

AD-A127 799

THE EFFECTS OF FREE-STREAM TURBULENCE ON THE TURBULENCE
STRUCTURE AND HEAT TRANSFER IN A CHANNEL FLOW
CENTER EAST HARTFORD CT W F BLAIR ET AL. NOV 82

1/2

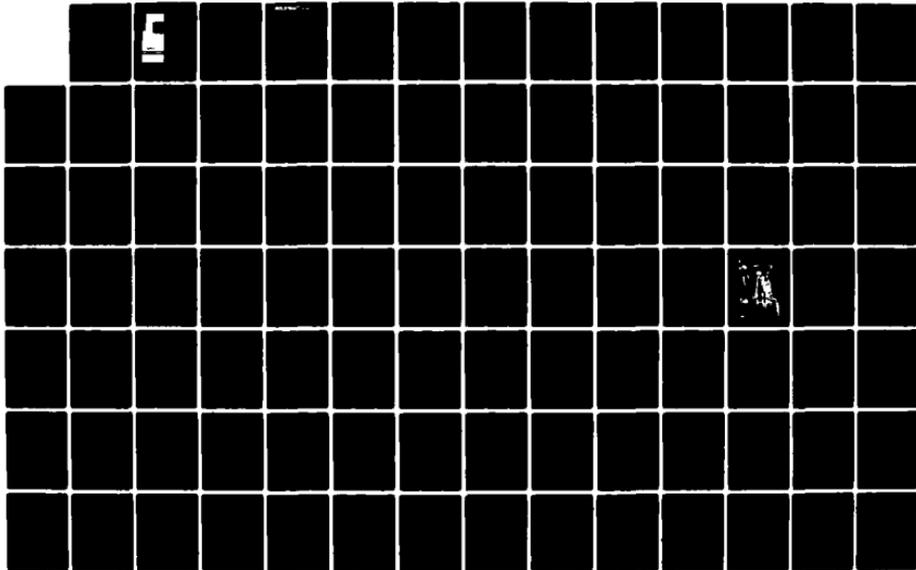
UNCLASSIFIED

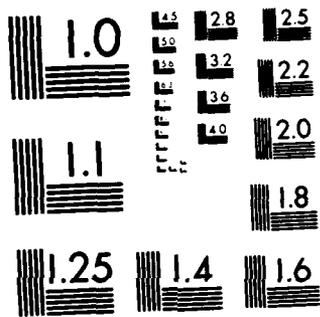
UTRC/R82-915634-2 AFOSR-TR-83-0355

F/G 20/4

NL

5





MICROCOPY RESOLUTION TEST CHART
NATIONAL BUREAU OF STANDARDS-1963-A

11

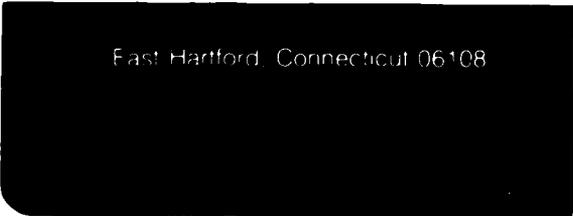
ADA 127799



Copy available to DTIC does not permit fully legible reproduction

UNITED TECHNOLOGIES RESEARCH CENTER

East Hartford, Connecticut 06108



DTIC FILE COPY

DTIC
MAY 9 1983
A

83 05 06 - 152

Approved for public release. Distribution unlimited.

DISCLAIMER NOTICE

THIS DOCUMENT IS BEST QUALITY PRACTICABLE. THE COPY FURNISHED TO DTIC CONTAINED A SIGNIFICANT NUMBER OF PAGES WHICH DO NOT REPRODUCE LEGIBLY.

UNITED TECHNOLOGIES RESEARCH CENTER



East Hartford, Connecticut 06108

(11)

R82-915634-2

The Effects of Free-Stream Turbulence
on the Turbulence Structure and Heat
Transfer in Zero Pressure
Gradient Boundary Layers

Contract No. F49620-81-C-0053

REPORTED BY

M. F. Blair

M. F. Blair

D. E. Edwards

D. E. Edwards

APPROVED BY

R. P. Dring

R. P. Dring

DATE November 1982

DTIC
ELECTE

MAY 9 1983

NO. OF PAGES _____

COPY NO. _____

AIR FORCE OFFICE OF SCIENTIFIC RESEARCH (AFSC)
NOTICE OF REPRODUCTION RIGHTS TO DTIC
This technical report has been reviewed and is
approved for distribution under AFR 190-12.
Distribution is unlimited.
MATTHEW J. KEENER
Chief, Technical Information Division

UNCLASSIFIED

SECURITY CLASSIFICATION OF THIS PAGE (When Data Entered)

REPORT DOCUMENTATION PAGE		READ INSTRUCTIONS BEFORE COMPLETING FORM
1. REPORT NUMBER AFOSR-TR- 83-0355	2. GOVT ACCESSION NO. AD-A127797	3. RECIPIENT'S CATALOG NUMBER
4. TITLE (and Subtitle) THE EFFECTS OF FREE-STREAM TURBULENCE ON THE TURBULENCE STRUCTURE AND HEAT TRANSFER IN ZERO PRESSURE GRADIENT BOUNDARY LAYERS		5. TYPE OF REPORT & PERIOD COVERED INTERIM
		6. PERFORMING ORG. REPORT NUMBER
7. AUTHOR(s) M F BLAIR D E EDWARD R P DRING		8. CONTRACT OR GRANT NUMBER(s) F49620-81-C-0053
9. PERFORMING ORGANIZATION NAME AND ADDRESS UNITED TECHNOLOGIES RESEARCH CENTER EAST HARTFORD, CT 06108		10. PROGRAM ELEMENT, PROJECT, TASK AREA & WORK UNIT NUMBERS 61102F 2307/A4
11. CONTROLLING OFFICE NAME AND ADDRESS AIR FORCE OFFICE OF SCIENTIFIC RESEARCH/NA BOLLING AFB, DC 20332		12. REPORT DATE November 1982
		13. NUMBER OF PAGES 186
14. MONITORING AGENCY NAME & ADDRESS (if different from Controlling Office)		15. SECURITY CLASS. (of this report) Unclassified
		15a. DECLASSIFICATION DOWNGRADING SCHEDULE
16. DISTRIBUTION STATEMENT (of this Report) Approved for Public Release; Distribution Unlimited.		
17. DISTRIBUTION STATEMENT (of the abstract entered in Block 20, if different from Report)		
18. SUPPLEMENTARY NOTES		
19. KEY WORDS (Continue on reverse side if necessary and identify by block number) FREE STREAM TURBULENCE HEAT TRANSFER BOUNDARY LAYER TURBULENCE MEASUREMENTS		
20. ABSTRACT (Continue on reverse side if necessary and identify by block number) In an earlier AFOSR funded investigation, experimental research was conducted to determine the influence of free-stream turbulence boundary layer heat transfer and mean profile development. The data obtained under this earlier contract indicated that both the skin friction and the heat transfer increased significantly with increased free-stream turbulence level. Under the present investigation detailed boundary layer turbulence structural data and turbulent heat transfer data were obtained for experimental test conditions and profile locations selected from the earlier test matrix.		

DD FORM 1473
1 JAN 73

EDITION OF 1 NOV 65 IS OBSOLETE

UNCLASSIFIED

83 05 06 - 152

SECURITY CLASSIFICATION OF THIS PAGE (When Data Entered)

UNCLASSIFIED

SECURITY CLASSIFICATION OF THIS PAGE(When Data Entered)

Numerous measurements assured that the present test conditions (boundary layer development and free-stream turbulence distributions) duplicated those of the earlier AFOSR contract. The purposes for making these present detailed boundary layer turbulence measurements were: (1) to provide data to which current finite-difference boundary layer turbulence models could be compared, and (2) to generate a data base for the development of new analytical models for boundary layer heat transfer prediction. The results from the present program have shown that the distributions of both the turbulence kinetic energy and the turbulence structural coefficients were affected by increased levels of free-stream turbulence. Local profile measurements indicated that the effect of increased free-stream turbulence was to decrease the near-wall turbulent Prandtl number relative to values expected for low free-stream turbulence. Turbulent Prandtl numbers in the outer region of the boundary layer were slightly increased for higher free-stream turbulence. A turbulence dependent correlation for the measured distribution of turbulent Prandtl number is given. With the completion of the experimental portion of this investigation, a theoretical effort was made to assess the capability of a finite difference boundary layer computer program, ABLE (Analysis of the Boundary Layer Equations) for predicting the effect of free-stream turbulence on momentum and thermal boundary layers. Comparisons with experimental data of mean flow velocity, mean flow temperature, Reynolds shear stress, turbulent heat transport, and turbulence kinematic energy were made in this investigation. In addition, the turbulent Prandtl number correlation deduced from the experimental measurements was used in the boundary layer analysis and its effect on surface heating evaluated. The results indicated that this boundary layer analysis, which uses a one equation eddy viscosity turbulence model, can provide adequate predictions of zero pressure gradient flows with high free-stream turbulence and wall heating.

UNCLASSIFIED

SECURITY CLASSIFICATION OF THIS PAGE(When Data Entered)

R82-915634-2

The Effects of Free-Stream Turbulence on the
Turbulence Structure and Heat Transfer in
Zero Pressure Gradient Boundary Layers

TABLE OF CONTENTS

<u>Section</u>	<u>Page</u>
ABSTRACT	1
INTRODUCTION	2
DESCRIPTION OF TEST EQUIPMENT	4
1. Wind Tunnel and Heat Transfer Test Surface	4
2. Turbulence Generating Grids	5
3. Boundary Layer Total Pressure and Thermocouple Probes and Traverse Control	6
HOT WIRE DATA ACQUISITION AND ANALYSIS TECHNIQUES	7
1. General	7
2. Description of the Hot Wire Probes	7
2.1 Probe Design	7
2.2 Probe Calibration	8
3. Description of the Data System	9
4. Data Analysis Techniques	10
4.1 Isothermal Flows (2 wire probes)	10
4.2 Flows with Wall Heating (3 wire probes)	11
4.3 Reynolds Stress and Turbulent Heat Flux Corrections for Sensor Separation	11
EXPERIMENTAL DATA	13
1. Experimental Test Program	13
2. Boundary Layer Profile Data Format	14
ANALYSIS AND DISCUSSION OF EXPERIMENTAL RESULTS	17
1. Comparisons of Present and Previous Results	17
1.1 Free-stream Turbulence Data	17
1.2 Heat Transfer Distributions	17
1.3 Boundary Layer Transition Locations	17
1.4 Mean Profile Data	18
1.5 Comparisons with Earlier Data - Conclusion	18
2. Profile Data with Low Free-Stream Turbulence	18
3. Effects of High Free-Stream Turbulence on the Fluctuating Velocities	19
4. Effects of High Free-Stream Turbulence on the Turbulent Prandtl Number	21

TABLE OF CONTENTS
(Continued)

	<u>Page</u>
THEORETICAL ANALYSIS	25
1. Prediction Method	25
2. Turbulence Model	26
3. Turbulent Prandtl Number Model	29
DISCUSSION OF ANALYTICAL RESULTS	31
CONCLUSIONS	36
LIST OF SYMBOLS	38
REFERENCES	41
FIGURES	
APPENDIX A - ERROR ANALYSIS	A-1
APPENDIX B - PROFILE DATA - PLOTS AND TABLES	B-1

ABSTRACT

In an earlier AFOSR funded investigation, experimental research was conducted to determine the influence of free-stream turbulence on turbulent boundary layer heat transfer and mean profile development. The data obtained under this earlier contract indicated that both the skin friction and the heat transfer increased significantly with increased free-stream turbulence level. Under the present investigation, detailed boundary layer turbulence structural data and turbulent heat transfer data were obtained for experimental test conditions and profile locations selected from the earlier test matrix. Numerous measurements assured that the present test conditions (boundary layer development and free-stream turbulence distributions) duplicated those of the earlier AFOSR contract. The purposes for making these present detailed boundary layer turbulence measurements were: (1) to provide data to which current finite-difference boundary layer turbulence models could be compared, and (2) to generate a data base for the development of new analytical models for boundary layer heat transfer prediction. The results from the present program have shown that the distributions of both the turbulence kinetic energy and the turbulence structural coefficients were affected by increased levels of free-stream turbulence. Local profile measurements indicated that the effect of increased free-stream turbulence was to decrease the near-wall turbulent Prandtl number relative to values expected for low free-stream turbulence. Turbulent Prandtl numbers in the outer region of the boundary layer were slightly increased for higher free-stream turbulence. A turbulence dependent correlation for the measured distribution of turbulent Prandtl number is given.

With the completion of the experimental portion of this investigation, a theoretical effort was made to assess the capability of a finite difference boundary layer computer program, ABLE (Analysis of the Boundary Layer Equations) for predicting the effect of free-stream turbulence on momentum and thermal boundary layers. Comparisons with experimental data of mean flow velocity, mean flow temperature, Reynolds shear stress, turbulent heat transport, and turbulence kinetic energy were made in this investigation. In addition, the turbulent Prandtl number correlation deduced from the experimental measurements was used in the boundary layer analysis and its effect on surface heating evaluated. The results indicated that this boundary layer analysis, which uses a one equation eddy viscosity turbulence model, can provide adequate predictions of zero pressure gradient flows with high free-stream turbulence and wall heating.

Accession For	
NTIS	<input checked="" type="checkbox"/>
DTIC	<input type="checkbox"/>
Unannounced	<input type="checkbox"/>
Justification	
Availability Codes	
Avail and/or	
Dist	Special
A	CP



INTRODUCTION

The search for improved gas turbine performance has led steadily in the direction of higher turbine inlet temperatures. The last twenty years have seen an increase in turbine inlet temperatures of roughly 1400°F but an increase in allowable blade metal temperature of only roughly 200°F. The difference between these two increases in temperature can be related directly to improved cooling technology. As an integral part of this advancing cooling technology, engine manufacturers are continually seeking improved techniques for calculating heat transfer coefficient distributions on gas turbine airfoils. As the level of cooling technology has been driven upward, and with it turbine inlet temperature, it is not surprising that the result is a design methodology which is extremely unforgiving of even small errors. The temptation is always present to overcool the airfoils but this is unacceptable due to the powerful negative impact of cooling air on the cycle and on turbine efficiency. It is this dilemma which has often led to extremely long and expensive developmental testing of advanced technology turbines.

Gas turbine thermal design systems are typically not based on fundamental fluid mechanics and heat transfer data and analysis alone but rather they are calibrated, or adjusted, to provide agreement with engine experience. Without the benefit of a first-principles understanding of the effects involved there is the likelihood that a designer will unknowingly either overcool the component or go beyond the range of validity of the design system calibration. There is, then, a clear requirement for the development of airfoil heat transfer distribution prediction procedures which are based on fundamental fluid mechanics and heat transfer data. The great emphasis placed on the development of accurate boundary layer calculation techniques over the past few years reflects the recognition of these needs.

One particularly important topic in the general context of turbine airfoil convective heat transfer is the influence of the free-stream turbulence on fully turbulent boundary layer development. It has, of course, long been recognized that increasing the free-stream turbulence level can cause a forward shift of the laminar to turbulent transition region. This particular phenomenon, the reduction of the boundary layer transition Reynolds number with increased free-stream turbulence level, is well documented in the open literature for zero pressure gradient flows and can be adequately predicted with currently available boundary layer prediction schemes. In addition, a number of investigators have studied the effects of free-stream turbulence level on turbulent boundary layer growth, profile structure, skin friction distribution and heat transfer. The consensus of these studies, is that free-stream turbulence has a very large and important influence on both the heat transfer and the boundary layer characteristics. As an example, it has been shown in a recently completed AFOSR funded contract at UTRC that a free-stream turbulence intensity of 5 percent produces an increase in Stanton number of approximately 15 percent over the value expected for a low turbulence freestream. While a number of existing boundary layer analysis procedures (including the UTRC ABLE code) account reasonably well for the influence of free-stream turbulence on mean velocity profile development and skin friction, no currently available analysis satisfactorily predicts the observed increased heat transfer rates.

The present program was designed to provide detailed boundary layer turbulence and turbulent heat flux distribution data for a range of free-stream turbulence levels. As part of this program these experimental data were employed to evaluate the analytical turbulence models currently incorporated in the UTRC ABLE code. It is anticipated that in the future these experimental data will be used by both UTRC and other workers in the field of boundary layer computation for development of new analytical turbulence models.

The contract effort consisted of acquiring, documenting and analyzing experimental flat wall boundary layer mean and fluctuating profile data to determine the influence of free-stream turbulence on fully turbulent boundary layer flows. For fully turbulent, zero pressure gradient flows, the following profile data were obtained for a range of free-stream turbulence intensities; boundary layer mean and fluctuating velocities and temperatures, turbulent shear stresses, and turbulent Prandtl numbers. In addition, in order to improve the ability of the UTRC boundary layer code to predict the effects of free-stream turbulence on heat transfer rates, a turbulent Prandtl number distribution model was incorporated into the UTRC boundary layer code. Calculations were carried out employing the measured turbulent Prandtl number distributions and comparisons made between the predicted and measured heat transfer distributions.

DESCRIPTION OF TEST EQUIPMENT

1. Wind Tunnel and Heat Transfer Test Surface

All experimental data for the present investigation were obtained in the United Technologies Research Center (UTRC) Boundary Layer Wind Tunnel (Fig. 1). This tunnel was designed specifically to generate large-scale, two-dimensional, incompressible boundary layers with Reynolds numbers and free-stream turbulence levels typical of turbomachinery airfoils. Complete descriptions of this facility including measurements documenting the tunnel flow uniformity and two-dimensionality of the test boundary layers are given in Refs. 1 and 2. The tunnel test section consisted of a flat upper wall instrumented for heat transfer measurements which served as the boundary layer test surface, plexiglass vertical sidewalls and a flexible lower wall. The test section was 34-in. wide, 96-in. long and 8-in. high at the entrance. For all test flows in this study the lower flexible test section wall was adjusted to produce a constant velocity along the test section.

A photograph of the Boundary Layer Wind Tunnel is presented in Fig. 2. Also shown in Fig. 2 are both the telescope used to position probes relative to the test wall and the computer controlled probe traverse mechanism.

The boundary layer test surface (upper wall of test section) consisted of a uniform heat flux electrically heated plate instrumented for the measurement of local convective coefficients (Fig. 3). The heated flat plate was constructed from a block of rigid urethane foam 34-in. wide by 96-in. long by 4-in. thick mounted in a plexiglass frame with 6-in. wide strips of 316 stainless steel foil cemented to the test surface. Details of the flat plate model and its instrumentation are presented in Refs. 1 and 2. Rigid foam was employed for the substrate of the heated flat plate model because of its extremely low thermal conductivity (0.025 Btu/hr ft²°F). Less than 1/2 percent of the heat generated on the surface of the plate was conducted away from the test surface. Electric current passing through the metal foil strips cemented to the test surface produced the surface heating. The metal foil strips were wired in series and were powered by a single low-ripple, regulated dc power supply. The foil test surface was instrumented with an array of 203 Cr-Al 0.13 mm diameter bead welded thermocouples. Each thermocouple was welded to the back surface of the foil through a hole in the rigid foam plate. Forty-eight surface static pressure taps were also installed along the test surface.

The dc power current passing through the surface strips was measured using two precision shunt resistors and a digital voltmeter. The temperatures of the test surface thermocouples were measured relative to a single test section free-stream reference junction using a digital voltmeter.

In order to insure a known, constant test surface emissivity and hence a known radiation loss, the completed foil test surface was coated with 3M C-101 high emissivity flat black paint ($\epsilon = 0.99$). Test results indicated that this surface was aerodynamically smooth, producing no premature boundary layer transition.

Local convective coefficients were determined by ignoring the negligible conduction losses, subtracting power lost through thermal radiation and dividing by the temperature difference from the wall (T_w) to the freestream (T_e). To illustrate the magnitude of the radiation losses from the test surface, for $U_e = 100$ ft/sec, for turbulent boundary layer flow with $T_w - T_e = 25^\circ\text{F}$, the radiation loss was approximately 4 percent of the total surface power.

As shown in Fig. 1, at the test section entrance a bleed scoop formed the leading edge of the heated boundary layer test surface. The scoop, which was mounted smoothly on the front edge of the heated test wall, provided a very short unheated starting length ($\xi = 1.7$ in.) upstream of the heated test surface. The leading edge of the scoop was a 4×1 ellipse in order to prevent a local separation bubble and premature transition of the test surface boundary layer. Details of the scoop including its instrumentation and adjustment are given in Refs. 1 and 2.

2. Turbulence Generating Grids

As described in Ref. 1, this wind tunnel has a relatively low residual test section turbulence level ($< 1/4\%$). Higher turbulence levels required for this study were generated by inserting various square array biplane grids constructed from rectangular bars at the entrance to the main tunnel contraction (see Fig. 1). Four turbulence generating grids were designed using the correlations of Ref. 3. The grids will be referred to as Grids 1, 2, 3 and 4 corresponding to mesh widths, M , of $7/8$, $2 \frac{9}{16}$, 7 and 9 in. The minimum turbulence configuration (no grid) will be referred to as Grid 0. Details of the grid configuration are given in Ref. 1. This present arrangement differs from that used for nearly all the earlier investigations of this subject in which the turbulence grids were located in the test section just upstream of the boundary layer test surface. The benefits derived from locating the grids at the contraction entrance were that the generated turbulence was more homogeneous and had a lower decay rate along the test section. Since grid generated turbulence decays approximately as $u'/U \propto (x/b)^{-5/7}$ (Ref. 3), the change in turbulence level with distance along the test section was reduced by increasing the distance from the grid to the test section entrance. In addition, the results of Ref. 3 indicate that approximately 10 grid mesh lengths are required to establish a uniform turbulent flow. Locating the grid a distance upstream of the test section requires, of course, a more coarse grid to achieve a given test section turbulence intensity.

Another effect considered was the expected influence of the contraction of the components of the grid generated turbulence. It was recognized that rearrangement of the relative magnitudes of the turbulence components would occur due to the contraction. However, since the contraction ratio was small (2.8), it was concluded that any effects of induced anisotropy would be small in comparison to the advantages gained in homogeneity and reduced decay rate. To determine the validity of the assumption, all three components of the test section turbulence were documented for all test cases.

3. Boundary Layer Total Pressure and Thermocouple Probes and Traverse Control

Boundary layer mean velocity profile data were measured using United Sensor Model Ba-0.020 impact probes with flattened tips. The probes used in the program were inspected for defects using both a Nikon Model II toolmakers microscope and a Jones and Lamson Model PC14 Shadowgraph. Mean temperature data were measured with miniature thermocouple probes designed using the results of Ref. 4. The thermocouple sensing elements for these probes were constructed from 0.001 in. dia. Chromel-Alumel bead welded wires. The thermocouple bead (\approx 0.003 in. dia.) was located at the center of the probe support prongs which were fabricated of heavier Chromel and Alumel wire. The results of Ref. 4 indicate that a probe of this design was virtually free of wire conduction errors and was capable of measuring boundary layer mean temperature profile data into the viscous sublayer region.

Movement of the boundary layer probes was achieved using an L.C. Smith ball/screw traverse drive with an optical shaft encoder capable of resolving relative probe location to within 0.0005 in. The traverse mechanism was suspended on a linear ball bearing track beneath the test section. The traverse could be located anywhere in the center 75 percent of the test section width from the leading to trailing edges of the test wall. A telescope sighted through the tunnel sidewall was used to accurately position probes relative to the test wall. Estimated absolute accuracy of measured probe distance from the test surface was 0.002 in. for any location in the test boundary layers.

HOT WIRE DATA ACQUISITION AND ANALYSIS TECHNIQUES

1. General

Measurements of fluctuating velocities and temperatures in the test boundary layers were obtained using multi-element hot wire anemometry techniques. For a large number of the test cases the wind tunnel was operated without wall heating, the resulting boundary layers being isothermal. Both vertical and horizontal x-type 2 wire probes were employed for these isothermal test cases. For cases with wall heating the velocity and temperature fluctuations in the flows were determined by using specially designed 3 wire probes, one wire of which was operated at a lower overheat than the other two. Detailed descriptions of both the 2 and 3 wire probes are given below in section 2. The voltage signals from the various hot wire probes were digitized, recorded and subsequently reduced to fluctuating velocity and temperature records using a minicomputer. A detailed description of the data system is provided in section 4. An analysis of the uncertainties of the various hot wire measurements is given in Appendix A.

2. Description of the Hot Wire Probes

2.1 Probe Design

The present study involved the measurement, using arrays of inclined hot wires, of fluctuating velocities and temperatures within boundary layer flows. In order to minimize potential errors for these measurements (errors largely arising from the inherent mean velocity and temperature gradients in the flows and the finite probe size) the hot wire probes were custom-designed and fabricated specifically for this program. The results from a large number of previous boundary layer turbulence and general hot wire studies were incorporated into the probe designs (Refs. 5-14). For the 2 wire x-type probes used in the isothermal tests three important design principles were adopted from these earlier studies. (1) To reduce the effects of the mean gradients in the flows the active length (or the size of the array of wires in the direction of the gradients) of the wires should be minimized. (2) To reduce end effects (nonuniform temperature along the active length) and to insure that a "Champagne k^2 " (Ref. 7) form of angular sensitivity could be employed, an active length/diameter ratio of 200 was chosen. (3) To maximize the spatial correlation coefficient (maximum accuracy of cross-products such as Reynolds stress) without introducing wire cross-talk effects a transverse wire spacing of $3\lambda/4$ was chosen.

Considerations (1) and (2) required that the diameter (d) be as small as possible--the limitation being practical considerations of probe fabrication and sensor survivability. A probe development program (UTC funded) conducted jointly with DISA, Inc. resulted in the conclusion that the minimum practical wire diameter for these probes was $2.5 \mu\text{m}$ (0.0001 in.) for platinum plated tungsten wires. From consideration (2) the active length of the wires was chosen as .50 mm (0.020 in.) and from (3) the transverse spacing was selected as 0.015 in. These wire arrays were employed

for x-type configurations oriented in both the vertical and horizontal planes. As will be discussed in the results section, cross-checks indicate that the fluctuating data measured with probes of this design are consistent and accurate.

The special 3 wire probes consisted of vertical x-type wire arrays with a third wire mounted equidistant between the wires of the x. This third wire was parallel to one of the wires of the x array. All three wires were constructed from the same material (platinum plated tungsten) and had the same diameter (0.0001 in.) and active length (0.020 in.). The transverse separation between adjacent wires of the 3 wire array was 0.015 in. With this wire arrangement the two parallel wires of the array were exposed to equal effective velocities during any given data sample period. Details of the techniques used to determine instantaneous velocities and temperatures with the 3 wire probes are given in section 4.

2.2 Probe Calibration

Prior to calibration all probe sensors were operated for approximately two hours in the 100 ft/sec mainstream of the wind tunnel. During this "wire curing" step the sensors were set to operate at overheat ratios slightly higher than those used during actual testing. These "curing" steps (1) provided some assurance that the sensors on a given probe would be likely to survive the calibration and testing environments, and (2) improved the stability of the calibration constants of a given sensor. Each probe was calibrated for temperature-resistance characteristics in a low temperature recirculating oven. Typically five temperature vs. resistance points were measured for each sensor. A least-squares data reduction program was used to find a best temperature-resistance coefficient.

$$R_w = R_{32} [1 + \alpha(T_w - 32)] \quad (1)$$

where R_w = resistance of the active sensor
 R_{32} = sensor resistance at 32°F
 T_w = sensor temperature
 α = temperature-resistance coefficient

Following the preliminary "burn-in" and the resistance temperature calibration each sensor was calibrated for velocity and angular sensitivity in a low-turbulence 1½-in. dia. jet flow. The sensors of the 2 wire probes were calibrated to an overheat ($R_{w-hot}/R_{adiabatic}$) of 1.5. For the 3 sensor probes the outside two sensors were calibrated at an overheat of 1.5 while the center sensor was calibrated at an overheat of 1.2. With the main probe support stem oriented perpendicular to the jet axis (wires ± 45° to the jet axis) mean velocity and bridge output voltage were recorded for approximately 20 jet speeds ranging from 7 to 130 ft/sec. The mean response equation of each sensor was assumed to be of the form

$$N_u = A_1 + B_1 R_e^{0.45} \quad (2)$$

which can be algebraically manipulated to

$$E_w^2 = \frac{A_2(R_s+R_w)^2}{R_w} T^{0.76} (T_w-T) + \frac{B_2(R_s+R_w)^2}{R_w} (T_w-T) U_E^{0.45} \quad (3)$$

where E_w = wire voltage
 R_s = probe body, cable and internal anemometer resistance
 R_w = sensor resistance
 T = air temperature
 T_w = sensor temperature
 U_E = effective velocity
 A_2, B_2 = empirical constants

The constants A_2 and B_2 were determined for each sensor from a least-squares fit of the data to Equation (3). Next, using a pitching fixture, pitch angle versus voltage data were obtained with the probes rotated from $+20^\circ$ to -20° in steps of 5° . The center of pitch coincided with the intersection of the wires of the x. Pitch sensitivity data were obtained for three jet velocities, 50, 80 and 100 ft/sec. The angular sensitivity of the wires was assumed to conform to Champagne's k^2 law (Ref. 7),

$$U_E^2(\phi) = U^2(\phi=0) (\cos^2 \phi + k^2 \sin^2 \phi) \quad (4)$$

where ϕ = angle between wire and direction normal to the flow ($\pm \approx 45^\circ$
with wall probe stem normal to the flow)
 U_E = effective velocity

Using a least-squares routine to find a best fit of the pitch-voltage data to Champagne's equation, optimum values of k were determined for each sensor.

In summary, the temperature-resistance, mean velocity and pitch calibrations were used to determine the following calibration constants.

- (1) R_{32} - sensor resistance at 32°F
- (2) α - temperature-resistance coefficient
- (3) A_2 and B_2 - empirical constants (Eq. 3)
- (4) k - empirical constant (Eq. 4).

3. Description of the Data System

For all test cases, both for isothermal flows and for flows with wall heating, the multi-element hot-wires were driven by Thermo Systems, Inc. (TSI) Model 1050 constant temperature anemometers. Signals from the anemometers were first passed through a wide band amplifier (Preston Model 8300 XWB) and then digitized using a TSI Model 1075 Multichannel Digitizer. A feature of this particular analog-to-digital converter which is important to this application is that the various

channels are sampled and held simultaneously. This simultaneous sample-hold feature permits cross-products of the various fluctuating quantities to be computed. Storage restrictions of the main memory of the minicomputer limited the total number of samples taken in a continuous stream to 18,432. The anemometer signals were sampled at 3906 Hz (6144 total samples) per channel or 2604 Hz (9216 total samples) per channel for 2 or 3 wire applications, respectively. The sampling rates resulted in total continuous sample periods of 2.36 sec for both 2 and 3 wire applications. The digitized voltage samples were stored on magnetic disks using a DEC Model RX02 floppy disk recorder.

The RX02 is a "double density" system and can record up to 512 K bytes of information on a single floppy disk. Reduction of the voltage-time records to either velocity-time records (isothermal flow - 2 wires) or velocity-temperature-time records (flows with wall heating - 3 wires) was accomplished off-line using an LSI 11-03 minicomputer. The reduced temperature-velocity-time or velocity-time results were written onto double-density magnetic disks and copied onto magnetic tape for purposes of plotting and tabulation.

4. Data Analysis Techniques

The digitized voltage vs. time records from the multi-wire probes were reduced to turbulence quantities using an LSI 11-03 minicomputer. For this reduction step the digitized data were read into the computer from the RX02 unit while the following constants for each sensor were input through a terminal.

- R_{32} - sensor resistance at 32°F
- Radiabatic - sensor resistance in flow with no overheat
- R_{hot} - sensor resistance at operating temperature
- R_s - probe, cable and anemometer (40 Ω for TSI-1050 sets) series resistance
- α - temperature-resistance coefficient
- A_2, B_2 - calibration constants from Eq. 3
- k - calibration constant from Eq. 4

4.1 Isothermal Flows (2 Wire Probes)

Solution for the velocity components (u and v for the vertical wire arrays, u and w for the horizontal arrays) for each time step proceeded as follows. First, using the adiabatic resistances (no sensor overheat) measured for the sensors in the test flow and Eq. 1, the temperature (T) of the flow was computed. The hot sensor temperatures (T_w) were then computed from R_{hot} and Eq. 1. For each time step the voltages for each of the sensors were input to Eq. 3 to determine the sensor effective velocity (U_E). Next, assuming that the wires of the x array were perpendicular to each other and at $\pm 45^\circ$ to the mainstream flow direction, the simultaneously measured effective sensor velocities were combined using Eq. 4 to find either u and v (vertical array) or u and w (horizontal arrays). As a check on the accuracy of the assumption that the wires were at exactly $\pm 45^\circ$ to the mainstream one of the probes was also calibrated using the "wire effective angle" method of Refs. 14 and 15.

Voltage vs. time records were reduced to fluctuating velocity components using these two different calibration-reduction techniques and the results were in very close agreement. Once the velocity component vs. time record was generated it remained a straightforward matter to compute any desired statistical quantities for the entire time record. The following turbulence quantities were computed for the u-v (vertical array) components. Similar quantities with the transverse velocity component (w) substituted for the vertical component (v) were computed for the horizontal probe arrays.

- $\bar{u}, \overline{u'^2}, \overline{u'^3}, \overline{u'^4}$ - first through the fourth moments of the streamwise fluctuating velocity
- $\bar{v}, \overline{v'^2}, \overline{v'^3}, \overline{v'^4}$ - first through the fourth moments of the normal fluctuating velocity
- $\overline{u'v'}, \overline{u'^2v'}, \overline{u'v'^2}$ - double and triple cross-products (and their correlation coefficients)
- S_u, S_v, F_u, F_v - skewness and flatness of both velocity components

4.2 Flows with Wall Heating (3 Wire Probes)

For the 3 wire probes employed for these measurements the two parallel wires of the array were operated at different overheats ($R_{w-hot}/R_{adiabatic} = 1.5$ and 1.2). The data reduction technique used for these measurements was based upon the assumption that during any time step the effective velocities over the two parallel wires were equal (for velocity scales equal to or larger than the separation distance between the wires). The solution technique proceeded by first finding the fluid temperature (T) for a given time step. Using the voltages (E_w) from the two parallel sensors and assuming that U_e was equal for both wires, Eq. 3 was iteratively solved for T. Once T was known the solution for the velocity components (u and v) for each step proceeded as in 4.1. In addition to computing the turbulence quantities listed in section 4.1, the following items were determined for the cases with wall heating.

- $\bar{t}, \overline{t'^2}, \overline{t'^3}, \overline{t'^4}$ - first through the fourth moments of the fluctuating temperature
- $\overline{v't'}, \overline{v't'^2}$ - velocity-temperature cross-products (and their correlation coefficients)
- S_T, F_T - skewness and flatness of temperature

4.3 Reynolds Stress and Turbulent Heat Flux Corrections for Sensor Separation

The accuracy of cross-products of correlated turbulent quantities is directly influenced by the spacing between the sensors used to measure these quantities (Ref. 13). The contributions of the smallest scales of the turbulence (smaller than the transverse sensor spacing) are excluded from the correlated products. As examples of the impact of this effect Refs. 9, 10, 11, 12 and 14 all present Reynolds stress measurements ($-u'v'$) which are about 30 percent lower than expected. Unfortunately this effect cannot be eliminated completely because a minimum limit for sensor separation is reached when sensor "cross-talk" errors become significant. As discussed in Section 2.1, the probes used for the present program were specifically designed to minimize these effects.

Correction factors for the cross-product terms measured in this program were determined by the following technique. Using the parallel wires of the 3 wire probes, the transverse spatial correlation coefficient (ψ parallel) was determined as a function of position (y/δ) in the test boundary layers. Wire separation distances (r) for each of the probes were accurately measured using a Nikon Model II toolmaker's microscope. By assuming that the correlation coefficient fell with the square of the separation distance (r^2) (Ref. 16) an appropriate spatial correlation coefficient could then be calculated for any x-type probe/boundary layer location combination. Next, assuming that the contributions to the cross-products were directly proportional to the spatial correlation coefficient, a correction factor for the probe/location combination was determined:

$$\beta = (1 - \Psi_{\text{parallel}}) \left[\frac{r(\text{x probe})}{r(\text{parallel})} \right]^2 \quad (5)$$

As an example the correction procedure for a measured Reynolds stress was as follows:

$$(\overline{u'v'})_{\text{corrected}} = \frac{(\overline{u'v'})_{\text{measured}}}{1 - \beta} \quad (6)$$

Typical correction factors (β) for the various probes, quantities and locations ranged from 0.12 to 0.2. A journal article documenting the development of this correction technique is currently in preparation.

EXPERIMENTAL DATA

1. Experimental Test Program

Measurements were obtained for three test flow conditions of incompressible, zero pressure gradient flow along a flat, uniform heat flux, test wall. The three test cases of this program reproduced conditions employed for an earlier AFOSR Contract (Ref. 2). For all test cases the free-stream velocity was 100 ft/sec and the test surface boundary layer passed through natural transition, i.e., no artificial trips were employed to promote boundary layer transition. Data were obtained for three levels of free-stream turbulence, (1) at the tunnel minimum turbulence level and (2) and (3) at higher levels of free-stream turbulence generated with bi-plane grids. Using the nomenclature of Refs. 1 and 2, the free-stream turbulence levels of this program are designated as follows

- (1) No grid-low free-stream turbulence (Grid 0) - $T_e \approx \frac{1}{2}$ percent
- (2) Grid number 2 ($\frac{1}{2}$ in. bars) - $T_{enom} = 2$ percent
- (3) Grid number 4 (2 in. bars) - $T_{enom} = 6$ percent

A complete documentation of the multi-component turbulence decay, integral length scale growth and spectral distributions generated by these particular test grids is available in Refs. 1 and 2.

For each of these flow conditions experimental boundary layer profile data were obtained at three streamwise locations ($x = 52, 68$ and 84 inches) for both an adiabatic test surface (no wall heating) and with a uniform surface heat flux condition. With no wall heating the following data were measured:

Type of Data	Instrumentation	Measurement Stations Per Profile
Profiles of streamwise velocity (mean and fluctuating)	Single, horizontal, linearized hot wire	30
Profiles of streamwise and normal velocities	Vertical x-type wires with analog-digital data system	17
Profiles of streamwise and transverse velocities (mean and fluctuating)	Horizontal x-type wires with analog-digital data system	17

With the uniform wall heat flux conditions the following data were measured:

Type of Data	Instrumentation	Measurement Stations Per Profile
Surface Stanton Number distribution	Thermocouple instrumentation incorporated into uniform heat flux test surface	210 surface locations
Profiles of streamwise mean velocity	Miniature boundary layer pitot probes	90
Profiles of mean temperature	Miniature boundary layer thermocouple probes	90
Profiles of temperatures and streamwise and normal velocities (mean and fluctuating)	3-Wire probes with analog-digital data system	17

In summary, for each of the three flow conditions surface heat transfer distributions and three stations of profile data were measured. In total (3 conditions) x (3 profiles) x (6 types of profile data) = 54 profile surveys were documented.

2. Boundary Layer Profile Data Format

The mean and fluctuating quantities measured for the various flow conditions and profile locations have been assembled in both graphical and tabular form. Comparisons of these results for the various flow conditions and with similar results from other experiments will be presented in the Analysis of Results section below. The compiled data for all the measurements stations are given in Appendix B - Experimental Data. As a guide to the format of the presentation and results, the data for a single sample profile are given in Figs. 4 through 5E and in Tables 1 through 3B. These particular sample profile data were obtained at the "middle" free-stream turbulence level (Grid 2, $T_e = 1.6\%$) at $X = 68$ in. The mean profile (total pressure and thermocouple probe) data for the sample set are presented in graphical form in Fig. 4 and in tabular form in Table 1. Table 2 presents a compilation of the test flow conditions and values computed from the mean profile data. These mean profile data are presented both in the form of velocity and temperature ratios versus y/δ and in the coordinates of the universal velocity and temperature "laws of the wall". As discussed in the Data Analysis Techniques section, the digitized data reduction system made possible the computation of any desired moments and cross-products of the various measured fluctuating quantities. A total of 38 quantities were selected for presentation on the grounds that they met either or both of the following criterion: (1) the quantity is employed in some existing boundary layer turbulence modeling method or (2) the quantity serves as a diagnostic of the characteristics of the turbulence, e.g.,

intermittency. These various quantities were both plotted and tabulated for each profile station (see Appendix B). The results for the sample profile are presented in Figs. 5A through E and in Table 3A and B. The distribution of a series of turbulence quantities computed from the fluctuating velocity data are presented in Fig. 5A. Starting in the upper left-hand corner of the figure, the distributions of the individual components of the turbulence are compared with the results of Klebanoff (obtained for near-zero free-stream turbulence). Moving clockwise, the next figure presents the measured distributions of the Reynolds stress nondimensionalized by the friction velocity (U_τ was determined independently from the mean profile data) and the distribution of the shear stress correlation coefficient. Also included in this plot is the distribution of shear stress computed from the mean profile data using the technique of Ref. 17. The lower right-hand corner plot of Fig. 5A presents the transport velocities of turbulent shear stress and kinetic energy defined as follows

$$V_\tau = \frac{\overline{u'v'^2}}{\overline{u'v'}} \quad (7)$$

$$V_q = \frac{\overline{v'(u'^2 + v'^2)}}{\overline{u'^2 + v'^2}} \quad (8)$$

See Ref. 14, pp. 220-239, and Ref. 18 for the development of these terms. (Due to a software error V_τ was not computed for the data of Grid 2 and does not appear for this sample plot.) The remaining plot of Fig. 5A presents the structural coefficient (as defined in Ref. 19) distributions for this case.

Turbulence quantities computed from the fluctuating velocities and temperatures are presented in Fig. 5B. Distributions of the turbulent heat flux and its correlation coefficient are given for the plot in the upper left-hand corner. The turbulent heat flux distribution is shown nondimensionalized by the independently measured wall heat flux. Also shown in this plot is the distribution of heat flux through the boundary layer as computed from the mean velocity and temperature profile data (see Ref. 17). The upper right-hand corner plot of Fig. 5B presents the fluctuating temperature distributions in two forms: (1) nondimensionalized by the friction temperature and (2) nondimensionalized by the temperature difference between the wall and free-stream. The lower two plots of Fig. 5B give the distributions of the turbulent Prandtl number, Pr_t , and two structural coefficients, $a_{1,\theta}$ and $G_{1,\theta}$ (see Ref. 19).

$$Pr_t = \frac{\overline{-u'v'} \frac{\partial T}{\partial y}}{\overline{v'v'} \frac{\partial U}{\partial y}} \quad (9)$$

$$a_{1,\theta} = \frac{\overline{v'v'}}{\overline{v'v'} \sqrt{-\overline{u'v'}}} \quad (10)$$

$$G_{i,\theta} = \frac{\overline{v'v'^2}}{2v'^2 U_T} \quad (11)$$

Triple product distributions of the fluctuating velocity components are given in Fig. 5C. Streamwise-transverse ($u'w'$) products are grouped on the left-hand plot while streamwise-normal ($u'v'$) products appear in the right.

For Fig. 5D the left-hand figure presents the skewness factor distributions for the fluctuating temperatures and velocities. Note that skewness factor distributions of the streamwise component (u') were determined both from the data from the vertical x probes (S_{UV}) and the horizontal x probes (S_{UH}). The correlation coefficients for the triple products of Fig. 5C are given in the right-hand plot of Fig. 5D.

Flatness factor distributions for the fluctuating temperatures and velocities are given in Fig. 5E. As with the skewness factors of Fig. 5D flatness factor distributions of the streamwise component are given for both the data for the vertical x (F_{UV}) and horizontal x (F_{UH}) probes. To avoid crowding on the figure the flatness of the temperature fluctuations was plotted after dividing by 2.

Tabulated values of these fluctuating quantities are given in Table 3A and 3B.

ANALYSIS AND DISCUSSION OF EXPERIMENTAL RESULTS

The present experimental test program was designed to examine, in detail, the effects of free-stream turbulence on the heat transfer through turbulent boundary layers. The test conditions for the present program were intended to reproduce cases for which other experimental data had been obtained previously under an earlier AFOSR contract (Ref. 2).

1. Comparisons of Present Results with Results of Ref. 2

A number of the measurements reported in Ref. 2 were repeated during the present program providing a measure of consistency for the two sets of data. Note that these various measurements were obtained in the same test facility and on the same test surface but about three years apart.

1.1 Free-Stream Turbulence Data

Measurements of the components of the free-stream turbulence (outside the boundary layer) were generally in excellent agreement (absolute levels of individual components agreed within 0.3 percent) with the measurements of Ref. 2. The exceptions to this rule were the transverse (w') fluctuation levels measured for Grid 4. These data were consistently higher (relatively 15% higher) than those measured for the same flow condition of Ref. 2. This discrepancy will be discussed in more detail in a following section.

1.2 Heat Transfer Distributions

For all three test conditions the agreement between the Stanton numbers measured for the present program and for Ref. 2 was excellent ($\pm 1\%$). The heat transfer distributions for the no grid (Grid 0), Grid 2 and Grid 4 cases were virtually identical with those presented in Ref. 2 in Figs. 41, 49 and 57, respectively and are not repeated here.

1.3 Boundary Layer Transition Location

For the Grid 2 and 4 test cases the agreement between the present transition location data and the similar data of Ref. 2 was within ± 3 percent. For the no-grid case, however, the transition Reynolds number increased from $Re_x = 1.2 \times 10^6$ to $Re_x = 1.35 \times 10^6$. This change in observed transition location was related to the three dimensional character of the transition process for the no-grid case. As discussed in Refs. 1 and 2, test section corner flows contaminate the flat test wall laminar boundary layer for the low free-stream turbulence case and produce premature transition along the tunnel centerline. This sidewall contamination was not important for the higher levels of free-stream turbulence because two-dimensional natural transition resulted well upstream of these effects. For the present no-grid test conditions the leading edge scoop adjustment was improved over the setting of the

tests of Ref. 2 producing reduced secondary corner flows and an increased transition Reynolds number. The transition Reynolds number (Re_{θ}) for the present tests was in excellent agreement with classic two-dimensional transition vs. turbulence correlations. The turbulent boundary layers which developed downstream of transition for both the present test and the test of Ref. 2 both exhibited the classic characteristics for zero-pressure gradient, low free-stream turbulence, two-dimensional flow.

1.4 Mean Profile Data

Integral thicknesses (δ^* and θ) computed from the profile data for the Grid 2 and 4 cases agreed within 2 percent with the results from Ref. 2 at the respective locations. For the no-grid case the integral thicknesses were about 12 percent reduced from those computed for Ref. 2 at similar stations. This reduction in boundary layer thickness resulted from the increased length of laminar flow upstream of transition for the present no-grid data.

If comparisons are made only for profiles with equal Re_{θ} the results for all the grids (0, 2 and 4) are practically identical to the respective cases in Ref. 2. When plotted in U^+ or T^+ vs. Y^+ coordinates the mean velocity and temperature profiles exhibit significant regions ($30 < Y^+ < 300$) of logarithmic behavior. Both the velocity and temperature wakes showed significant reduction with increasing free-stream turbulence as did the similar data of Ref. 2. Skin friction coefficients computed from fits of the mean velocity data to the "law-of-the-wall" were in excellent agreement with the results of Ref. 2.

1.5 Comparison with Earlier Data - Conclusion

The conclusion reached from the free-stream turbulence, heat transfer and mean profile data was that the test conditions of Ref. 2 were closely duplicated for the present series of measurements. In effect these present measurements can be considered as an additional set of data for the same test conditions as Ref. 2.

2. Profile Data with Low Free-Stream Turbulence

A number of comparisons have been made between the data obtained for the present no-grid (low free-stream turbulence) profiles and measurements from other experiments. These comparisons are intended to provide a measure of the accuracy and consistency of the present boundary layer turbulence data.

The distributions of the u' and v' components of the turbulence profiles were in very good agreement with the classic results of Klebanoff (Ref. 20), see for example Appendix B—Fig. B-4A. The transverse component (w') measurements, however, were typically about 15 percent reduced from Klebanoff's results with w' only slightly greater than v' . These present w' distributions are thought to be accurate as they are in close agreement (as were the u' and v' distributions) with the recent results of Ref. 14. The turbulent shear stress distributions measured for these low free-stream turbulence cases were in excellent agreement with the shear stress

distributions computed from the mean profiles, Fig. B-4A. The accuracy of these particular measurements is also supported by the fact that for all cases the $u'v'$ correlation coefficient was near the widely accepted value of 0.44 (Ref. 21) across most of the boundary layer. The measured distributions of the "structural" coefficients were in good agreement with the widely accepted constants, $a_1 = u'v'/q^2 = 0.15$, $a_2 = u'^2/q^2 = 0.5$, $a_3 = v'^2/q^2 = 0.2$.

A number of the turbulence quantities computed from the present data were also determined for a similar zero-pressure gradient, low free-stream turbulence flow in the work of Ref. 14. The present distributions of transport velocity of turbulent shear stress (V_τ), transport velocity of turbulent kinetic energy (V_q) (Fig. B-4A), $u'v'$ and $u'w'$ triple products (Fig. B-4C), skewness factors (Fig. B-4D) and flatness factors (Fig. B-4E) were all in good agreement with the respective data of Ref. 14.

The turbulent heat flux distribution measurements were in reasonably close agreement with the distributions inferred from the mean profiles (Fig. B-4B). The accuracy of these mean profile distributions is unclear because of extreme sensitivity to uncertainties in the mean temperature profiles. The fluctuating temperature distributions agreed very closely with the distributions measured in Ref. 21 (Fig. B-4B). The values of the thermal coefficient a_{1e} were about 30 percent greater than those determined in Ref. 21 with the cause of the difference uncertain. The authors were unaware of any other measurements of G_{1e} to which the present data could be compared. Finally the turbulent Prandtl number distributions measured for the low free-stream turbulence cases were in excellent agreement with the proposed distribution of Rotta (Ref. 22).

