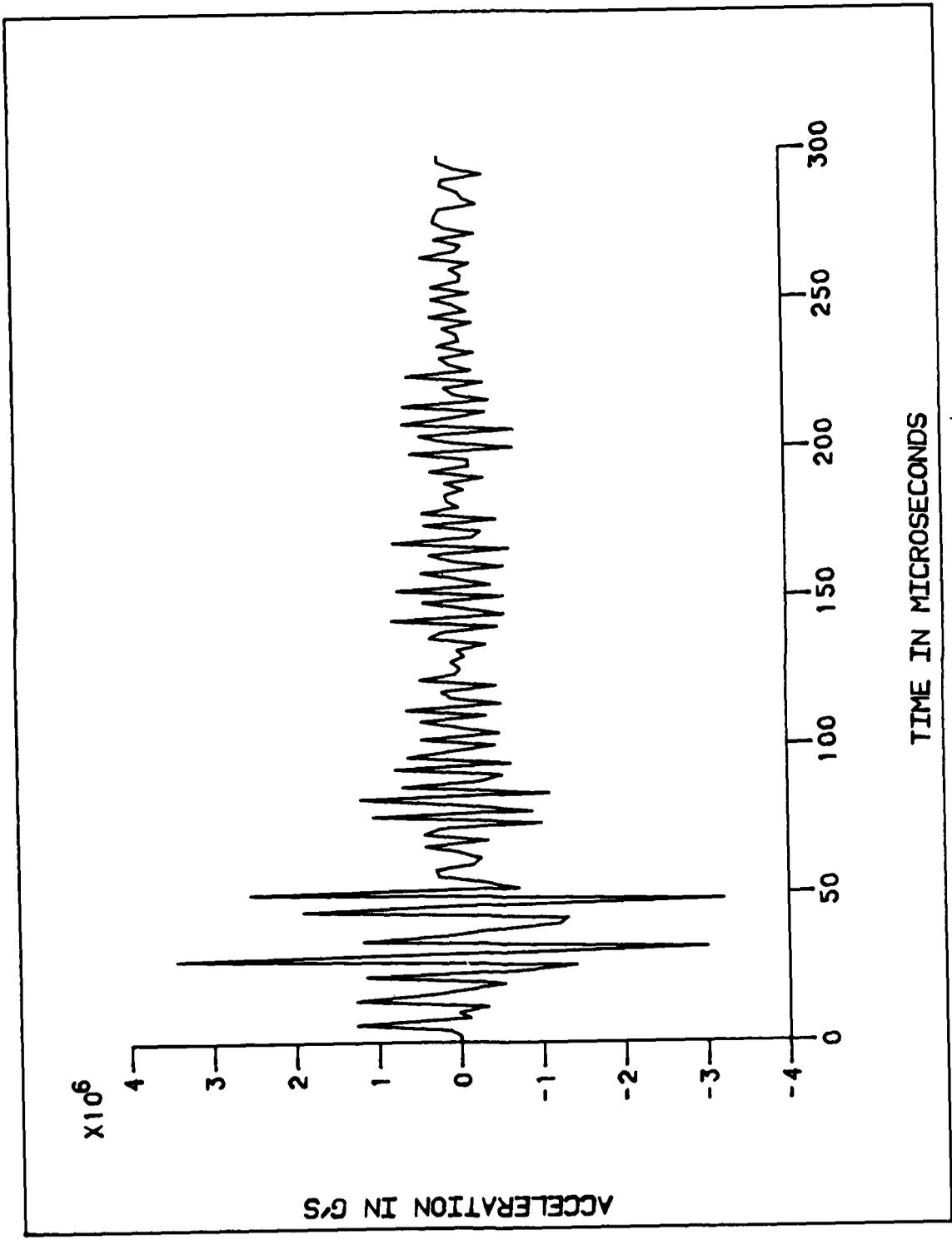


Figure 22. Axial acceleration history of 38 mm thick plate at 0 mm location for projectile impact velocity of 1012 m/s.