There were, then, a large number of experimentally determined turbulence quantities in the present program which agreed very closely with the results of other studies. The conclusions reached from this result are that one can have a high level of confidence in the present data acquisition and reduction technique and that the turbulence quantities reported here can be expected to be both accurate and self-consistent.

3. Effects of High Free-Stream Turbulence on the Fluctuating Velocities

The impact of increased free-stream turbulence on the boundary layer turbulence kinetic energy distribution is shown in Fig. 6. Presented in this figure are experimental data from the present program obtained at stations with nearly equal momentum thickness Reynolds numbers ($Re_\theta \approx 5500 \pm 100$) for Grids 0, 2 and 4. Also given in Fig. 6 are turbulence kinetic energy distributions measured for similar zero pressure gradient, low speed flows by other investigators (Refs. 12 and 21). Integral thicknesses were not computed for these other data but based upon the stated values of δ it is estimated that for Ref. 12 Re_θ was about 3500 while for Ref. 21 $Re_\theta \approx 700$. Agreement between the present low free-stream turbulence profile case (Grid 0) and the similar data of Ref. 21 is reasonably good except very close to the wall. As discussed earlier this near-wall discrepancy resulted from the

relatively higher values of w' determined in the study of Ref. 21. For the profiles with higher free-stream turbulence there were no cases where the present flow conditions (Re_∞ and Te) were identical with those of Ref. 12. Despite this a comparison of these results shows reasonably good agreement for both the trends and absolute magnitudes of the kinetic energy distributions. Both the present data and the independent results from Refs. 12 and 21 indicate a progressive increase in boundary layer turbulence kinetic energy with increasing free-stream turbulence. Increased levels of turbulence kinetic energy were measured across the entire thickness of the boundary layer.

The effects of the free-stream turbulence level on the individual components of the boundary layer turbulence for these same three profiles are shown in Fig. 7. The streamwise (u') component followed the same trends as the turbulence kinetic energy, increasing with free-stream turbulence level over the entire thickness of the boundary layer. The normal component (v'), however, was damped by the presence of the solid wall and showed virtually no change over the lower half of the boundary layer. See Ref. 23 for an in-depth study of the interaction of solid surfaces with turbulent fluctuations. The distribution of w' for $Te = 4.2$ percent showed a large increase over the distributions for the lower turbulence levels. Outside the boundary layer w' was also measured to be about 20 percent higher than u' or v' . On the grounds that earlier independent measurements of the free-stream turbulence for this grid showed the turbulence to be isotropic at this station (Ref. 1) and the unreasonably large "jump" in the w' distribution across the entire boundary layer it has been concluded that these w' measurements are in error. It is thought that there was an error in the calibration for the horizontal x wire probe used for the Grid 4 test cases. The Grid 4 w' data are reported here as measured, that is uncorrected for this probable error. It is estimated that by reducing the measured w' data by 20 percent a reasonably accurate set of distributions of the transverse component for this Grid 4 case would result.

With the Grid 4 w' data reduced by 20 percent the conclusion that can be reached from Fig. 7 is that the u' and w' component of turbulence increased progressively with increasing free-stream turbulence level. Both components increased at all locations in the boundary layer. The vertical component v' , however, was essentially constant and independent of free-stream turbulence level for the lower half of the boundary layer.

Distributions of the boundary layer turbulence structural coefficients (Bradshaw, et al., Ref. 19) are given in Figs. 8A and 8B. As can be seen from an inspection of Fig. 8A the ratio of shear stress to turbulence kinetic energy (a_1) decreased across the entire boundary layer with increasing free-stream turbulence. The observed decrease was most extreme over the outer 60 percent of the boundary layer. Also shown with the present a_1 distribution data are similar results from Refs. 14 and 21. Agreement between these similar (not identical, Re_∞ and Te were slightly different) sets of data was very good.

The ratios of the direct stress components to the turbulent kinetic energy (a_2 , a_3 and a_4) are given in the remaining plots of Figs. 8A and 8B. Employing the

previously described 20 percent reduction to the w' component for $T_e = 4.2$ percent (this also reduces q^2 for $T_e = 4.2$ percent), fairings of the corrected structural coefficient distributions for the highest turbulence level are given in the figures. Using the measured results for $T_e = 0.2$ percent and 1.5 percent and the corrected fairings for $T_e = 4.2$ percent the following conclusions were reached. As the free-stream turbulence stress level was increased a_2 (u'^2/q^2) increased slightly above 0.5, the value widely used for low free-stream turbulence boundary layers. The greatest percentage change was observed for the lower half of the boundary layer for a_3 ($a_3 = v'^2/q^2$). This ratio decreased progressively with increasing T_e dropping to about 0.12 (40% reduction from classic value of 0.2) for $T_e = 4.2$ percent. Only very small changes were observed for a_4 ($a_4 = w'^2/q^2$) with the measured values grouping around 0.3 for the lower half of the boundary layer.

A number of previous studies of free-stream turbulence effects on turbulent boundary layers (Refs. 8, 10, 11, 14 and 234) have reported finite turbulent shear stress levels beyond the edge of the velocity boundary layer. This effect was also observed for the present program. A comparison between the present results and those of the previous investigations is given in Fig. 9 where the turbulent shear level at the edge of the boundary layer ($\delta = 0.995$) is given as a function of T_e . The data from the present study and the results reported for most of the other experiments are tightly grouped. Taken together, these data indicate an increase of turbulent shear at the boundary edge directly proportional to the free-stream turbulence level. Huffman's results, which are believed (Ref. 2) to contain significant errors due to anisotropy, show much larger levels of turbulent shear than the other studies.

Additional evidence of the impact of free-stream turbulence on the characteristics of the turbulence near the boundary layer edge is provided by the measurements of flatness factor. The flatness factor ($u'^4/(u'^2)^2$) is an indication of the distribution of velocity fluctuations in a set of samples. For a normal Gaussian distribution the flatness factor is equal to 3 with larger values indicating contributions from intermittent turbulent fluctuations. Flatness factor distributions of the streamwise fluctuating velocity component are given in Fig. 10 as a function of position in the boundary layer. An examination of Fig. 10 shows that the intermittent character of the turbulence near the edge of the boundary layer was greatly reduced by increased free-stream turbulence. For a boundary layer beneath a low turbulence mainstream a relatively sharp irregular "edge" of turbulent boundary layer flow results adjacent to the non-turbulent freestream. With higher levels of free-stream turbulence this distinct border appears to have disappeared.

4. Effects of High Free-Stream Turbulence on the Turbulent Prandtl Number

The measured distributions of turbulent shear stress ($\overline{u'v'}$), the turbulent heat flux ($\overline{v't'}$) and the normal derivatives of the mean velocity and temperature were combined to form local turbulent Prandtl numbers.

$$Pr_t = \frac{\epsilon_m}{\epsilon_h} \quad (12)$$

where ϵ_m = eddy diffusivity of mass
 ϵ_h = eddy diffusivity of heat

$$Pr_t = \frac{\overline{-u'v'}}{\overline{v't'}} \frac{\partial T}{\partial y} \quad (9)$$

The distributions of turbulent Prandtl number measured for the various test cases are presented in Fig. 11. The results from all three profile locations ($x = 52, 68$ and 84 inches) for all three free-stream turbulence levels are included in Fig. 11 with an average free-stream turbulence level assigned to each set. For all points above the wall the turbulent Prandtl numbers were determined from the turbulent heat flux and shear stresses measured with the hot wire probes and from the derivatives of the mean profiles measured with the total pressure and thermocouple probes. At the wall the turbulent Prandtl numbers were determined from the mean temperature and velocity profile data by assuming that for at least some small distance the ratio of shear stress to heat flux remains at the wall value.

$$\frac{\tau}{\dot{q}} \approx \frac{\tau_{wall}}{\dot{q}_{wall}} = \frac{\rho_w \overline{u'v'}}{\rho_w c_p \overline{v't'}} \quad (13)$$

$$\frac{\overline{u'v'}}{\overline{v't'}} = \frac{\rho_w c_p U_\tau^2}{\dot{q}_{wall}} \quad (14)$$

$$Pr_t (wall) = \frac{\rho_w c_p U_\tau^2}{\dot{q}_{wall}} \frac{\partial T}{\partial U} \quad (15)$$

Near-wall values of the turbulent Prandtl number were evaluated from Eq. 15 using friction velocities (U_τ) determined from the mean velocity profile fits to the "law-of-the-wall". Values of $\partial T/\partial U$ were determined graphically from the near-wall velocity and temperature profile data.

Errors in the four measured terms of Eq. 9 combined to produce considerable scatter in the data of Fig. 11. This scatter, however, is much less than reported for the similar measurements of Refs. 21 and 25. It is expected that the consistency and absolute accuracy of such local turbulent Prandtl number measurements could be further improved by employing larger samples of the turbulent data.

The turbulent Prandtl number distributions measured for the low free-stream turbulence profiles were in good agreement with the similar data of Ref. 17. (Ref. 17 employed mean profile data only.) In addition, Rotta's (Ref. 22) suggested Pr_t distribution for low free-stream turbulence boundary layers appears to represent the present low turbulence data well.

$$Pr_t = 0.95 - 0.45 (y/\delta)^2 \quad (16)$$

An examination of Fig. 11 indicates that as the free-stream turbulence level was raised the turbulent Prandtl number increased over nearly the entire boundary layer. Values of Pr_t over unity were recorded at $y/\delta \approx 0.3$ for the highest free-stream turbulence level. These increased outer region turbulent Prandtl numbers for high free-stream turbulence levels had not been expected. At the outset of the program the turbulence characteristics of the outer portion of the boundary layer were known to be altered considerably by increased levels of free-stream turbulence. It was also known from the measurements of Ref. 1 that the free-stream turbulence had a large impact on the turbulent heat transfer with the Reynolds analogy factor increasing with increasing turbulence level. It was speculated that the increased Reynolds analogy factor might result from lowered Pr_t levels (relatively greater increase in $v't'$ as compared to $u'v'$) in the outer portions of the boundary layer. The experimental results of Fig. 11 indicate just the opposite effect. As the free-stream turbulence level was increased the outer region Pr_t levels increased while the near-wall values (determined from the mean profile data) indicate a small but progressive decrease. The following expression, a modification of Rotta's (Eq. 16) low free-stream turbulence equation, represents the measured results reasonably well.

$$Pr_t = \left[\left[0.95 - 0.45 (y/\delta)^2 \right] (1 + 2T)^2 \right] - \frac{5T}{\cosh^{10} y/\delta} \quad (17)$$

Equation 17 is shown in Fig. 11 for the three turbulence levels for which the experimental data were obtained. At $T_e = 0.2$ percent, Eq. 17 is practically identical to Rotta's (Eq. 16) expression.

The mean velocity and temperature profile data from both the present program and from Ref. 1 provide additional evidence that the near-wall turbulent Prandtl number decreased with increased free-stream turbulence level. (The arguments for this conclusion will be presented here in a highly abbreviated form. A more in-depth examination of these effects will be conducted during the preparation of a technical journal article on this contract work.) The effects of the free-stream turbulence on the similarity between the mean velocity and temperature profiles was examined by plotting the velocity ratio (U/U_e) versus the temperature ratio ($(T_w - T)/(T_w - T_e)$) across the boundary layers. Plots of these ratios for all profile locations and free-stream turbulence levels are given in Fig. 12. Also given in Fig. 12 are the similar data from the same stations and turbulence levels obtained in Ref. 1. An examination of Fig. 12 indicates that for all cases, independent of the free-stream turbulence level, the mean velocity and temperature profiles remained highly similar. This similarity between the velocity and temperature profiles extends across at least the outer 90 percent of the boundary layer thickness including all the wake and at least some of the logarithmic zone. It follows that the shapes of the velocity and temperature profiles should also be similar when plotted in universal (U^+ or T^+ vs. Y^+) coordinates. It has been observed in virtually every study of free-stream turbulence effects on turbulent boundary layers that the wake strength of the velocity boundary layer was progressively reduced with increasing free-stream turbulence. It was also

observed in Refs. 2 and 26 that for a given turbulence level the temperature wake was reduced by a larger amount than was the velocity wake. Implicit in the formulation of the temperature law-of-the-wall is the assumption that the turbulent Prandtl number is constant across the entire boundary layer. The large thermal wake depressions reported in Refs. 2 and 26 followed from the use of an average boundary layer turbulent Prandtl number for all the profiles. The following interpretation, however, is more consistent with the conclusion from Fig. 12, that the shapes of the outer region velocity and temperature profiles remained similar for all turbulence levels and streamwise locations. If the near-wall turbulent Prandtl numbers were assumed to be reduced with increased T_e (as Fig. 11 indicates) the slope of the temperature law-of-the-wall ($1/\kappa_\theta = Pr_t/\kappa$) would be reduced. With a reduced logarithmic region slope the apparent temperature wake strength, which is the maximum deviation from the log-law, would increase. An examination of the temperature profiles of Ref. 2 (in T^+ vs. Y^+ coordinates) indicated that good fits to the temperature law-of-the-wall could be achieved from $Y^+ \approx 30$ to $Y/\delta = 0.1$ if Π_θ was set equal to Π . For the present data, very good agreement between the thermal and velocity wake strengths resulted from the use of the near-wall turbulent Prandtl numbers of Fig. 11 for the respective profiles.

Finally, with regard to a potential physical mechanism producing the reduced near-wall Pr_t , the diffusion terms of the turbulence kinetic energy transport equation for velocity and temperature (Ref. 19) differ by the contribution of the pressure-velocity fluctuation product. Blom (Ref. 27) has pointed out that the pressure fluctuations serve to transfer energy from the relatively higher u' component of turbulence to the relatively smaller v' and w' components. Blom also argued that the absence of the $p'v'$ term in the temperature diffusion term could explain the reduction of Pr_t below unity. Since the effect of increased T_e was to increase the difference between the u' and v' components of turbulence near the wall the importance of the $p'v'$ term may grow with T_e . In other words, with increasing difference between u' and v' the effect of the $p'v'$ term may be to progressively decrease Pr_t .

The overall impact of free-stream turbulence on boundary layer heat transfer rates is, then, to depress the near-wall turbulent Prandtl number. Since the heat and momentum transport in turbulent boundary layers are dominated by the turbulent eddy contributions the result is that the Reynolds analogy factor rises with increasing free-stream turbulence level. It should be pointed out, however, that the results of Refs. 2 and 14 clearly show that the effects of free-stream turbulence on turbulent boundary layers are not dependent on turbulence intensity alone. For a fixed free-stream intensity the largest impact on a turbulent boundary layer results if the integral scale of the turbulence is about equal to the boundary layer thickness. Turbulence with integral scales significantly smaller or larger than the boundary layer thickness will produce reduced effects.

THEORETICAL ANALYSIS

The experimental data discussed previously in this report was used to assess the capability of a boundary layer analysis for predicting the effect of free-stream turbulence on momentum and thermal boundary layers. Previously, Blair and Werle (Ref. 2) examined the effects of free-stream turbulence on zero pressure gradient flows. They also evaluated the ability of a finite difference code (Ref. 28), which used a turbulence model of McDonald et al. (Refs. 29 and 30), to predict surface heating and skin friction. The present analytical investigation, which is a continuation of the work initiated by Blair and Werle, makes use of a boundary layer analysis (ABLE - Analysis of the Boundary Layer Equations) recently developed by Edwards, Carter and Werle (Ref. 31). This new boundary layer analysis contains the McDonald et al. turbulence model (Refs. 29 and 30) utilized by Blair and Werle's (Ref. 2) previous work. In addition, it was demonstrated in Ref. 31 that results obtained from the ABLE analysis and the boundary layer procedure employed by Blair and Werle were in excellent agreement for zero pressure gradient flows. In the present study, the capability of the ABLE code to accurately predict mean flow velocity, mean flow temperature, Reynolds shear stress and turbulent heat transport profiles is determined. In addition, the turbulent Prandtl number distribution deduced from the experimental measurements discussed earlier in this report is used in the boundary layer analysis and its effect on surface heating is evaluated.

1. Prediction Method

The ABLE boundary-layer code provides a rapid computation of two dimensional or axisymmetric boundary-layer flows subject to a prescribed distribution of edge Mach number, streamwise velocity, or static pressure. At the surface a distribution of either wall temperature or heat flux may be imposed. This analysis is applicable to attached flows which are laminar, transitional, or turbulent. A detailed description of the theory used in the ABLE code is given in Ref. 31 and a flow chart of the code is shown in Fig. 13. An implicit finite-difference technique is used in the ABLE code to solve the boundary-layer equations which are written in nondimensional form for two dimensional flow as follows.

continuity

$$\frac{\partial \rho u}{\partial s} + \frac{\partial \rho v}{\partial n} = 0 \quad (18)$$

momentum

$$\rho u \frac{\partial u}{\partial s} + \rho v \frac{\partial u}{\partial n} = -\frac{\partial p}{\partial s} + \frac{\partial}{\partial n} \left(\mu \frac{\partial u}{\partial n} - \rho \overline{u'v'} \right) \quad (19)$$

energy

$$\rho u \frac{\partial H}{\partial s} + \rho v \frac{\partial H}{\partial n} = \frac{\partial}{\partial n} \left[\mu \left(1 - \frac{1}{Pr} \right) u \frac{\partial u}{\partial n} + \frac{\mu}{Pr} \frac{\partial H}{\partial n} - \overline{\rho h'v'} - \overline{\rho u u'v'} \right] \quad (20)$$

In the above equations, s is the coordinate along the surface, n is the coordinate normal to the surface, u is the streamwise velocity, v is the normal velocity, ρ is the static density, P is the static pressure, h is the static enthalpy and H is the total enthalpy where (in non-dimensional form)

$$H = T + \frac{1}{2} u^2 = h + \frac{1}{2} u^2$$

The placement of a bar over several terms is used to denote the time average of various turbulent fluctuating quantities which are generally considered to represent the dominant Reynolds stress terms in the turbulent boundary layer equations.

The ABLE code currently contains two turbulence models, the Cebeci-Smith algebraic model (Ref. 32) and the McDonald et al., one equation turbulence model (Refs. 29 and 30). Both models are based on an eddy viscosity concept in which the Reynolds shear stress is related to the mean flow velocity gradient by

$$-\overline{\rho u'v'} = \mu_T \frac{\partial u}{\partial n} \quad (21)$$

In addition, the turbulent heat transport is related to the Reynolds shear stress and mean flow quantities through a Reynolds analogy type of argument

$$-\overline{\rho h'v'} = \frac{-\overline{\rho u'v'}}{Pr_t} \frac{\frac{\partial h}{\partial n}}{\frac{\partial u}{\partial n}} = \frac{\mu_T}{Pr_t} \frac{\partial h}{\partial n} \quad (22)$$

where Pr_t is the turbulent Prandtl number. This code presently contains two transition models, the first of which is the Dhawan and Narasimha (Ref. 33) forced transition model which requires the specification of the start and length of transition. The second model is a natural transition model developed by McDonald and Fish (Ref. 29) where the prediction of transition is controlled by the integrated form of the turbulence kinetic energy equation. In the present investigation the ABLE code is applied to the experimental flows discussed previously using the one equation turbulence model of McDonald and Kreskovsky (Ref. 30), the details of which are given in the next section.

2. Turbulence Model

The one equation turbulence model of McDonald and Kreskovsky (Ref. 30) permits the effect of free-stream turbulence to be included in the computed boundary layer

analysis. This turbulence model, which accounts for the proper approach of the turbulence level in the outer region of the boundary layer to the local edge value, is based on the integral form of the turbulence kinetic energy equation. This model is an extension of a turbulence model developed earlier by McDonald and Fish (Ref. 29). The eddy viscosity coefficient is expressed in nondimensional form as

$$\left(\frac{\mu_T}{\mu}\right) = \frac{\rho}{\mu} \ell^2 \frac{\partial u}{\partial n} Re^{1/2} \quad (23)$$

where Re is a reference Reynolds number, and ℓ , the local mixing length, is expressed as a function of the mixing length ℓ_e at the boundary layer edge through the relation

$$\ell = \mathcal{D} \ell_e \left\{ \tanh\left(\frac{\kappa n}{\ell_e}\right) + \frac{1}{2} \left(1 - \tanh\left(\frac{\kappa \delta}{\ell_e}\right)\right) \left|1 - \cos\left(\frac{n\pi}{\delta_\tau}\right)\right| \right\} \quad (24)$$

and

$$\mathcal{D} = \sqrt{\frac{1}{2} \left[1 + \operatorname{erf}\left(\frac{n^+ - 23}{8}\right)\right]} \quad (25)$$

where κ is the von Karman constant, δ is the boundary-layer thickness, and δ_τ is the "shear stress" thickness which is defined as the first location from the outer edge of the boundary layer where

$$\frac{\tau}{\tau_{\max}} \geq 0.02 \quad (26)$$

and τ_{\max} is the maximum shear stress at each streamwise location.

The local value of ℓ_e is obtained through the solution of the integral form of the turbulence kinetic energy equation which is expressed in nondimensional form as

$$\frac{d}{ds} \left[\frac{\rho_e u_e^3}{2 \sigma_1} \phi_1 \right] = \rho_e u_e^3 (\phi_2 - \phi_3) + E \quad (27)$$

where

$$E = \frac{q_e^2}{2} \left[\rho_e u_e \frac{d\delta_\tau}{ds} - (\rho v)_e \sqrt{Re} \right] \quad (28)$$

$$\phi_1 = \int_0^{\delta_\tau} \frac{\rho u}{\rho_e u_e} \left[\ell \frac{\partial}{\partial n} \left(\frac{u}{u_e} \right) + \sigma_1 f(n/\delta_\tau) \frac{q_e^2}{u_e^2} \right]^2 dn \quad (29)$$

$$\phi_2 = \sqrt{\text{Re}} \int_0^{\delta_\tau} \frac{\rho}{\rho_e} l^2 \left[\frac{\partial}{\partial n} \left(\frac{u}{u_e} \right) \right]^3 (1 - l/L) \, dn \quad (30)$$

$$\phi_3 = \int_0^{\delta_\tau} \frac{\rho}{\rho_e} \left(\frac{a_2 - a_3}{a_1} \right) \left\{ \left[l \frac{\partial}{\partial n} \left(\frac{u}{u_e} \right) \right]^2 + a_1 f \left(\frac{n}{\delta_\tau} \right) \frac{q_e^2}{u_e^2} \right\} \frac{1}{u_e} \frac{du}{ds} \, dn \quad (31)$$

and

$$L = .118 \mathcal{D} \tanh \left[\frac{\kappa n}{.118} \right] \quad (32)$$

$$q_e = \sqrt{(u'u' + v'v' + w'w')_e} \quad (33)$$

$$f \left(\frac{n}{\delta_\tau} \right) = \frac{1}{2} \left[1 - \cos \left(\frac{n\pi}{\delta_\tau} \right) \right] \quad (34)$$

where the subscript e denotes the flow quantities at the boundary-layer edge. The influence of the free-stream turbulence in the one equation model comes through the term, q_e^2 , which acts as a source term in Eq. 27. In the above relation, L is a dissipation length and a_1 , a_2 , and a_3 are structural coefficients that relate the Reynolds shear stress and turbulence intensity components to the turbulent kinetic energy as suggested by Townsend (Ref. 34) and Bradshaw and Ferris (Ref. 19). These coefficients are given by

$$-\overline{u'v'} = a_1 \left\{ \overline{q^2} - f \left(\frac{y}{\delta_\tau} \right) \overline{q_e^2} \right\} \quad (35)$$

$$\overline{u'u'} = a_2 \overline{q^2} \quad (36)$$

$$\overline{v'v'} = a_3 \overline{q^2} \quad (37)$$

$$\overline{w'w'} = (1 - a_2 - a_3) \overline{q^2} \quad (38)$$

where McDonald and Fish (Ref. 29) suggest the values $a_2 = .5$ and $a_3 = .2$ and that a_1 be expressed as follows

$$a_1 = \frac{a_0 \left(\frac{\bar{R}_\theta}{100} \right)}{1 + 6.666 a_0 \left(\frac{\bar{R}_\theta}{100} - 1 \right)} \quad (39)$$

where

$$a_0 = 0.0115$$

and \bar{R}_θ , which is referred to as the Reynolds number based on momentum thickness by Shamroth and McDonald (Ref. 35) but is in fact a correlation given in terms of an integrated turbulent Reynolds number R_T .

$$\bar{R}_\theta = \begin{cases} 100 R_T^{0.22} & R_T \leq 1 \\ 0.0098215 (R_T - 1)^3 + 1.165 (R_T - 1)^2 + 22 (R_T - 1) + 100 & 1 < R_T < 40 \\ 68.26 R_T - 614.33 & R_T \geq 40 \end{cases} \quad (40)$$

and

$$R_T = \frac{\frac{1}{\delta} \int_0^{\delta_T} v_T \, dn}{\frac{1}{\delta_s} \int_0^{\delta_s} v \, dn} \quad (41)$$

where δ_s is an estimate of the inner wall layer. The computational transition process is controlled by the structural coefficient a_1 as it varies from zero in laminar flow to .15 in fully turbulent flow. For the present analysis, the structural coefficients a_1 , a_2 and a_3 are constant over the boundary layer using the values suggested by McDonald and Fish (Ref. 29). However the experimental structural coefficients were observed to vary across the boundary layer (Figs. 8(a) and 8(b) and the effect of varying the structural coefficients in the analysis should be assessed in the future. A detailed description of the one equation turbulence model is given in Ref. 36.

3. Turbulent Prandtl Number Model

In the present investigation an evaluation is made of the effect of a variable turbulent Prandtl number, Pr_t , across the boundary layer on the ABL code prediction of the turbulent heat transport and surface heating. Three different functional forms of Pr_t have been applied in this investigation. They are given by

1) McDonald (Ref. 28)

$$Pr_t = .9P \left(\frac{n^+ - 23}{8} \right) / P \left(\frac{n^+ - 23.6}{10} \right) \quad (42)$$

where

$$P(x) = \frac{1}{2} \left[1 + \operatorname{erf} \left(\frac{x}{\sqrt{2}} \right) \right]$$

2) Rotta (Ref. 22)

$$Pr_t = .95 - .45(n/\delta)^2 \quad (43)$$

3) Present experimental investigation

$$Pr_t = \left[.95 - .45(n/\delta)^2 \right] \left[1 + 2Tu \right]^2 - 5Tu / \cosh[10n/\delta] \quad (44)$$

where

$$Tu = 1/3 \overline{qe^2}$$

A comparison of these three turbulent Prandtl number distributions are shown in Fig. 14. McDonald's function (Eq. 42) has a maximum value of 1.7 at the wall and decreases rapidly in the laminar sublayer of the turbulent boundary layer but is nearly constant and equal to .9 for $\eta^+ > 50$. Rotta's function (Eq. 43) has a maximum value of Pr_t of .95 at the wall and decreases linearly with respect of $(n/\delta)^2$ to the edge of the boundary layer to a value of .5. The Pr_t distribution obtained in the experimental portion of the present investigation (Eq. 44) is essentially a modification of Rotta's distribution to account for the effects of free-stream turbulence. An assessment of the accuracy of each of these turbulent Prandtl number formulations is made in the next section using the ABLE code and the experimental results presented above. However, since the turbulent heat transport $\overline{h'v'}$ is modeled in terms of $u'v'$, the turbulent Prandtl number and the normal derivatives of u and h as given in Eq. 22, then all of these quantities are compared with the experimental data before an assessment is made of the effect of Pr_t on the calculation of the turbulent heat transport.

DISCUSSION OF ANALYTICAL RESULTS

A series of calculations have been made with the ABLE code for the flow over a heated flat plate for each of the nominal inlet free-stream turbulence levels of 1%, 2%, 4% and 6% generated by the use of inlet turbulence Grids 1, 2, 3 and 4, respectively. A calculation was not performed for the case of the flow with .25 percent inlet turbulence since it was concluded by Blair and Werle (Ref. 2) that the transition process of this flow is three dimensional and thus the turbulence model of McDonald et al. (Refs. 29 and 30) cannot accurately predict the location and length of the transition region. For each case, a calculation is made with the ABLE code for each of the turbulent Prandtl number formulations discussed in the previous section. The mean flow quantities, Reynolds shear stress and turbulence kinetic energy predicted from the ABLE code were found to be insensitive to the different turbulent Prandtl number formulations; hence, these quantities are presently for only the present Pr_t formulation given in Eq. 44. This result was expected since the experimental flows are low speed and thus the momentum equation (Eq. 19) is essentially uncoupled from the energy equation (Eq. 20).

For all of the test cases analyzed in this investigation with the ABLE code, the gas is assumed to be air with a constant ratio of specific heats, γ , equal to 1.4 and a constant Prandtl number equal to .72. The von Karman constant for turbulent flow is set to .43 as suggested by McDonald and Kreskovsky (Ref. 30). The following flow conditions were used in all test cases

$$\begin{aligned}U_e &= 100 \text{ ft/sec} \\P_{T_e} &= 14.78 \text{ lb/in}^2\end{aligned}$$

and the streamwise variation of the free-stream turbulence for each flow with a specified inlet grid is obtained from the following expression

$$\tau_u = .78 \left(\frac{2.54x + 132}{b} \right)^{-5/7} \quad (45)$$

where b (grid bar width) = .48, 1.27, 3.81, and 5.08 for Grids 1, 2, 3, and 4, respectively. This relation was shown to be accurate in earlier testing reported in Ref. 2. The measured wall temperature levels are tabulated in Table I of Ref. 2, with the free-stream static temperature set to 530°R ($T_e = 530.83^\circ\text{R}$) for all calculations. The temperature distributions were numerically smoothed to eliminate spurious variations in the computed wall results due to minor experimental error. The smoothed temperature distributions were used as input to the ABLE code. Comparison of the measured and smoothed temperature distributions for each of the flow cases are shown in Fig. 15. The smoothing procedure is a least squares polynomial curve fit described in Ref. 37.

A computational mesh consisting of 101 grid points in the normal direction and 100 points in the streamwise direction was used in each of the calculations. A grid stretching based on a geometric progression was applied in each direction to insure

that a fine grid distribution was placed in the high gradient regions. The initial profile for the boundary layer calculation is the Blasius profile (Ref. 38) which was imposed at the flat plate leading edge. In addition, two iterations per streamwise station are applied in the computational procedure due to large streamwise temperature gradients which are encountered in the transition region of the flow. The current calculations required approximately 1 minute of CPU time on a UNIVAC 1180 operating system to compute the flow over the 8 foot length of the test section.

In this section comparisons are presented in Figs. 16-27 between the results obtained from the ABLE code with those measured experimentally both in the present investigation and in the previous investigation by Blair and Werle (Ref. 2). These comparisons are made for zero pressure gradient flows for the two cases with inlet turbulence levels of 2 percent and 6 percent. The following quantities are compared:

1. skin friction
2. displacement thickness
3. momentum thickness
4. mean velocity profile
5. mean temperature profile
6. Reynolds shear stress profile
7. turbulent kinetic energy profile
8. profiles of the components of turbulence intensity
9. turbulent Prandtl number profile
10. turbulent heat transport profile

The profile comparisons are shown only at $X = 68$ inches from the leading edge of the test section; similar comparisons were obtained at the other measuring stations.

Figure 16 is a comparison of the computed skin friction coefficient distribution with that obtained experimentally along the flat plate surface for inlet turbulence levels of 2 percent and 6 percent. The theoretical distribution is slightly higher (approximately 5%) than the experimental distribution for the flow with an inlet turbulence of 2 percent while the result obtained for the flow with an inlet turbulence of 6 percent is in excellent agreement with the experimental result. A comparison of the theoretical and experimental displacement and momentum thickness distributions are shown in Figs. 17 and 18, respectively. It is apparent from these figures that the computed integral quantities are in good agreement with the experiment over most of the test section. Figures 19 and 20 are comparisons of the computed mean velocity and temperature profiles with the experimental measurements for inlet turbulence levels of 2 percent and 6 percent. In all profile comparisons, the theoretical boundary layer thickness, δ , was used to nondimensionalize the normal coordinate. It is observed from Figs. 19 and 20 that the computed results are in excellent agreement with the experimental results for both flows. From Figs. 16-20 it is concluded that the ABLE boundary layer analysis with the McDonald and Kreskovsky turbulence model (Ref. 30) produces good agreement with the mean flow quantities of zero pressure gradient flows with various levels of inlet turbulence.

In order to determine how well the boundary layer analysis will predict turbulent fluctuating quantities, comparisons of computed profiles with experimental data

were made of the flows with inlet turbulence levels of 2 percent and 6 percent. Figure 21 is a comparison of the computed Reynolds shear stress profile with that obtained experimentally. Several interesting features are noted in this figure. By using the turbulence modeling (Eqs. 28-32) developed by McDonald and Kreskovsky in the computational procedure, the predicted Reynolds shear stress in the inner layer of the turbulent boundary layer is in excellent agreement with the experimental measurements. The theoretical results exhibit the same shape and level as the experimental results. However, in the outer layer, as the free-stream turbulence increases, the deviation between the computed and experimental results grows significantly as it is observed that the computed $u'v' \rightarrow 0$ whereas the experimental $u'v'$ approaches a finite value. Figure 22 is a comparison of the measured and computed turbulence kinetic energy distributions. It is observed that the computed results are in good agreement with the experimental data for both levels of free-stream turbulence. This result indicates that McDonald and Kreskovsky's modeling of free-stream turbulence in the turbulence kinetic energy equation (Eq. 27) captures the correct shape of the turbulence kinetic energy across the boundary layer for zero pressure gradient flows with different levels of free-stream turbulence. Figures 23-25 are comparisons of the measured and computed components of turbulence intensity, $u'u'$, $v'v'$, $w'w'$. From these figures it is observed that the predicted results are in fair agreement with the experimental data across the boundary layer.

An anomaly appears in the results shown in Figs. 19-25. First, from Figs. 22-25, the computed turbulence kinetic energy and its various components are in relatively good agreement with the experimental results. Secondly, the computed mean velocity (Fig. 19) is in excellent agreement with the experiment. However, the computed Reynolds shear stress (Figs. 21) has a significant deviation from the experimental results in the outer region of the boundary layer as the inlet turbulence level increases. This difference tends to suggest that the eddy viscosity modeling of turbulence (Eq. 21) does not properly model flows with significant levels of free-stream turbulence since by applying this model, the computed Reynolds shear stress is forced to zero at the edge of the boundary layer. The inability of the turbulence model to predict accurate Reynolds shear stress distributions over the entire boundary layer will affect the transition model (Eq. 39) since the transition process is controlled by an integrated form of the turbulent Reynolds number R_T (Eq. 41). An investigation is needed to determine an analytical turbulence model that will properly represent the Reynolds shear stress in the outer region of the turbulent boundary layer for flows with significant levels of free-stream turbulence.

A comparison of McDonald's, Rotta's, and the present turbulent Prandtl number formulation with the experimentally determined distribution is shown in Fig. 26(a) and Fig. 26(b) for flows with 2 percent and 6 percent turbulence levels, respectively. In Fig. 26(a) it is observed that McDonald's distribution, which is essentially constant over the boundary-layer overpredicts the experimental results over most of the boundary layer while Rotta's and the present distribution exhibit the same general shape and level as the experiment distributions. For the high inlet turbulent flow (Fig. 26(b)), the present Pr_t distribution exhibits the same shape and level as the experimental distribution while the relatively constant McDonald distribution does not have the same shape or level as the experimental results.

Also, Rotta's distribution exhibits the best shape of the experimental data but is not at the same level. Figures 27(a) and 26(b) are comparisons of the measured and computed turbulent heat flux distributions for the flows with turbulence levels of 2 percent and 6 percent, respectively. From these figures, it is noted that except near the wall, the computed turbulent heat flux distribution across the boundary layer is essentially the same for each of the three different Pr_t formulations which are used. Figures 27(a) and 27(b) show that the computed results are in reasonable agreement with the experimental results except at the edge of the boundary layer where the theoretical results go to zero and the experimental data does not. Since the turbulent heat flux is determined from Eq. (22) and all the computed quantities in that equation are in reasonable agreement with experimental data except the Reynolds shear stress, this suggests that the inaccuracy of predicting the turbulent heat flux in the outer portion of the boundary layer is due to the modeling of the Reynolds shear stress in the outer region of a turbulent boundary layer with significant levels of free-stream turbulence. Further investigation of this feature of the flow is needed.

In Figure 28, comparisons are presented between measured and computed Stanton number distributions for the flows with inlet turbulence levels of 1, 2, 4 and 6 percent. The following observations about the Stanton number prediction in the fully turbulent region of the flows are made. The predicted Stanton number using McDonald's formulation is in good agreement with the experiment for the flow with an inlet turbulence level of 1 percent; however, as the inlet turbulence level increases, the computational procedure using McDonald's formulation underpredicts the measured Stanton number distribution. The computational procedure using Rotta's formulation overpredicts the Stanton number for the flow with the 1 percent inlet turbulence level. However, as the inlet turbulence level increases the computed results tend to slightly overpredict the measured Stanton number distribution. The present turbulent Prandtl number formulation yields essentially the same results as the computed with Rotta's formulation since the present formulation is a perturbation of Rotta's Pr_t formulation. The results shown in Fig. 28 indicate that the computation using the present distribution shows no marked improvement over the computation using Rotta's distribution. However, for the flows with inlet turbulence of 2 percent or larger, computations using either Rotta's or the present formulation result in predicted Stanton number distributions which are in better agreement with the experimental results than that obtained using McDonald's formulation. A further indication of the advantage of Rotta's or the present formulation is shown in Fig. 29 where the predicted Reynolds analogy factor, $2 S_t/C_f$, from calculations using the present and McDonald's turbulent Prandtl number distributions are compared to the experimentally deduced Reynolds analogy. In this figure it is observed that the calculation using the present (or Rotta's) formulation predicted Reynolds analogy factors that are in better agreement with the experiment than the computation which uses McDonald's formulation. The overall implication of these results is that the analysis using the McDonald and Kreskovsky turbulence model (Ref. 30) with either Rotta's or the present turbulent Prandtl number formulation can accurately represent the momentum and energy transport mechanisms for zero pressure gradient flows in the wall region of the boundary layer but that there is a severe weakness in its

ability to represent the momentum and energy transport mechanisms in the region near the edge of the boundary layer for flows with significant levels of free-stream turbulence.

CONCLUSIONS

The present program was designed to examine, both experimentally and analytically, the effect of free-stream turbulence on the heat transfer through turbulent boundary layers. The experimental test conditions for the present program were intended to reproduce cases for which numerous other experimental data had been obtained under an earlier AFOSR contract (Ref. 2). Measurements of multi-component free-stream turbulence intensities, test surface Stanton number distributions, transition Reynolds numbers and boundary layer integral thicknesses were in excellent agreement with the respective quantities of the earlier contract. It has been concluded that these present measurements can be viewed as additional data for the same test conditions as were previously studied. A number of comparisons were made between low free-stream turbulence boundary layer turbulence data obtained in the present study and similar results from other investigations. These comparisons showed excellent agreement indicating that the present boundary layer turbulence data are of high quality.

The conclusions reached from the experimental measurements obtained for higher levels of free-stream turbulence were as follows:

1. The present data indicate a progressive increase of boundary layer turbulence kinetic energy with increasing free-stream turbulence. Increased levels of turbulence kinetic energy were measured across the entire thickness of the boundary layer. These results are in agreement with data from other independent studies.
2. Both the u' and w' components of turbulence increased progressively with increasing free-stream turbulence level. The u' component increased more than the w' component. The vertical component (v'), however, was essentially constant and independent of free-stream turbulence level for the inner half of the boundary layer.
3. The ratio of shear stress to turbulence kinetic energy decreased across the entire boundary layer with increasing free-stream turbulence level. The decrease was most extreme over the outer 60 percent of the boundary layer.
4. The effects of free-stream turbulence level on the ratios of the direct stress components to the turbulence kinetic energy were to a) increase $u'u'/q^2$, b) decrease $v'v'/q^2$ and c) leave $w'w'/q^2$ nearly constant.
5. Reynolds stress distribution measurements indicated that at high levels of free-stream turbulence the turbulent shear stresses extend beyond the mean velocity boundary layer. The present data and results from other sources indicate an increase in turbulent shear at the boundary layer edge directly proportional to the free-stream

turbulence level. Flatness factor measurements indicated that as the free-stream turbulence level was increased the "border" between the fluid in the boundary layer and the free-stream fluid became less distinct.

6. Measurements of the boundary layer turbulent Prandtl number distribution for the case of the low free-stream turbulence were in good agreement with a model suggested by Rotta. The present data indicate that as the free-stream turbulence level was increased, the near-wall Pr_t decreased while Pr_t over the outer region of the boundary layer slightly increased. A correlation, $Pr_t(y/\delta, Te)$, which fit the observed data reasonably well was suggested.

The experimental data was used to assess the capability of a boundary-layer computer program, ABLE (Analysis of the Boundary Layer Equations) for predicting the effect of free-stream turbulence on momentum and thermal boundary layers. In addition the turbulent Prandtl number formulation deduced from the experimental measurements was used in the boundary layer analysis and its effect on surface heating was determined. The following conclusions were reached from the theoretical portion of this investigation:

1. The modeling of free-stream turbulence in the one equation turbulence model of McDonald and Kreskovsky captures the correct shape and level of the turbulence kinetic energy.
2. For increased levels of free-stream turbulence, the Reynolds shear stress and turbulent heat flux determined from the turbulence model is significantly smaller than that observed experimentally in the wake region of the turbulent boundary layer. This discrepancy could be due to the eddy viscosity concept used in McDonald and Kreskovsky's model and further investigation of turbulence models is needed.
3. Analytical calculations using either Rotta's turbulent Prandtl number correlation or the correlation of the present investigation predicted Reynolds analogy factors ($2S_t/C_f$) that are in reasonable agreement with experimental measurements and accurately predict the increase in surface heat transfer due to increased free-stream turbulence.

LIST OF SYMBOLS

$a_1, a_2, a_3, a_4, a_{1,e}$	Turbulence structural coefficients
c_p	Specific heat at constant pressure
d	Hot wire sensor diameter
D	Wall damping function in turbulence
F_{uv}	Flatness factor (vertical x probe) = $\overline{u'^4}/(\overline{u'^2})^2$
F_{uh}	Flatness factor (horizontal x probe) = $\overline{u'^4}/(\overline{u'^2})^2$
F_{vv}	Flatness factor (vertical x probe) = $\overline{v'^4}/(\overline{v'^2})^2$
F_{wh}	Flatness factor (horizontal x probe) = $\overline{w'^4}/(\overline{w'^2})^2$
F_T	Flatness factor (tri-x probe) = $\overline{t'^4}/(\overline{t'^2})^2$
$G_{1,e}$	Turbulence structural coefficient (Eq. 11)
h	Height of hot wire sensor array
h	Static enthalpy
$\overline{h'v'}$	Reynolds thermal flux
H	Total enthalpy
l	Active length of hot wire sensor
l	Mixing length function in turbulence model
l_e	Free-stream mixing length
L	Dissipation length scale
n	Distance normal to surface
n^+	Dimensionless normal distance to surface,
Nu	Nusselt number of hot wire sensor
p	Static pressure
Pr_t	Turbulent Prandtl number (Eq. 9)
\dot{q}	Heat flux
q^2	Turbulence kinetic energy
r	Separation distance between hot wire sensors
Re	Reynolds number
Re_θ	Reynolds number based on momentum thickness
R_T	Turbulent Reynolds number
\tilde{R}_θ	McDonald's correlation of Re_θ

s	Coordinate along wall surface
S_{uV}	Skewness factor (vertical x probe) = $\overline{u'^3}/(\overline{u'^2})^{3/2}$
S_{uH}	Skewness factor (horizontal x probe) = $\overline{u'^3}/(\overline{u'^2})^{3/2}$
S_{vV}	Skewness factor (vertical x probe) = $\overline{v'^3}/(\overline{v'^2})^{3/2}$
S_{vH}	Skewness factor (horizontal x probe) = $\overline{v'^3}/(\overline{v'^2})^{3/2}$
S_T	Skewness factor (tri-x probe) = $\overline{t'^3}/(\overline{t'^2})^{3/2}$
T	Mean static temperature
T_e	Free-stream turbulence level
T_T	Total temperature
T_τ	Friction temperature = $q_w/\rho_w c_p U_\tau$
T^+	Dimensionless temperature = $(T_w - T)/T_\tau$
t'	Fluctuating temperature
u	Streamwise velocity
U	Mean streamwise velocity
U_τ	Friction velocity
U^+	Dimensionless velocity - U/U_τ
$\overline{u'u'}, \overline{v'v'}, \overline{w'w'}$	Components of turbulent intensity
u', v', w'	Streamwise normal and transverse fluctuating velocities
$-\overline{u'v'}$	Reynolds shear stress
v	Normal velocity
V_q	Transport velocity of turbulence kinetic energy (Eq. 8)
V_τ	Transport velocity of turbulent shear stress (Eq. 7)
x	Distance from plate leading edge
y	Distance from wall
y^+	Dimensionless distance from wall = yU_τ/ν
δ	Boundary layer thickness
δ_τ	Thermal boundary layer thickness
δ_s	Thickness of inner wall region of boundary layer
δ^*	Boundary layer displacement thickness
θ	Boundary layer momentum thickness
κ	von Karman constant for velocity of law-of-the-wall
κ_θ	von Karman constant for temperature law-of-the-wall

μ	Molecular viscosity
μ_T	Eddy viscosity
ν	Kinematic viscosity
Π	Wake strength for velocity boundary layer
Π_θ	Wake strength for temperature boundary layer
ρ	Density
τ	Shear stress
Ψ_{B_V}	Correlation coefficient = $\frac{\overline{u'^2 v'}/\overline{u'^2}}{\sqrt{v'^2}}$
Ψ_{B_W}	Correlation coefficient = $\frac{\overline{u'^2 w'}/\overline{u'^2}}{\sqrt{w'^2}}$
Ψ_{C_V}	Correlation coefficient = $\frac{\overline{u' v'^2}/\overline{u'^2}}{\sqrt{v'^2}}$
Ψ_{C_W}	Correlation coefficient = $\frac{\overline{u' w'^2}/\overline{u'^2}}{\sqrt{w'^2}}$

Subscripts

e	Freestream
w	Wall
0.995	where $U = 0.995 U_e$

REFERENCES

1. Blair, M. F., D. A. Bailey and R. H. Schlinker: Development of a Large-Scale Wind Tunnel for the Simulation of Turbomachinery Airfoil Boundary Layers, ASME Paper 81-GT-6 presented at ASME Gas Turbine Conference, March 1981, ASME Journal of Engineering for Power, Vol. 103, pp. 678-687, 1981.
2. Blair, M. F. and M. J. Werle: The Influence of Freestream Turbulence on the Zero Pressure Gradient Fully Turbulent Boundary Layer, UTRC Report R80-915388-12, September 1980.
3. Baines, W. D. and E. G. Peterson: An Investigation of Flow Through Screens, Trans. of ASME, Vol. 73, pp. 467-480, July 1951.
4. Blackwell, B. F. and R. J. Moffat: Design and Construction of a Low Velocity Boundary-Layer Temperature Probe, AIAA Paper No. 74-709, ASME Paper No. 74-HT-29, July 1974.
5. Guitton, D. E. and R. P. Patel: An Experimental Study of the Thermal Wake Interference Between Closely Spaced Wires of a X-Type Hot-Wire Probe. McGill University Report 69-7, June 1969.
6. Guitton, D. E.: Correction of Hot Wire Data for High Intensity Turbulence, Longitudinal Cooling and Probe Interference. McGill University Report 68-6, 1968.
7. Champagne, F. H., C. A. Sleicher and O. H. Wehrmann: Turbulence Measurements with Inclined Hot-Wires Part I - Heat Transfer Experiments with Inclined Hot Wires, Part II Hot Wire Response Equations, JFM, Vol. 28, 1967, pp. 153-182.
8. Charnay, G., J. P. Schon and M. Sunyach: Isolation and Sampling of Random Signals Transmitted by Several Hot Wire Anemometers. Entropie No. 50, March-April 1973 (in French).
9. Johnson, D. S.: Velocity and Temperature Fluctuation Measurements in a Turbulent Boundary Layer Downstream of a Stepwise Discontinuity in Wall Temperature, Journal of Applied Mechanics, Vol. 26, 1959.
10. Charnay, G., G. Comte-Bellot and J. Mathieu: Development of a Turbulent Boundary Layer on a Flat Plate in an External Turbulent Flow. AGARD, CP 93, paper No. 27, 1971.
11. Huffman, F. D., D. R. Zimmerman and W. A. Bennet: The Effect of Free-Stream Turbulence Level in Turbulent Boundary Layer Behavior. AGARD AG164, pp. 91-115, 1972.
12. Charnay, G., J. Mathieu and G. Comte-Bellot: Response of a Turbulent Boundary Layer to Random Fluctuations in the External Stream, Physics of Fluids, Vol. 19, No. 9, September 1976.