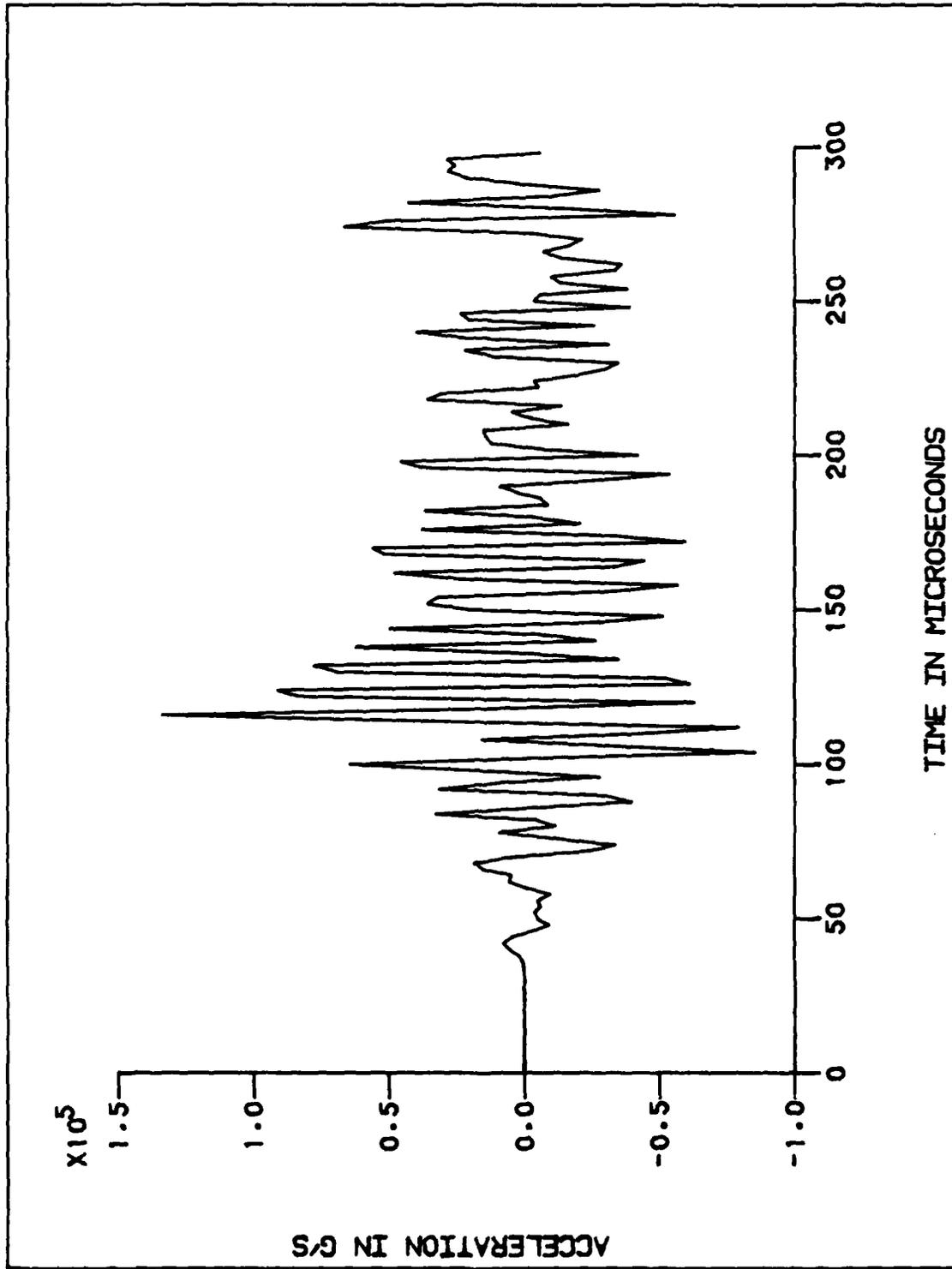


Figure 23. Axial acceleration history of 38 mm thick plate at 240 mm location for projectile impact velocity of 1012 m/s.