13. Sandborn: Resistance Temperature Transducers, Metrology Press, Fort Collins, CO, 1972, pp. 205-209.
14. Hancock, P. D.: Effect of Free-Stream Turbulence in Turbulent Boundary Layers, Ph.D. thesis, Imperial College, London University, 1980.
15. Bradshaw, P.: An Introduction to Turbulence and Its Measurement, Pergamon Press, 1971, pp. 121-126.
16. Batchelor, G. K.: The Theory of Homogeneous Turbulence, Cambridge University Press, 1967, pp. 46-47.
17. Simpson, R. L., D. G. Whitten and R. J. Moffat: An Experimental Study of the Turbulent Prandtl Number of Air with Injection and Suction. Int. J. Heat and Mass Transfer, Vol. 13, 1970, pp. 124-143.
18. Bradshaw, P.: The Turbulence Structure of Equilibrium Boundary Layers, JFM, Vol. 29, Part 4, pp. 624-645, 1967.
19. Bradshaw, P., D. H. Ferriss, and N. P. Atwell: Calculation of Boundary Layer Development Using the Turbulent Energy Equation, JFM, Vol. 28, pp. 593-616, 1967.
20. Klebanoff, P. S.: Characteristics of Turbulence in a Boundary Layer with Zero Pressure Gradient, NACA Report 1247 (1955).
21. Subramanian, C. S. and R. A. Antonia: Effect of Reynolds Number on a Slightly Heated Turbulent Boundary Layer, Int. J. of Heat and Mass Transfer, Vol. 25, No. 11, pp. 1833-1846, 1981.
22. Rotta, J. C.: Temperaturverteilungen in der Turbulenten Grezschicht an der Ebenen Platte, Int. J. of Heat and Mass Transfer 7, 1964, pp. 215-228.
23. Thomas, N. H. and P. E. Hancock: Grid Turbulence Near a Moving Wall, JFM, Vol. 82, Part 3, 1977, pp. 481-496.
24. Evans, R. L.: Free-Stream Turbulence Effects on the Turbulent Boundary Layer, A.R.C. C.P. 1282, 1974.
25. Senda, M., K. Suzuki and T. Sato: Turbulence Structure Related to the Heat Transfer in a Turbulent Boundary Layer with Injection, Turbulent Shear Flows, Vol. 2, Selected Papers from the 2nd Int. Symposium on Turbulent Shear Flows, Springer-Verlag, 1980.
26. Simonich, J. C. and P. Bradshaw: Effect of Free-Stream Turbulence on Heat Transfer through a Turbulent Boundary Layer, ASME Journal of Heat Transfer, Vol. 100, No. 4, 1978.

27. Blom, J.: An Experimental Determination of the Turbulent Prandtl Number in a Developing Temperature Boundary Layer, Ph.D. thesis, Technological University, Eindhoven, The Netherlands, 1970.
28. McDonald, H.: User's Manual for the Finite-Difference Boundary Layer Prediction Deck (M093). United Aircraft Research Laboratories Report UAR-J228, 1970.
29. McDonald, H. and R. W. Fish: Practical Calculations of Transitional Boundary Layers. *Int. J. Heat Mass Transfer*, Vol. 16, pp. 1729-1744, 1973.
30. McDonald, H. and J. P. Kreskovsky: Effect of Free Stream Turbulence on the Turbulent Boundary Layer. *Int. J. Heat Mass Transfer*, Vol. 17, pp. 705-716, 1974.
31. Edwards, D. E., J. E. Carter and M. J. Werle: Analysis of Boundary Layer Equations Including a New Composite Coordinate Transformation, UTRC Report No. UTRC81-30, May 1982.
32. Cebeci, T. and A. M. O. Smith: *Analysis of Turbulent Boundary Layers*, Academic Press, New York, 1974.
33. Dhawan, S. and R. Narasimha: Some Properties of Boundary Layer Flow During the Transition from Laminar to Turbulent Motion, *J. Fluid Mech.*, Vol. 3, 1958.
34. Townsend, A. A.: *The Structure of Turbulent Shear Flow*. Cambridge University Press, 1956.
35. Shamroth, S. J. and H. McDonald: Assessment of a Transitional Boundary Layer Theory at Low Hypersonic Mach Numbers. NASA CR-2131, November 1972.
36. Walker, J. D. and M. J. Werle: Summary and Critique of a Turbulence Model for Free-Stream Turbulence Effects on Boundary Layer Characteristics, UTRC Report UTRC82-18.
37. IMSL Library 2 Reference Manual, Edition 66, International Mathematical and Statistics Libraries, Inc., 1977.
38. Schlichting, H.: *Boundary Layer Theory*, 6th Ed., McGraw-Hill Co., Inc., New York, 1978.

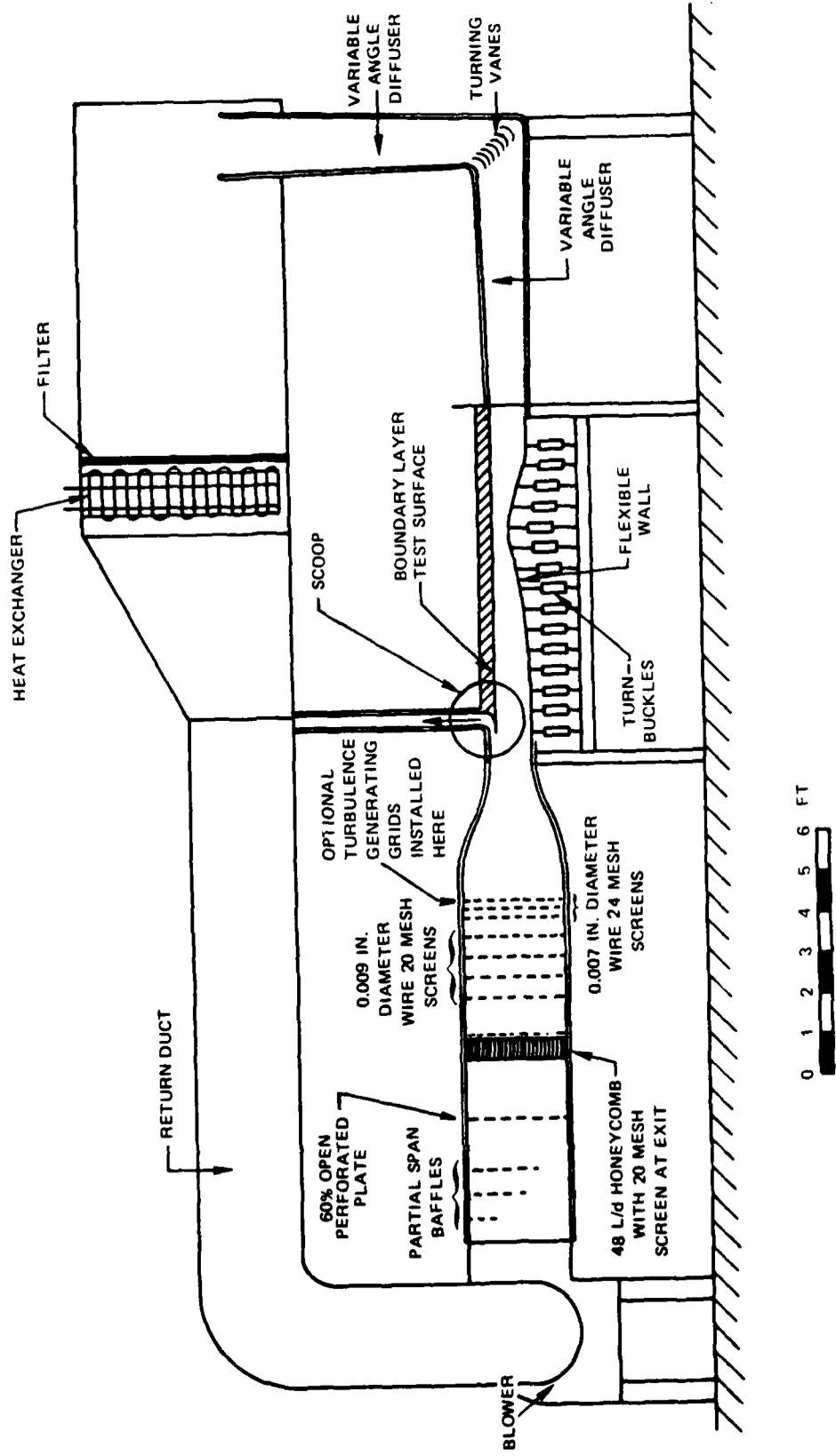


Figure 1. United Technologies Research Center Boundary Layer Wind Tunnel

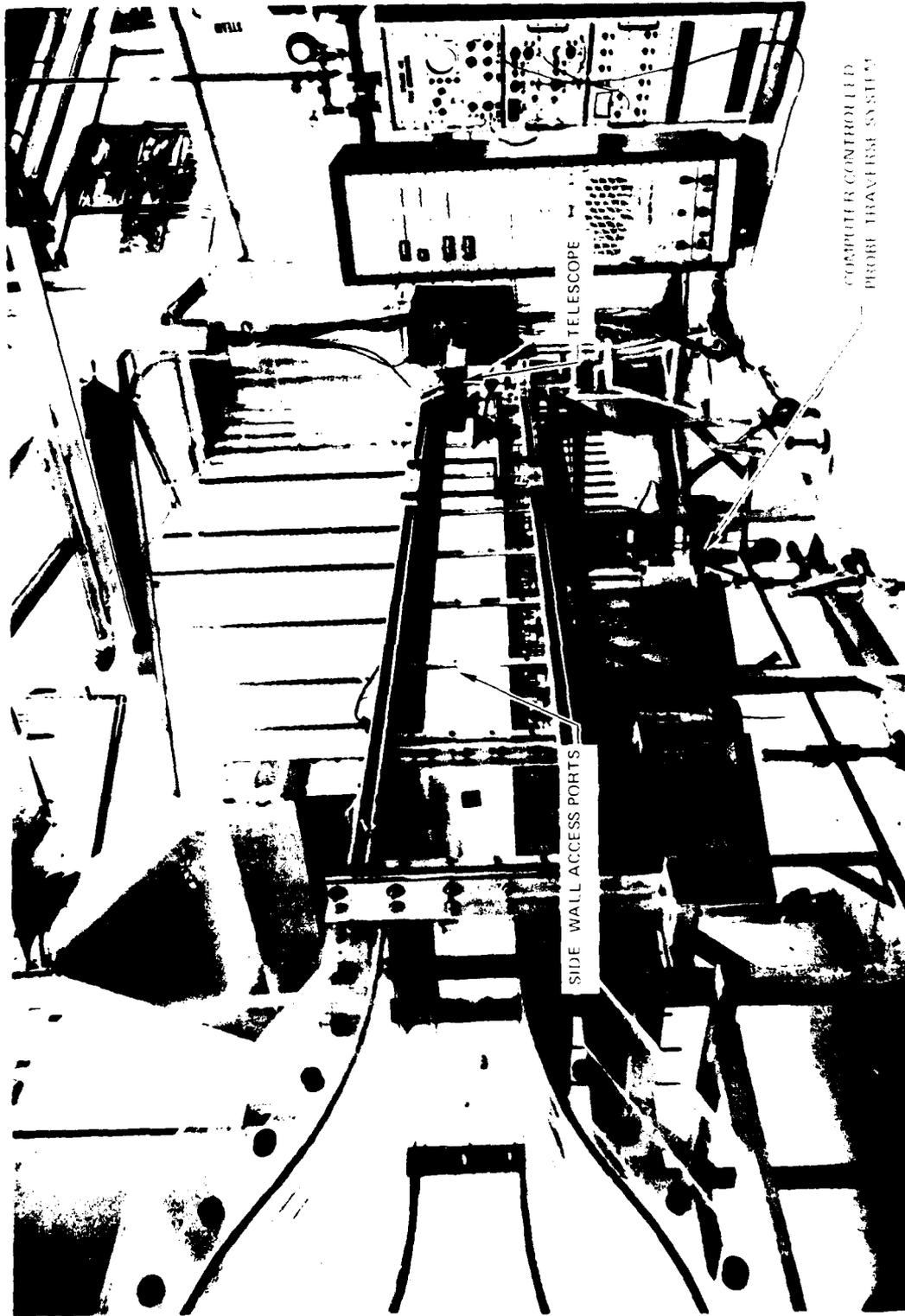


Figure 2. Photograph of UTRC Boundary Layer
Wind Tunnel Test Section

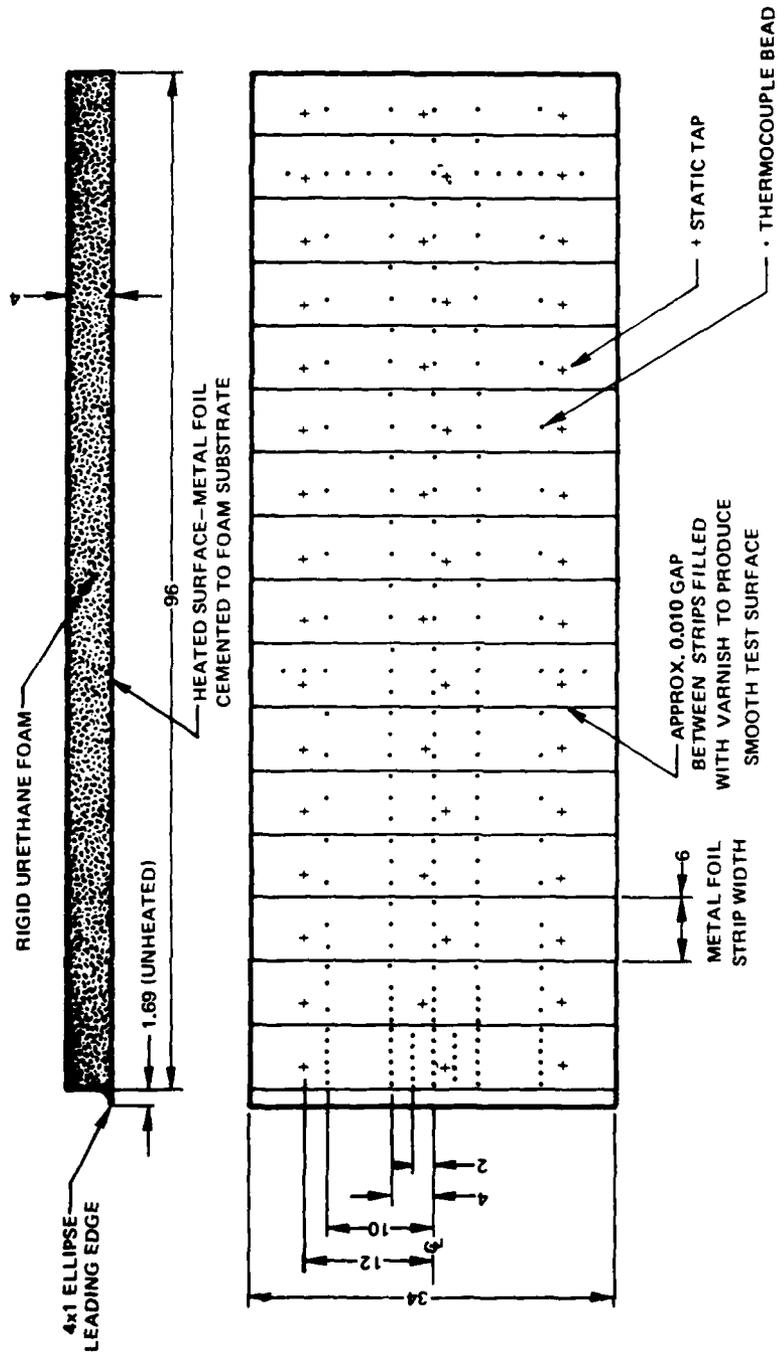


Figure 3. Instrumentation Diagram for the Uniform Heat Flux Flat Wall Model

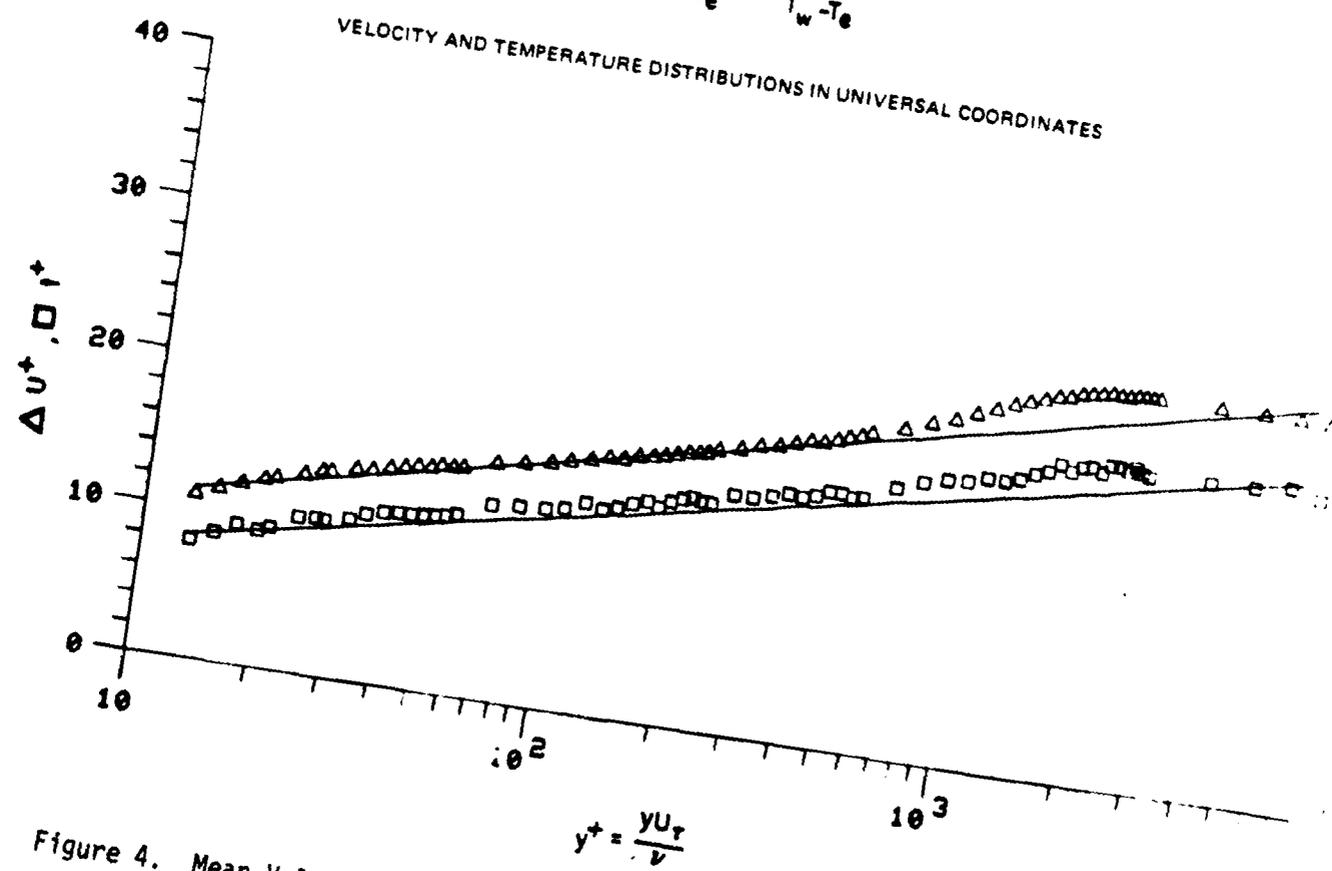
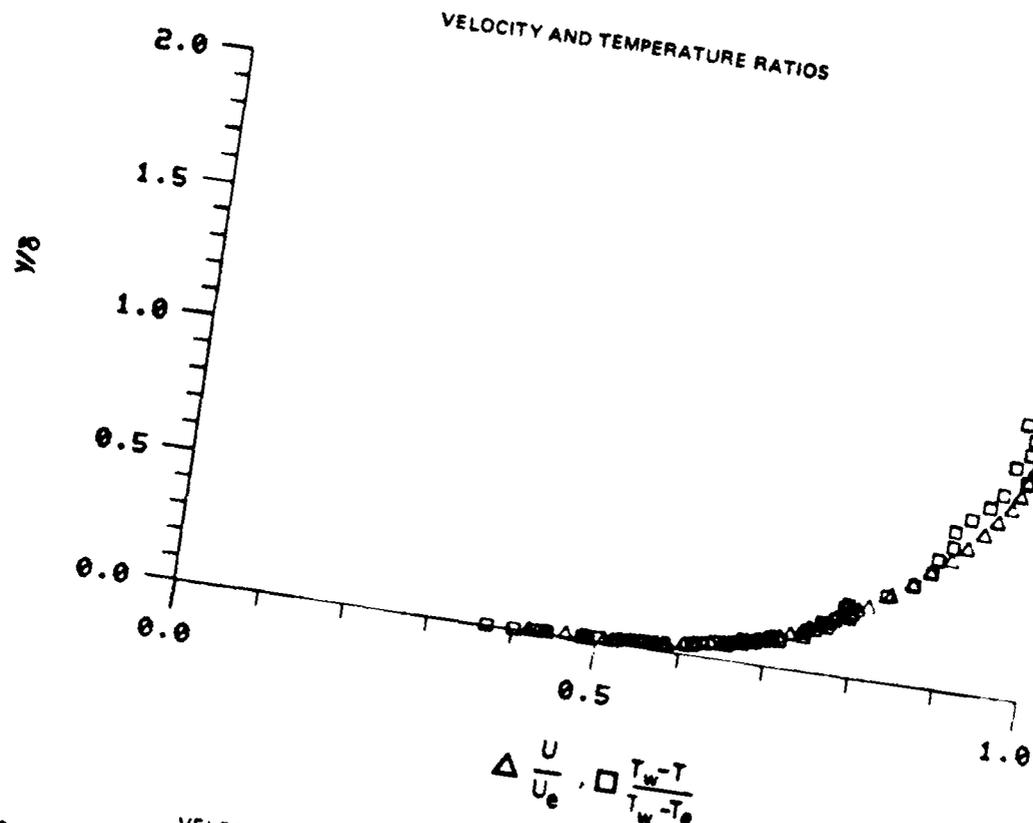


Figure 4. Mean Velocity and Temperature Profiles $x = 68$ in., $Te = 1.6\%$

RUN NO. 3. POINT 2.
 BOUNDARY LAYER PROPERTIES

	LINEAR INTERPOLATION TO WALL	STANDARD SUBLAYER FUNCTION FROM WALL TO $y^+=30$
FREE STREAM VELOCITY	98.170	98.170
FREE STREAM TEMPERATURE	70.180	
WALL TEMPERATURE	95.750	
WALL HEAT FLUX	.07954	
FREE STREAM DENSITY	.07552	
FREE STREAM KINEMATIC VISCOSITY	.0001619	
DENSITY OF FLUID AT WALL	.07204	
KINEMATIC VISCOSITY OF FLUID AT WALL	.0001760	
WALL/FREE STREAM DENSITY RATIO	.95396	
LOCATION REYNOLDS NUMBER (REX)	3435559.84	
INPUT VALUE OF VELOCITY DELTA	1.30000	
INPUT VALUE OF TEMPERATURE DELTA	1.30000	
CALCULATED DELTA		1.12046
DELTA 99.5% INPUT	.00300	
DISPLACEMENT THICKNESS (DELSTAR)	.15429	.15443
MOMENTUM THICKNESS (THETA)	.10938	.10967
ENERGY-DISSIPATION THICKNESS	.19539	.19556
ENTHALPY THICKNESS	.00597	.00598
SHAPE FACTOR 12 (DELSTAR/THETA)	1.41057	1.40819
SHAPE FACTOR 32 (ENERGY/THETA)	1.78634	1.76327
MOMENTUM THICKNESS REYNOLDS NUMBER	5526.16	5540.66
DISPLACEMENT THICKNESS REYNOLDS NUMBER	7795.01	7602.27
SKIN FRICTION COEFFICIENT	.002865	
FRICTION VELOCITY	3.81748	
LAW OF THE WALL CONSTANT (K)	.41000	
LAW OF THE WALL CONSTANT (C)	5.00000	
WAKE STRENGTH		.44004
CLAUSERS 'DELTA' INTEGRAL	-3.62559	-3.91767
CLAUSEPS 'G' INTEGRAL	24.97566	24.86843
DISPLACEMENT THICKNESS - CONSTANT DENSITY	.14465	.14845
MOMENTUM THICKNESS - CONSTANT DENSITY	.11055	.11085
SHAPE FACTOR 12 - CONSTANT DENSITY	1.30848	1.33924
LOCATION -X-	68.00000	
$Te = 1.6\%$		

Table 2

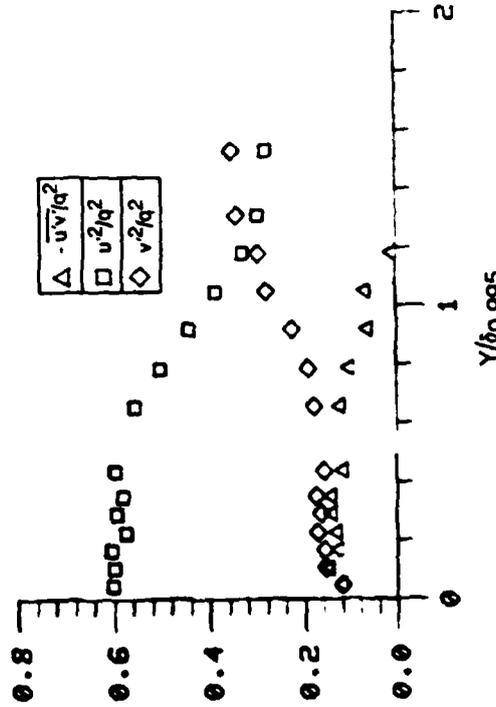
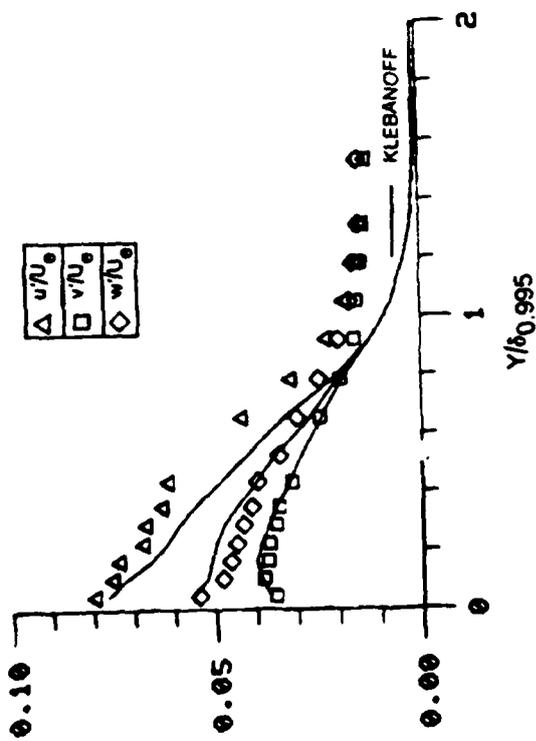
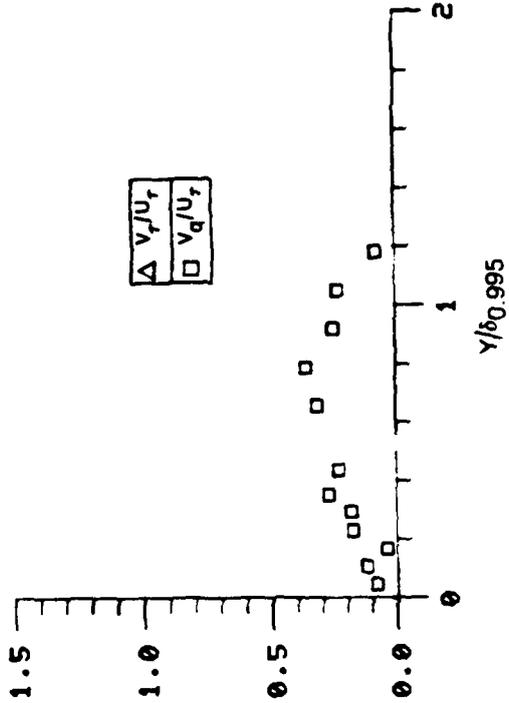
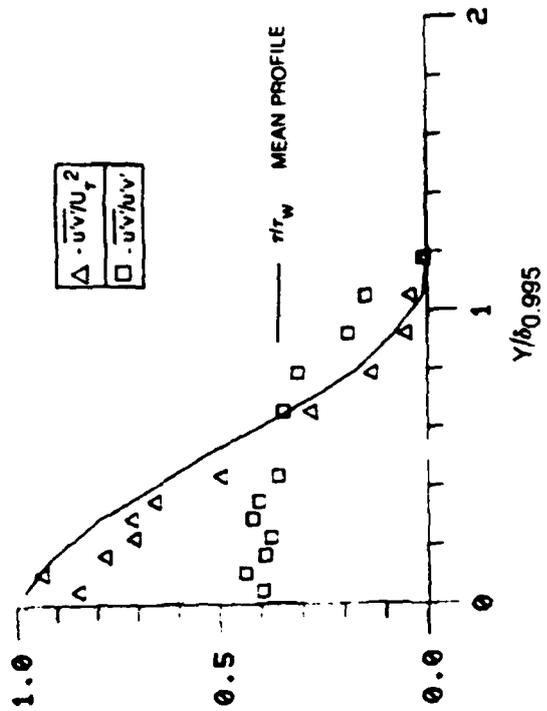


Figure 5A. Boundary Layer Turbulence Quantities $x = 68$ in, $Te = 1.6\%$

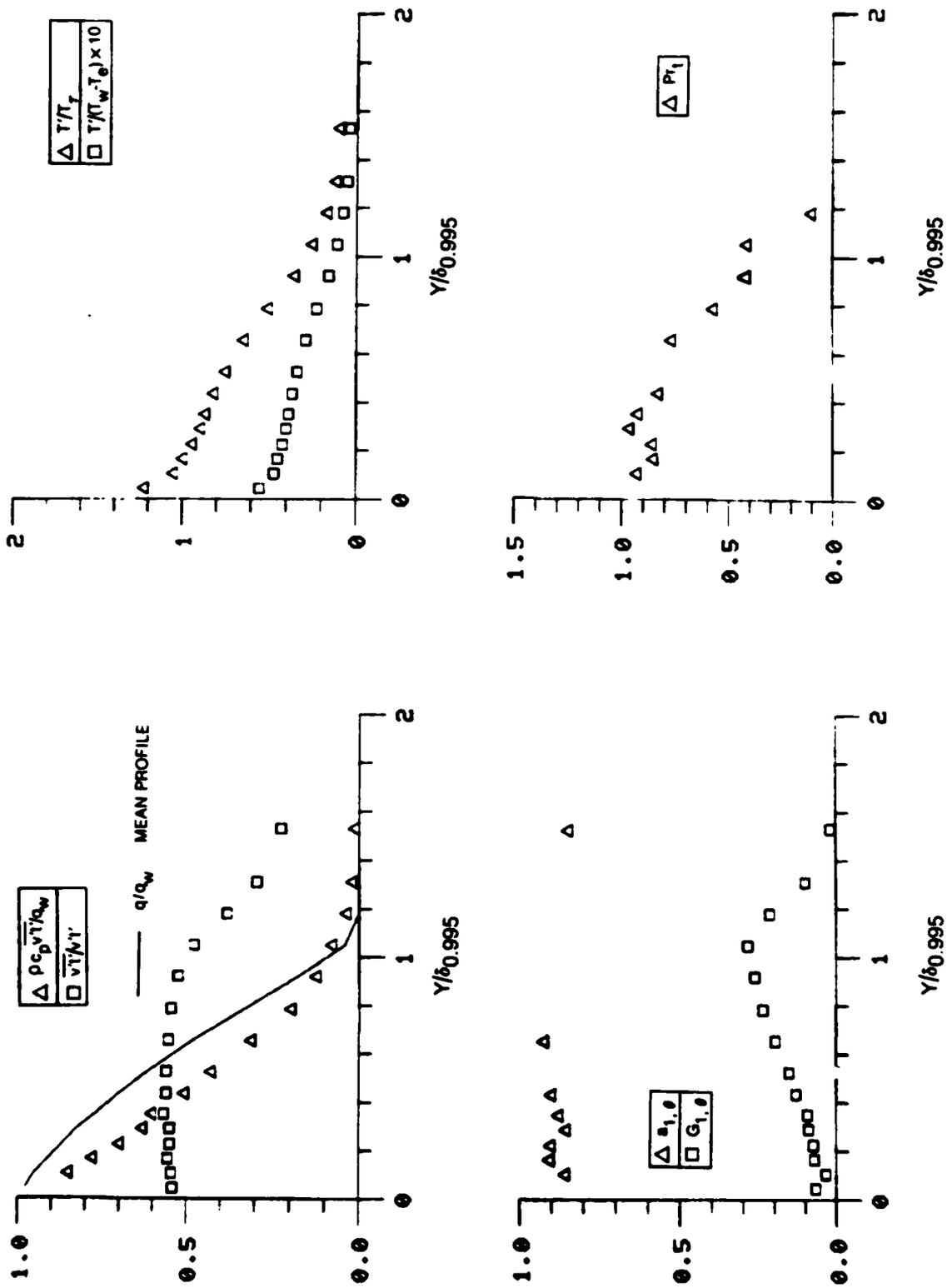


Figure 58. Boundary Layer Turbulence Quantities $x = 68$ in, $T_e = 1.6\%$

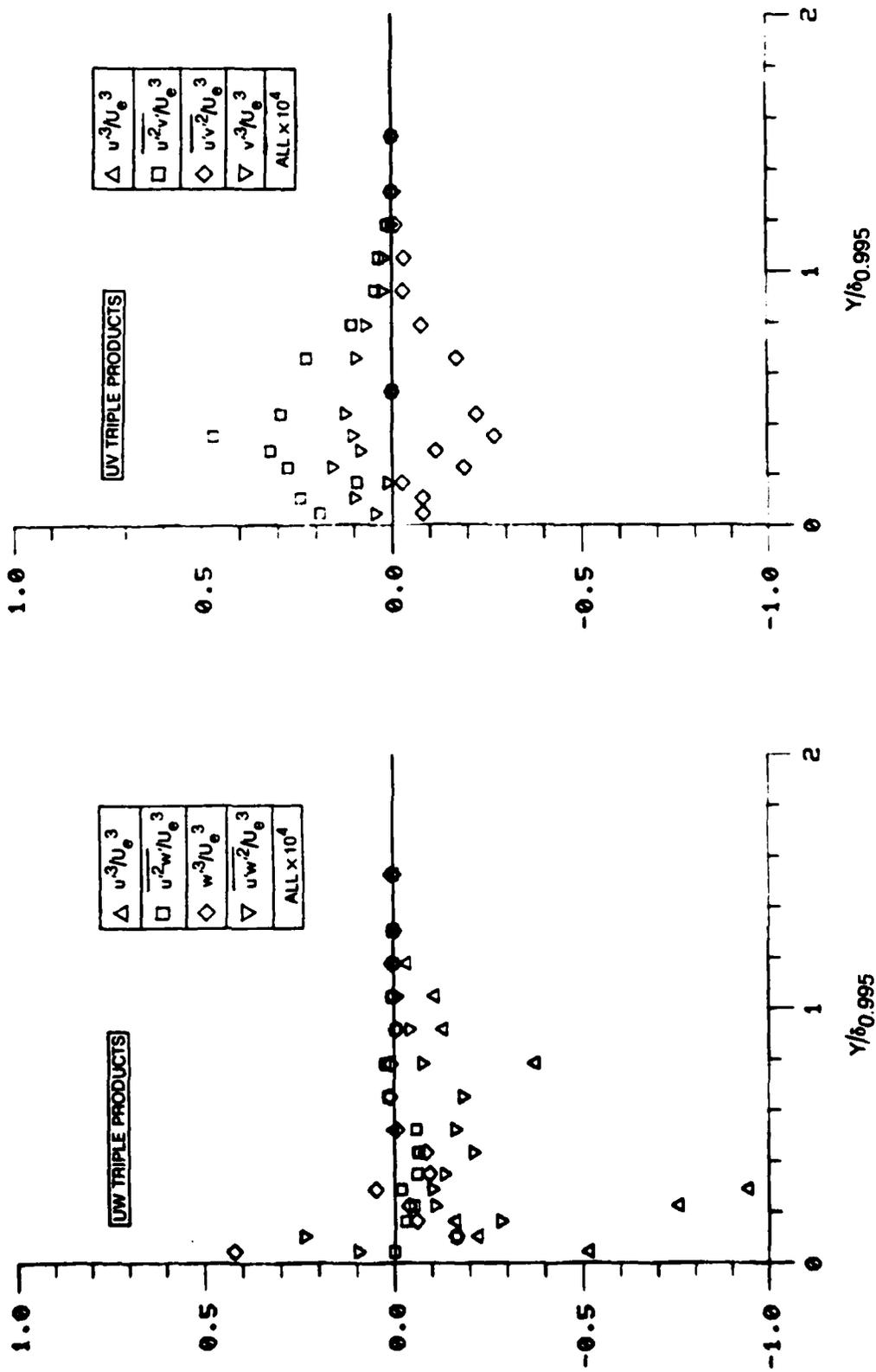


Figure 5C. Boundary Layer Triple Product Distributions $x = 68$ in, $T_e = 1.6\%$

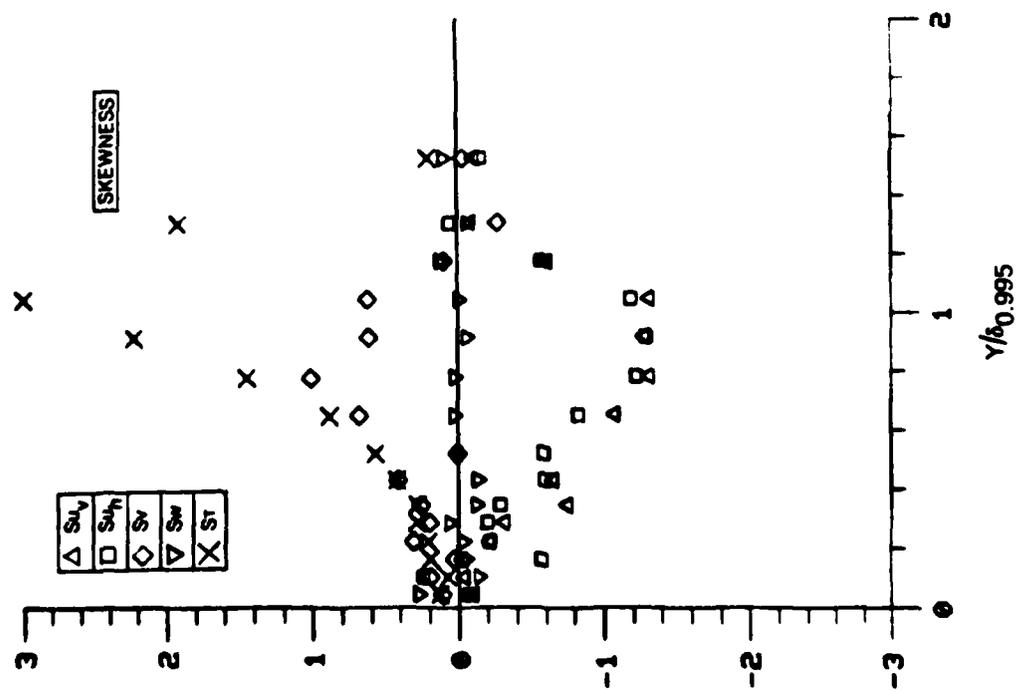
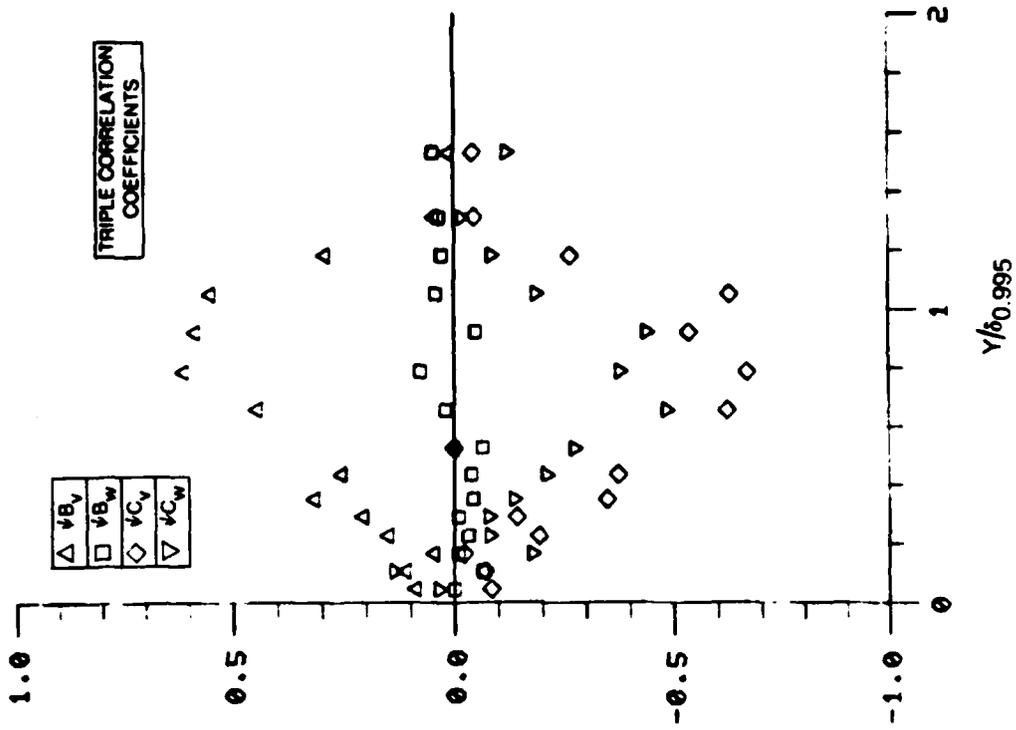


Figure 50. Boundary Layer Skewness and Triple Product Correlation Coefficient Distributions $x = 68$ in, $T_e = 1.6\%$

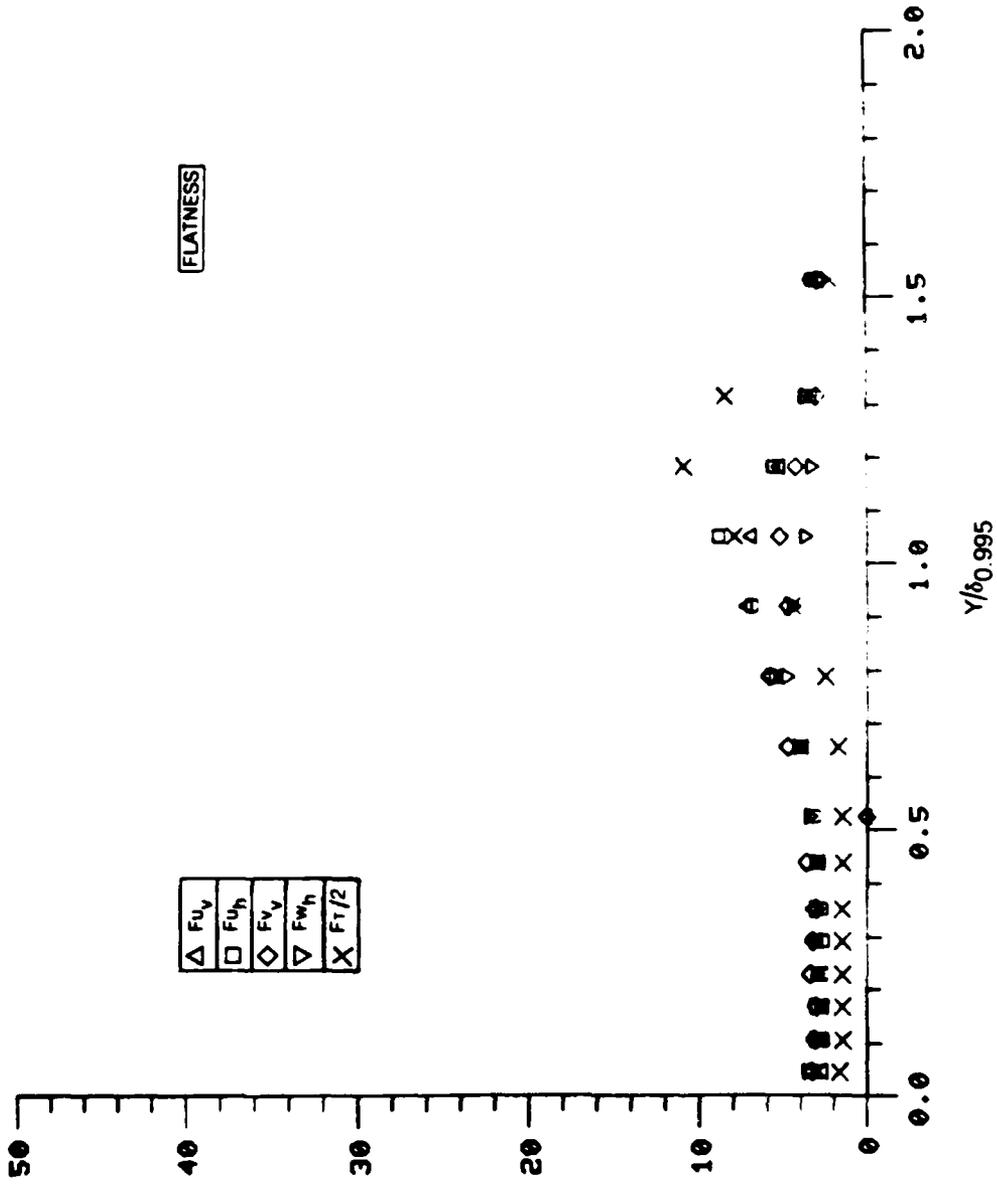


Figure 5E. Boundary Layer Flatness Distributions $x = 68$ in, $T_e = 1.6\%$

KEY		
SYMBOL	SOURCE	T_e %
○	PRESENT	0.2
□	PRESENT	1.5
△	PRESENT	4.2
---	REF 20	0.1
—	REF 12	AS SHOWN

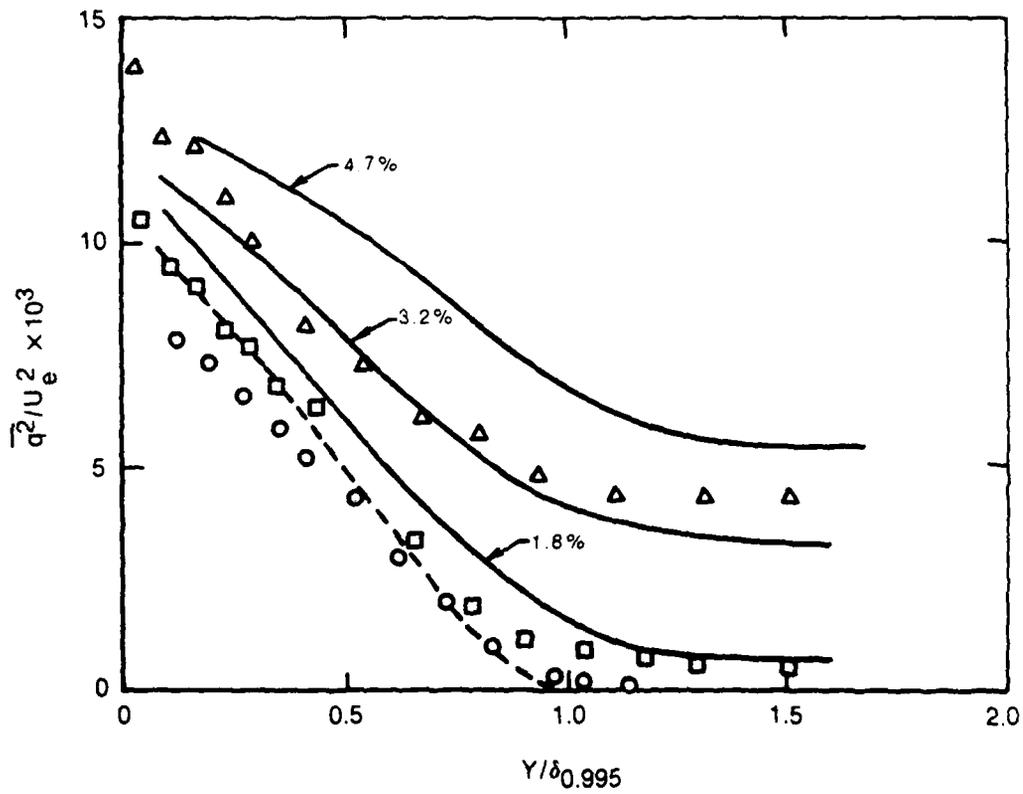


Figure 6. Boundary Layer Turbulent Kinetic Energy Distribution

SYM	FAIRING	T_e %
○	—	0.2
□	- · - · -	1.5
△	- - -	4.2

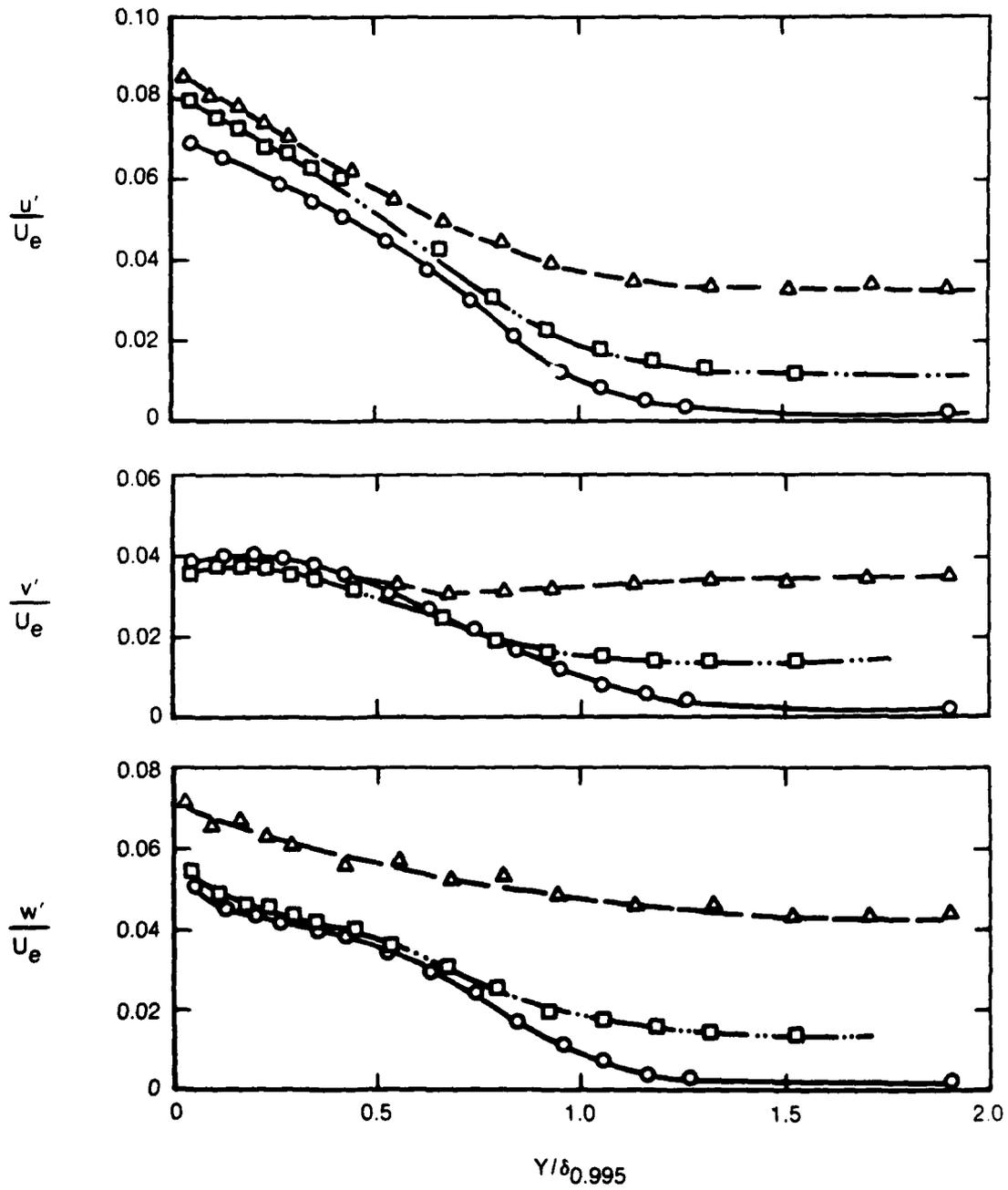


Figure 7. Distribution of the Components of Boundary Layer Turbulence

SYMBOL	T_e %
○	0.2
□	1.5
△	4.2

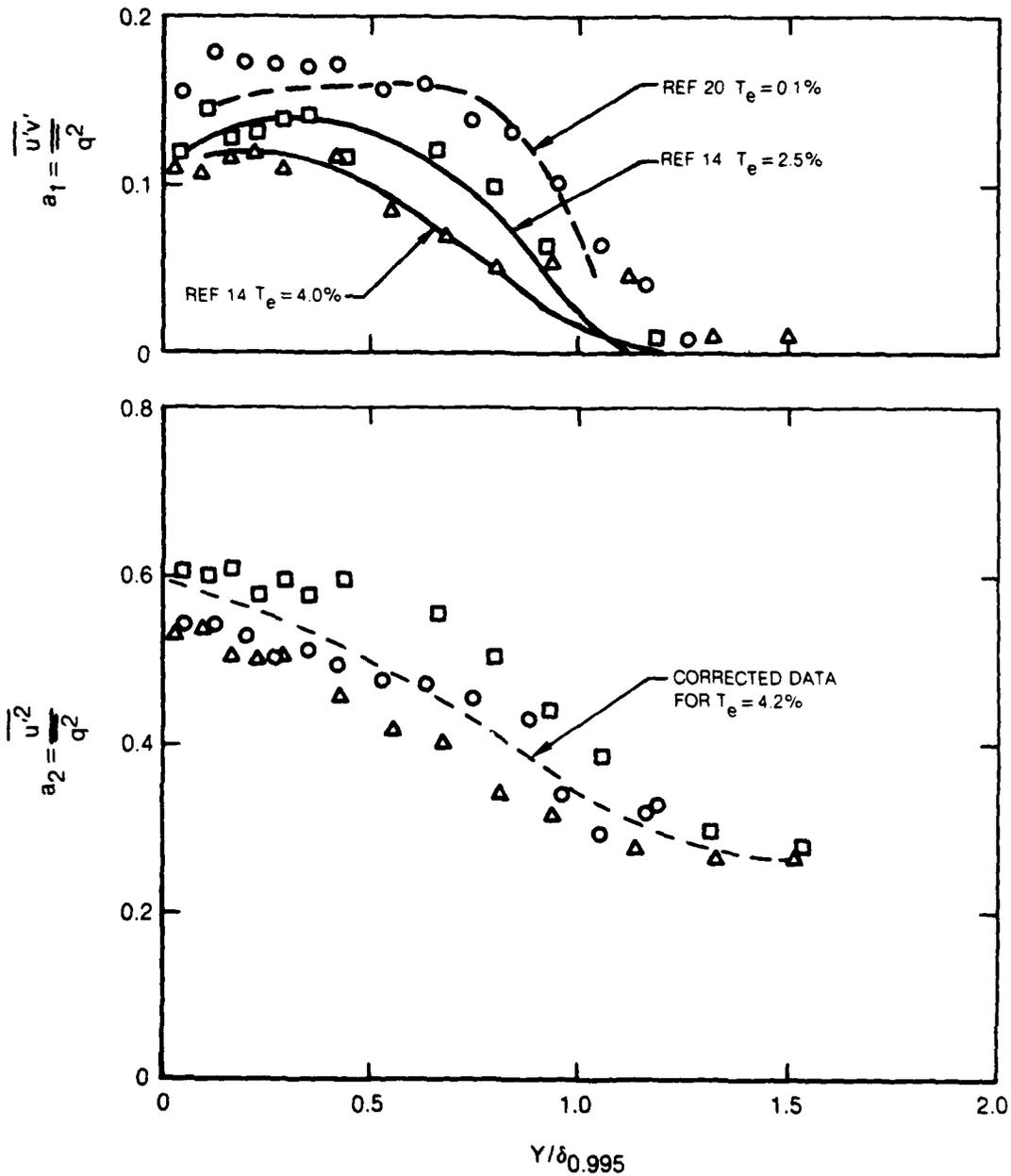


Figure 8A. Distribution of the Turbulence Structural Coefficients a_1 and a_2 Across the Boundary Layer

SYMBOL	T_e %
○	0.2
□	1.5
△	4.2

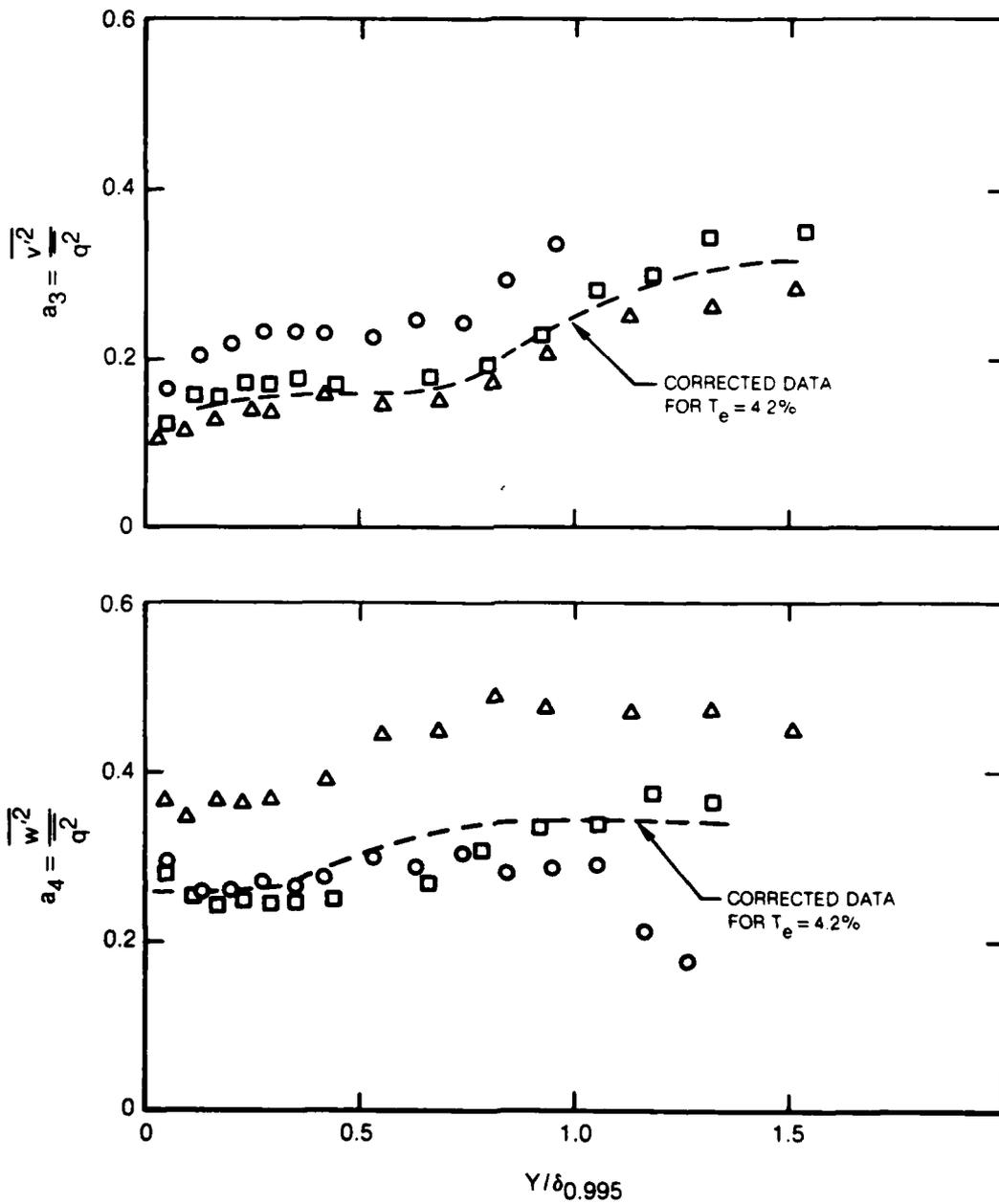


Figure 8B. Distribution of the Turbulence Structural Coefficients a_3 and a_4 Across the Boundary Layer

KEY	
SYM	SOURCE
●	PRESENT DATA
△	CHARNAY, et al (10)
▽	CHARNAY, et al (12)
□	HANCOCK (14)
◇	HUFFMAN (11)
○	EVANS (24)

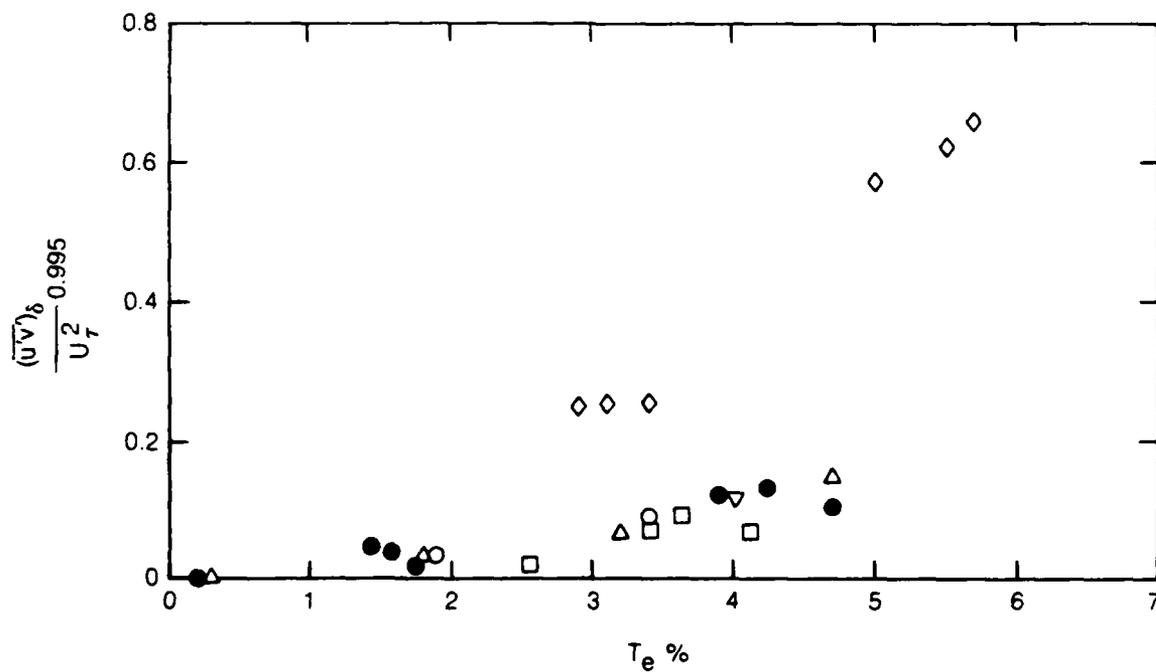


Figure 9. Influence of Free-Stream Turbulence on the Turbulent Shear Stress at $\delta_{0.995}$

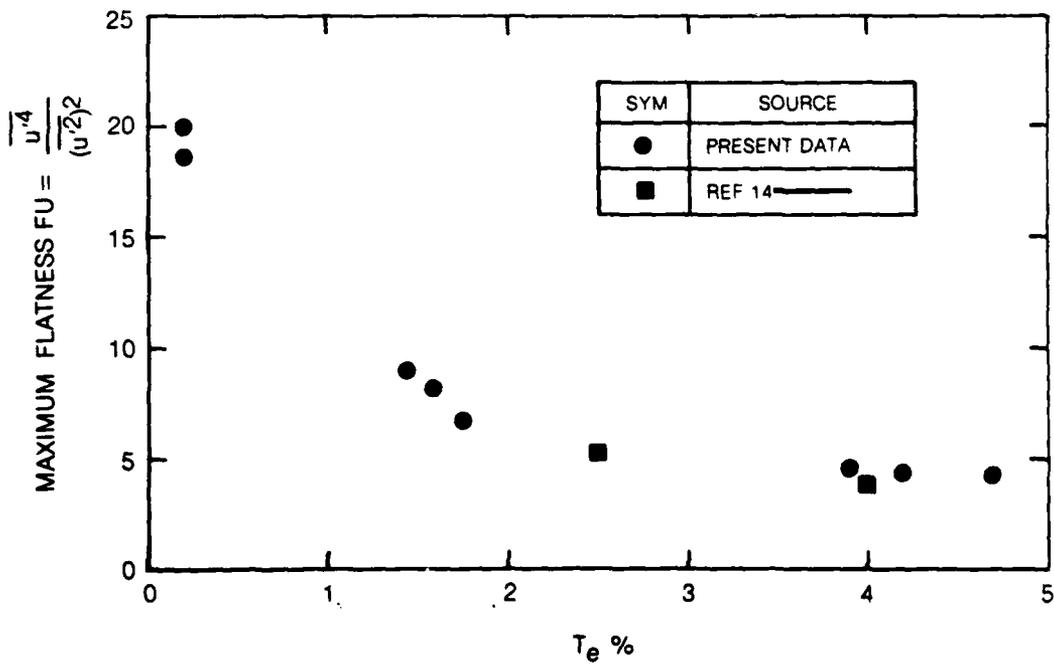
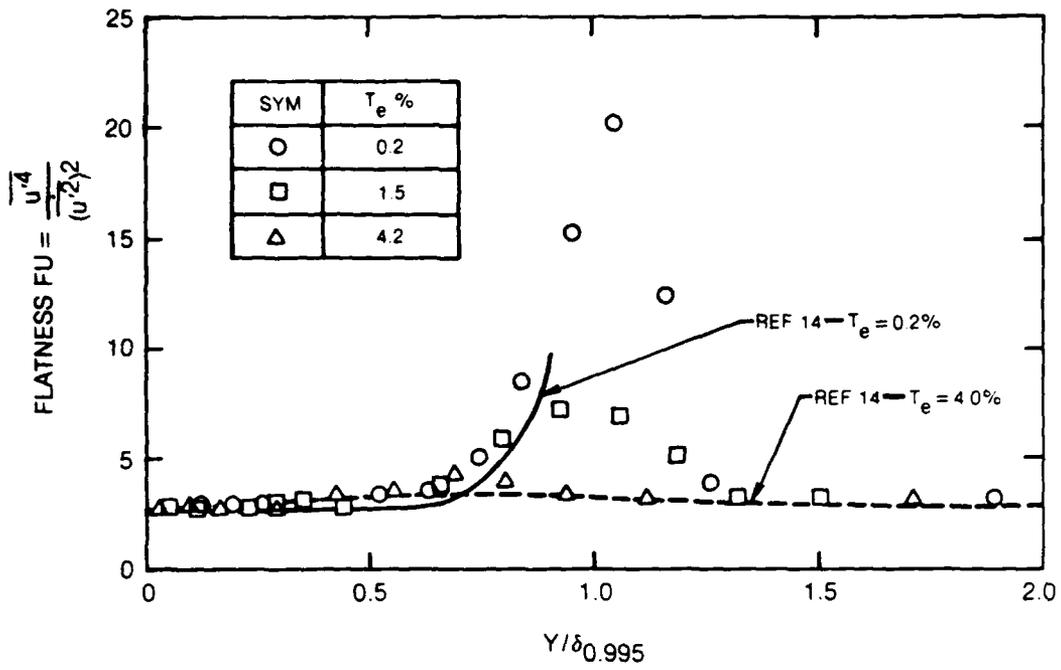


Figure 10. Effect of Free-Stream Turbulence on the u' Flatness Factor Distributions

SYMBOL CODE		
DATA	EQ 17	T_e %
○	—	0.2
▲	- - -	1.4
□	- · - ·	4.0

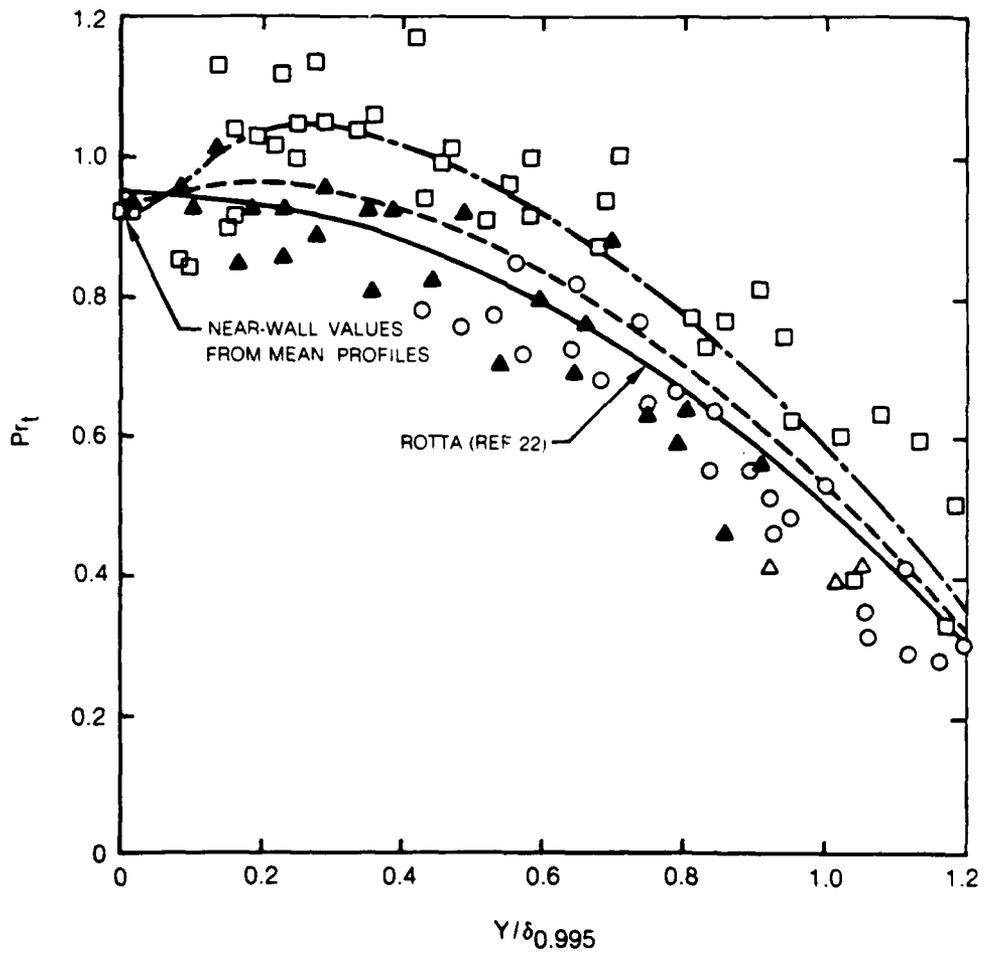


Figure 11. Effect of Free-Stream Turbulence on the Turbulent Prandtl Number Distribution

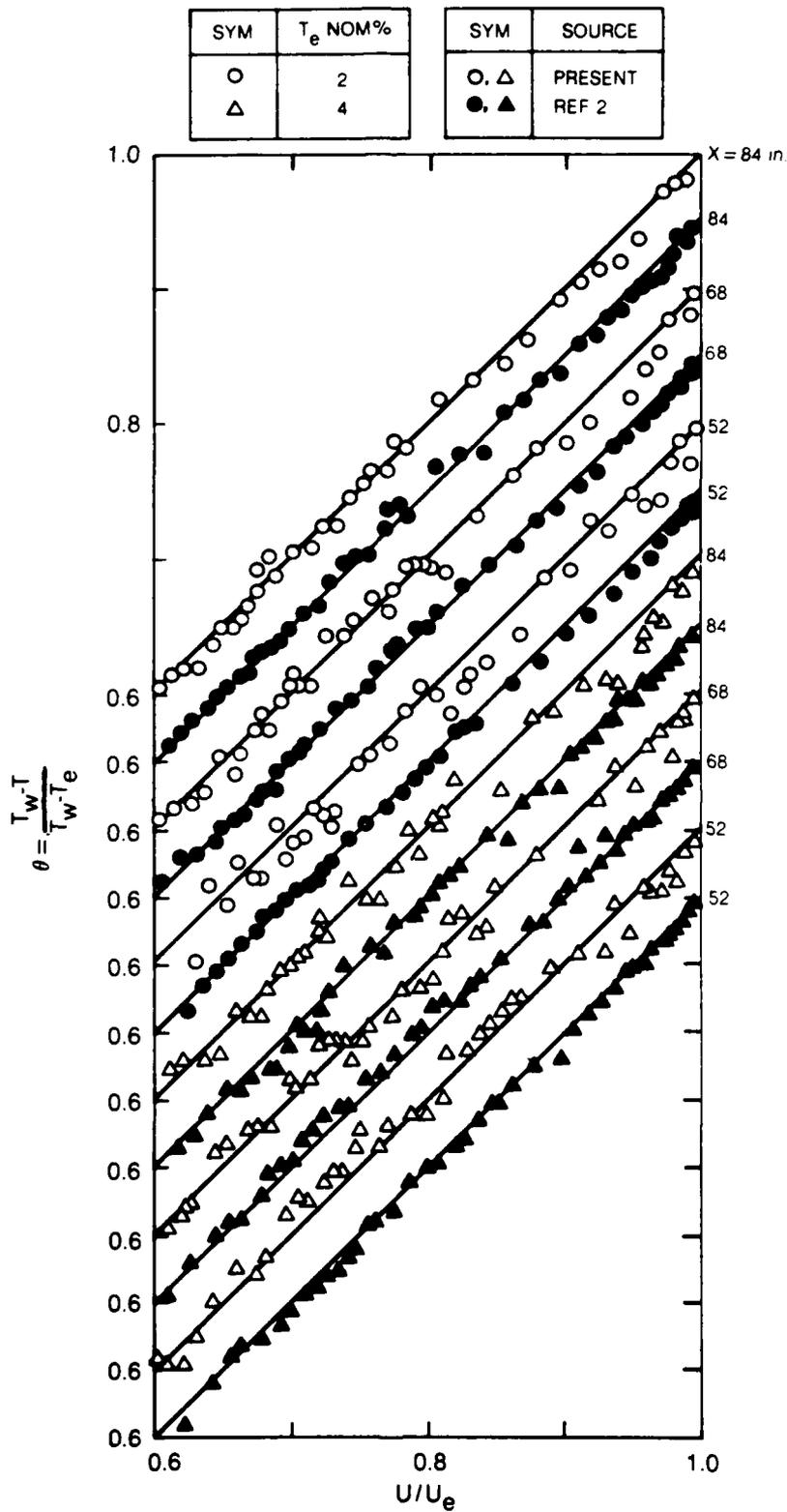


Figure 12. Mean Velocity and Temperature Profiles for Various Free-Stream Turbulence Levels

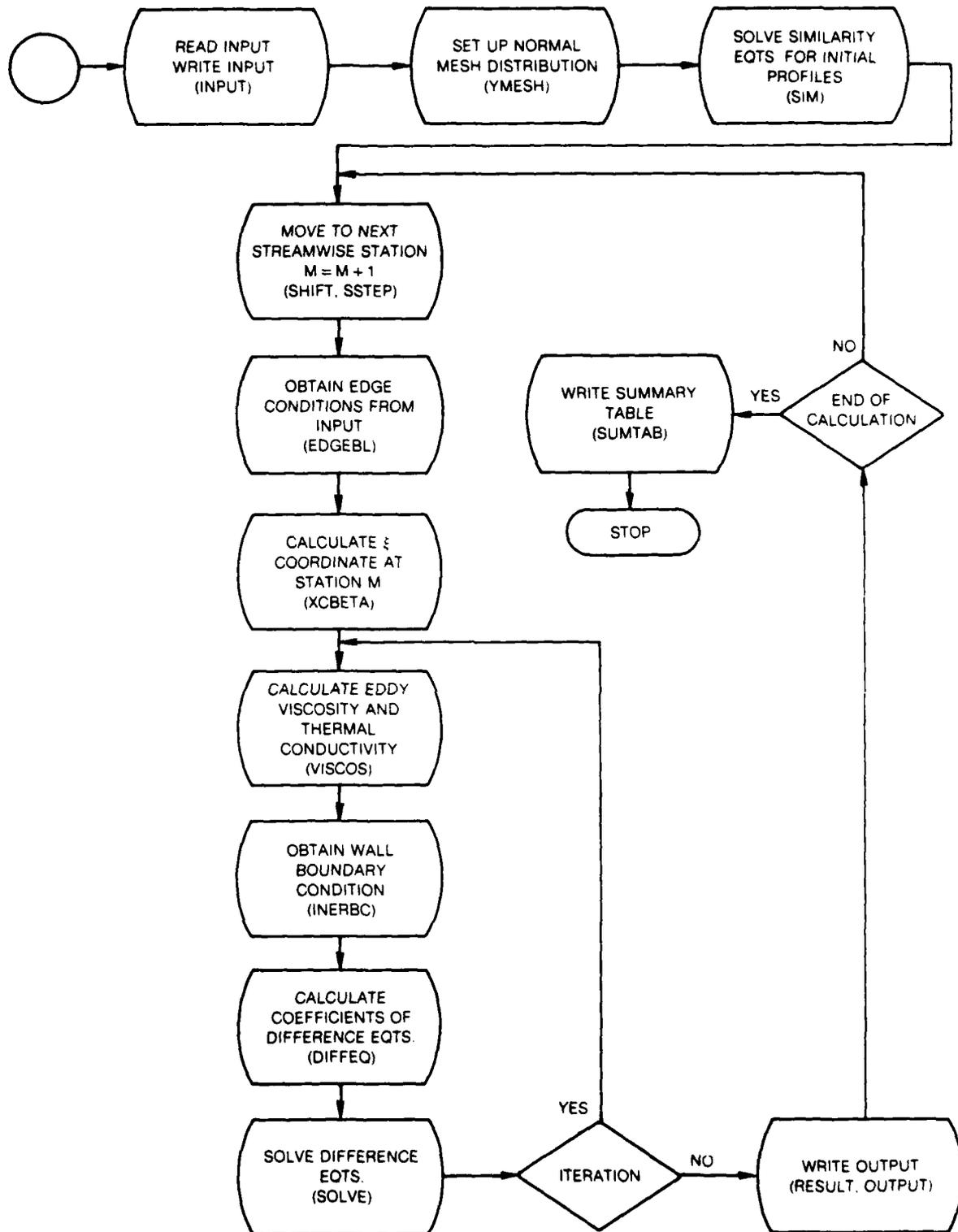


Figure 13. Flow Chart for ABLE Code - Module MAIN

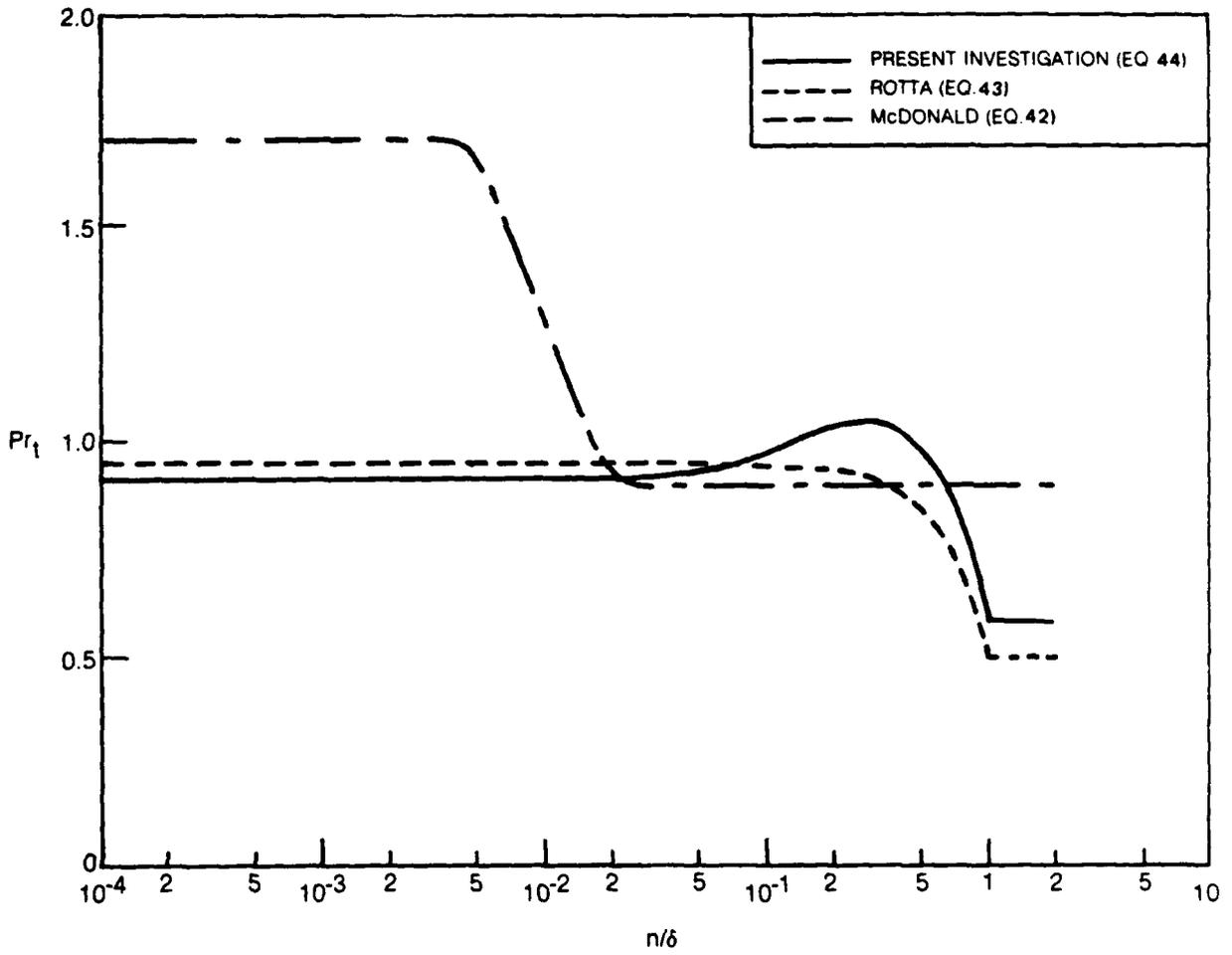


Figure 14. Comparison of Turbulent Prandtl Number Distributions for Flow with 2% Freestream Turbulence

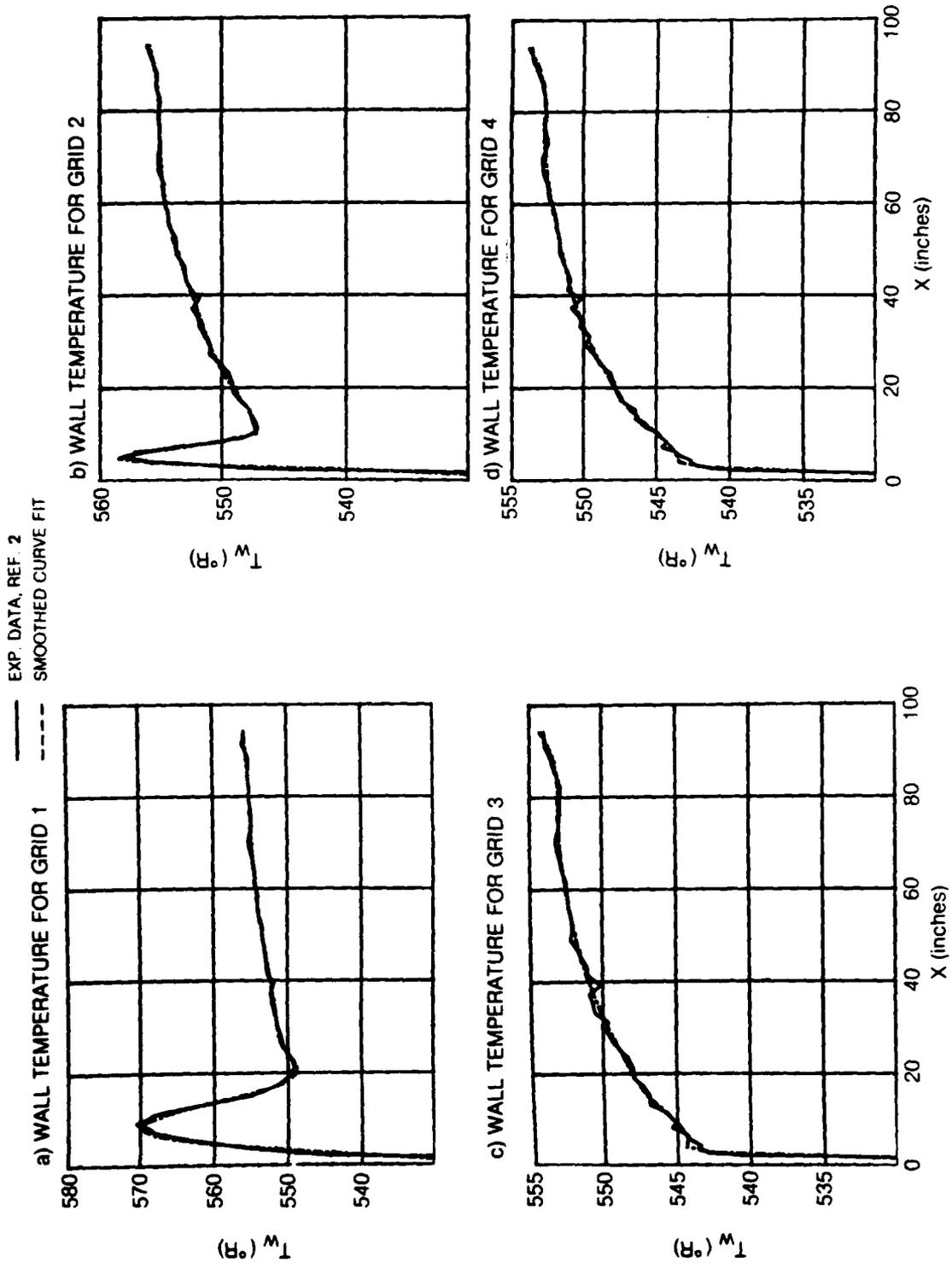
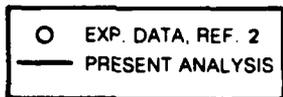
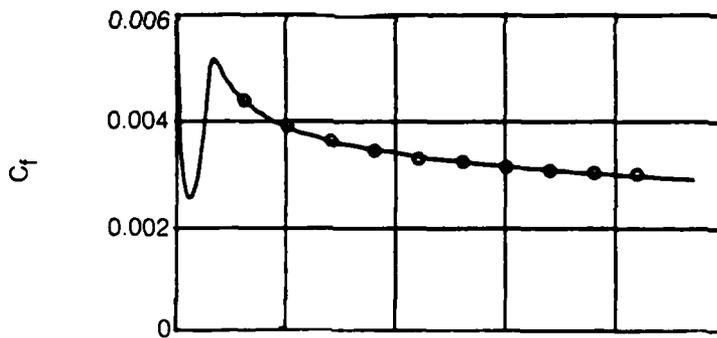


Figure 15. Numerically Smoothed Experimental Wall Temperature Distribution



a) FLOW WITH 6% INLET TURBULENCE



b) FLOW WITH 2% INLET TURBULENCE

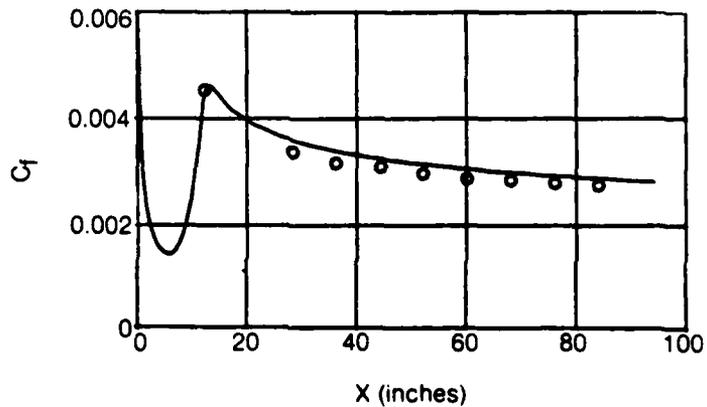


Figure 16. Comparison of Theoretical Skin Friction with Experimental Data for Different Inlet Turbulence Levels

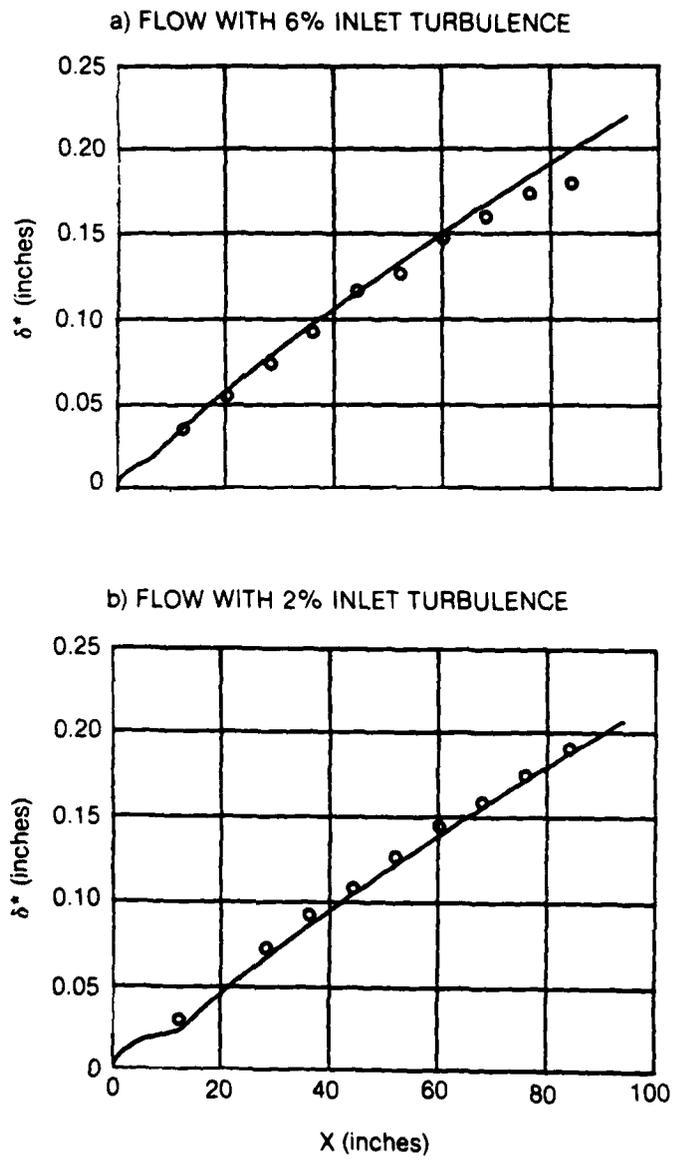
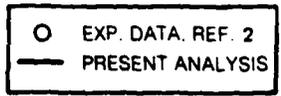
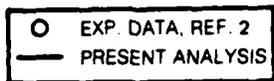
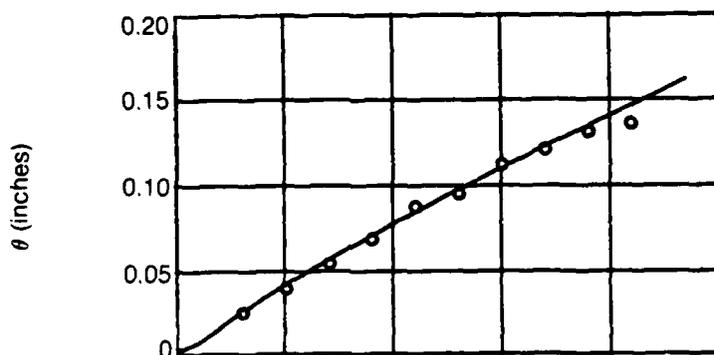


Figure 17. Comparison of Theoretical Displacement Thickness with Experimental Data for Different Inlet Turbulence Levels



a) FLOW WITH 6% INLET TURBULENCE



b) FLOW WITH 2% INLET TURBULENCE

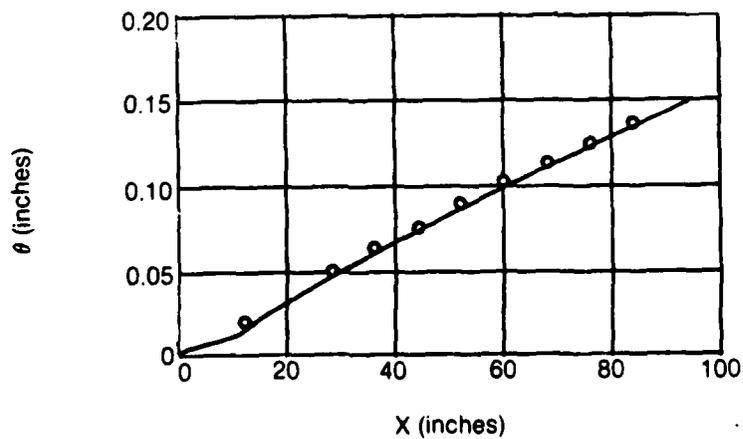


Figure 18. Comparison of Theoretical Momentum Thickness with Experimental Data for Different Inlet Turbulence Levels

INLET TURBULENCE	δ (inches)	
6%	1.256	□ EXP. DATA, REF. 2
2%	1.055	— PRESENT ANALYSIS

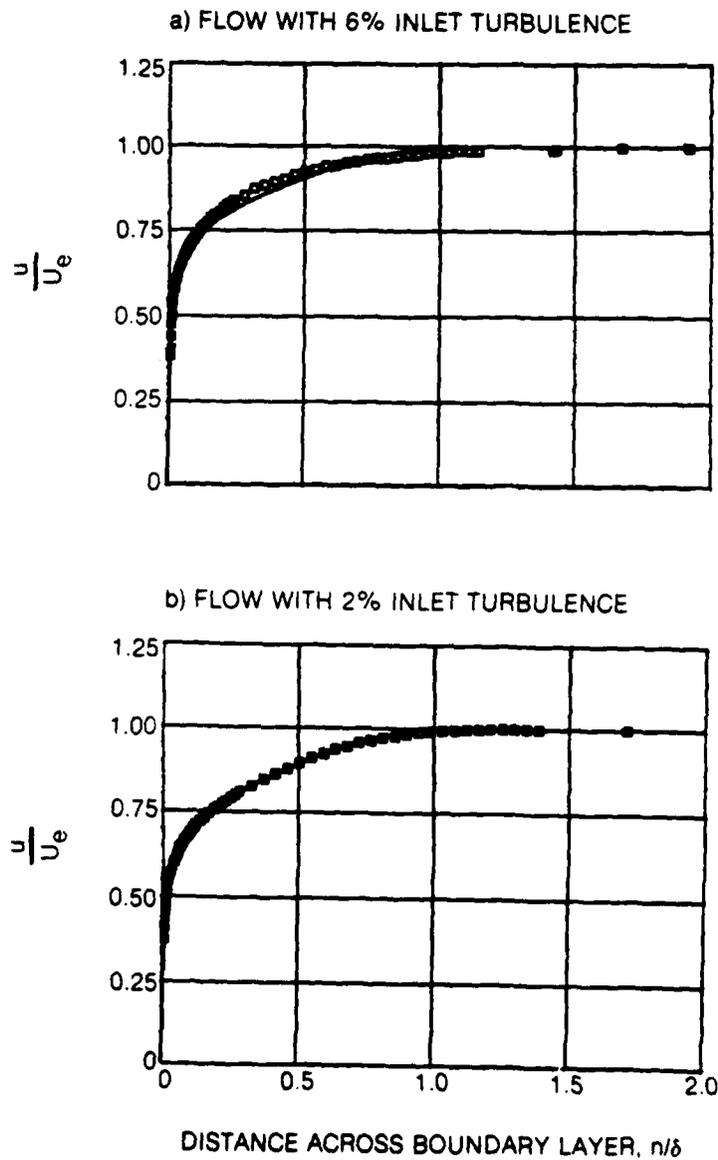
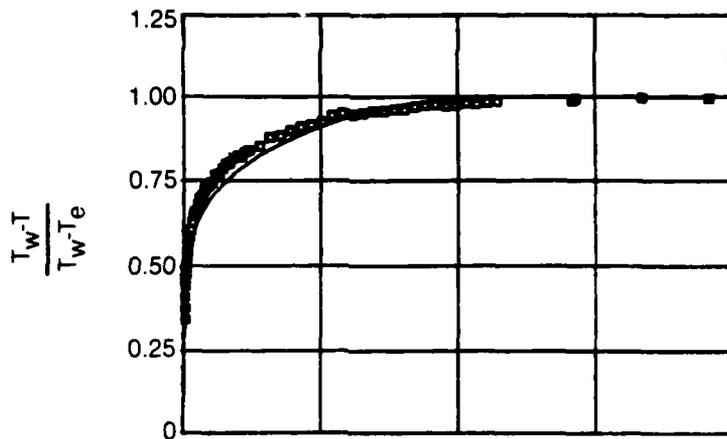


Figure 19. Comparison of Theoretical Mean Velocity With Experimental Data at $X = 68$ inches for Different Inlet Turbulence Levels

INLET TURBULENCE	δ (inches)	
6%	1.256	□ EXP. DATA, REF. 2
2%	1.055	— PRESENT ANALYSIS

a) FLOW WITH 6% INLET TURBULENCE



b) FLOW WITH 2% INLET TURBULENCE

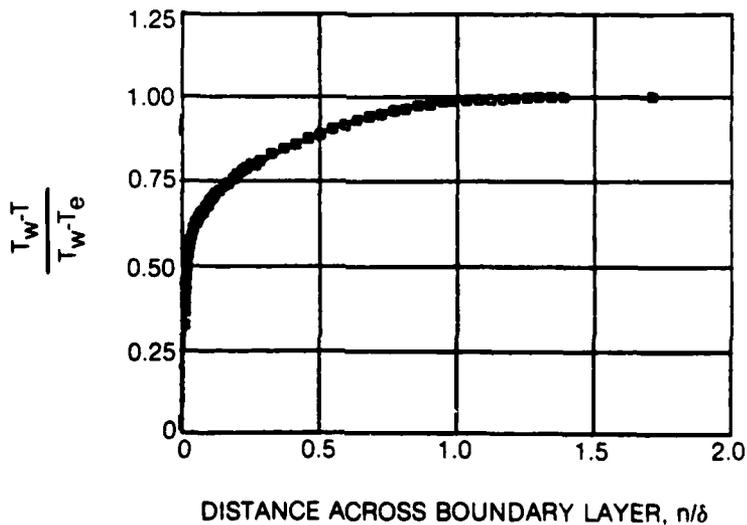


Figure 20. Comparison of Theoretical Mean Temperature with Experimental Data at X = 68 inches for Different Inlet Turbulence Levels