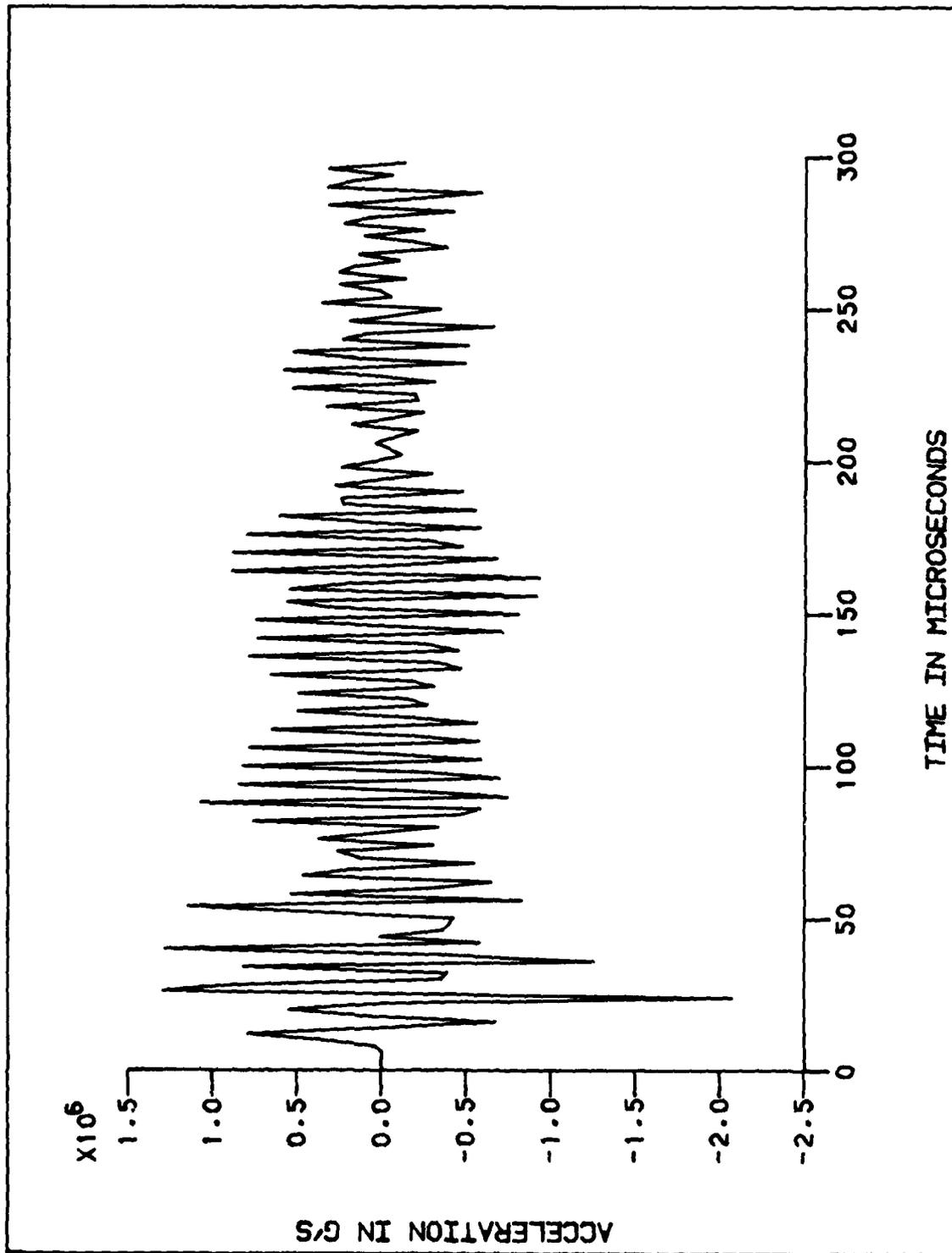


Figure 24. Axial acceleration history of 70 mm thick plate at 0 mm location for projectile impact velocity of 1508 m/s.