INLET TURBULENCE	δ (inches)	
6%	1.256	□ EXP. DATA
2%	1.055	— PRESENT ANALYSIS

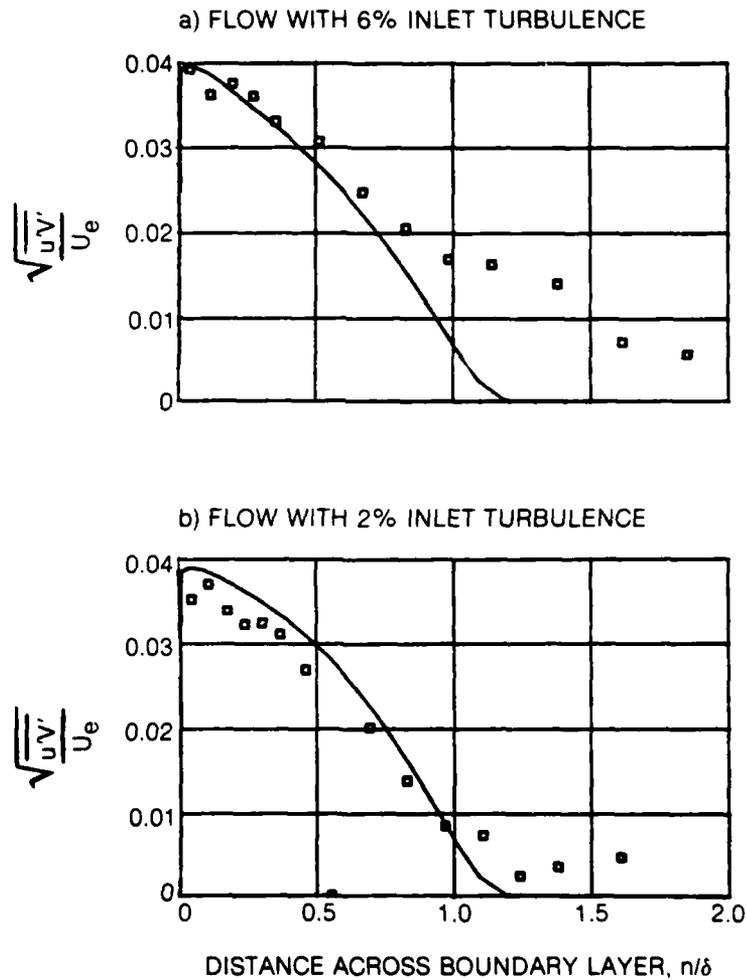


Figure 21. Comparison of Theoretical Reynolds Shear Stress with Experimental Data at $X = 68$ inches for Different Inlet Turbulence Levels

INLET TURBULENCE	δ (inches)	
6%	1.256	□ EXP DATA
2%	1.055	— PRESENT ANALYSIS

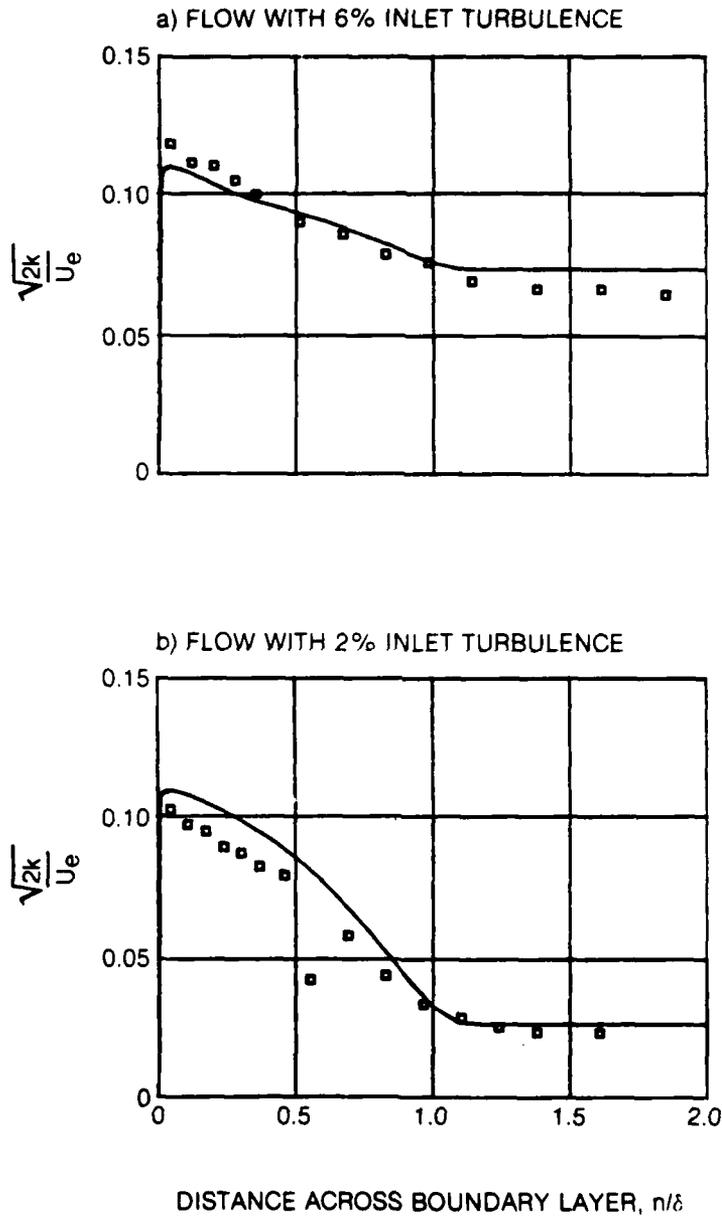


Figure 22. Comparison of Theoretical Turbulent Kinetic Energy with Experimental Data at $X = 68$ inches for Different Inlet Turbulence Levels

INLET TURBULENCE	δ (inches)	□ EXP. DATA
6%	1.256	— PRESENT ANALYSIS
2%	1.055	

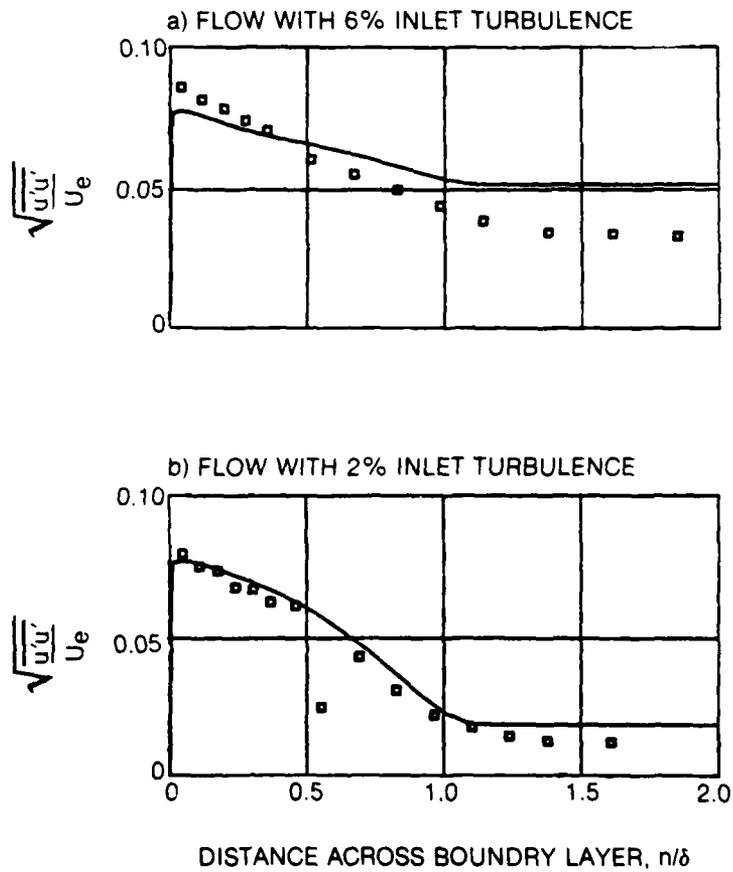


Figure 23. Comparison of Theoretical Component of Turbulent Intensity, $\sqrt{u'u'}$, with Experimental Data at $X = 68$ inches for Different Inlet Turbulence Levels

INLET TURBULENCE	δ (inches)	<div style="display: inline-block; width: 10px; height: 10px; border: 1px solid black; margin-right: 5px;"></div> EXP. DATA <div style="display: inline-block; width: 10px; height: 1px; border-bottom: 1px solid black; margin-right: 5px;"></div> PRESENT ANALYSIS
6%	1.256	
2%	1.055	

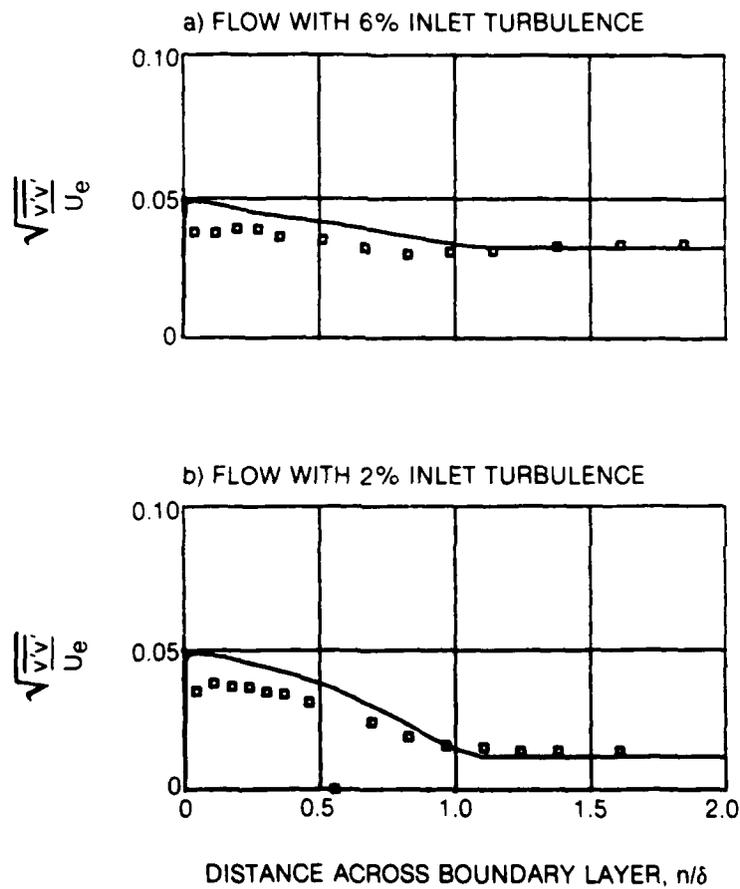


Figure 24. Comparison of Theoretical Component of Turbulent Intensity, $\sqrt{v'v'}$, with Experimental Data at $X = 68$ inches for Different Inlet Turbulence Levels

INLET TURBULENCE	δ (inches)	
6%	1.256	□ EXP. DATA
2%	1.055	— PRESENT ANALYSIS

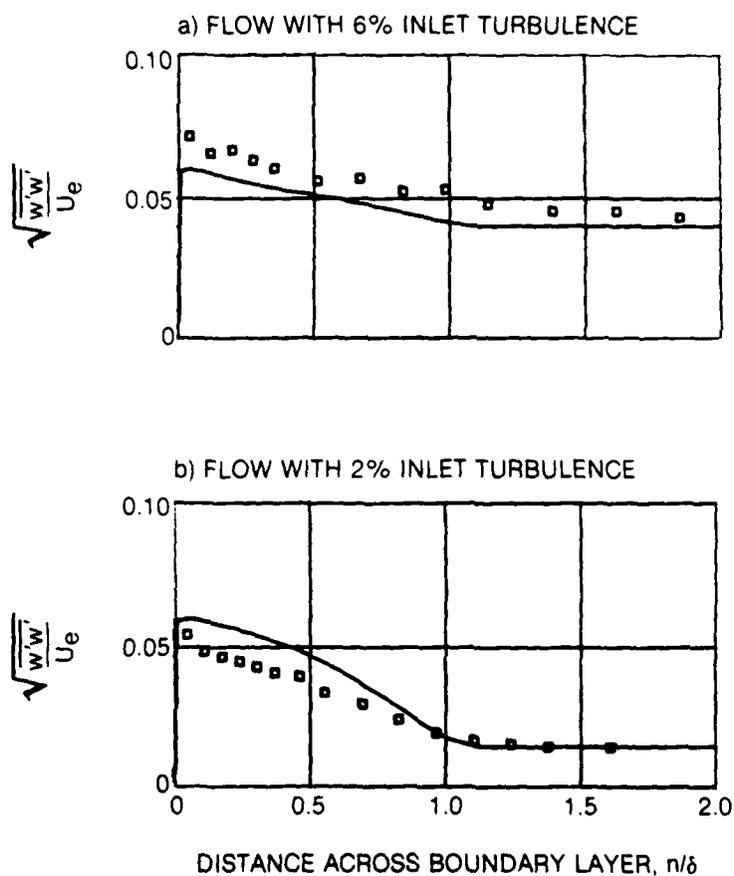


Figure 25. Comparison of Theoretical Component of Turbulent Intensity, $\sqrt{w'w'}$, with Experimental Data at $X = 68$ inches for Different Turbulence Levels

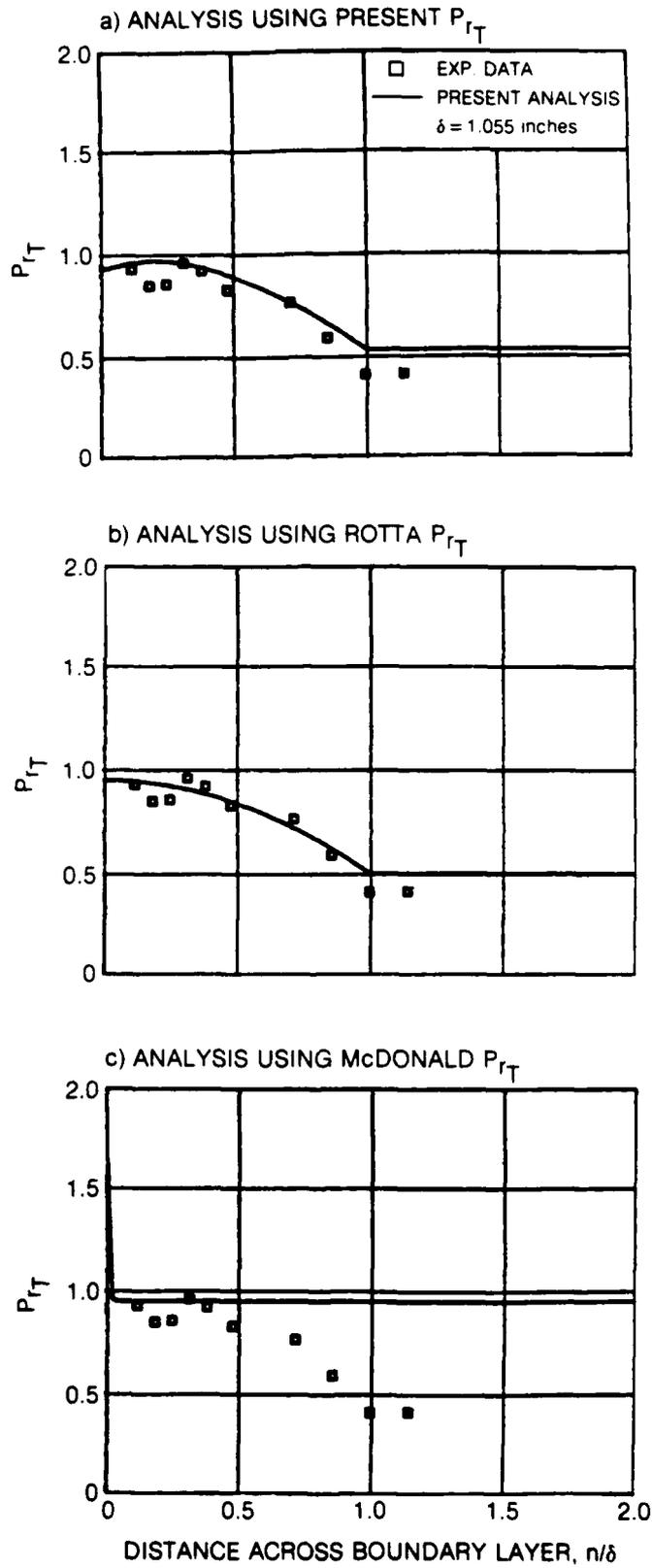


Figure 26a. Comparison of Theoretical Turbulent Prandtl Number with Experimental Data at $X = 68$ inches for 2% Inlet Turbulence Level

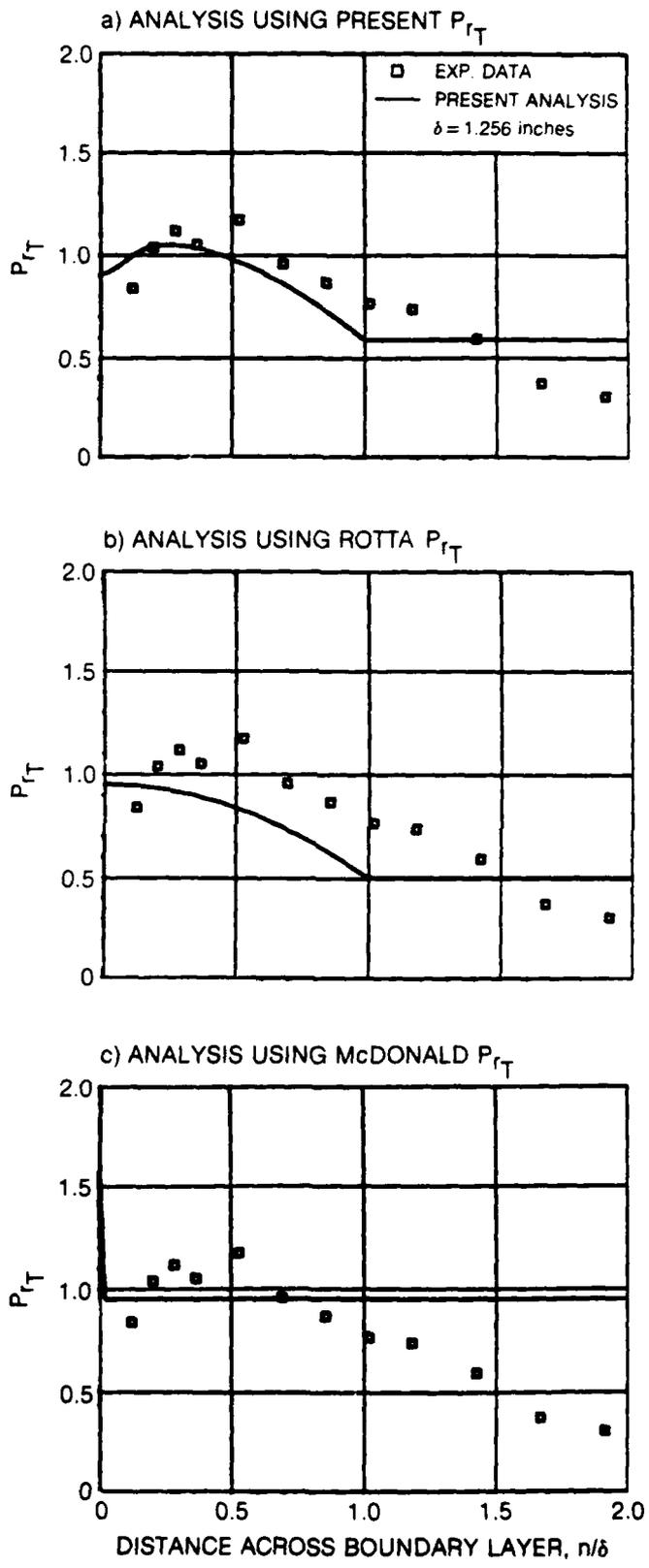


Figure 26b. Comparison of Theoretical Turbulent Prandtl Number with Experimental Data at $X = 68$ inches for 6% Inlet Turbulence Level

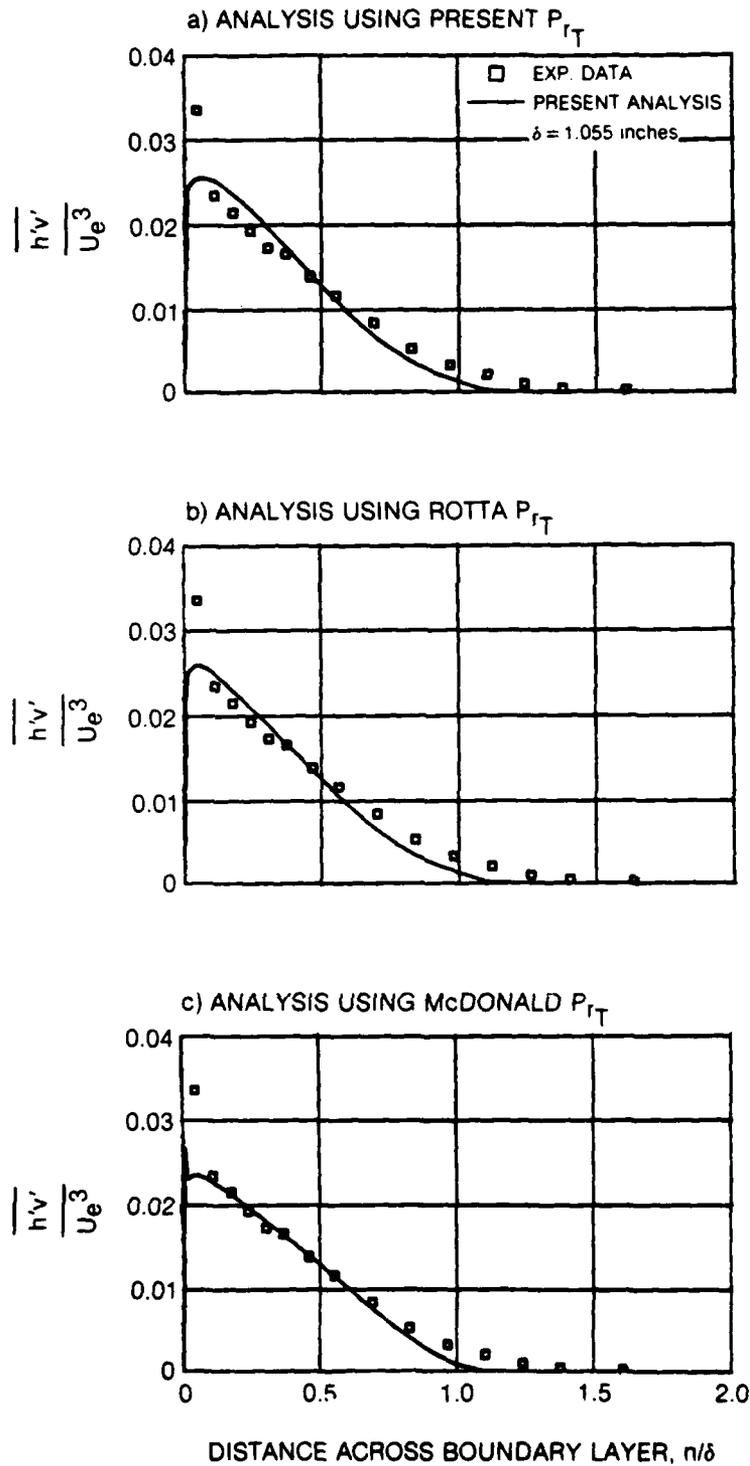


Figure 27a. Comparison of Theoretical Thermal Heat Flux with Experimental Data at $X = 68$ inches for a 2% Inlet Turbulence Level

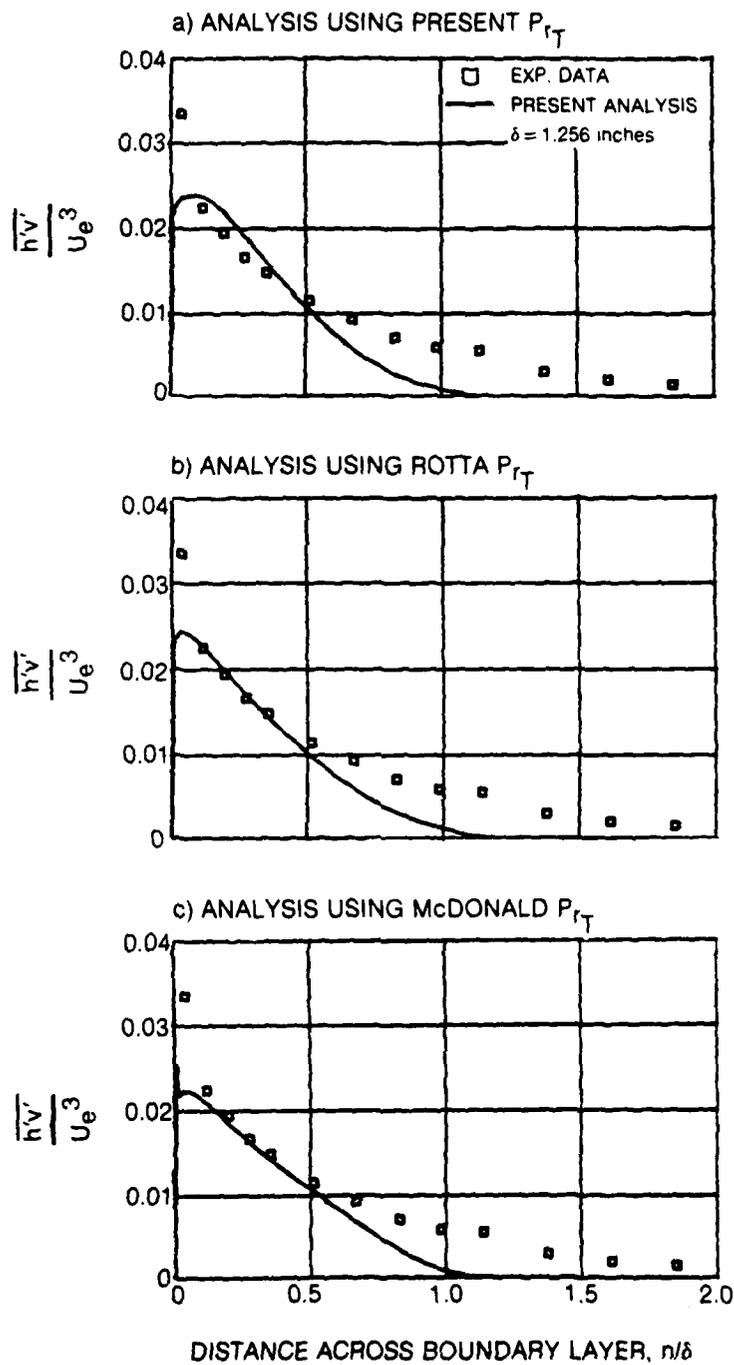


Figure 27b. Comparison of Theoretical Thermal Heat Flux with Experimental Data at $X = 68$ inches for a 6% Inlet Turbulence Level

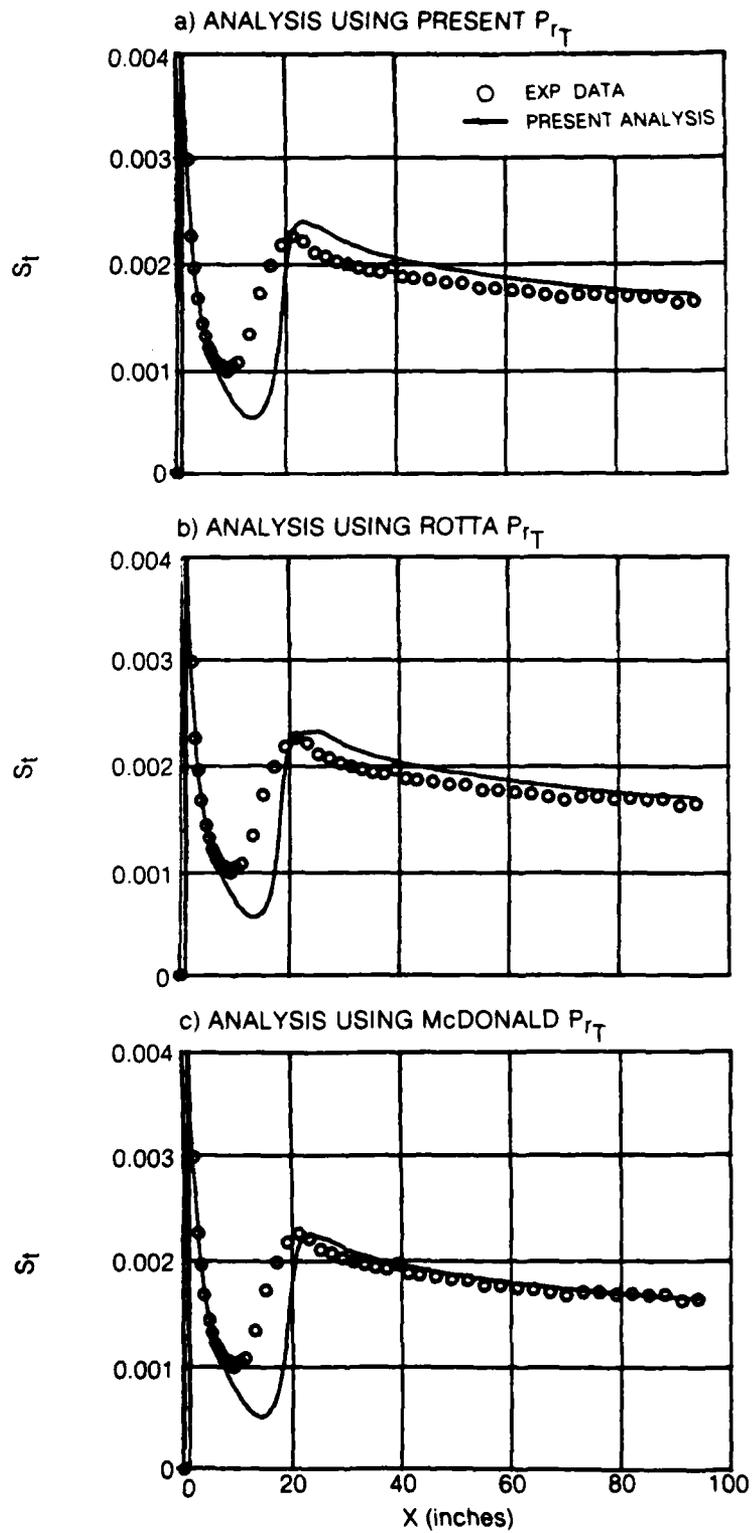


Figure 28a. Comparison of Theoretical Stanton Number with Experimental Data for 1% Inlet Turbulence Level

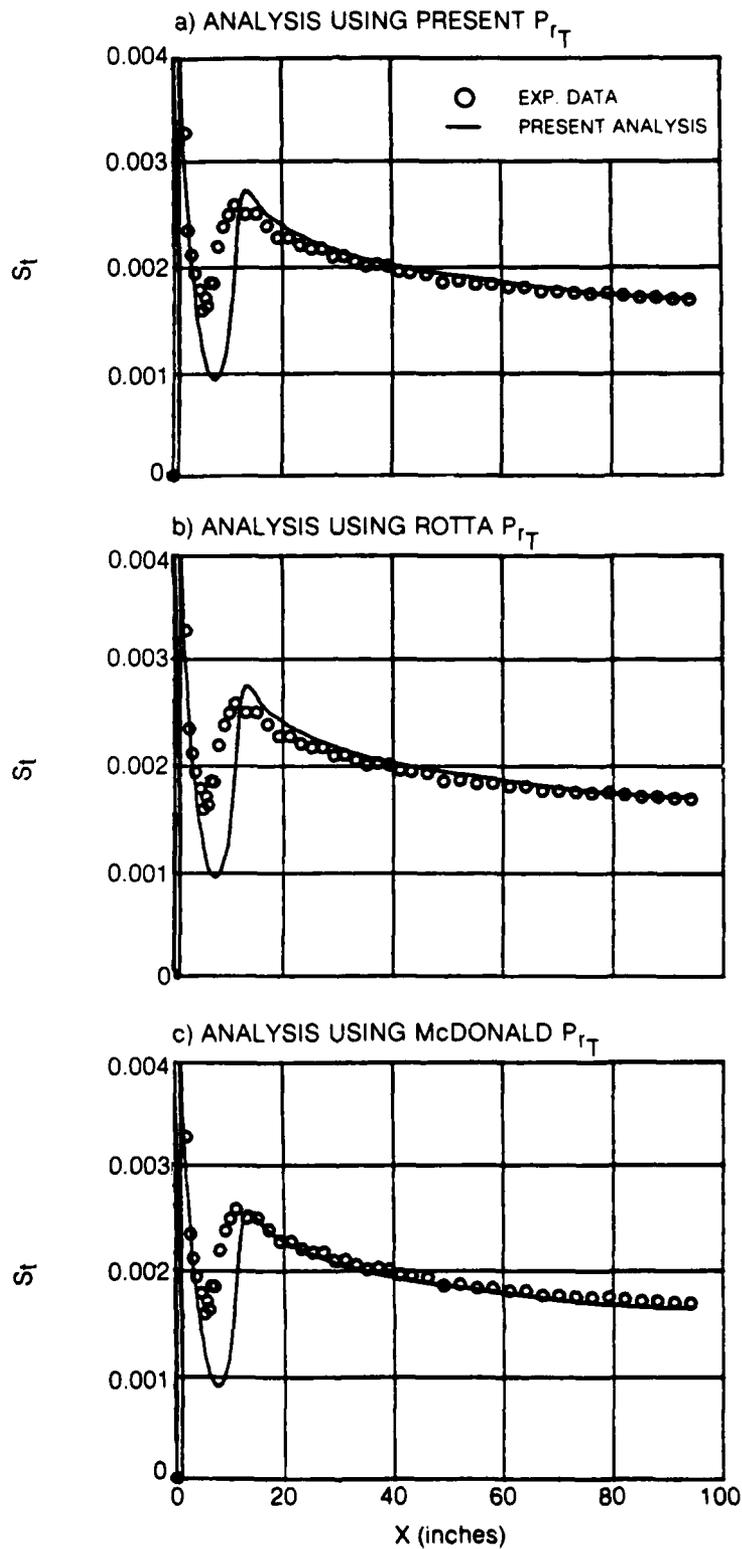


Figure 28b. Comparison of Theoretical Stanton Number with Experimental Data for 2% Inlet Turbulence Level

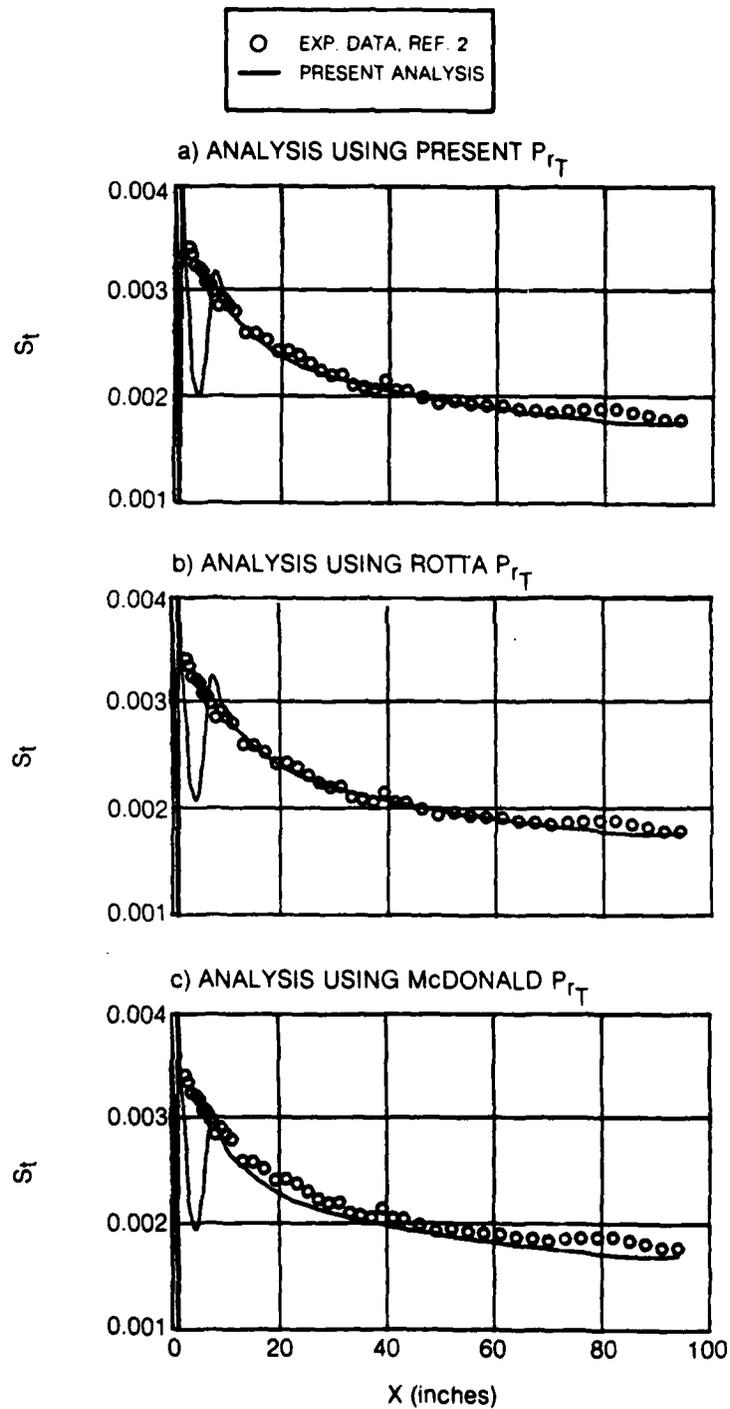


Figure 28c. Comparison of Theoretical Stanton Number with Experimental Data for 4% Inlet Turbulence Level

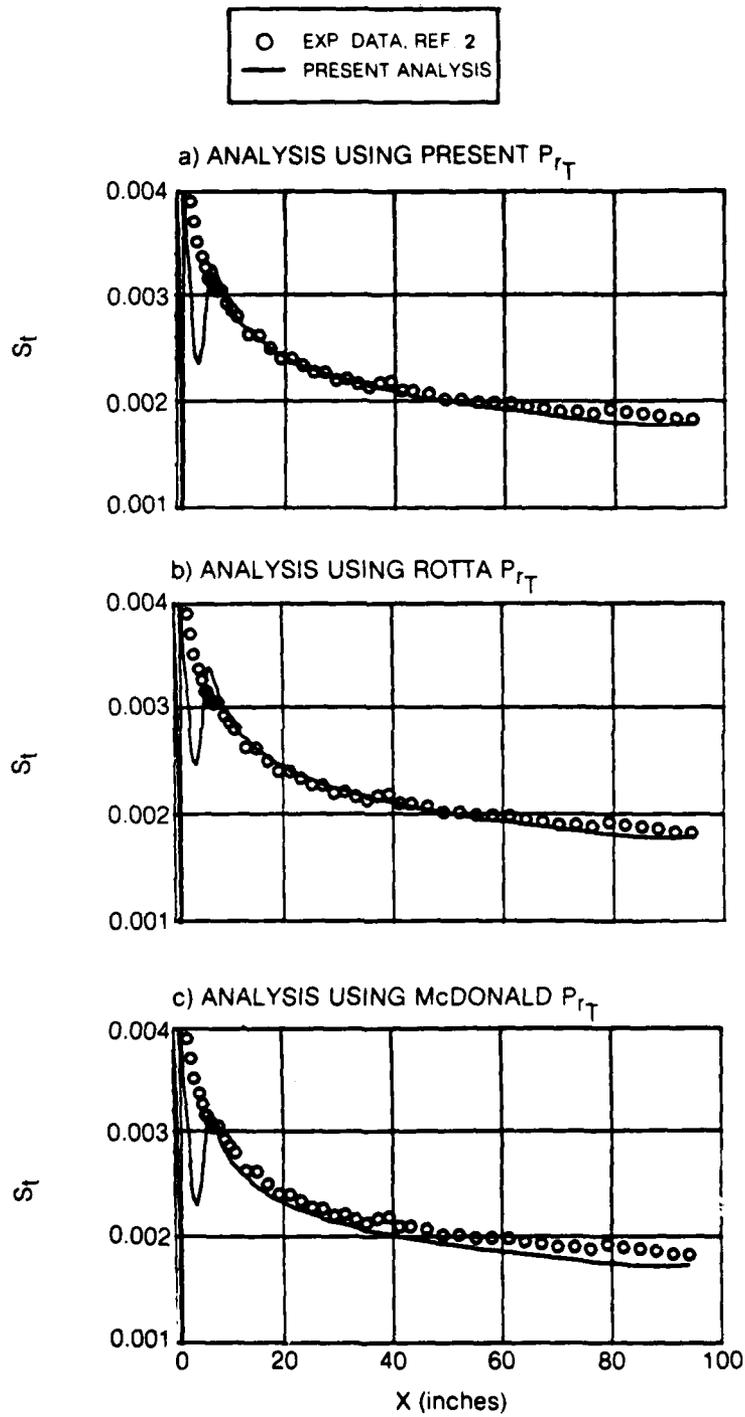


Figure 28d. Comparison of Theoretical Stanton Number with Experimental Data for 6% Inlet Turbulence Level

● ○ ▽ △	EXP. DATA, REF. 2
————	PRESENT ANALYSIS USING PRESENT P_{rT}
— · — · — ·	PRESENT ANALYSIS USING McDONALD P_{rT}

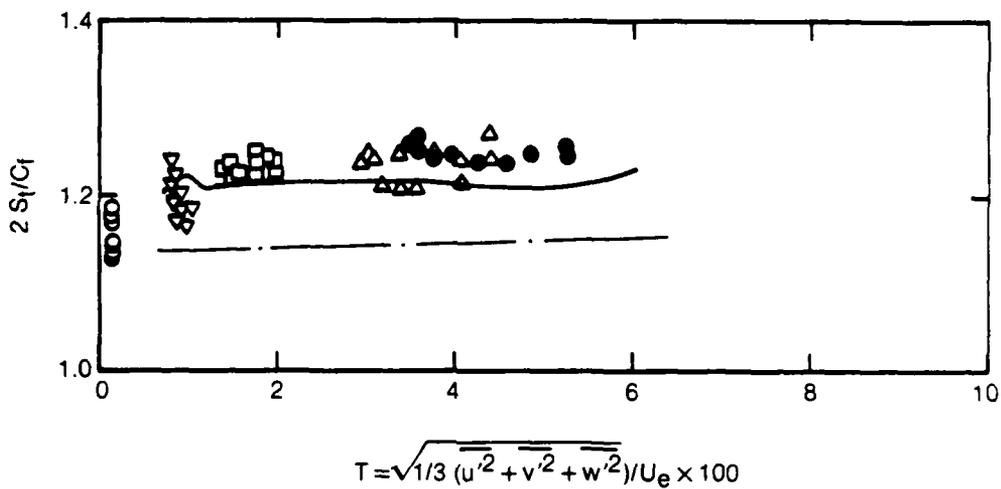


Figure 29. Theoretical Prediction of Reynolds Analogy Factor - Varying Edge Turbulence Levels

APPENDIX A - ERROR ANALYSIS

In this program mean velocity (U) profiles were determined by means of Pitot, single wire, x wire and 3 sensor hot wire probes. Mean temperatures (T) were measured by both 3 sensor hot wire and thermocouple probes. Fluctuating velocities and Reynolds stress distributions (u' , v' , w' , $u'v'$) were determined both with x wire and 3 wire techniques.

Assessment of absolute errors for the analog signals measured in this program would be a relatively straightforward matter. For example, the possible errors in measured pressures from the pitot probes or the recorded, digitized voltages from the hot-wire anemometer can be computed from the individual expected uncertainties. Computation of the absolute errors of the measured physical quantities (e.g., u' , v' , t'), however, is practically impossible because the true accuracy of factors such as Pitot probe wall proximity corrections, Pitot probe turbulence corrections, hot wire wall radiation effects, high turbulence sensor cross-talk, etc. are unknown. For this reason the uncertainties for the various quantities measured for this program will be assessed by (1) comparing the measured quantities with independently determined or computed results or (2) comparisons of like quantities measured using different probes and instrumentation techniques.

Mean velocity and temperature profile data obtained with different measurement techniques (four techniques for velocity, two techniques for temperature) are presented in Fig. A-1. These profile data were all obtained at $X = 84$ inches at three levels of free-stream turbulence. In-depth descriptions of the Pitot and thermocouple probes and data systems used for the mean velocity and temperature profiles are provided in Ref. 2. These probes were designed and constructed specifically for these types of boundary layer flows and a number of well established near-wall correction terms (see Ref. 2) were applied to the data. In addition, a number of comparison checks (see Refs. 1 and 2) showed that these mean profile data were very accurate. For these reasons the mean velocity profiles from the Pitot probe and the mean temperature profiles from the thermocouple probe were selected as the "true" respective profiles. For the data of Fig. A-1 all the other velocity profile measurements were compared to the Pitot probe data while the triple sensor temperature data were compared to the temperature profiles from the thermocouple probe.

An examination of the mean velocity profile data of Fig. A-1 indicates that 85 percent of all the measurements fell within ± 3 percent of the "true" Pitot profile. Only one set of data, the vertical x probe results for $Te = 0.2$ percent, had any discrepancies larger than 5 percent. Discrepancies in the mean temperature profile measurements were slightly larger than those for the velocity measurements with only 70 percent of the measurements falling within ± 5 percent of the thermocouple probe profile. The 3 sensor probe temperature data also showed a clear bias to read slightly lower temperatures than the thermocouple probe. The relatively larger errors for the temperature profiles as compared to the velocity profiles is not surprising. Because of the relatively large size of the 3 sensor probes they span a gradient of both velocity and temperature in the boundary layer flow. The data reduction system

is forced to assume that a single effective velocity and flow temperature apply over the entire probe.

Plots of the average bias $\overline{(x_i - \bar{x})}$ and standard error $\sqrt{\overline{(x_i - \bar{x})^2}}$ (x_i is the measured quantity and \bar{x} is the "true" quantity at a given profile location) for the various profiles are given in Fig. A-2. These results are plotted as a function of the ratio between the overall boundary layer thickness and the probe sensor height (h). The overall profile errors are plotted in Fig. A-2, were largest for the thinnest boundary layers (with the relatively steepest velocity and temperature gradients). In addition, the largest local bias errors of Fig. A-1 were located near the wall where the steepest gradients exist. This result has led to the conclusion that the local gradients across the sensor arrays were a significant cause of the discrepancies between the x-type probe mean data and the "true" profiles.

Distributions of the various measurements of the streamwise and vertical velocity fluctuations and the Reynolds stress distributions are given in Figs. A-3 and A-4. No "true" or best distributions of these quantities are known for these profiles and so the data at a given location in a profile were compared with the average of all the like data taken at that location. The plots of Figs. A-3 and A-4, then, show distributions of the inconsistencies between the various measurement techniques for the various profiles. The agreement between the separate measurements for u'/U_e and v'/U_e were generally very good ($\pm \approx 1/2$ percent for u'/U_e and $\pm \approx 1/5$ percent for v'/U_e). If the average levels of u'/U_e and v'/U_e are approximated as 0.06 and 0.04 respectively the above inconsistencies are equivalent to ± 8 percent and ± 5 percent uncertainties in the fluctuating velocities (u' and v') themselves. Note that for the u'/U_e plot of Fig. A-3 data were included for a "single horizontal wire". These particular data were obtained with an analog data system consisting of a polynomial linearizer and a true RMS voltmeter. All the other data of Fig. A-3 were obtained with the previously described analog-digital system. The consistency between the measurements for these two very different techniques is excellent. The largest inconsistencies were observed for the Reynolds stress distributions of Fig. A-4. This is not at all surprising as it is far more difficult to measure correlated than single fluctuating quantities. For these Reynolds stress data a ± 15 percent inconsistency band encompasses nearly all the measurements.

Plots of the average "bias" and "standard error" for the fluctuating quantities of Fig. II-3 and II-4 are presented in Fig. A-5. The overall profile errors for the stress measurements were clearly much larger than the errors for the individual velocity fluctuations. Unlike the mean profile results of Fig. A-2 the uncertainties of the fluctuating quantities for the various boundary layer thicknesses were nearly constant.

TYPE PROBE	SINGLE HORIZ	TRI-X	X-H	X-V	T_e %		
	○	□	▽	△	02	14	39
SYMBOL	○	□	▽	△	OPEN	HALF FILLED	SOLID

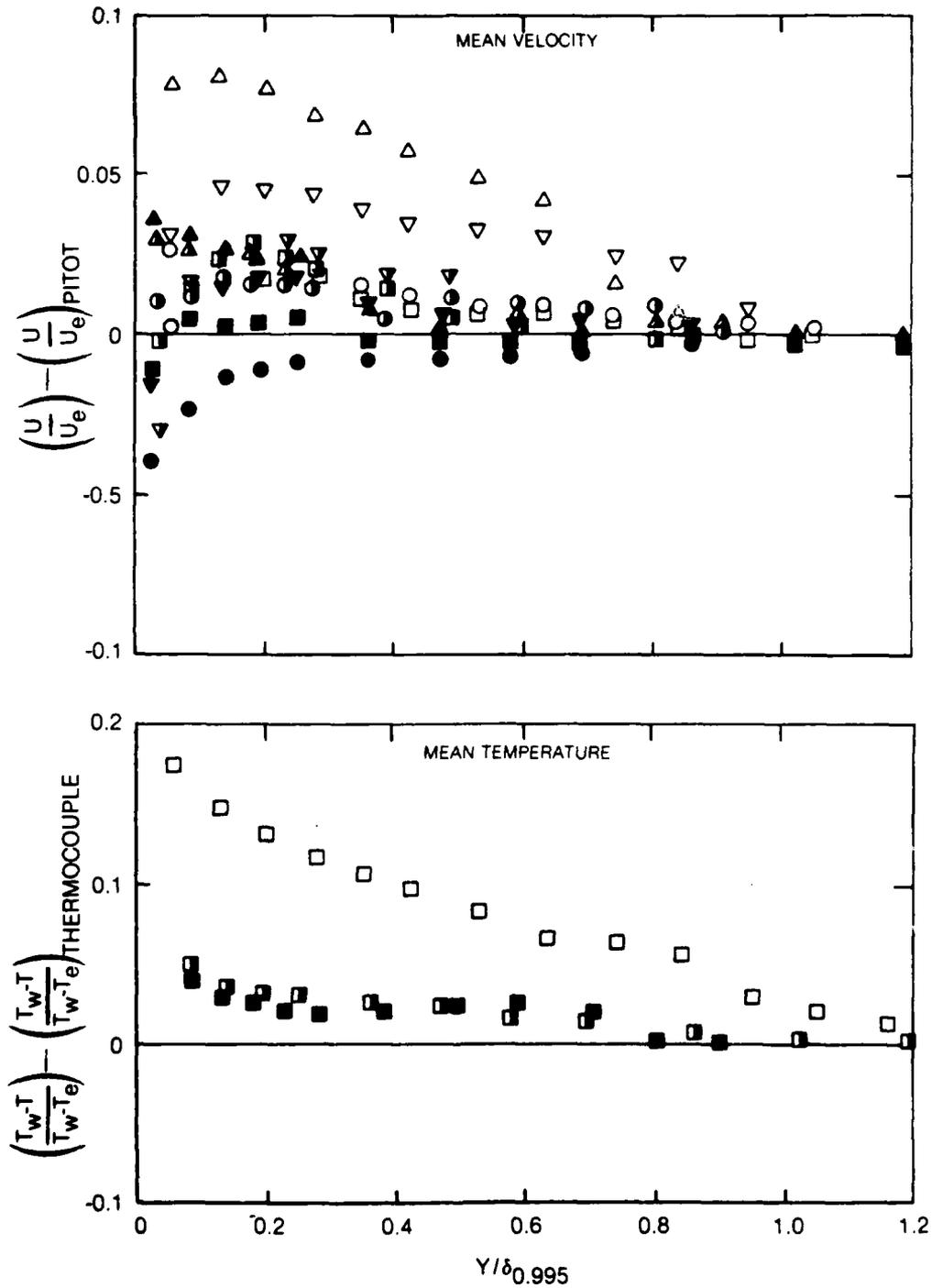


Figure A-1. Discrepancies Between Various Local Measurements of Mean Velocity and Temperature Profile Data

AD-A127 799

THE EFFECTS OF FREE-STREAM TURBULENCE ON THE TURBULENCE
STRUCTURE AND HEAT FLUXES (U) UNITED TECHNOLOGIES RESEARCH
CENTER EAST HARTFORD CT M F BLAIR ET AL. NOV 82

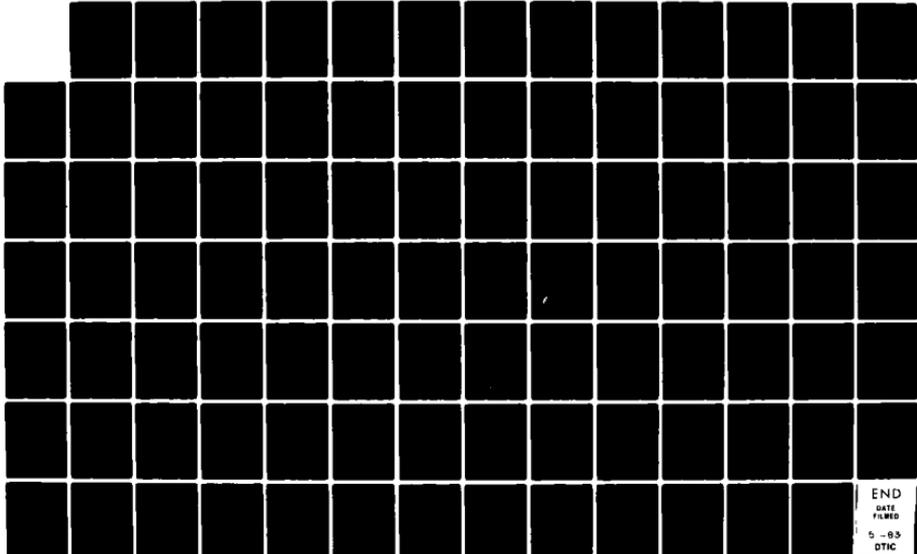
2/2

UNCLASSIFIED

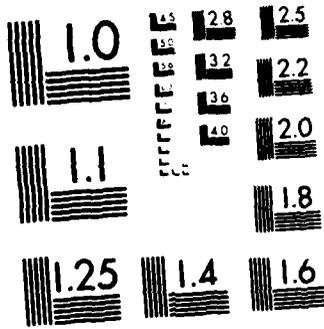
UTRC/R82-915634-2 AFOSR-TR-83-0355

F/G 20/4

NL



END
DATE
FILMED
5 - 83
DTIC



MICROCOPY RESOLUTION TEST CHART
NATIONAL BUREAU OF STANDARDS 1963-A

MEASURED QUANTITY	U/U _e			T _w -T/T _w -T _e
	TRI-X	X-H	X-V	TRI-X
SYMBOL	□	▽	△	◇

T _e %	0.2	1.4	3.9
SYMBOLS	OPEN	HALF FILLED	SOLID

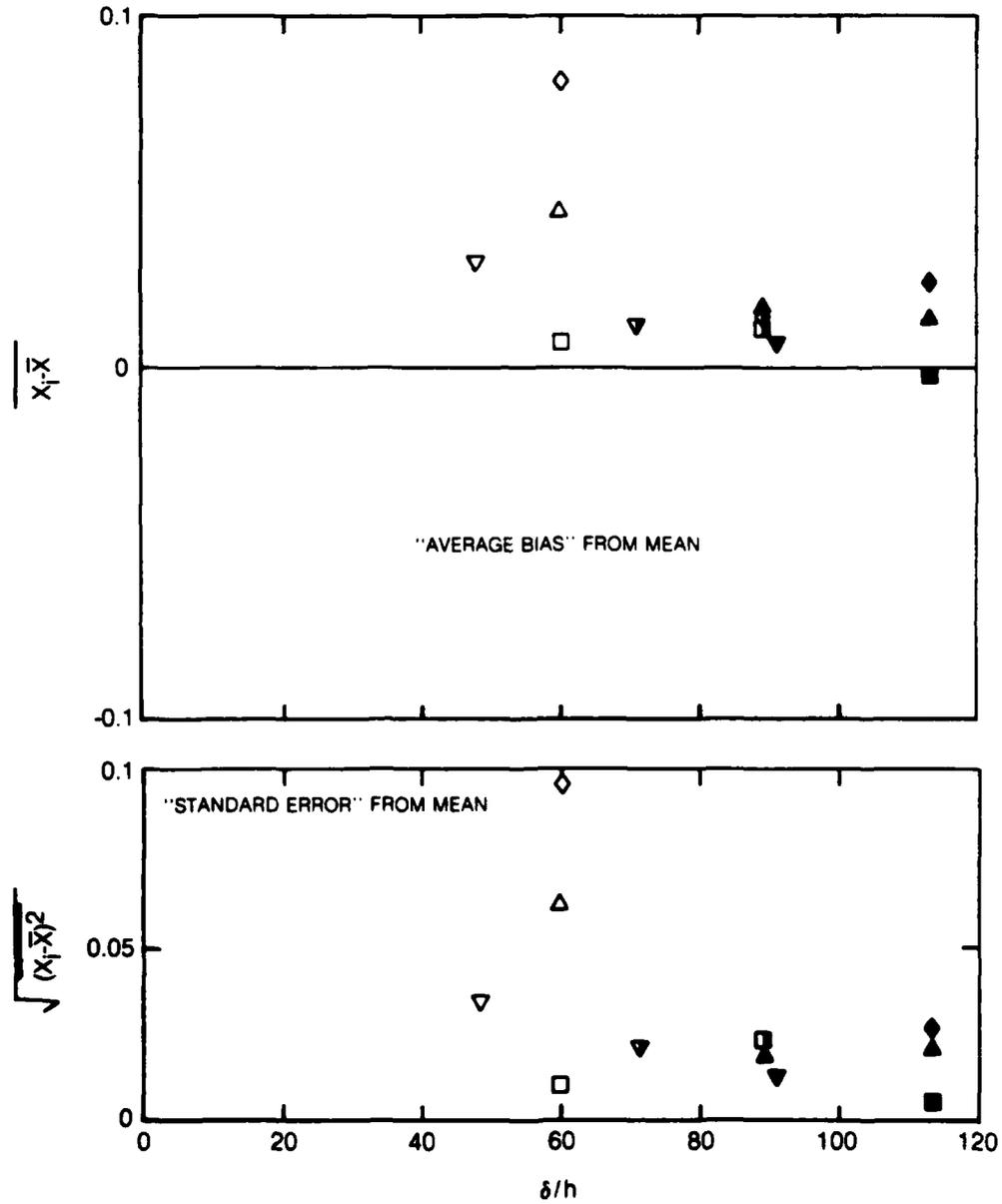


Figure A-2. Overall Discrepancies Between Different Mean Measurement Techniques for Various Profiles

TYPE PROBE	SINGLE HORIZ	TRI-X	X-H	X-V ₁	X-V ₂
SYMBOL	○	□	▽	△	▲

T _e %	0.2	1.4	3.9
SYMBOL	OPEN	HALF FILLED	SOLID

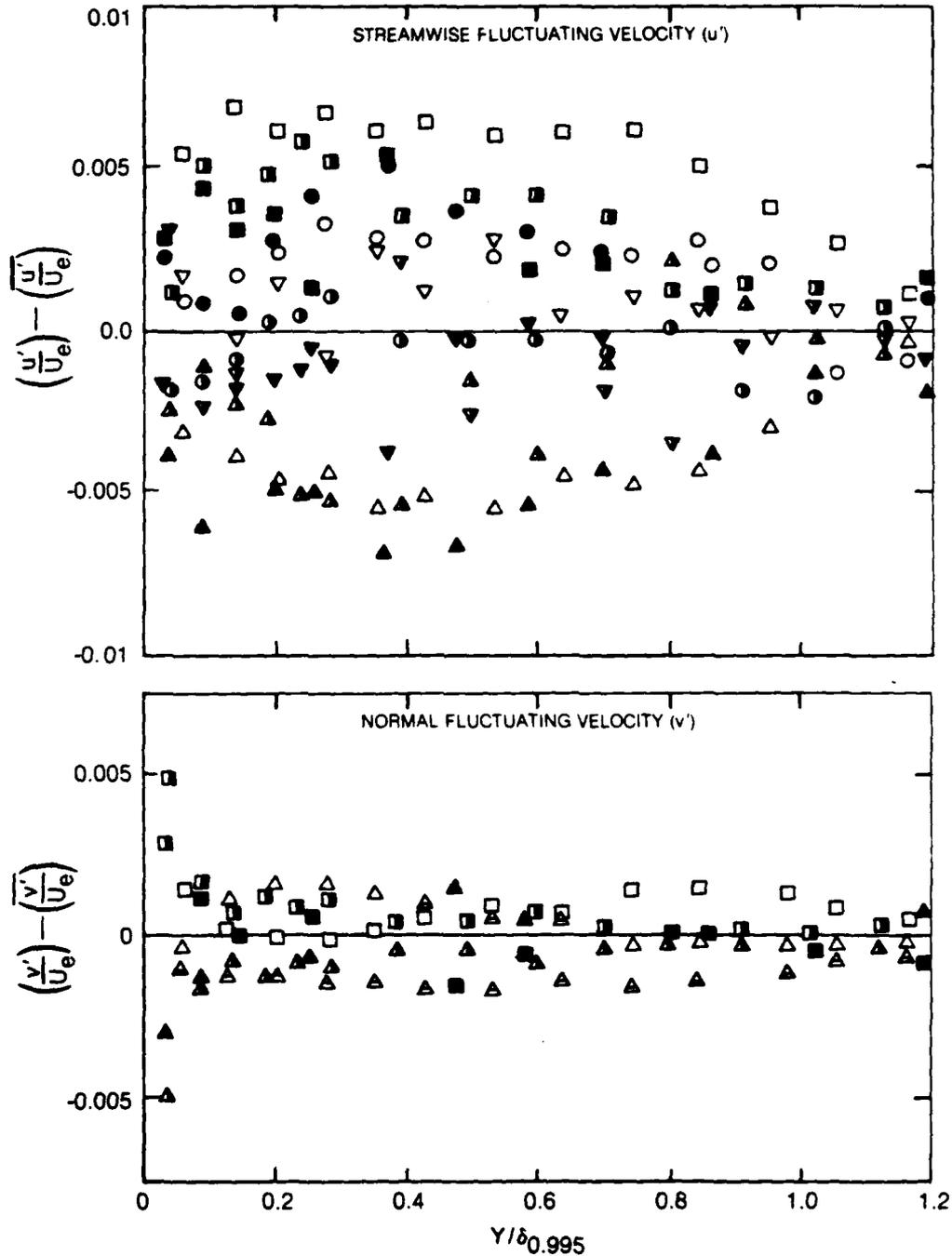


Figure A-3. Discrepancies Between Various Local Measurements of Fluctuating Velocity Components

TYPE PROBE	TR-X	X-V ₁	X-V ₂
SYMBOL	○	△	▲

T _e %	0.2	0.14	3.9
SYMBOL	OPEN	HALF FILLED	SOLID

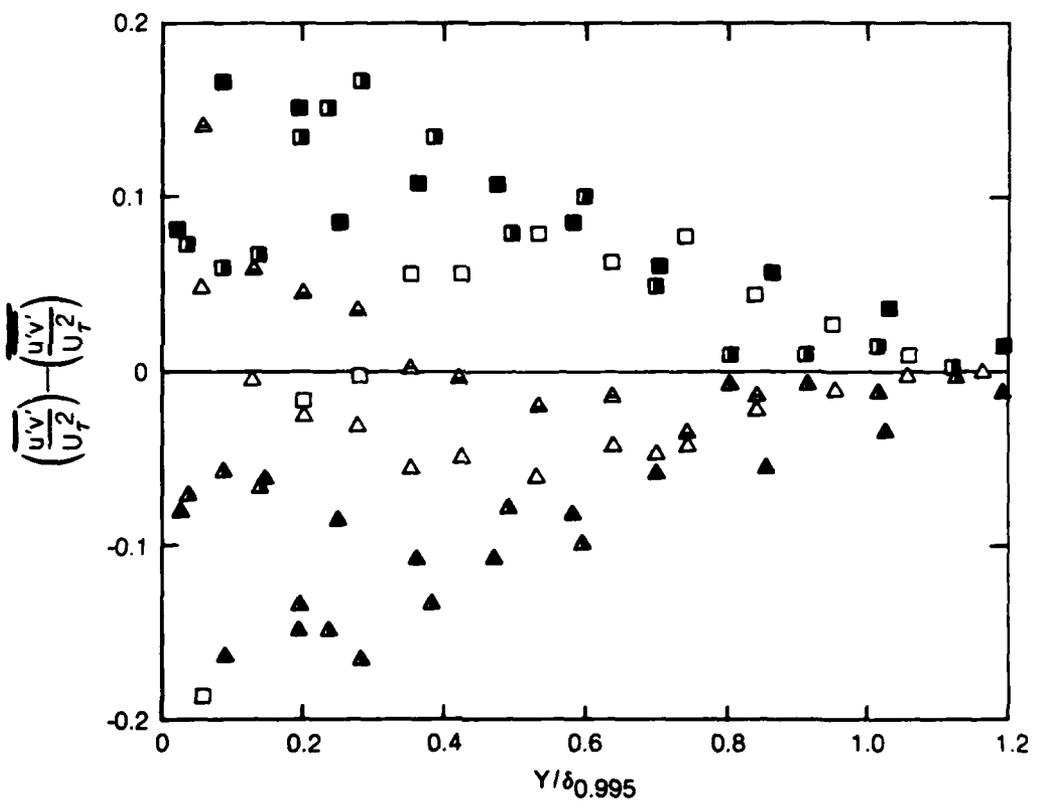


Figure A-4. Discrepancies Between Various Local Turbulent Shear Stress Measurements

MEASURED QUANTITY	u'/U_e			v'/U_e		$\overline{u'v'}/U_e^2$	
	TRI-X	X-H	X-V	TRI-X	X-V	TRI-X	
TYPE PROBE	TRI-X	X-H	X-V	TRI-X	X-V	TRI-X	
SYMBOL	□	▽	△	◇	▽	▽	○

T_e %	0.2	1.4	3.9
SYMBOLS	OPEN	HALF FILLED	SOLID

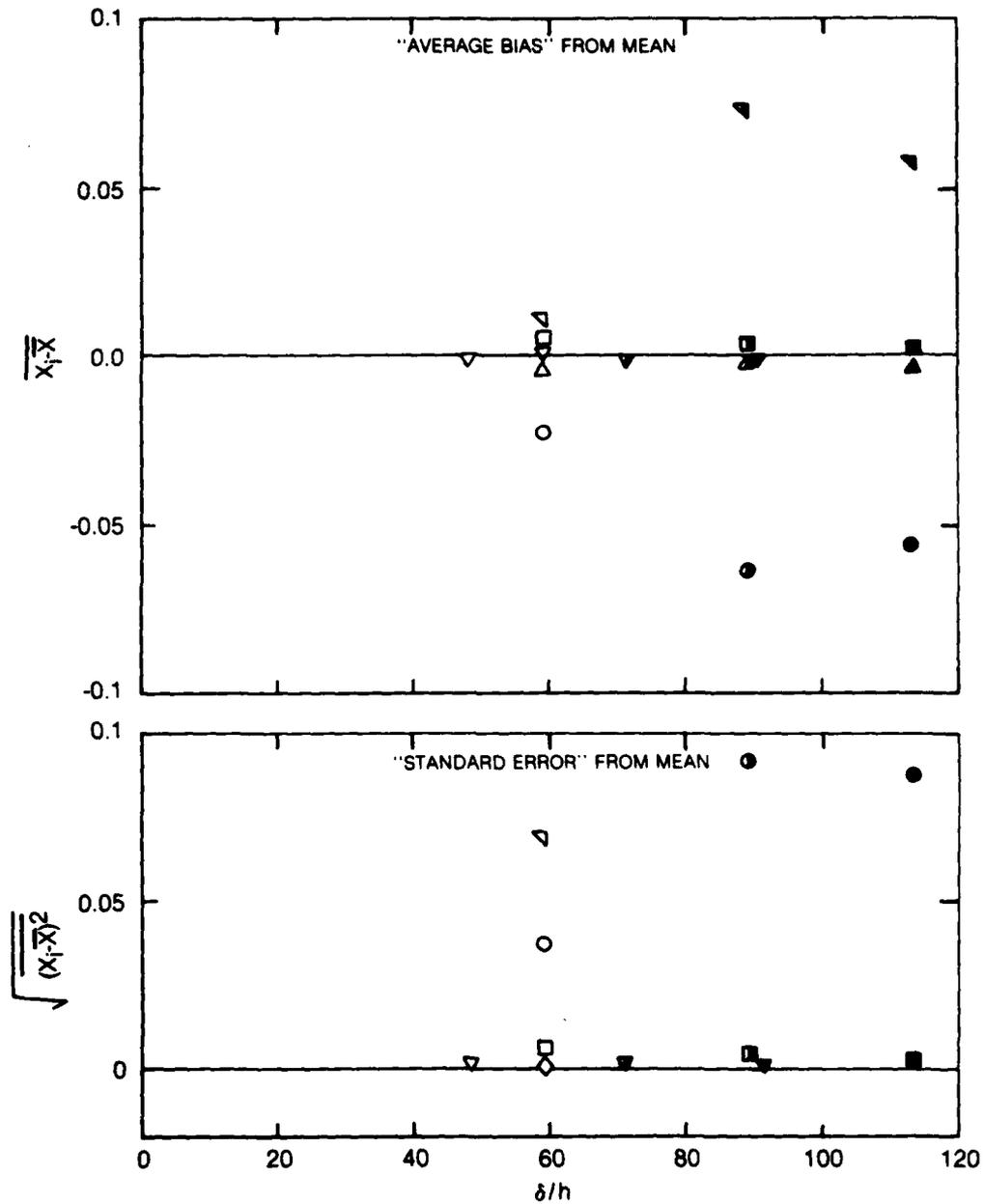


Figure A-5. Overall Discrepancies Between Different Fluctuating Measurement Techniques for Various Profiles

APPENDIX B - EXPERIMENTAL PROFILE DATA

All data for each profile are grouped together as follows:

- a) mean profile data
- b) mean profile data tabulation
- c) boundary layer property tabulation
- d) fluctuating profile plots (A-E)
- e) fluctuating profile data tabulation (A-B).

The profile data are presented in the following order:

	<u>X</u>	<u>Te%</u>
1	52	0.2
2	68	0.2
3	84	0.2
4	52	1.8
5	68	1.6
6	84	1.4
7	52	4.7
8	68	4.2
9	84	3.9

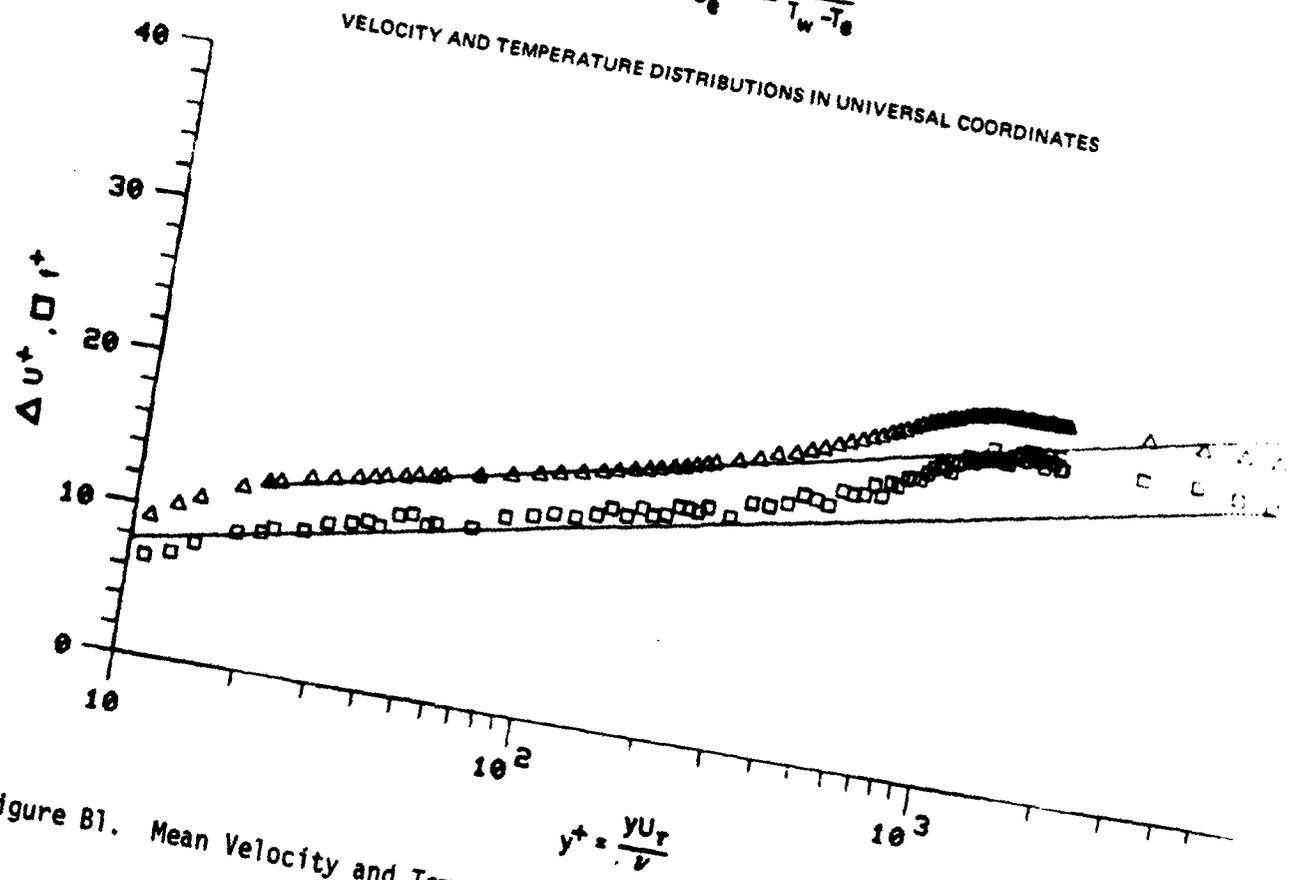
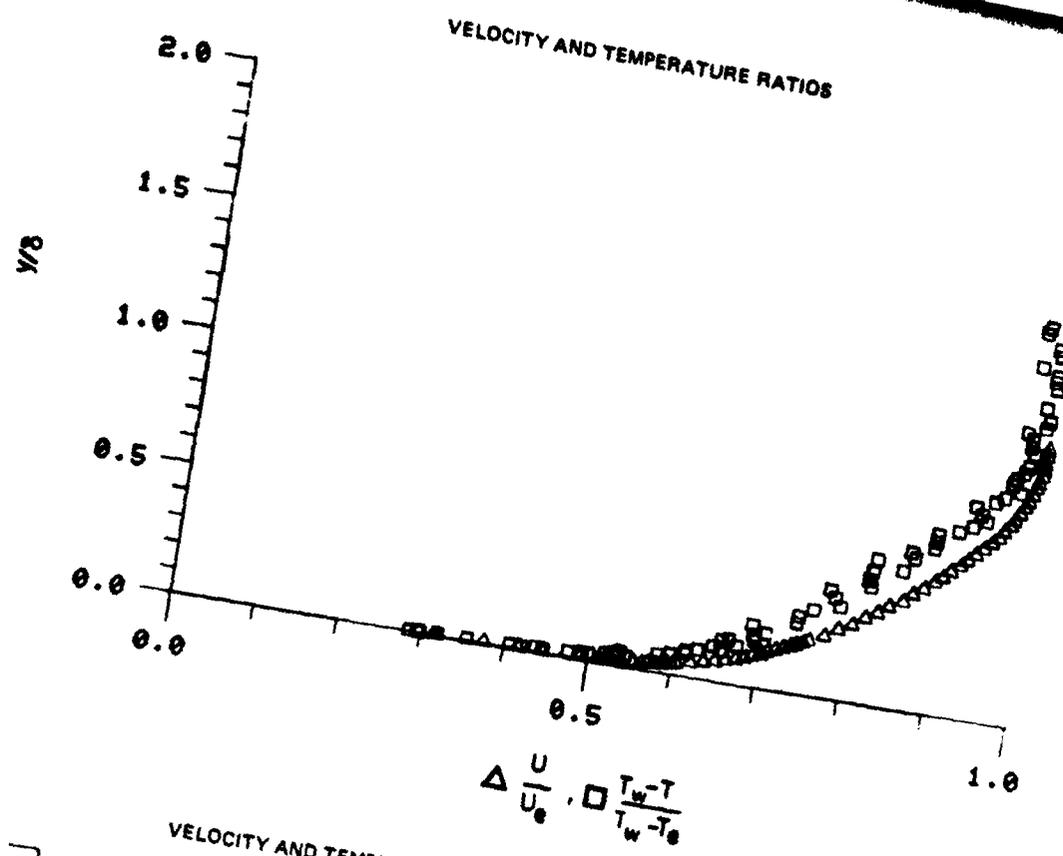


Figure B1. Mean Velocity and Temperature Profiles $x = 52$ in., $Te = 0.2\%$

RUN NO. 3. POINT 3.

BOLNDARY LAYER PROPERTIES

	LINEAR INTERPOLATION TO WALL	STANDARD SUBLAYER FUNCTION FROM WALL TO $Y^+=35$
FREE STREAM VELOCITY	98.922	98.922
FREE STREAM TEMPERATURE	71.150	
WALL TEMPERATURE	87.410	
WALL HEAT FLUX	.05120	
FREE STREAM DENSITY	.07523	
FREE STREAM KINEMATIC VISCOSITY	.0001628	
DENSITY OF FLUID AT WALL	.07300	
KINEMATIC VISCOSITY OF FLUID AT WALL	.0001717	
WALL/FREE STREAM DENSITY RATIO	.97028	
LOCATION REYNOLDS NUMBER (REX)	2633499.75	
INPUT VALUE OF VELOCITY DELTA	.61000	
INPLT VALUE OF TEMPERATURE DELTA	.67000	
CALCULATED DELTA		.60504
DELTA 99.5% INPUT	.00000	
DISPLACEMENT THICKNESS (DELSTAR)	.09026	.09063
MOMENTUM THICKNESS (THETA)	.06280	.06280
ENERGY-DISSIPATION THICKNESS	.11090	.11078
ENTHALPY THICKNESS	.00274	.00273
SHAPE FACTOR 12 (DELSTAR/THETA)	1.43730	1.44315
SHAPE FACTOR 32 (ENERGY/THETA)	1.76594	1.76410
MOMENTUM THICKNESS REYNOLDS NUMBER	3180.47	3180.34
DISPLACEMENT THICKNESS PEYNOLDS NUMBER	4571.29	4589.72
SKIN FRICTION COEFFICIENT	.003195	
FRICTION VELOCITY	4.01412	
LAW OF THE WALL CONSTANT (K)	.41000	
LAW OF THE WALL CONSTANT (C)	5.00000	
WAKE STRENGTH		.49082
CLAUSEPS 'DELTA' INTEGRAL	-2.07555	-2.16672
CLAUSEPS 'G' INTEGRAL	14.68922	14.91446
DISPLACEMENT THICKNESS - CONSTANT DENSITY	.08589	.08792
MOMENTUM THICKNESS - CONSTANT DENSITY	.06337	.06336
SHAPE FACTOR 12 - CONSTANT DENSITY	1.35544	1.36758

LOCATION -X- 52.00000

Te = 0.2%

Table B2

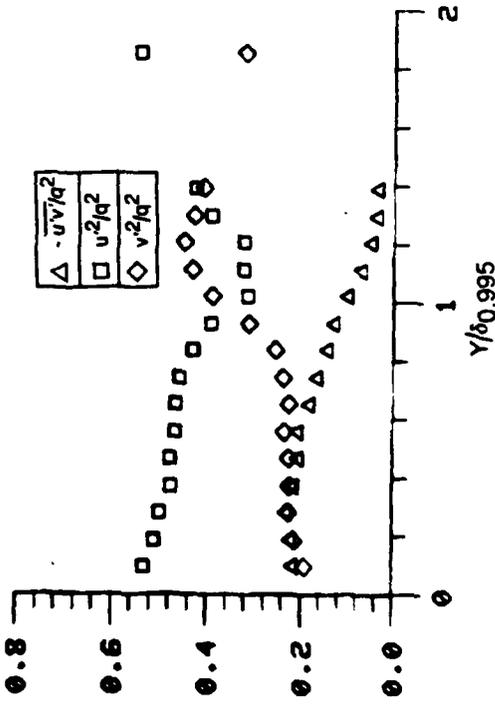
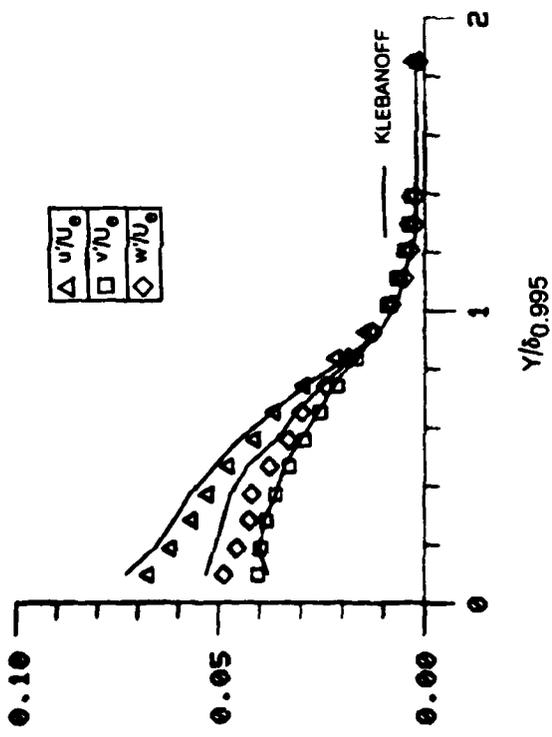
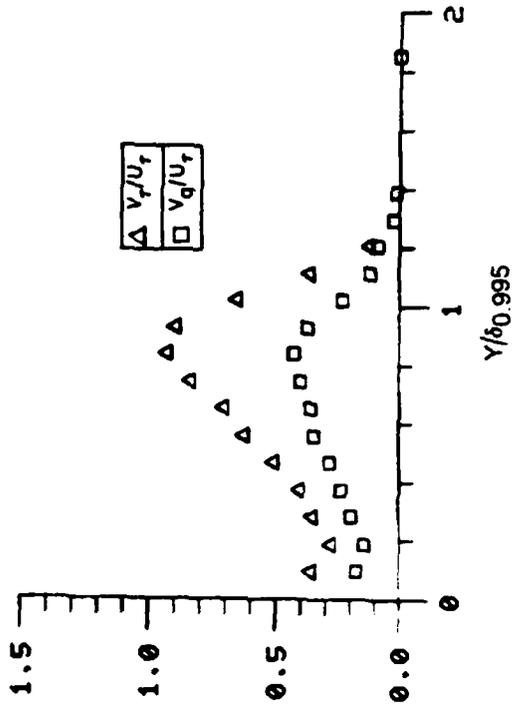
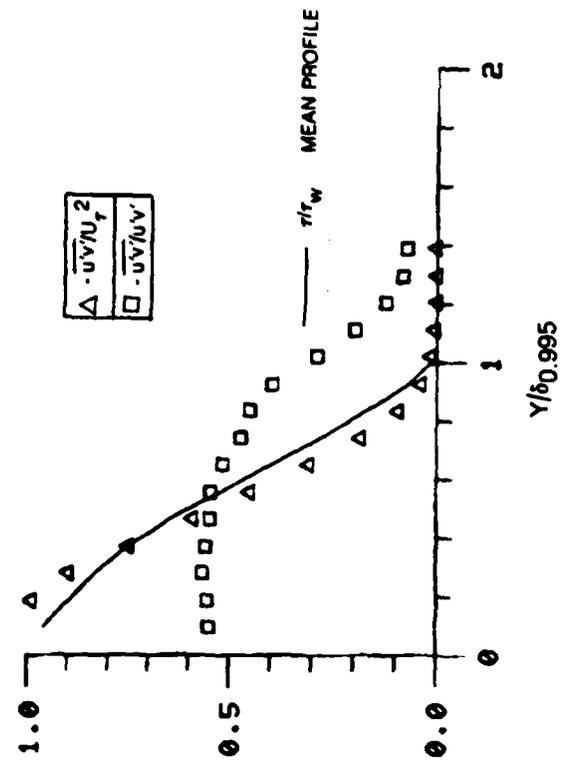


Figure B2A. Boundary Layer Turbulence Quantities $x = 52$ in, $T_e = 0.2\%$

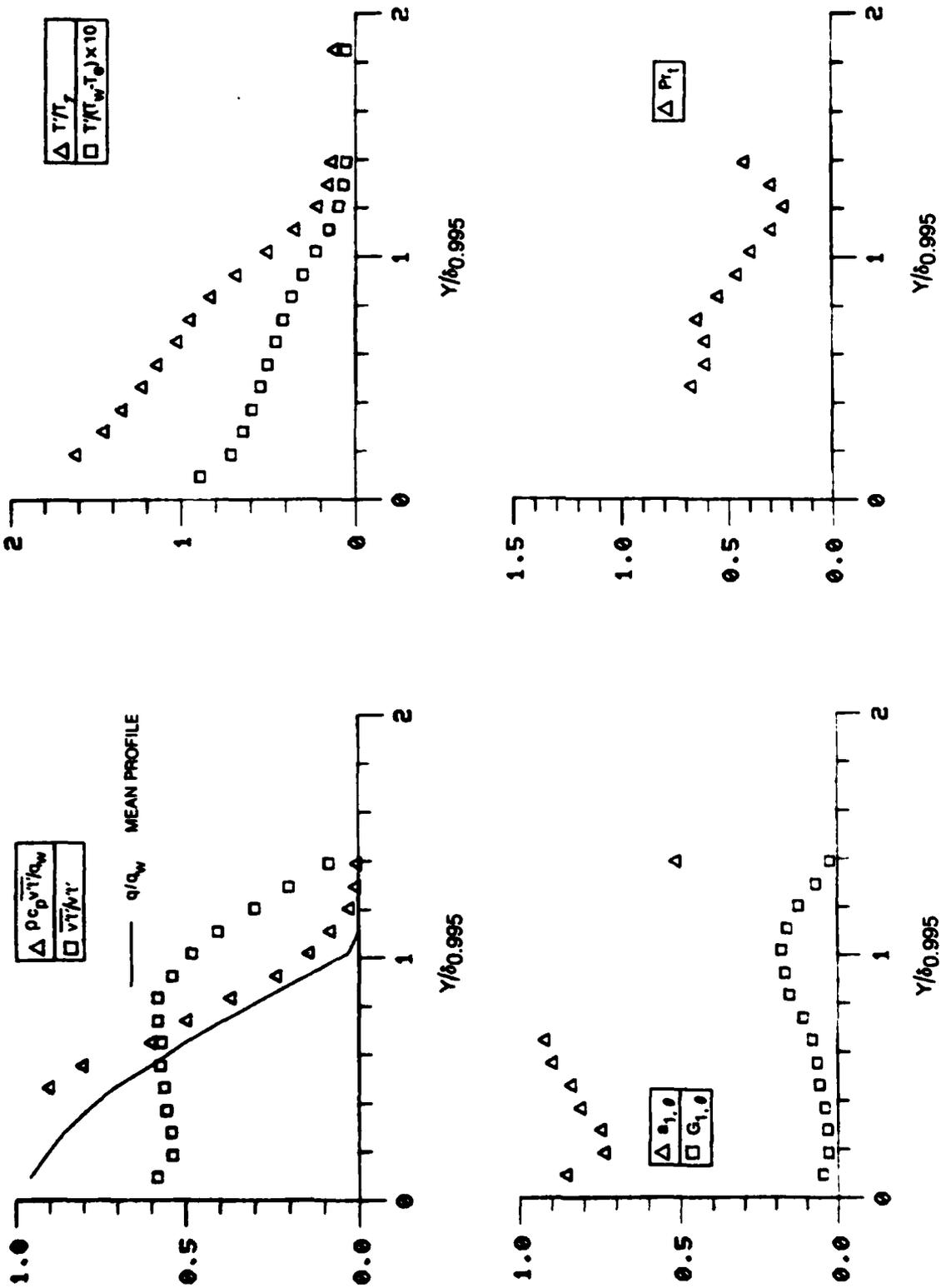


Figure B2B. Boundary Layer Turbulence Quantities, $x = 52$ in, $T_e = 0.2\%$

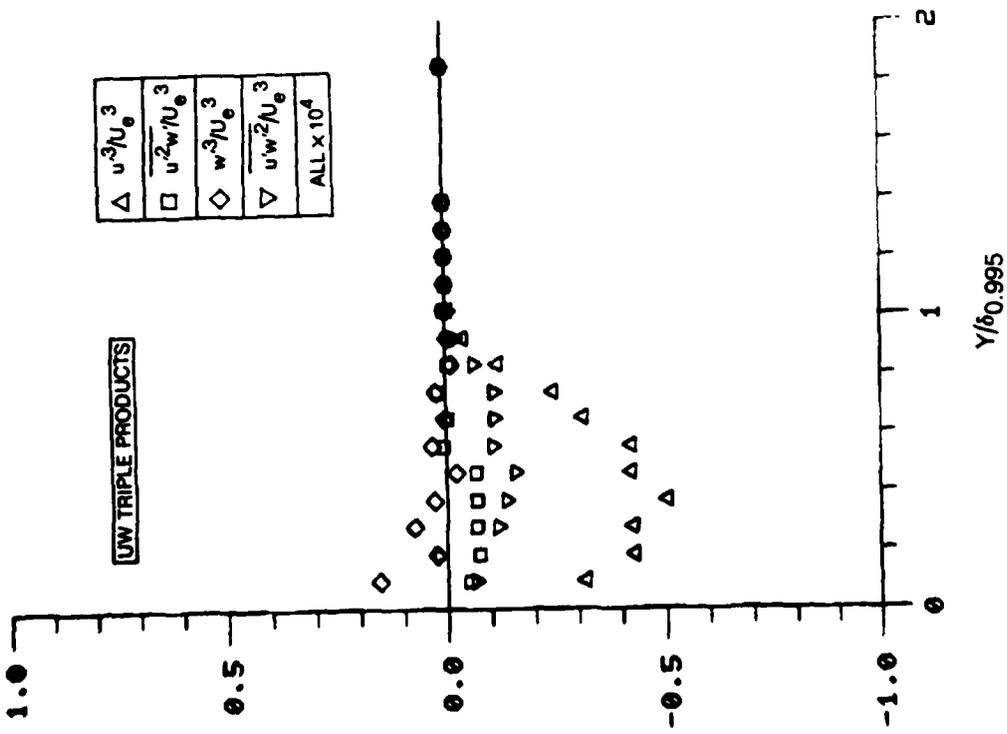
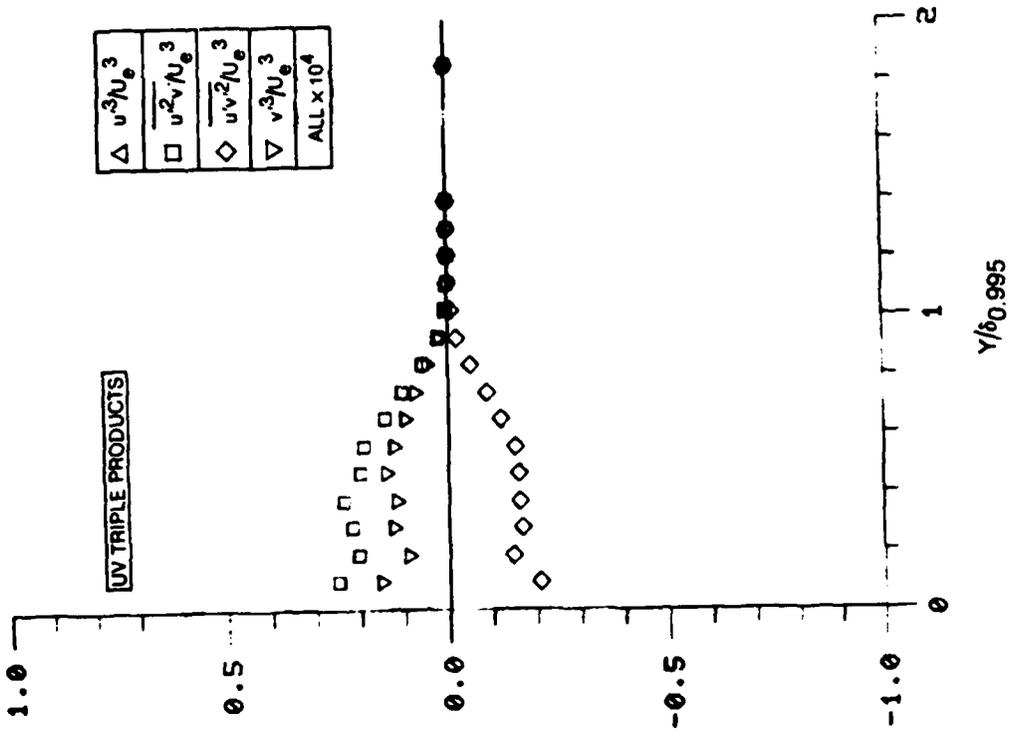


Figure B2C. Boundary Layer Triple Product Distributions $x = 52$ in, $Te = 0.2\%$

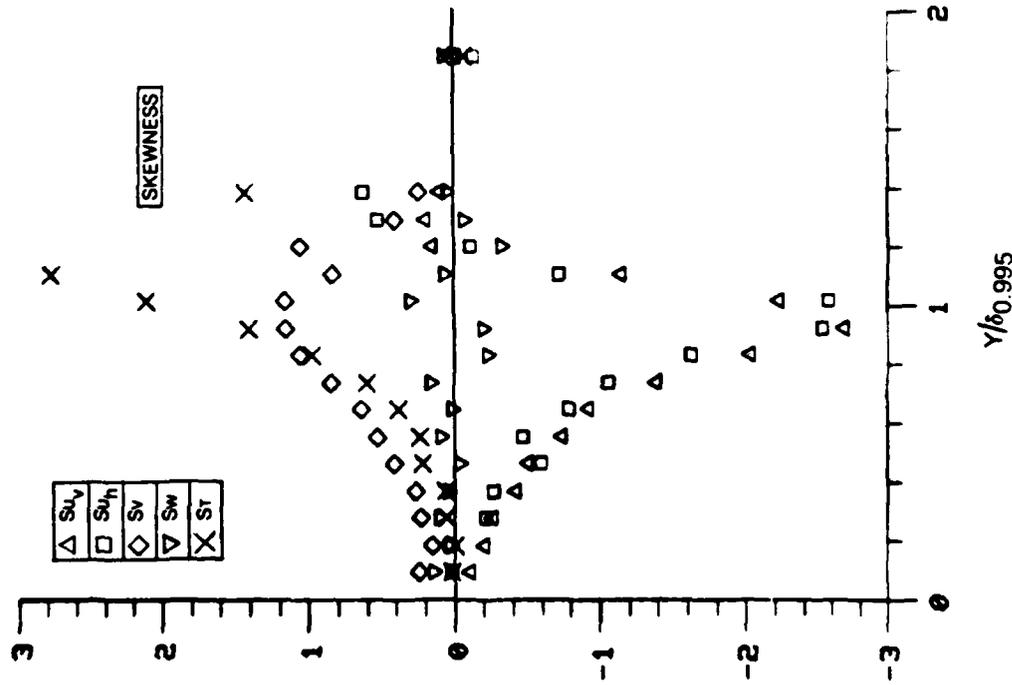
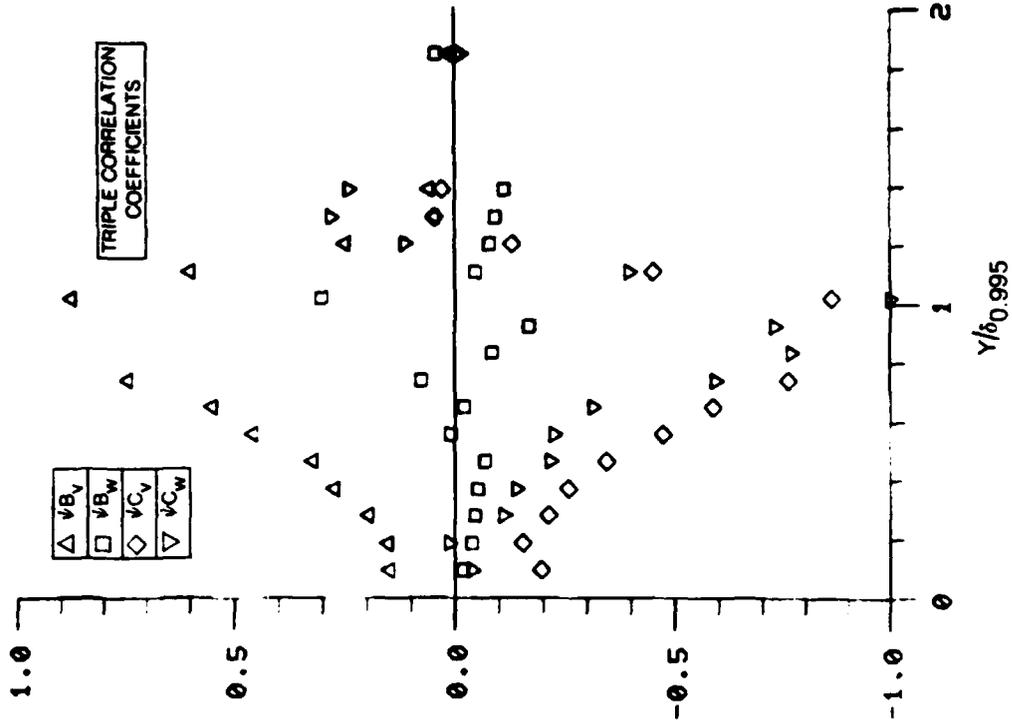


Figure B2D. Boundary Layer Skewness and Triple Product Correlation Coefficient Distributions $x = 52$ in, $T_e = 0.2\%$