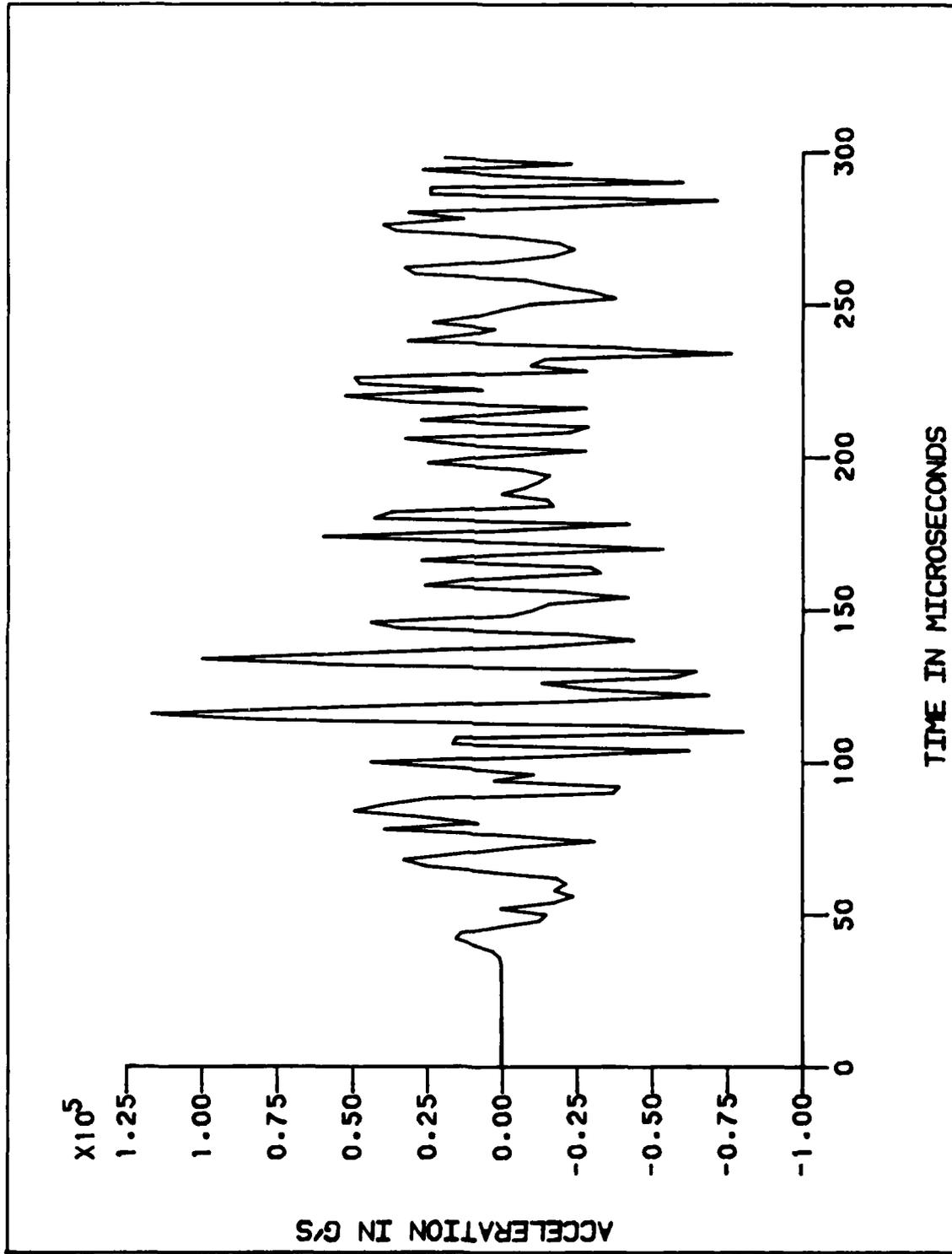


Figure 25. Axial acceleration history of 70 mm thick plate at 240 mm location for projectile impact velocity of 1508 m/s.

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