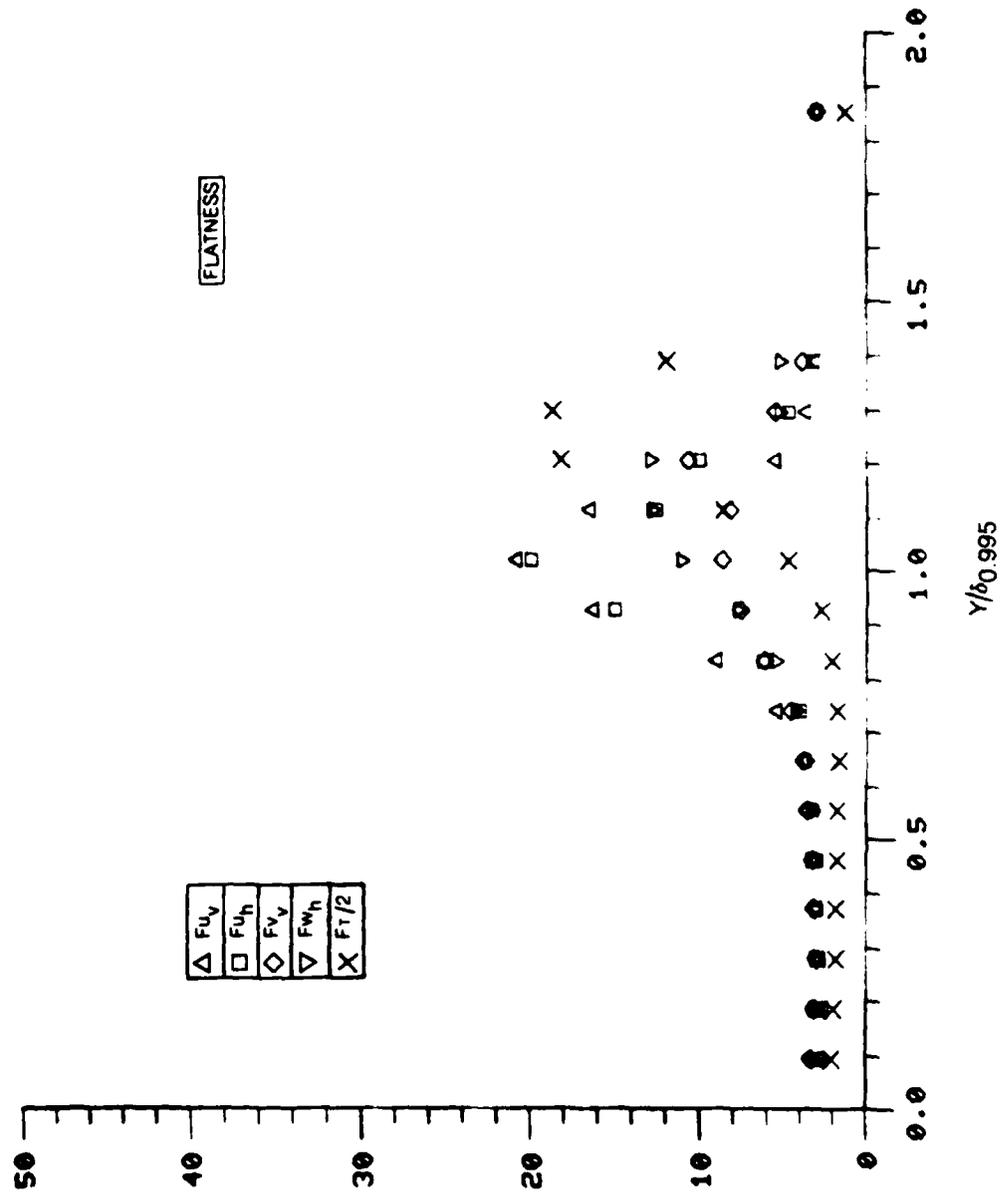


Figure B2E. Boundary Layer Flatness Distributions $x = 52$ in, $T_e = 0.2\%$

Fluctuating Profile Data

x = 52 in, Te = 0.2%

N	Y: INCHES	Y/ DELTA	U'/UE	V'/UE	W'/UE	$\sqrt{U'V'}/UE$	$\frac{U'V'}{U^2AU^2}$	$\frac{U'V'}{U'V'}$	C/UE
1	3.000	5.556	.0011	.0016	.0000	.0002	-.0000	-.0101	.0000
2	.050	.093	.0672	.0003	.0466	.0427	1.1054	.5476	.0000
3	.100	.185	.0617	.0000	.0456	.0404	.9912	.5565	.0000
4	.150	.278	.0567	.0000	.0423	.0385	.8960	.5656	.0000
5	.200	.370	.0523	.0000	.0419	.0381	.7479	.5592	.0000
6	.250	.463	.0476	.0000	.0377	.0312	.5861	.5496	.0000
7	.300	.556	.0409	.0000	.0329	.0272	.4455	.5451	.0000
8	.350	.648	.0360	.0000	.0251	.0225	.3068	.5171	.0000
9	.400	.741	.0291	.0000	.0210	.0174	.1837	.4712	.0000
10	.450	.833	.0214	.0000	.0184	.0124	.0931	.4525	.0000
11	.500	.926	.0141	.0000	.0123	.0086	.0391	.3967	.0000
12	.550	1.019	.0064	.0000	.0079	.0054	.0123	.2911	.0000
13	.600	1.111	.0000	.0000	.0048	.0026	.0040	.1954	.0000
14	.650	1.204	.0000	.0000	.0035	.0016	.0015	.1246	.0000
15	.700	1.296	.0000	.0000	.0025	.0011	.0007	.0827	.0000
16	.750	1.389	.0000	.0000	.0021	.0009	.0005	.0733	.0000
17	1.000	1.852	.0000	.0000	.0014	.0004	.0001	.0216	.0000
18	3.000	5.556	.0000	.0000	.0011	.0005	-.0001	-.0498	.0000

N	Y: INCHES	Y/ DELTA	$\frac{V'U'}{U^2}$	$\frac{U'^2}{U^2}$	$\frac{V'^2}{U^2}$	$\frac{W'^2}{U^2}$	$\frac{V'U'}{U^2AU}$	$\frac{V'U'}{U^2AU}$	$\frac{V'U'}{U^2AU}$
1	3.000	5.556	-.0144	.3261	.6739	.0000	.4930	.0021	.0019
2	.050	.093	.2144	.5212	.1909	.2779	.3548	.1721	.0000
3	.100	.185	.2186	.5099	.2121	.276	.2753	.1469	.0000
4	.150	.278	.2266	.4971	.2261	.2768	.3522	.1941	.0000
5	.200	.370	.2142	.4744	.2215	.3041	.4059	.2376	.0000
6	.250	.463	.2043	.4622	.2253	.2985	.5053	.2845	.0000
7	.300	.556	.1808	.4662	.2328	.3009	.6197	.3420	.0000
8	.350	.648	.1808	.4637	.2250	.3113	.7033	.3589	.0000
9	.400	.741	.1629	.4558	.2369	.3074	.8136	.4027	.0000
10	.450	.833	.1435	.4498	.2539	.3163	.9265	.4266	.0000
11	.500	.926	.1266	.4414	.3123	.2962	.8932	.3730	.0000
12	.550	1.019	.0971	.4345	.3855	.2966	.6531	.2312	.0000
13	.600	1.111	.0693	.4225	.4307	.2468	.3220	.1171	.0000
14	.650	1.204	.0466	.4141	.4477	.2288	.2377	.0915	.0000
15	.700	1.296	.0355	.4062	.4273	.2225	.1559	.0229	.0000
16	.750	1.389	.0331	.4049	.4045	.1706	.0443	.0115	.0000
17	1.000	1.852	.0104	.3991	.3153	.1416	.0131	.0000	.0000
18	3.000	5.556	.0255	.5770	.2831	.1399	.0540	.0006	.0000

N	Y: INCHES	Y/ DELTA	$\frac{V'T'}{V'T'}$	$\frac{T'}{T'AU}$	$\frac{T'}{(T'-T_e)}$	A10	G10	PRT
1	3.000	5.556	-.0385	.1470	.0065	-.8704	.0012	-1.5202
2	.050	.093	.5613	2.0110	.0891	.8526	.0496	.0000
3	.100	.185	.5389	1.6112	.0714	.7351	.0312	.0000
4	.150	.278	.5404	1.4474	.0641	.7457	.0327	.0000
5	.200	.370	.5535	1.3417	.0594	.8058	.0426	.0000
6	.250	.463	.5604	1.2262	.0543	.8372	.0599	.6707
7	.300	.556	.5726	1.1369	.0505	.8978	.0654	.6062
8	.350	.648	.5697	1.0209	.0452	.9203	.0800	.6041
9	.400	.741	.5780	.9221	.0417	1.0504	.1096	.6456
10	.450	.833	.5789	.8222	.0367	1.2202	.1562	.5457
11	.500	.926	.5384	.7422	.0317	1.4047	.2178	.4556
12	.550	1.019	.4797	.6800	.0224	1.8409	.3200	.3898
13	.600	1.111	.4029	.5500	.0153	2.2637	.4622	.2886
14	.650	1.204	.2995	.4474	.0096	2.7649	.6272	.2274
15	.700	1.296	.1995	.3535	.0068	3.2742	.8116	.2076
16	.750	1.389	.0858	.2700	.0057	3.9111	.1028	.4156
17	1.000	1.852	.0108	.1200	.0053	4.6053	.0017	-.6188
18	3.000	5.556	.0079	.1203	.0050	5.0799	.0000	-.0404

Table B3A

Fluctuating Profile Data

x = 52 in., Te = 0.2%

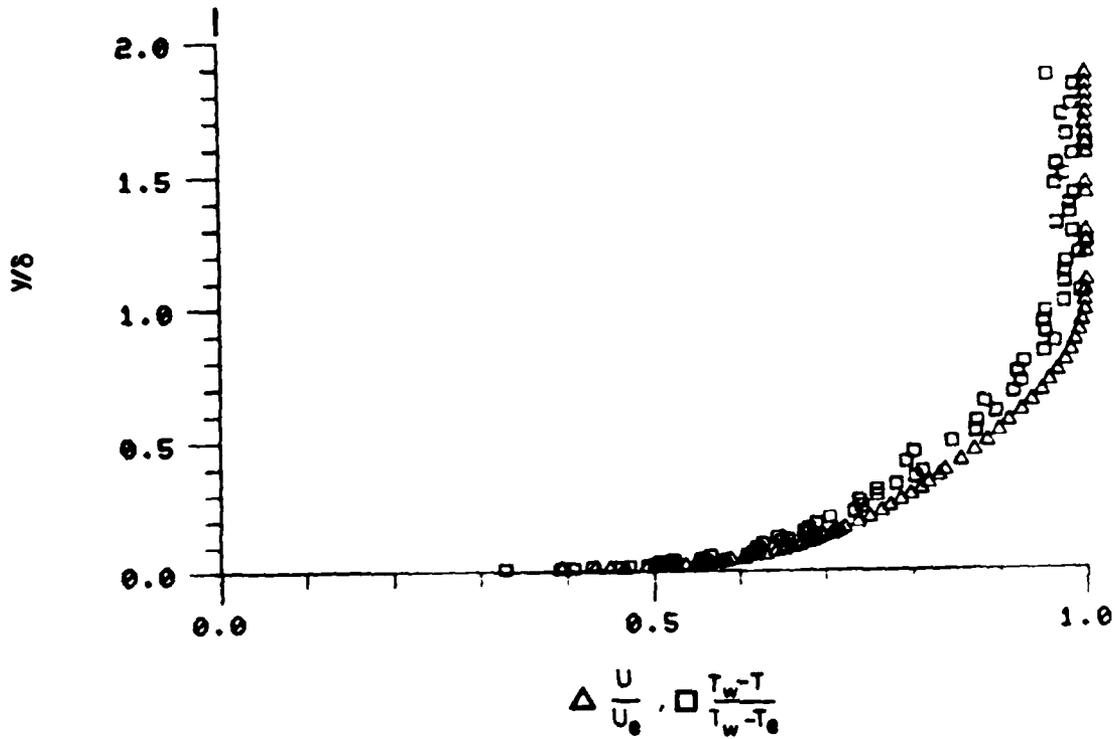
N	Y: INCHES	Y/ DELTA	U ² W ² /					
			UE3 x 10000					
1	3.0000	5.5556	-.0000	.0000	.0000	-.0000	.0000	.0000
2	3.0000	.0953	-.0316	.0000	.1535	-.0659	.0250	.0000
3	1.5000	.1806	-.0429	.0000	.0236	-.0186	.0250	.0000
4	1.5000	.2709	-.0427	.0000	.0130	-.0116	.0250	.0000
5	2.0000	.3772	-.0503	.0000	.0291	-.0135	.0250	.0000
6	2.5000	.4633	-.0422	.0000	.0218	-.0158	.0250	.0000
7	3.0000	.5556	-.0423	.0000	.0337	-.0106	.0250	.0000
8	3.5000	.6444	-.0313	.0000	.0040	-.0109	.0250	.0000
9	4.0000	.7411	-.0246	.0000	.0222	-.0110	.0250	.0000
10	4.5000	.8333	-.0120	.0000	.0138	-.0653	.0250	.0000
11	5.0000	.9222	-.0403	.0000	.0038	-.0185	.0250	.0000
12	5.5000	1.0111	-.0077	.0000	.0014	-.0057	.0250	.0000
13	6.0000	1.1111	-.0013	.0000	.0001	-.0006	.0250	.0000
14	6.5000	1.2044	.0001	.0000	.0002	.0001	.0250	.0000
15	7.0000	1.2966	.0001	.0000	.0000	.0001	.0250	.0000
16	7.5000	1.3889	.0000	.0000	.0000	.0000	.0250	.0000
17	8.0000	1.4852	.0000	.0000	.0000	.0000	.0250	.0000
18	8.5556	1.5556	.0000	.0000	.0000	.0000	.0250	.0000

N	Y: INCHES	Y/ DELTA	SU V	SU H	SV	SW	ST	PSIBV	PSIEA
			1	3.0000	5.5556	-.0089	.0000	.1593	.0000
2	3.0000	.0953	-.1185	.0000	.2349	.1338	-.0078	-.0078	-.0199
3	1.5000	.1806	-.2137	.0000	.1462	.0263	-.0000	-.0106	-.0364
4	1.5000	.2709	-.2698	.0000	.2255	.0999	-.0531	-.0531	-.0444
5	2.0000	.3772	-.4216	.0000	.2600	.0429	.0055	-.0253	-.0516
6	2.5000	.4633	-.5138	.0000	.4093	-.0416	.0055	-.0253	-.0516
7	3.0000	.5556	-.7500	.0000	.5202	.0915	.2334	-.0568	-.0516
8	3.5000	.6444	-.9244	.0000	.6275	.0063	.3377	-.0568	-.0516
9	4.0000	.7411	-.3927	-.1.0619	.6290	-.1506	.5987	-.0568	-.0516
10	4.5000	.8333	.0364	-.1.6232	.0497	-.2327	.9722	.0512	-.0837
11	5.0000	.9222	.6934	-.2.5814	.1390	-.2045	1.3393	.2750	-.1697
12	5.5000	1.0111	.2307	-.2.5805	.1372	.2927	2.0928	.8817	-.3076
13	6.0000	1.1111	-.1474	-.1.7285	.8223	.0555	2.7547	.6047	-.0736
14	6.5000	1.2044	.1456	-.1.128	.0417	-.3304	3.7660	.2607	-.0769
15	7.0000	1.2966	.1866	.5162	.3969	-.0802	3.1489	.0471	-.0699
16	7.5000	1.3889	.0864	.6099	.2289	.0367	1.4124	.0622	-.1112
17	8.0000	1.4852	-.0262	.7236	.0079	.0576	-.0760	.0180	-.0775
18	8.5556	1.5556	-.1217	-.2463	.0061	.0468	-.0199	.0211	-.0124

N	Y: INCHES	Y/ DELTA	PSICV	PSICW	FU V	FU H	FU V	FU H	FT
			1	3.0000	5.5556	-.0500	-.0000	.0000	.0000
2	3.0000	.0953	-.1956	-.0001	.0000	.0000	.0000	.0000	.0000
3	1.5000	.1806	-.1564	-.0133	.0000	.0000	.0000	.0000	.0000
4	1.5000	.2709	-.2113	-.0193	.0000	.0000	.0000	.0000	.0000
5	2.0000	.3772	-.2579	-.0384	.0000	.0000	.0000	.0000	.0000
6	2.5000	.4633	-.3446	-.0247	.0000	.0000	.0000	.0000	.0000
7	3.0000	.5556	-.4743	-.0229	.0000	.0000	.0000	.0000	.0000
8	3.5000	.6444	-.5865	-.0311	.0000	.0000	.0000	.0000	.0000
9	4.0000	.7411	-.7598	-.0593	.0000	.0000	.0000	.0000	.0000
10	4.5000	.8333	-.0323	-.0769	.0000	.0000	.0000	.0000	.0000
11	5.0000	.9222	-.1411	-.0728	.0000	.0000	.0000	.0000	.0000
12	5.5000	1.0111	.8598	-.0999	.0000	.0000	.0000	.0000	.0000
13	6.0000	1.1111	-.4516	-.0368	.0000	.0000	.0000	.0000	.0000
14	6.5000	1.2044	.1306	.1146	.0000	.0000	.0000	.0000	.0000
15	7.0000	1.2966	.0495	.2824	.0000	.0000	.0000	.0000	.0000
16	7.5000	1.3889	.0338	.2456	.0000	.0000	.0000	.0000	.0000
17	8.0000	1.4852	.0022	.2401	.0000	.0000	.0000	.0000	.0000
18	8.5556	1.5556	.0672	-.0366	.0000	.0000	.0000	.0000	.0000

Table B3B

VELOCITY AND TEMPERATURE RATIOS



VELOCITY AND TEMPERATURE DISTRIBUTIONS IN UNIVERSAL COORDINATES

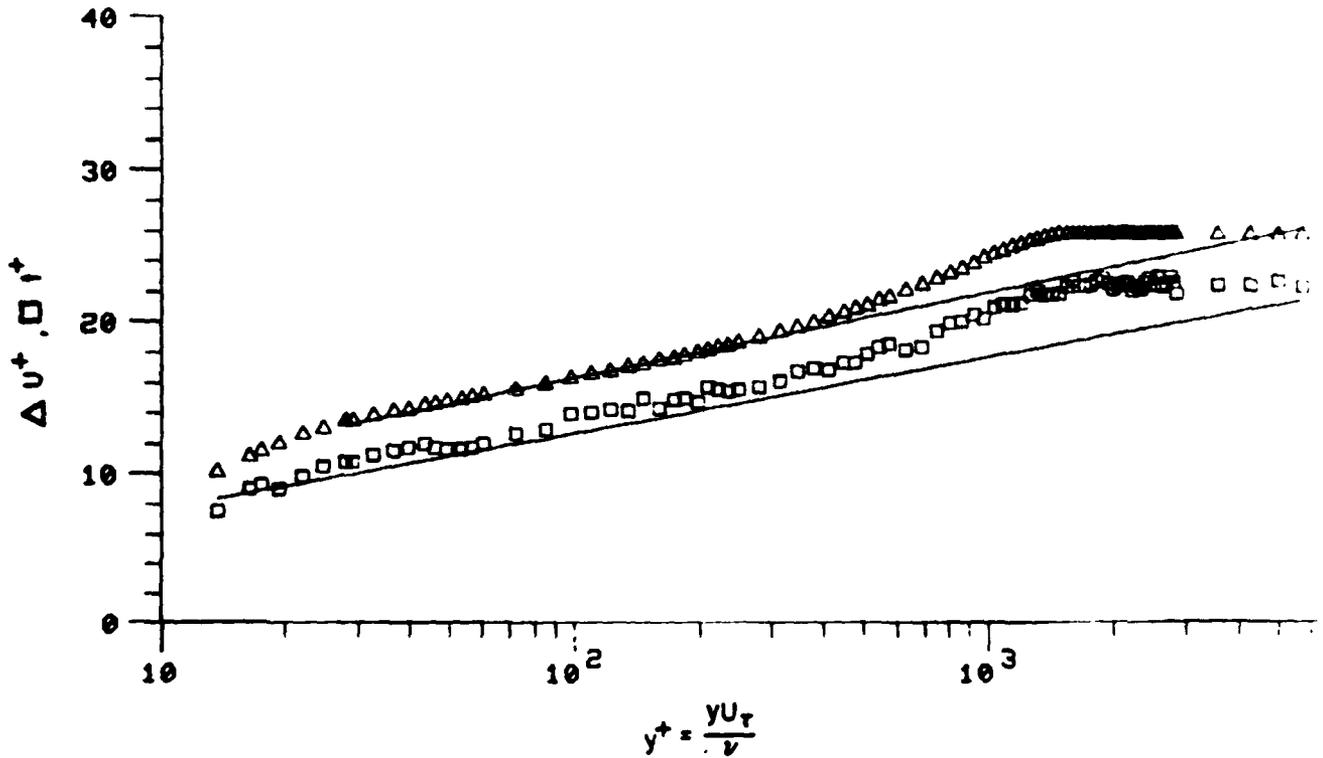


Figure B3. Mean Velocity and Temperature Profiles $x = 68$ in., $Te = 0.2\%$

RUN NO. 3. POINT 2.
 BOUNDARY LAYER PROPERTIES

	LINEAR INTERPOLATION TO WALL	STANDARD SUBLAYER FUNCTION FROM WALL TO $y^+ = 35$
FREE STREAM VELOCITY	99.171	99.171
FREE STREAM TEMPERATURE	73.270	
WALL TEMPERATURE	90.270	
WALL HEAT FLUX	.05020	
FREE STREAM DENSITY	.07493	
FREE STREAM KINEMATIC VISCOSITY	.0001639	
DENSITY OF FLUID AT WALL	.07262	
KINEMATIC VISCOSITY OF FLUID AT WALL	.0001733	
WALL/FREE STREAM DENSITY RATIO	.96909	
LOCATION REYNOLDS NUMBER (REX)	3428173.22	
INPUT VALUE OF VELOCITY DELTA	.90000	
INPUT VALUE OF TEMPERATURE DELTA	.97000	
CALCULATED DELTA		.61093
DELTA 99.5% INPUT	.00000	
DISPLACEMENT THICKNESS (DELSTAR)	.12319	.12306
MOMENTUM THICKNESS (THETA)	.08551	.08564
ENERGY-DISSIPATION THICKNESS	.15118	.15148
ENTHALPY THICKNESS	.00370	.00371
SHAPE FACTOR 12 (DELSTAR/THETA)	1.44071	1.43351
SHAPE FACTOR 32 (ENERGY/THETA)	1.76804	1.76469
MOMENTUM THICKNESS REYNOLDS NUMBER	4310.75	4327.68
DISPLACEMENT THICKNESS REYNOLDS NUMBER	6210.57	6203.78
SKIN FRICTION COEFFICIENT	.002940	
FRICTION VELOCITY	3.86252	
LAW OF THE WALL CONSTANT (K)	.41000	
LAW OF THE WALL CONSTANT (C)	5.00000	
WAKE STRENGTH		.57972
CLAUSERS 'DELTA' INTEGRAL	-2.88141	-3.06592
CLAUSERS 'G' INTEGRAL	21.95430	21.63072
DISPLACEMENT THICKNESS - CONSTANT DENSITY	.11589	.11941
MOMENTUM THICKNESS - CONSTANT DENSITY	.08625	.08660
SHAPE FACTOR 12 - CONSTANT DENSITY	1.34363	1.37890
LOCATION -X-	68.00000	
Te = 0.2%		

Table B5

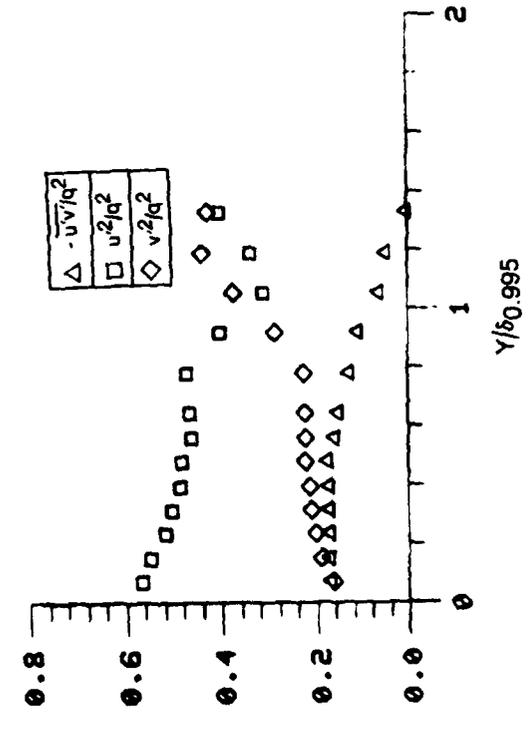
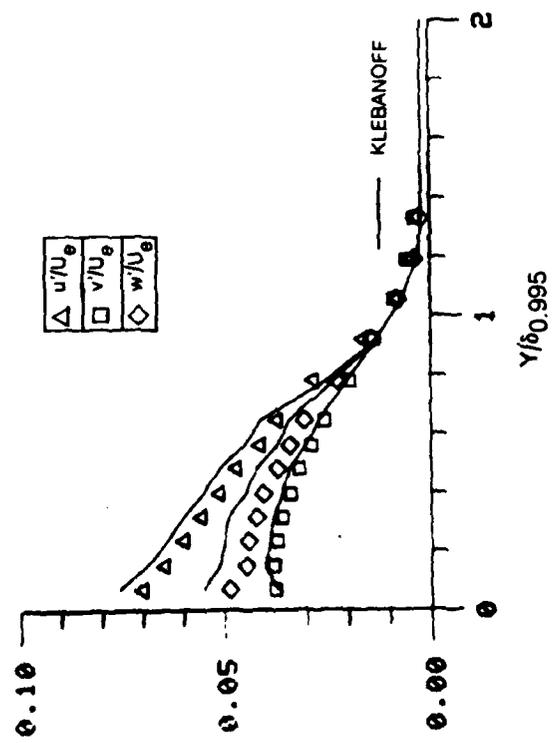
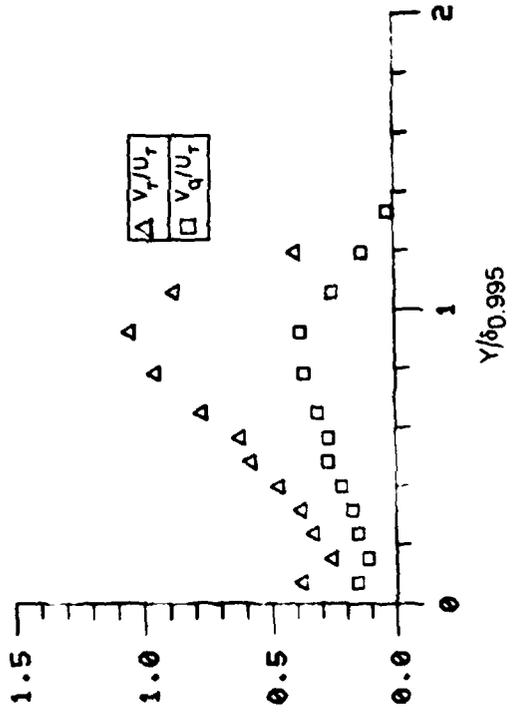
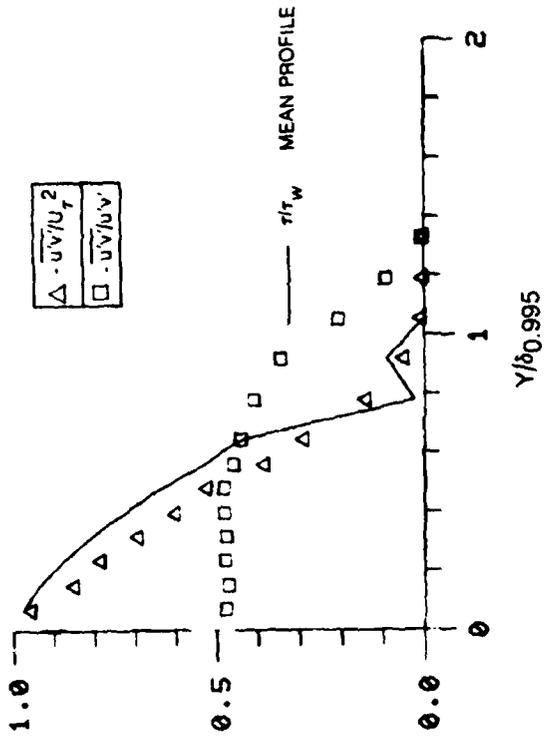


Figure B4A. Boundary Layer Turbulence Quantities $x = 68$ in, $T_e = 0.2\%$

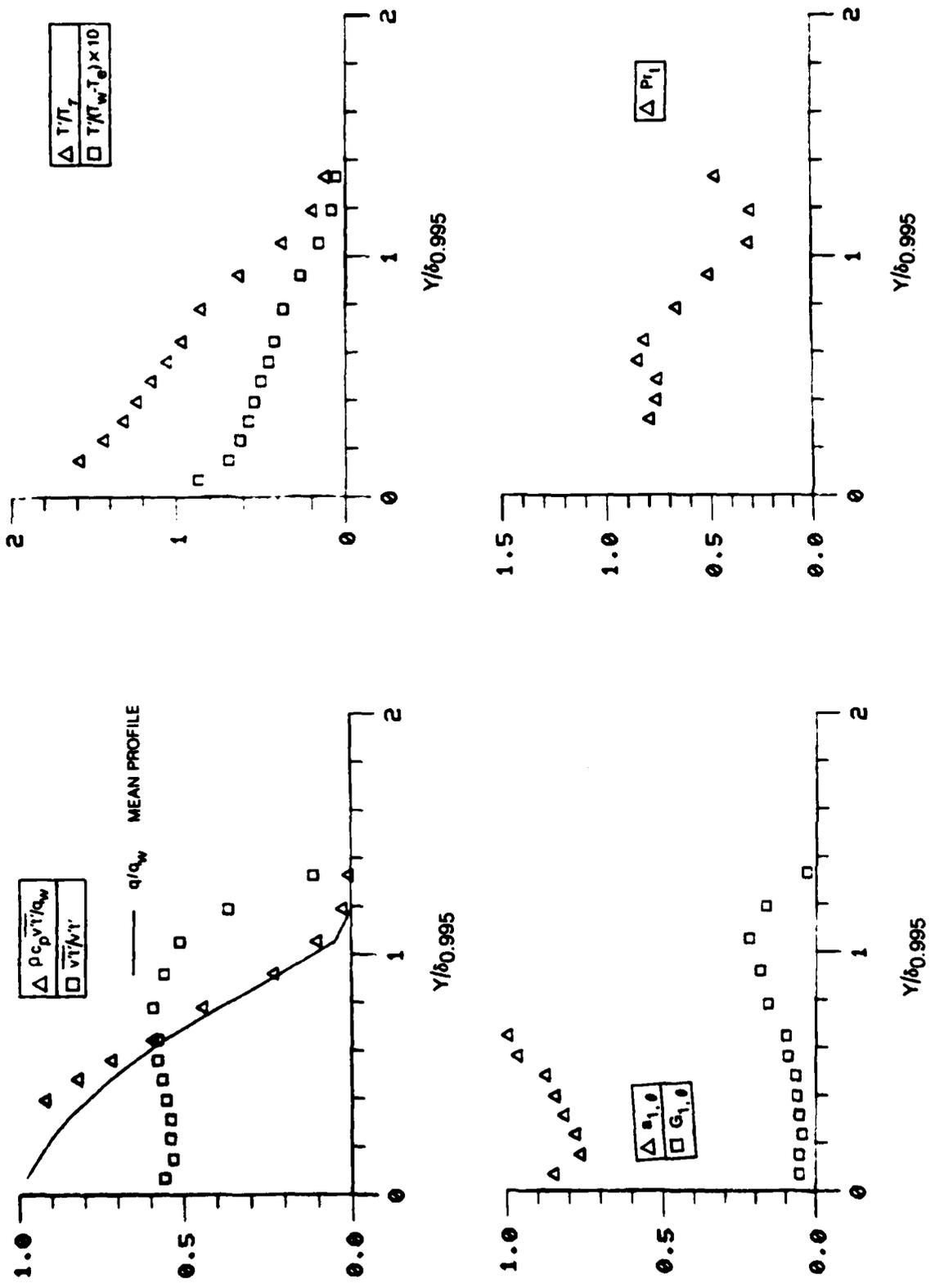


Figure B4B. Boundary Layer Turbulence Quantities, $x = 68$ in, $T_e = 0.2\%$

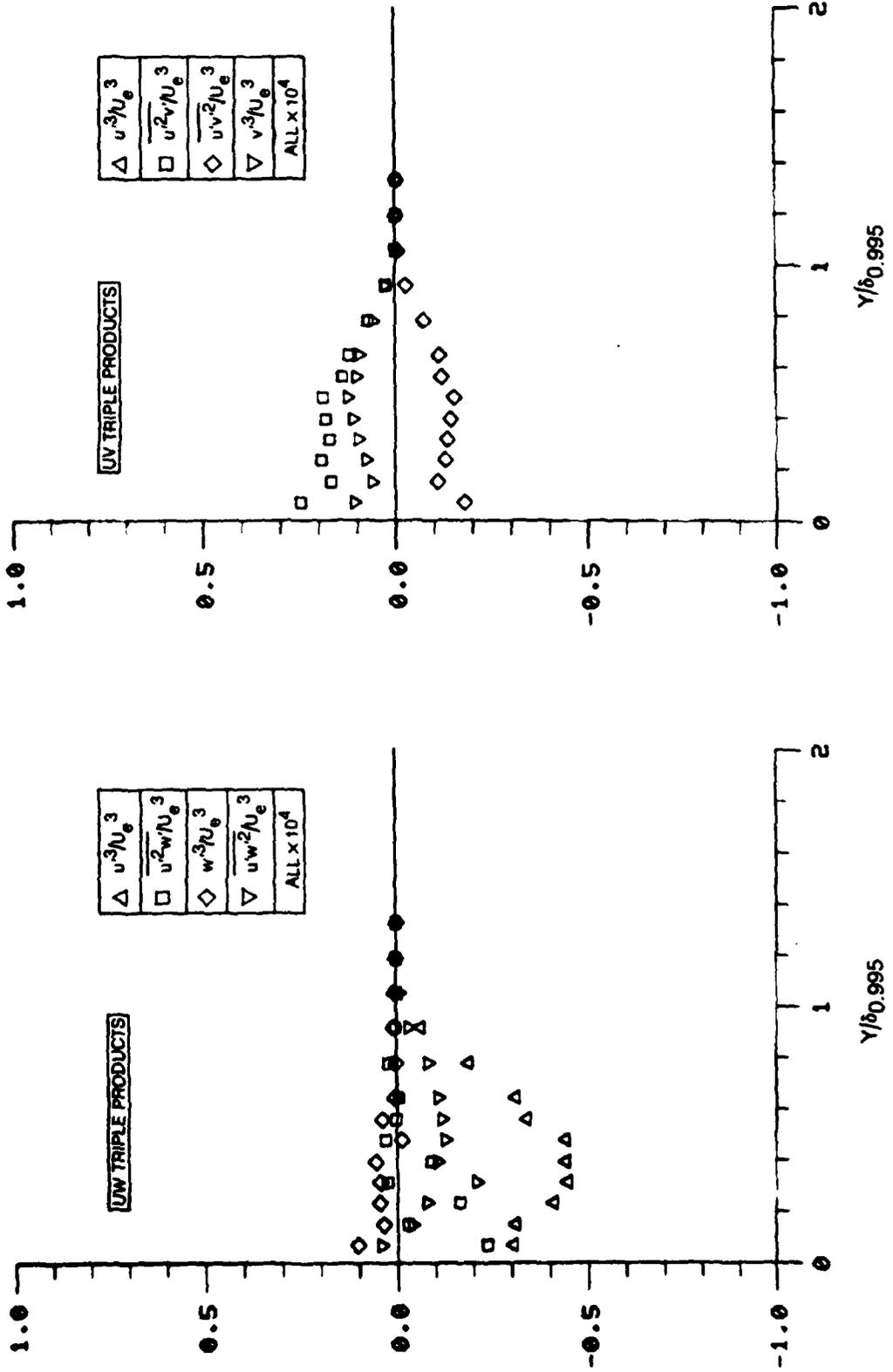


Figure B4C. Boundary Layer Triple Product Distributions $x = 68$ in, $Te = 0.2\%$

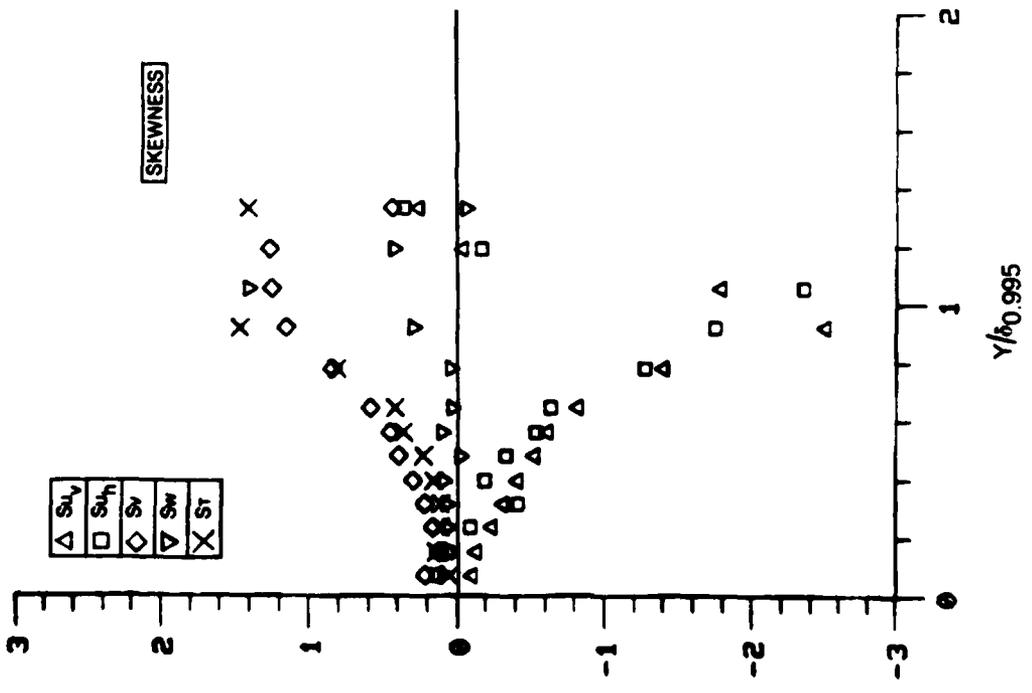
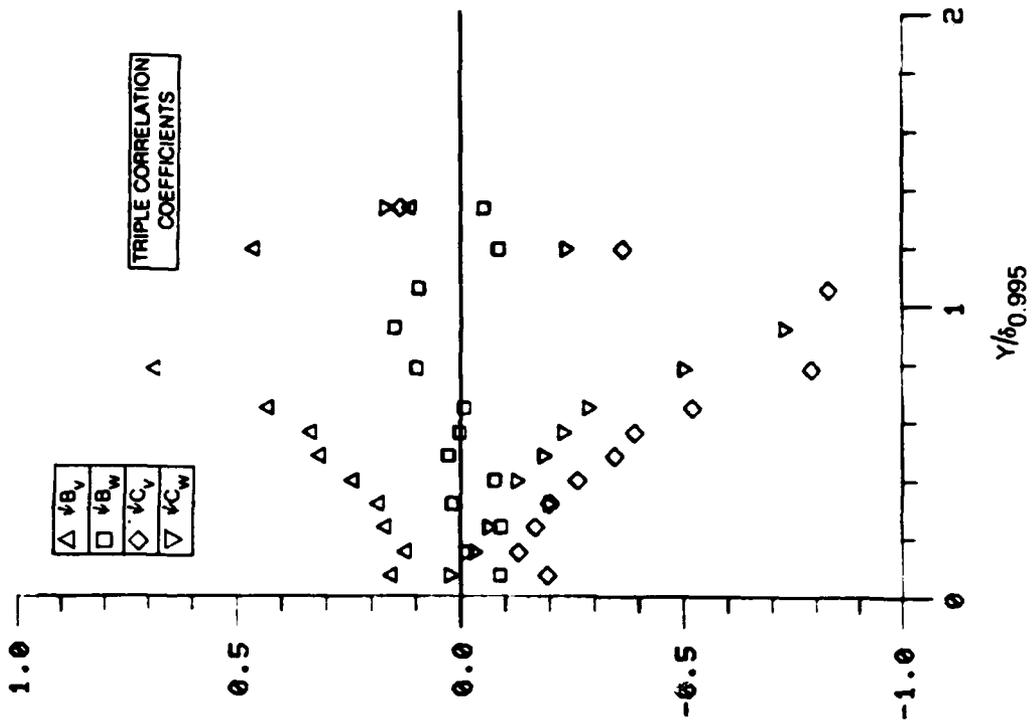


Figure B4D. Boundary Layer Skewness and Triple Product Correlation Coefficient Distributions $x = 68$ in, $T_e = 0.2\%$

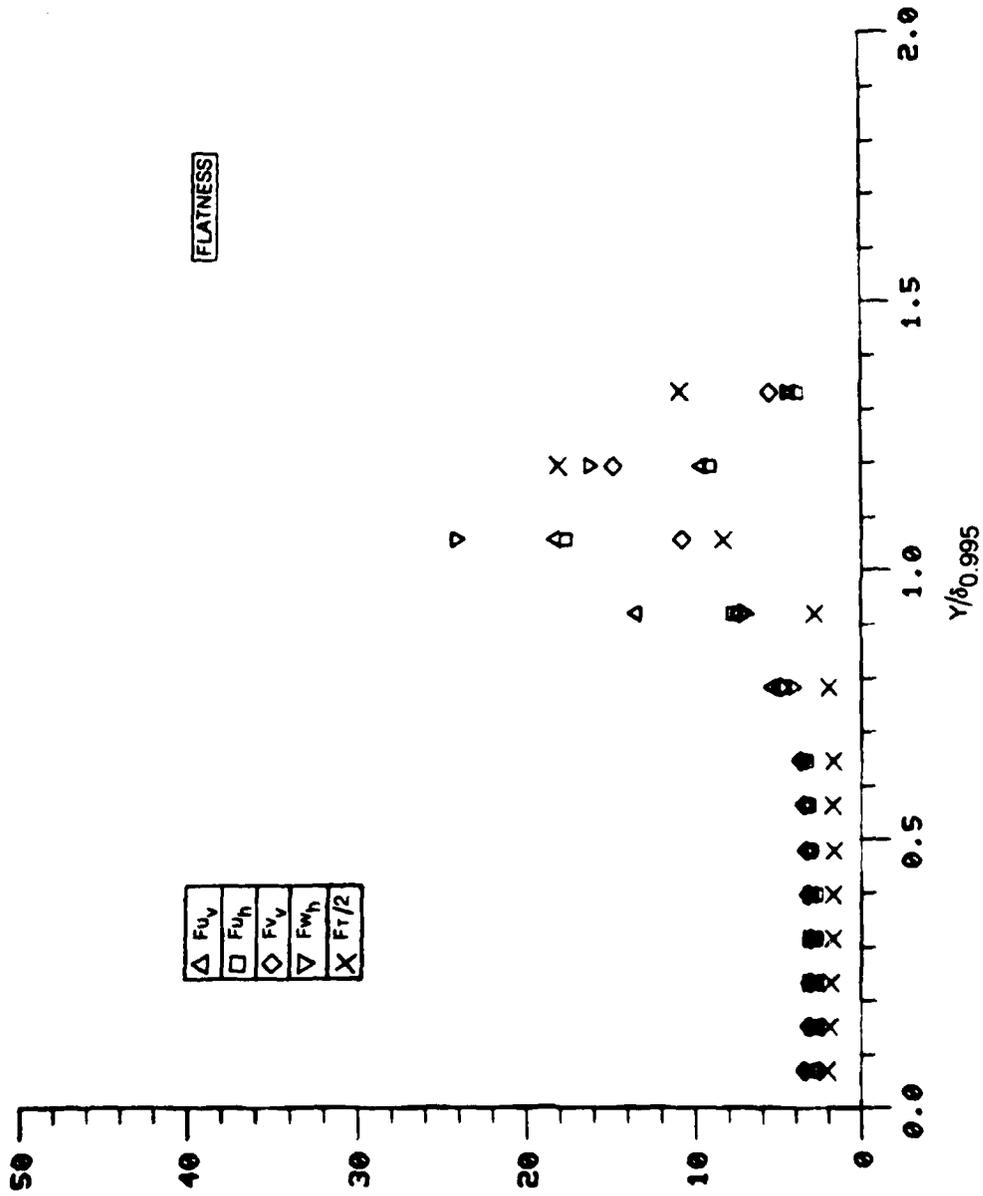


Figure B4E. Boundary Layer Flatness Distributions $x = 68$ in, $T_e = 0.2\%$

Fluctuating Profile Data

x = 68 in., Te = 0.2%

N	Y: INCHES	Y/ DELTA	U'/UE	V'/UE	W'/UE	$\sqrt{U'V'}/UE$	$\frac{U'V'}{U\tau U^2}$	$\frac{U'V'}{U'V'}$	C/UE
1	3.000	4.110	.0026	.0019	.0011	.002	.00048	-.1956	.0000
2	.050	.068	.0043	.0034	.0018	.002	.00059	.4797	.0000
3	.100	.136	.0059	.0047	.0025	.003	.00078	.4719	.0000
4	.150	.204	.0075	.0059	.0033	.004	.00104	.4607	.0000
5	.200	.272	.0091	.0071	.0041	.005	.00139	.4766	.0000
6	.250	.340	.0107	.0083	.0049	.006	.00181	.4813	.0000
7	.300	.408	.0123	.0095	.0057	.007	.00230	.4833	.0000
8	.350	.476	.0139	.0107	.0065	.008	.00286	.4819	.0000
9	.400	.544	.0155	.0119	.0073	.009	.00354	.4710	.0000
10	.450	.612	.0171	.0131	.0081	.010	.00434	.3461	.0000
11	.500	.680	.0187	.0143	.0089	.011	.00524	.0933	.0000
12	.550	.748	.0203	.0155	.0097	.012	.00614	.0076	.0000
13	.600	.816	.0219	.0167	.0105	.013	.00704	.0000	.0000
14	.650	.884	.0235	.0179	.0113	.014	.00794	.0000	.0000
15	.700	.952	.0251	.0191	.0121	.015	.00884	.0000	.0000
16	.750	1.020	.0267	.0203	.0129	.016	.00974	.0000	.0000
17	.800	1.088	.0283	.0215	.0137	.017	.01064	.0000	.0000
18	.850	1.156	.0299	.0227	.0145	.018	.01154	.0000	.0000

N	Y: INCHES	Y/ DELTA	$\frac{V'U'}{U^2}$	$\frac{U'^2}{U^2}$	$\frac{V'^2}{U^2}$	$\frac{W'^2}{U^2}$	$\frac{V'U'}{U\tau U}$	$\frac{V'U'}{U\tau U}$	$\frac{V'U'}{U'V'}$
1	3.000	4.110	.6413	.5756	.3090	.1154	-.0071	-.0004	.0000
2	.050	.068	.1669	.5658	.1621	.2721	.0385	.1622	.0000
3	.100	.136	.1720	.5491	.1894	.2611	.0426	.1556	.0000
4	.150	.204	.1771	.5324	.2167	.2502	.0467	.1487	.0000
5	.200	.272	.1822	.5157	.2440	.2393	.0508	.1418	.0000
6	.250	.340	.1873	.5000	.2713	.2284	.0549	.1349	.0000
7	.300	.408	.1924	.4843	.2986	.2175	.0590	.1280	.0000
8	.350	.476	.1975	.4686	.3259	.2066	.0631	.1211	.0000
9	.400	.544	.2026	.4529	.3532	.1957	.0672	.1142	.0000
10	.450	.612	.2077	.4372	.3805	.1848	.0713	.1073	.0000
11	.500	.680	.2128	.4215	.4078	.1739	.0754	.1004	.0000
12	.550	.748	.2179	.4058	.4351	.1630	.0795	.0935	.0000
13	.600	.816	.2230	.3901	.4624	.1521	.0836	.0866	.0000
14	.650	.884	.2281	.3744	.4897	.1412	.0877	.0797	.0000
15	.700	.952	.2332	.3587	.5170	.1303	.0918	.0728	.0000
16	.750	1.020	.2383	.3430	.5443	.1194	.0959	.0659	.0000
17	.800	1.088	.2434	.3273	.5716	.1085	.1000	.0590	.0000
18	.850	1.156	.2485	.3116	.5989	.0976	.1041	.0521	.0000

N	Y: INCHES	Y/ DELTA	$\frac{V'T'}{V'T'}$	$\frac{T'}{T\tau U}$	$\frac{T'}{(T\tau - T\epsilon)}$	A10	G10	PRT
1	3.000	4.110	-.0007	.0997	.0043	.0000	.0000	.0000
2	.050	.068	.0008	.0874	.0870	.8470	.0000	.0000
3	.100	.136	.0016	.5867	.0888	.7619	.0000	.0000
4	.150	.204	.0024	.4331	.0821	.7793	.0000	.0000
5	.200	.272	.0032	.3199	.0572	.8165	.0000	.7933
6	.250	.340	.0040	.2348	.0335	.8462	.0000	.7628
7	.300	.408	.0048	.1451	.0496	.8749	.0000	.7569
8	.350	.476	.0056	.0541	.0457	.8937	.0000	.8496
9	.400	.544	.0064	.9665	.0419	.9954	.0000	.8172
10	.450	.612	.0072	.8448	.0366	1.1985	.0000	.6602
11	.500	.680	.0080	.6254	.0271	1.4081	.0000	.5079
12	.550	.748	.0088	.5066	.0161	1.6007	.0000	.3105
13	.600	.816	.0096	.3878	.0085	1.7548	.0000	.2992
14	.650	.884	.0104	.2690	.0052	1.8659	.0000	.4731
15	.700	.952	.0112	.1502	.0048	1.9569	.0000	.8446
16	.750	1.020	.0120	.0314	.0045	2.0822	.0000	.5663
17	.800	1.088	.0128	.0000	.0000	2.2075	.0000	.0000
18	.850	1.156	.0136	.0000	.0000	2.3328	.0000	.0000

Table B6A

Fluctuating Profile Data

x = 68 in., Te = 0.2%

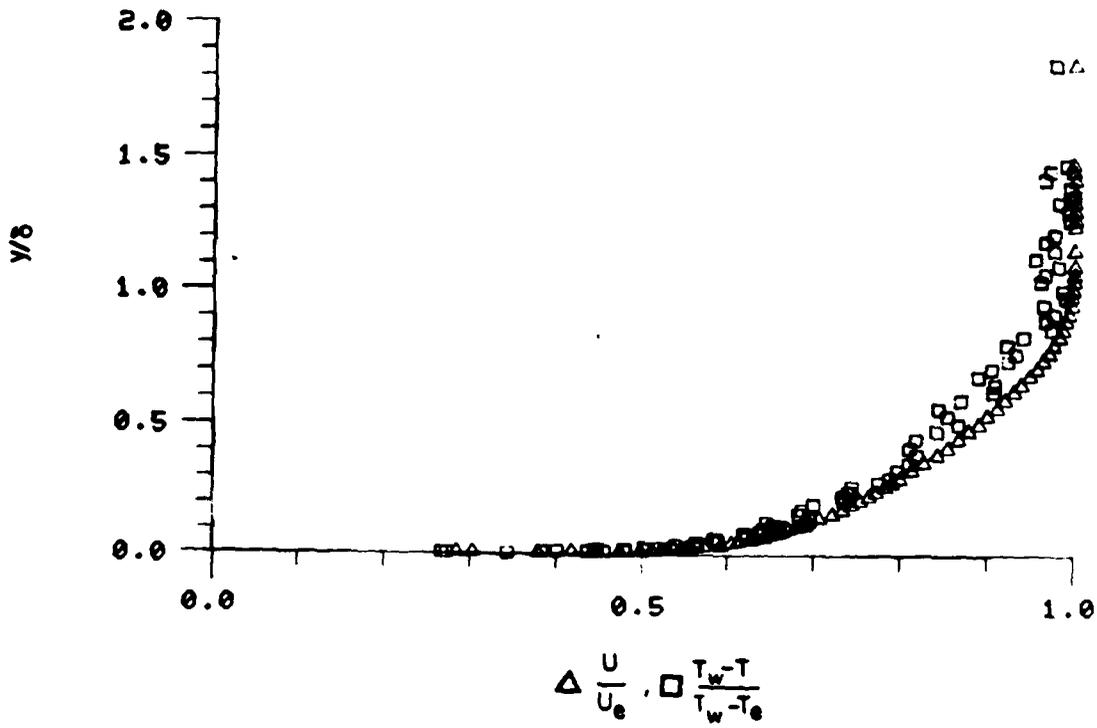
N	Y: INCHES	Y/ DELTA	U ³ /UE ³ x 10000	L ² U ³ / UE ³ x 10000	U ³ /UE ³ x 10000	U ⁴ /UE ⁴ x 10000	L ² U ⁴ / UE ⁴ x 10000	U ⁴ /UE ⁴ x 10000	V ³ /UE ³ x 10000
1	3.000	4.110	-.0000	-.0000	-.0000	-.0000	.0000	-.0000	-.0000
2	.050	.060	-.0000	-.0000	-.0000	-.0000	.0000	-.0000	-.0000
3	.110	.150	-.0000	-.0000	-.0000	-.0000	.0000	-.0000	-.0000
4	.170	.230	-.0000	-.0000	-.0000	-.0000	.0000	-.0000	-.0000
5	.230	.310	-.0000	-.0000	-.0000	-.0000	.0000	-.0000	-.0000
6	.290	.390	-.0000	-.0000	-.0000	-.0000	.0000	-.0000	-.0000
7	.350	.470	-.0000	-.0000	-.0000	-.0000	.0000	-.0000	-.0000
8	.410	.560	-.0000	-.0000	-.0000	-.0000	.0000	-.0000	-.0000
9	.470	.640	-.0000	-.0000	-.0000	-.0000	.0000	-.0000	-.0000
10	.570	.780	-.0000	-.0000	-.0000	-.0000	.0000	-.0000	-.0000
11	.670	.910	-.0000	-.0000	-.0000	-.0000	.0000	-.0000	-.0000
12	.770	1.050	-.0000	-.0000	-.0000	-.0000	.0000	-.0000	-.0000
13	.870	1.190	-.0000	-.0000	-.0000	-.0000	.0000	-.0000	-.0000
14	.970	1.320	-.0000	-.0000	-.0000	-.0000	.0000	-.0000	-.0000
15	1.500	2.050	-.0000	-.0000	-.0000	-.0000	.0000	-.0000	-.0000
16	3.000	4.110	-.0000	-.0000	-.0000	-.0000	.0000	-.0000	-.0000
17	.000	.000	-.0000	-.0000	-.0000	-.0000	.0000	-.0000	-.0000
18	.000	.000	-.0000	-.0000	-.0000	-.0000	.0000	-.0000	-.0000

N	Y: INCHES	Y/ DELTA	SU V	SU H	SV	SH	ST	PSIEV	PSIEH
1	3.000	4.110	-.1433	-.0034	-.0398	-.4208	.0273	.0090	-.1112
2	.050	.060	-.1033	-.1372	-.2109	-.0843	.0569	.1552	-.0067
3	.110	.150	-.1133	-.0548	-.1158	-.0345	.1433	.1211	-.0067
4	.170	.230	-.2336	-.0950	-.1523	-.0542	.1036	.1682	-.0067
5	.230	.310	-.3241	-.4190	-.2117	-.0649	.1094	.1828	-.0067
6	.290	.390	-.4141	-.1899	-.2855	-.0846	.1525	.2407	-.0067
7	.350	.470	-.5241	-.3355	-.3844	-.0251	.2221	.3149	-.0067
8	.410	.560	-.6191	-.5386	-.4444	-.0937	.3597	.3339	-.0067
9	.470	.640	-.6898	-.6376	-.5799	-.0326	.4162	.4319	-.0067
10	.570	.780	-.8399	-.7800	-.8360	-.0376	.5076	.6876	-.0067
11	.670	.910	-1.1119	-.7551	1.1376	-.2880	.5502	1.0418	-.0067
12	.770	1.050	-1.7931	-.3528	1.2399	1.3923	.0275	1.0351	-.0067
13	.870	1.190	-.0481	-.1697	1.4168	1.4168	.1170	.4642	-.0067
14	.970	1.320	.2444	-.3479	1.2267	-.0666	.1700	.1169	-.0067
15	1.500	2.050	.1154	-.0666	1.0225	-.1279	.1452	.0128	-.0067
16	3.000	4.110	-.0000	-.0000	-.0000	-.0000	-.0000	-.0000	-.0067
17	.000	.000	-.0000	-.0000	-.0000	-.0000	-.0000	-.0000	-.0067
18	.000	.000	-.0000	-.0000	-.0000	-.0000	-.0000	-.0000	-.0067

N	Y: INCHES	Y/ DELTA	PSICV	PSICH	FU V	FU H	FU V	FU H	FT
1	3.000	4.110	-.0267	-.2917	.0291	.2710	.0379	.7406	2.6777
2	.050	.060	-.1246	-.0260	.8096	.6306	.3938	.3530	4.6154
3	.110	.150	-.1274	-.0291	.7371	.6681	.1789	.1543	3.6061
4	.170	.230	-.1679	-.0617	.7174	.6968	.0549	.2251	3.3333
5	.230	.310	-.1969	-.1963	.8463	.7914	.1224	.2488	3.3333
6	.290	.390	-.2617	-.1210	.9808	.7615	.2415	.2765	3.3333
7	.350	.470	-.3345	-.1838	.6222	.9865	.3747	.5555	3.3333
8	.410	.560	-.5204	-.2304	.6550	.1745	.4584	.3665	3.3333
9	.470	.640	-.5204	-.2333	.2922	.3490	.7791	.5881	3.3333
10	.570	.780	-.7887	-.5004	.4422	.6673	.9047	.1493	3.3333
11	.670	.910	-1.0312	-.7246	.4379	7.7491	.7122	.0297	3.3333
12	.770	1.050	-.0322	-.0129	1.1795	17.5620	.0941	.9625	3.3333
13	.870	1.190	-.3642	-.2370	.5933	.0941	.7623	.5272	3.3333
14	.970	1.320	.1345	-.7266	.2827	.0941	.9937	.1627	3.3333
15	1.500	2.050	.0841	-.0535	.1062	.0941	.9937	.0261	3.3333
16	3.000	4.110	-.0000	-.0000	-.0000	-.0000	-.0000	-.0000	3.3333
17	.000	.000	-.0000	-.0000	-.0000	-.0000	-.0000	-.0000	3.3333
18	.000	.000	-.0000	-.0000	-.0000	-.0000	-.0000	-.0000	3.3333

Table B6B

VELOCITY AND TEMPERATURE RATIOS



VELOCITY AND TEMPERATURE DISTRIBUTIONS IN UNIVERSAL COORDINATES

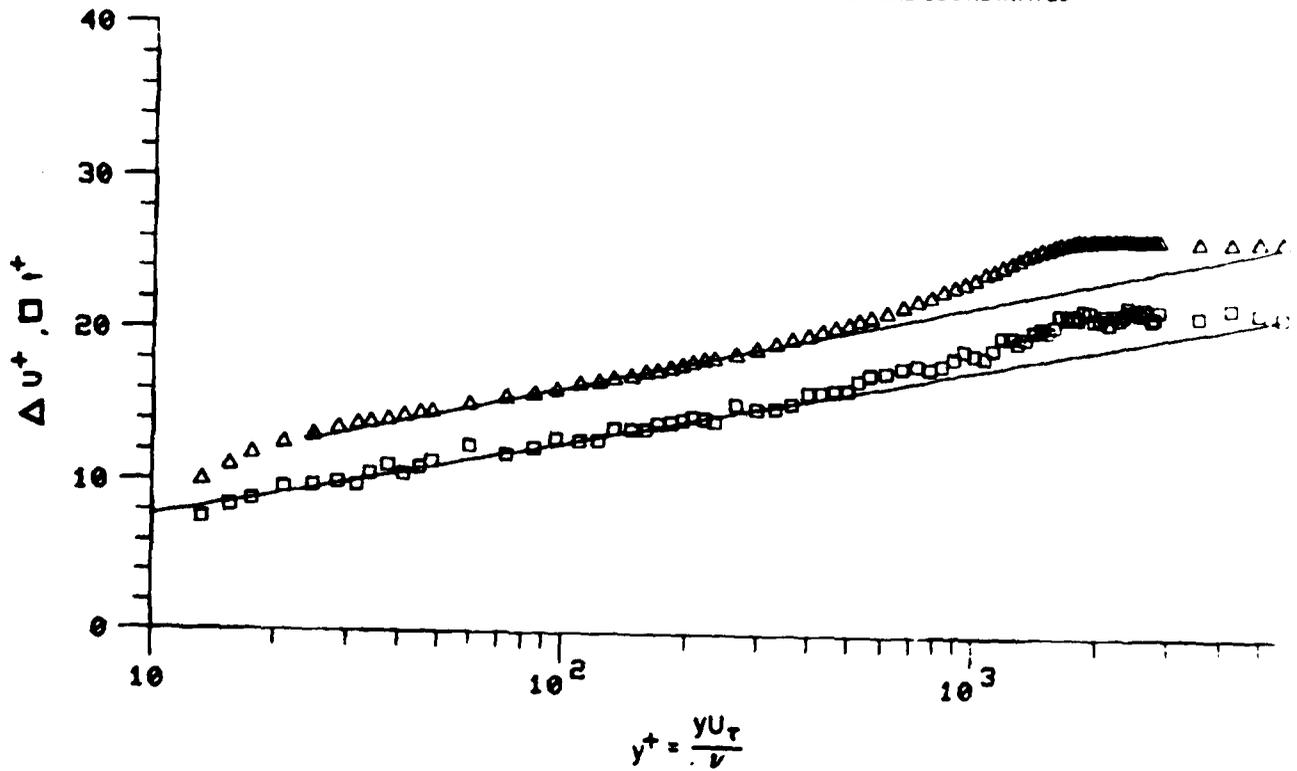


Figure B5. Mean Velocity and Temperature Profiles $x = 84$ in., $Te = 0.2\%$

RUN NO. 3. POINT 1.

BOUNDARY LAYER PROPERTIES

	LINEAR INTERPOLATION TO WALL	STANDARD SUBLAYER FUNCTION FROM WALL TO $Y^+=35$
FREE STREAM VELOCITY	99.264	99.264
FREE STREAM TEMPERATURE	71.940	
WALL TEMPERATURE	88.630	
WALL HEAT FLUX	.05020	
FREE STREAM DENSITY	.07512	
FREE STREAM KINEMATIC VISCOSITY	.0001632	
DENSITY OF FLUID AT WALL	.07283	
KINEMATIC VISCOSITY OF FLUID AT WALL	.0001724	
WALL/FREE STREAM DENSITY RATIO	.96956	
LOCATION REYNOLDS NUMBER (REX)	4257589.31	
INPUT VALUE OF VELOCITY DELTA	1.13000	
INPUT VALUE OF TEMPERATURE DELTA	1.25000	
CALCULATED DELTA		1.01545
DELTA 99.5% INPUT	.00000	
DISPLACEMENT THICKNESS (DELSTAR)	.15211	.15221
MOMENTUM THICKNESS (THETA)	.10696	.10706
ENERGY-DISSIPATION THICKNESS	.18910	.18914
ENTHALPY THICKNESS	.00448	.00448
SHAPE FACTOR 12 (DELSTAR/THETA)	1.42212	1.42180
SHAPE FACTOR 12 (ENERGY/THETA)	1.76801	1.76676
MOMENTUM THICKNESS REYNOLDS NUMBER	5421.16	5426.27
DISPLACEMENT THICKNESS REYNOLDS NUMBER	7709.54	7715.06
SKIN FRICTION COEFFICIENT	.002806	
FRICTION VELOCITY	3.77632	
LAW OF THE WALL CONSTANT (K)	.41000	
LAW OF THE WALL CONSTANT (C)	5.00000	
WAKE STRENGTH		.60112
CLAUSERS 'DELTA' INTEGRAL	-3.76703	-3.88635
CLAUSERS 'G' INTEGRAL	27.56274	27.56606
DISPLACEMENT THICKNESS - CONSTANT DENSITY	.14552	.14765
MOMENTUM THICKNESS - CONSTANT DENSITY	.10785	.10795
SHAPE FACTOR 12 - CONSTANT DENSITY	1.34934	1.36957
LOCATION -X-	84.00000	
Te = 0.2%		

Table B8

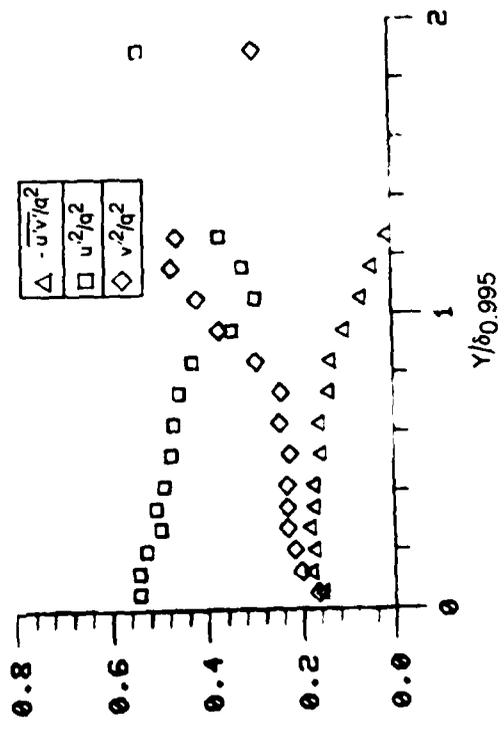
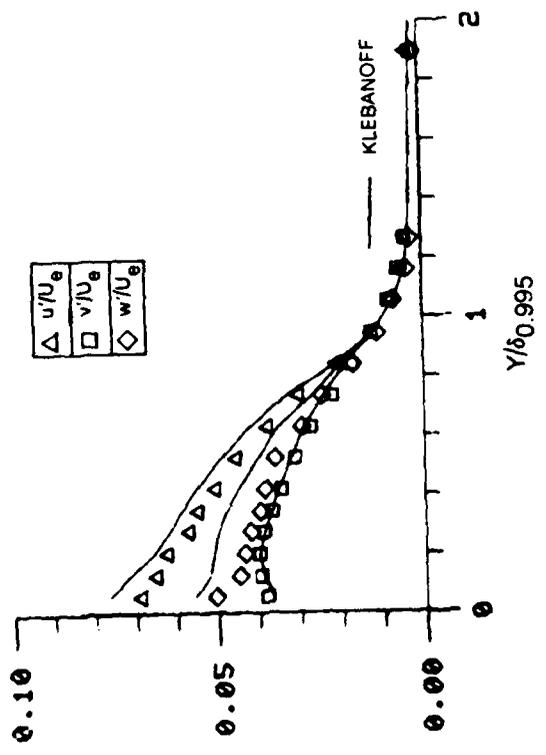
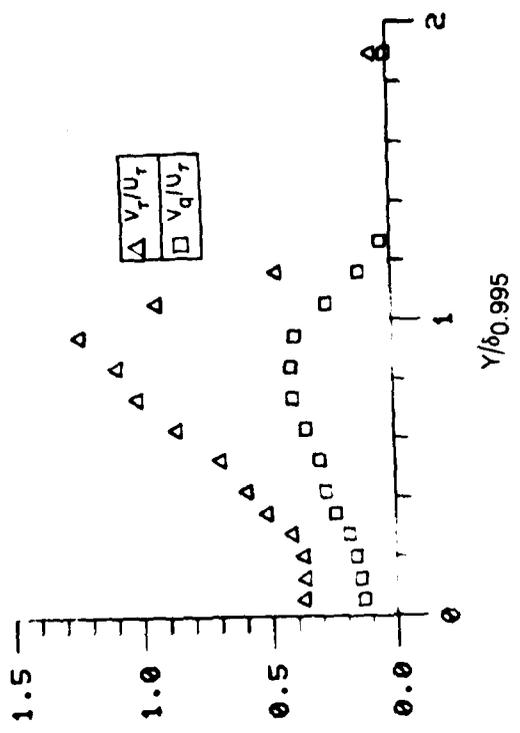
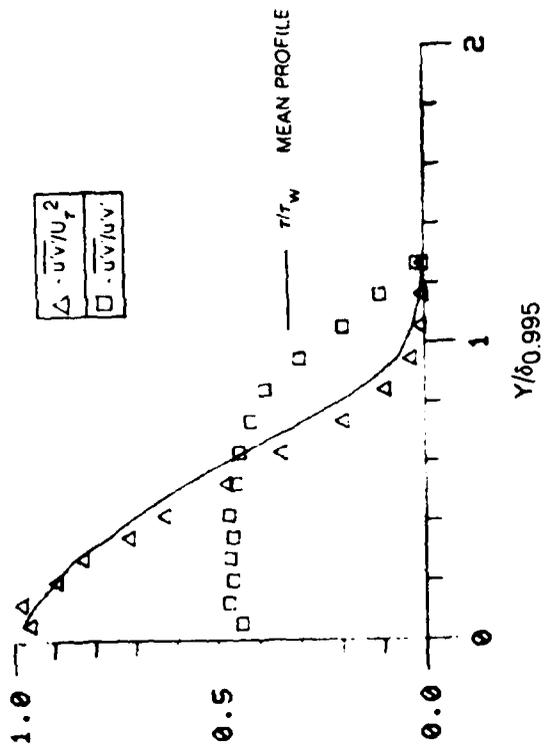


Figure B6A. Boundary Layer Turbulence Quantities $x = 84$ in, $T_e = 0.2\%$

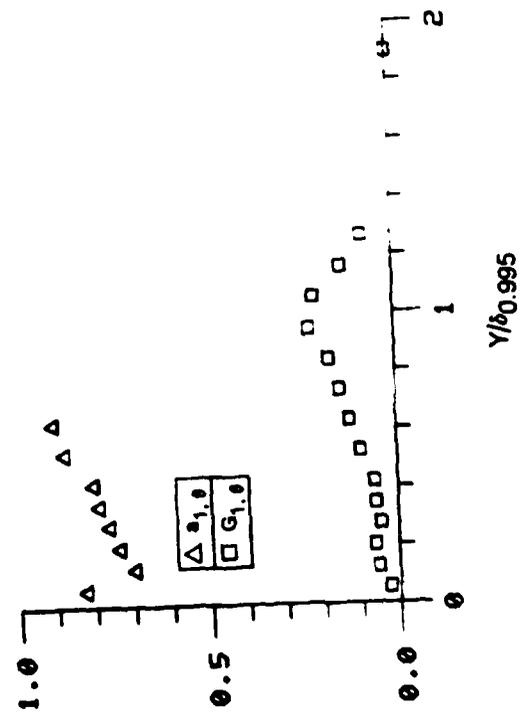
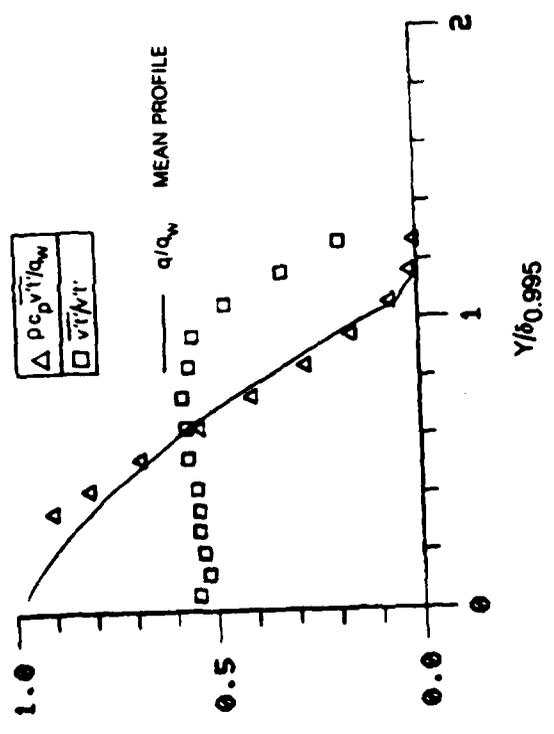
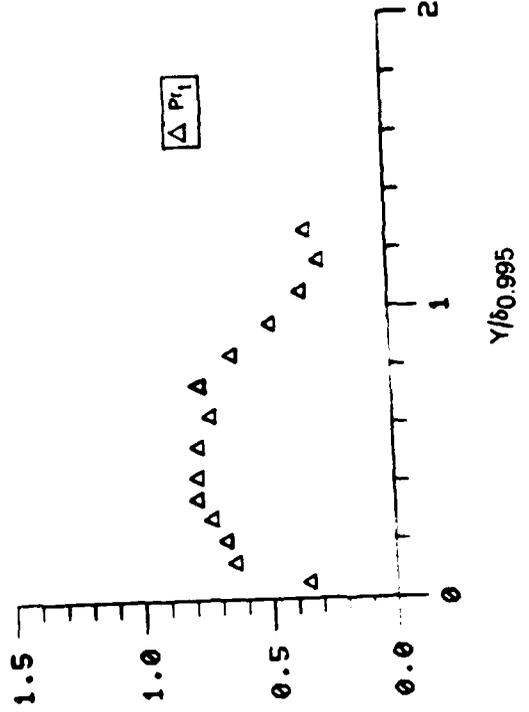
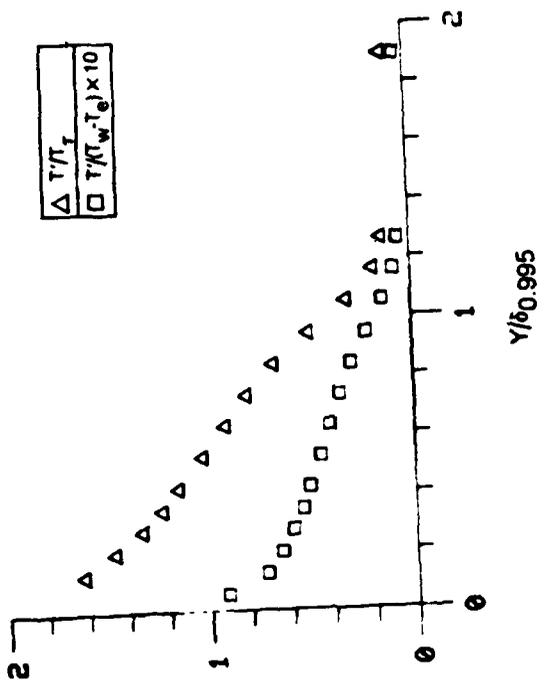


Figure B68. Boundary Layer Turbulence Quantities, $x = 84$ in, $T_e = 0.2\%$

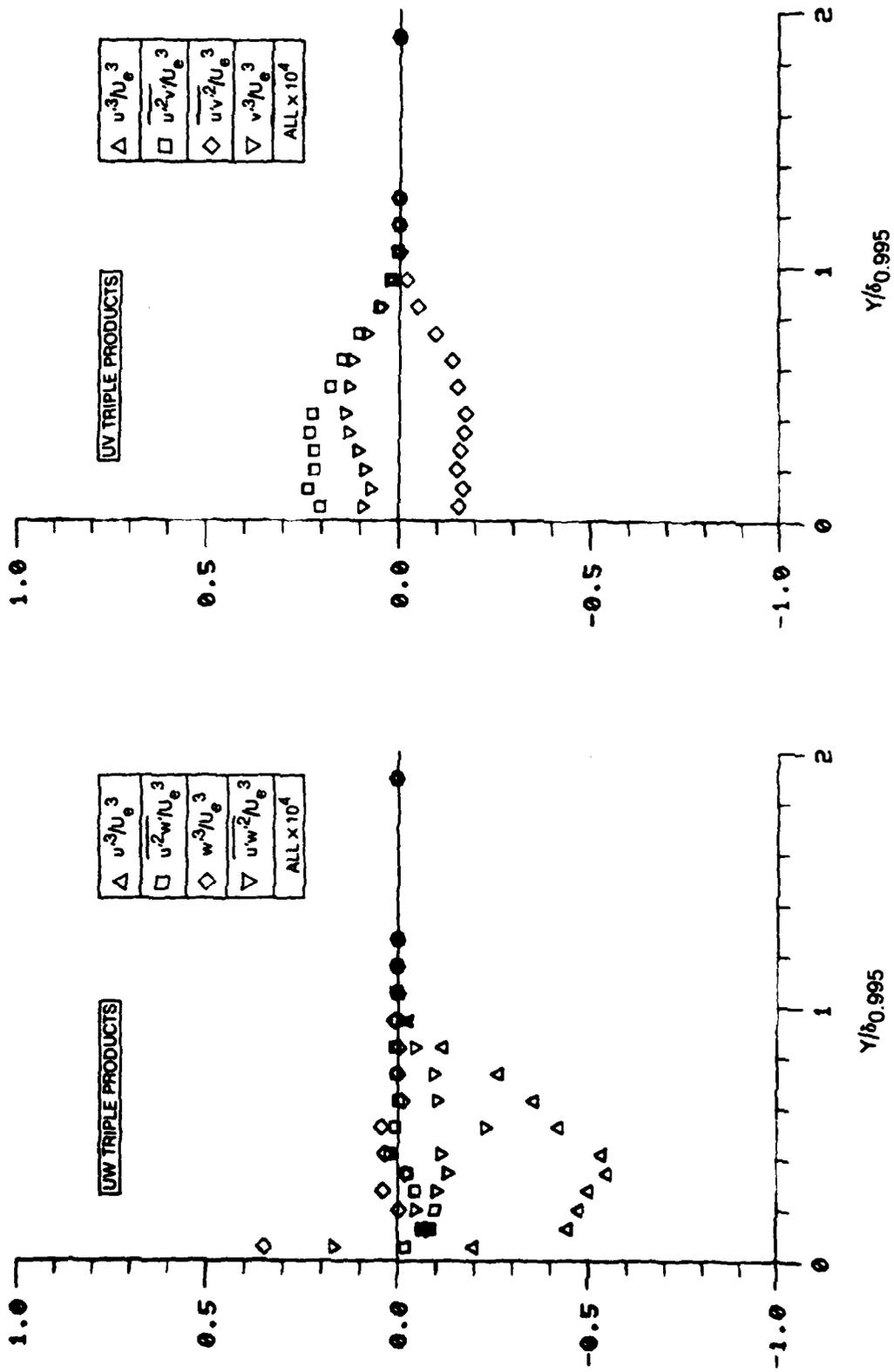


Figure B6C. Boundary Layer Triple Product Distributions $x = 84$ in, $Te = 0.2\%$

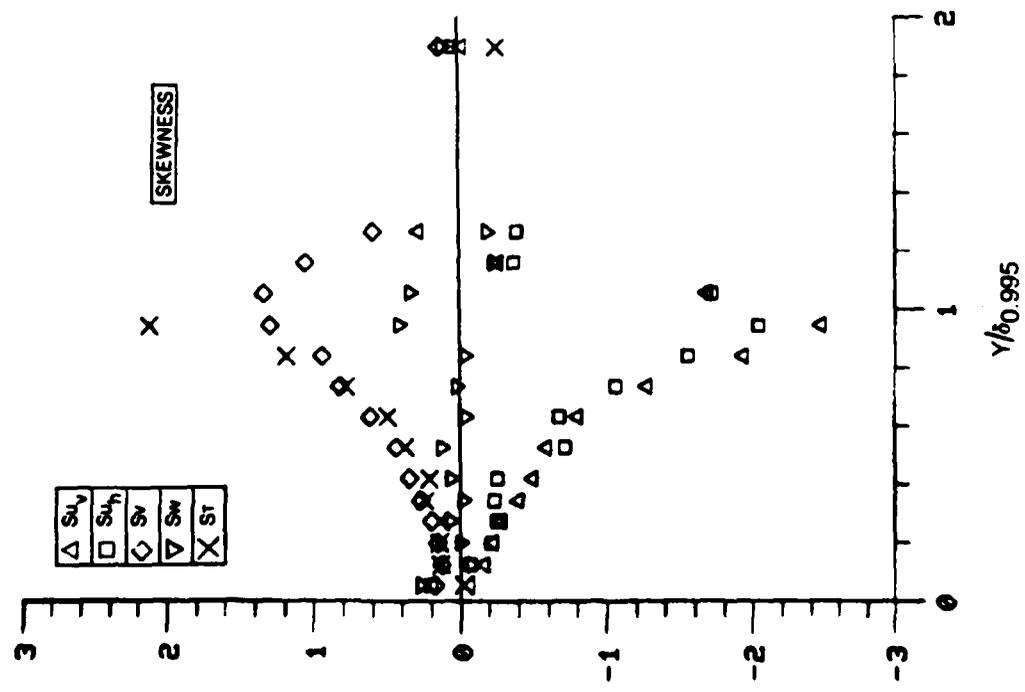
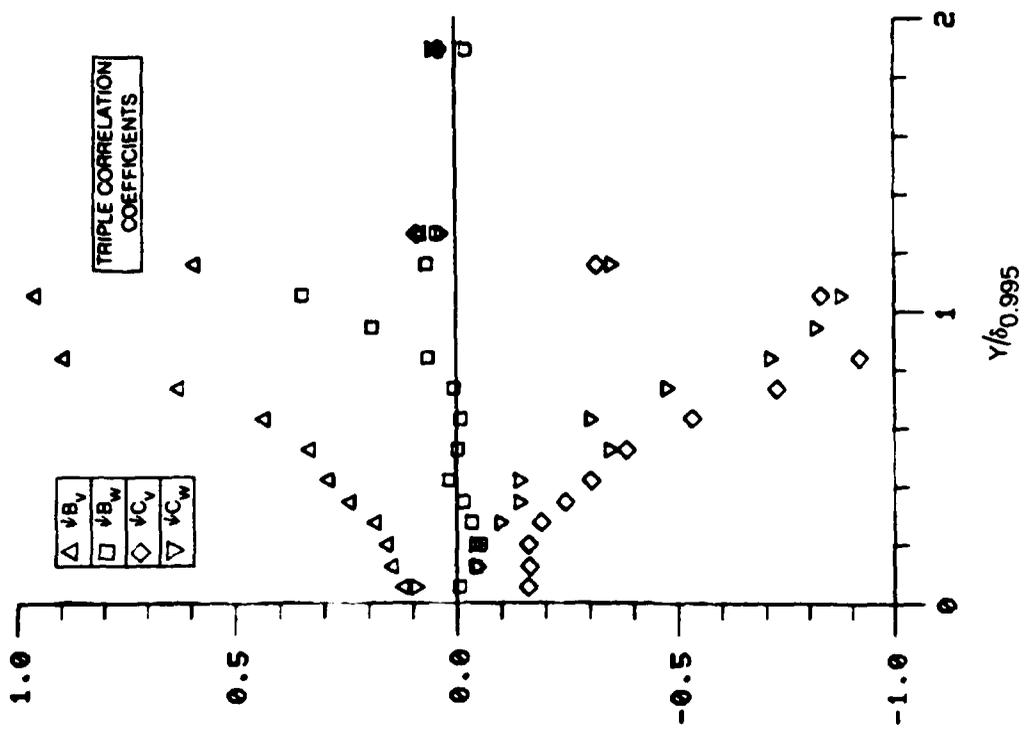


Figure B6D. Boundary Layer Skewness and Triple Product Correlation Coefficient Distributions $x = 84$ in, $Te = 0.2\%$

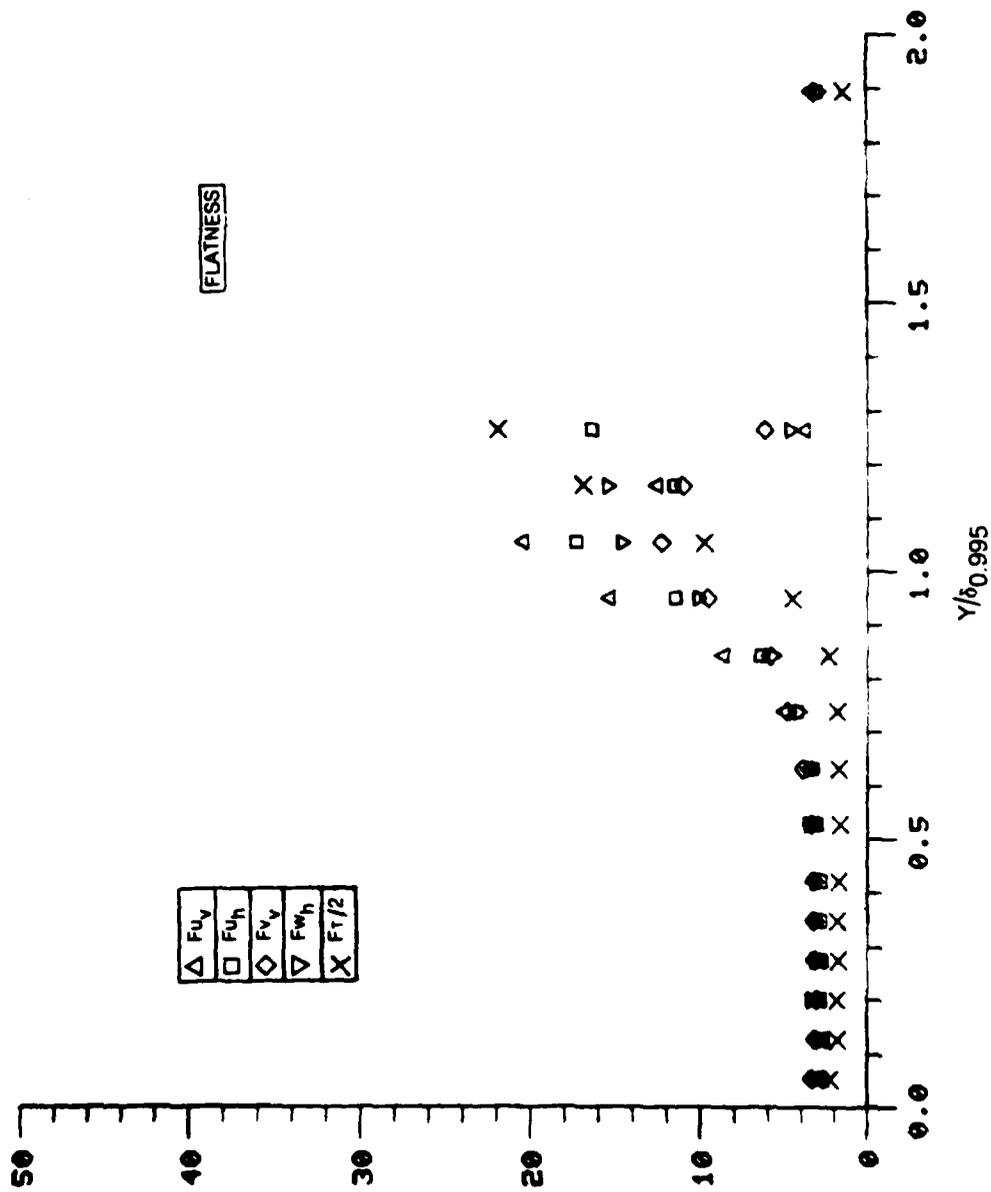


Figure B6E. Boundary Layer Flatness Distributions $x = 84$ in, $Te = 0.2\%$

Fluctuating Profile Data

x = 84 in., Te = 0.2%

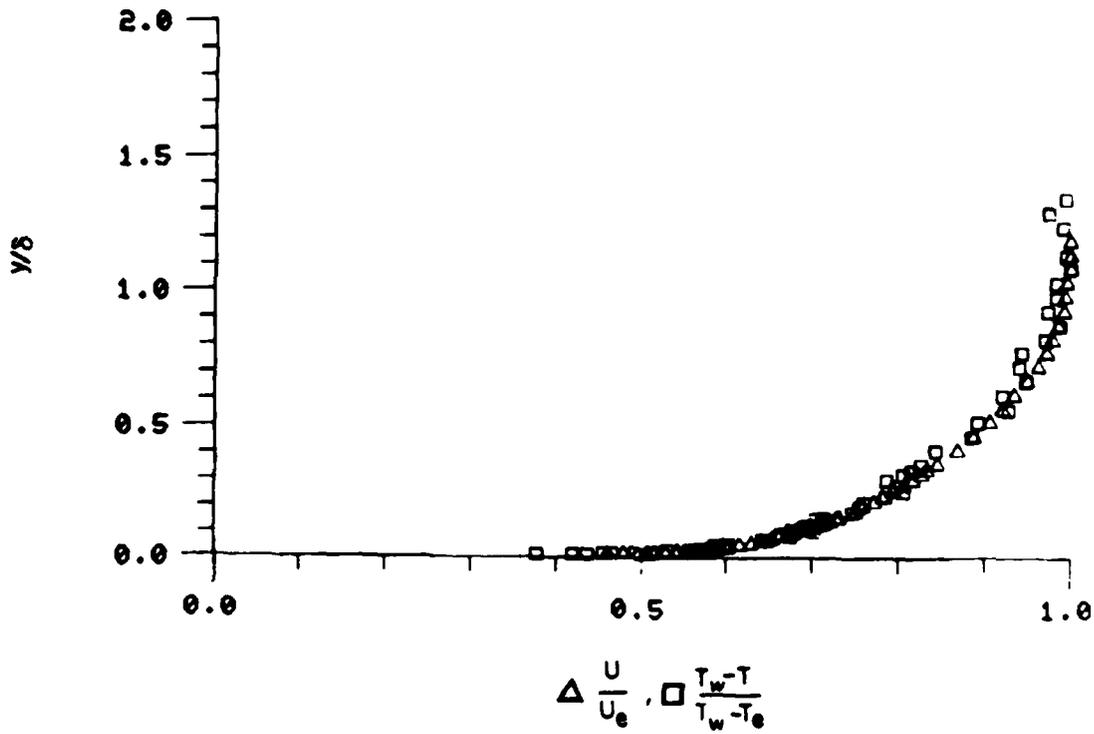
N	Y: INCHES	Y/ DELTA	U'/UE	V'/UE	W'/UE	$\sqrt{U'^2+V'^2}$ /UE	$\frac{U'V'}{UTAU^2}$	$\frac{U'V'}{U'V'}$	C/UE
1	3.00	3.15	.0022	.0017	.0011	.0005	-.0000	-.0514	.0000
2	.00	.15	.0690	.0481	.0507	.0370	.9580	.4416	.0000
3	.12	.30	.0651	.0397	.0450	.0373	.9704	.4716	.0000
4	.24	.45	.0623	.0359	.0436	.0357	.8916	.4620	.0000
5	.36	.60	.0571	.0318	.0421	.0344	.8262	.4702	.0000
6	.48	.75	.0548	.0307	.0396	.0319	.7138	.4636	.0000
7	.60	.90	.0505	.0274	.0381	.0299	.6279	.4715	.0000
8	.72	1.05	.0452	.0242	.0359	.0260	.4746	.4566	.0000
9	.84	1.20	.0377	.0219	.0294	.0220	.3408	.4442	.0000
10	.96	1.35	.0301	.0172	.0245	.0167	.1947	.4175	.0000
11	1.08	1.50	.0208	.0116	.0168	.0116	.0037	.3841	.0000
12	1.20	1.65	.0120	.0055	.0110	.0066	.0037	.3257	.0000
13	1.32	1.80	.0071	.0028	.0071	.0034	.0074	.2577	.0000
14	1.44	1.95	.0046	.0018	.0039	.0017	.0027	.2054	.0000
15	1.56	2.10	.0038	.0014	.0026	.0006	.0000	.1655	.0000
16	1.68	2.25	.0026	.0009	.0019	.0006	.0000	.1056	.0000
17	1.80	2.40	.0022	.0006	.0018	.0005	.0000	.0597	.0000
18	1.92	2.55	.0000	.0000	.0000	.0000	.0000	.0000	.0000

N	Y: INCHES	Y/ DELTA	$\frac{V'^2}{U'^2}$	$\frac{U'^2}{G^2}$	$\frac{V'^2}{G^2}$	$\frac{W'^2}{G^2}$	$\frac{VTAU}{UTAU}$	$\frac{VC}{UTAU}$	$\frac{V'T^2+U'^2+V'^2}{G^2}$
1	3.00	3.15	-.0265	.5492	.3245	.1263	-.0311	.0020	.0016
2	.00	.15	.1355	.5416	.1654	.2930	.3702	.1326	.0000
3	.12	.30	.1780	.5407	.2013	.2580	.3666	.1460	.0000
4	.24	.45	.1724	.5271	.2156	.2573	.3712	.1617	.0000
5	.36	.60	.1807	.5196	.2210	.2707	.4157	.1864	.0000
6	.48	.75	.1713	.5054	.2300	.2646	.5150	.2431	.0000
7	.60	.90	.1721	.4901	.2306	.2743	.5926	.2820	.0000
8	.72	1.05	.1571	.4749	.2257	.2994	.6944	.3030	.0000
9	.84	1.20	.1607	.4653	.2248	.2859	.8629	.3578	.0000
10	.96	1.35	.1395	.4559	.2246	.2024	1.0112	.4046	.0000
11	1.08	1.50	.1321	.4280	.2223	.2757	1.0841	.4139	.0000
12	1.20	1.65	.1033	.4227	.3717	.2856	1.2316	.3923	.0000
13	1.32	1.80	.0661	.4296	.4153	.2909	.9351	.2661	.0000
14	1.44	1.95	.0409	.3889	.4586	.2125	.4592	.1336	.0000
15	1.56	2.10	.0079	.3665	.4566	.1766	.7596	.0414	.0000
16	1.68	2.25	.0051	.3007	.2992	.1801	.0566	.0036	.0000
17	1.80	2.40	.0052	.2485	.2485	.3016	.0117	.0001	.0000
18	1.92	2.55	.0000	.0000	.0000	.0000	.0000	.0000	.0000

N	Y: INCHES	Y/ DELTA	$\frac{V'T^2}{V'^2}$	T'/TAU	T'/(TW-TE)	A16	G16	PRT
1	3.00	3.15	.0833	.1002	.0045	.6516	-.0023	.6734
2	.00	.15	.5444	.6271	.0915	.8264	.0244	.0000
3	.12	.30	.5201	.6271	.0722	.7012	.0545	.0000
4	.24	.45	.5298	.6066	.0650	.7377	.0612	.0000
5	.36	.60	.5405	.5306	.0590	.7617	.0484	.7302
6	.48	.75	.5415	.4244	.0547	.7871	.0585	.7829
7	.60	.90	.5452	.3241	.0512	.8035	.0639	.7776
8	.72	1.05	.5670	.2261	.0455	.8787	.0968	.7701
9	.84	1.20	.5704	.1338	.0405	.9054	.1267	.7215
10	.96	1.35	.5756	.0838	.0358	1.0112	.1536	.6880
11	1.08	1.50	.5592	.0478	.0298	1.1645	.1795	.6320
12	1.20	1.65	.5472	.0297	.0219	1.5421	.2305	.4773
13	1.32	1.80	.5268	.0137	.0137	1.8759	.2157	.3478
14	1.44	1.95	.4262	.0081	.0081	1.7551	.1430	.2735
15	1.56	2.10	.3264	.0057	.0057	1.0844	.0823	.2211
16	1.68	2.25	.0471	.0048	.0048	.2512	.0000	.0605
17	1.80	2.40	.0380	.0043	.0043	.0000	.0000	.0000
18	1.92	2.55	.0000	.0000	.0000	.0000	.0000	.0000

Table B9A

VELOCITY AND TEMPERATURE RATIOS



VELOCITY AND TEMPERATURE DISTRIBUTIONS IN UNIVERSAL COORDINATES

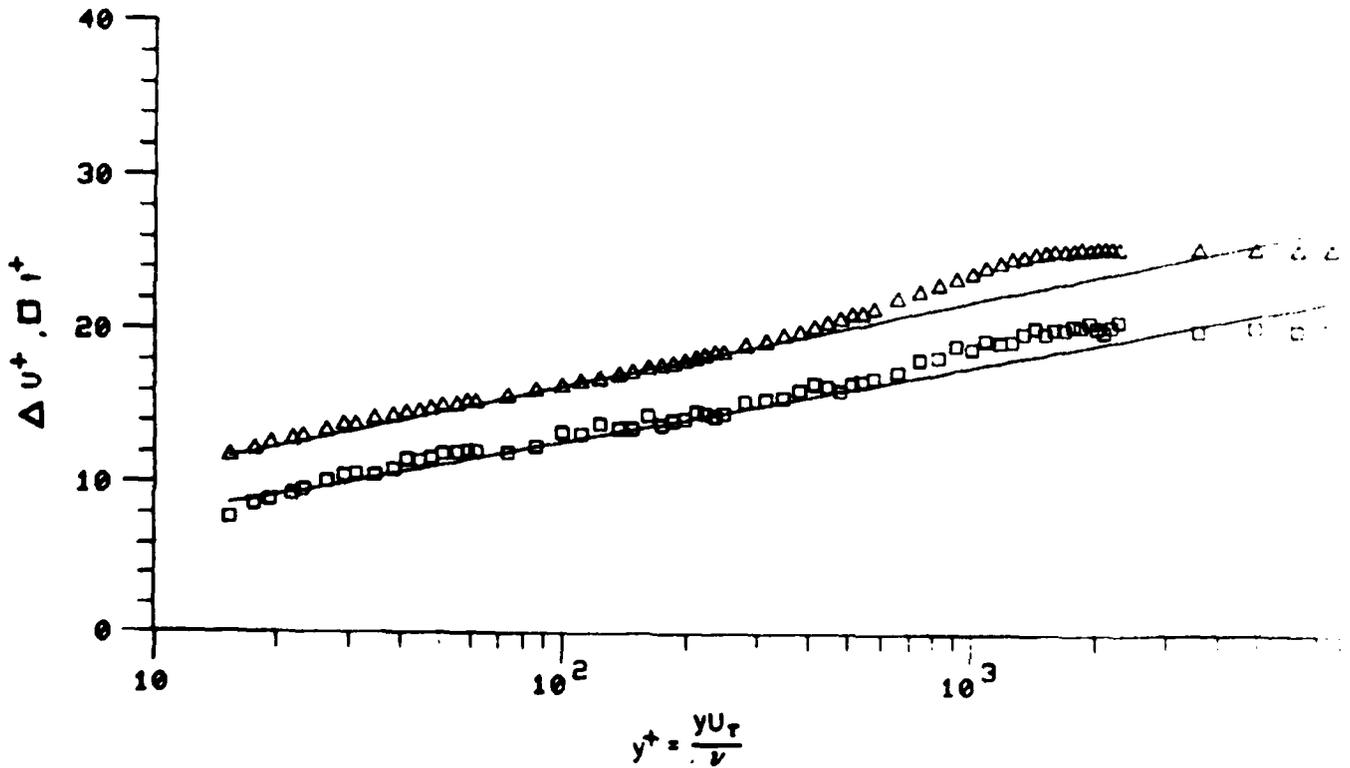


Figure B7. Mean Profile and Temperature Profiles $x = 52$ in., $Te = 1.8\%$

RUN NO. 3. POINT 3.

BOUNDARY LAYER PROPERTIES

	LINEAR INTERPOLATION TO WALL	STANDARD SUBLAYER FUNCTION FROM WALL TO $Y^+ = 35$
FREE STREAM VELOCITY	98.101	98.101
FREE STREAM TEMPERATURE	70.960	
WALL TEMPERATURE	94.980	
WALL HEAT FLUX	.07908	
FREE STREAM DENSITY	.07541	
FREE STREAM KINEMATIC VISCOSITY	.0001623	
DENSITY OF FLUID AT WALL	.07214	
KINEMATIC VISCOSITY OF FLUID AT WALL	.0001756	
WALL/FREE STREAM DENSITY RATIO	.95669	
LOCATION REYNOLDS NUMBER (REX)	2618495.47	
INPUT VALUE OF VELOCITY DELTA	1.00000	
INPUT VALUE OF TEMPERATURE DELTA	1.00000	
CALCULATED DELTA		.86723
DELTA 99.5% INPUT	.00000	
DISPLACEMENT THICKNESS (DELSTAR)	.12408	.12411
MOMENTUM THICKNESS (THETA)	.08716	.08756
ENERGY-DISSIPATION THICKNESS	.15548	.15581
ENTHALPY THICKNESS	.00436	.00437
SHAPE FACTOR 12 (DELSTAR/THETA)	1.42349	1.41709
SHAPE FACTOR 32 (ENERGY/THETA)	1.78378	1.77907
MOMENTUM THICKNESS REYNOLDS NUMBER	4389.16	4410.12
DISPLACEMENT THICKNESS REYNOLDS NUMBER	6247.94	6249.55
SKIN FRICTION COEFFICIENT	.002998	
FRICTION VELOCITY	3.86306	
LA _w OF THE WALL CONSTANT (K)	.41000	
LA _w OF THE WALL CONSTANT (C)	5.00000	
WAKE STRENGTH		.46566
CLAUSERS 'DELTA' INTEGRAL	-2.81396	-3.02501
CLAUSERS 'G' INTEGRAL	20.20020	19.93622
DISPLACEMENT THICKNESS - CONSTANT DENSITY	.11555	.11974
MOMENTUM THICKNESS - CONSTANT DENSITY	.08806	.08850
SHAPE FACTOR 12 - CONSTANT DENSITY	1.31209	1.35299

LOCATION -X- 52.00000

Te = 1.8%

Table B11

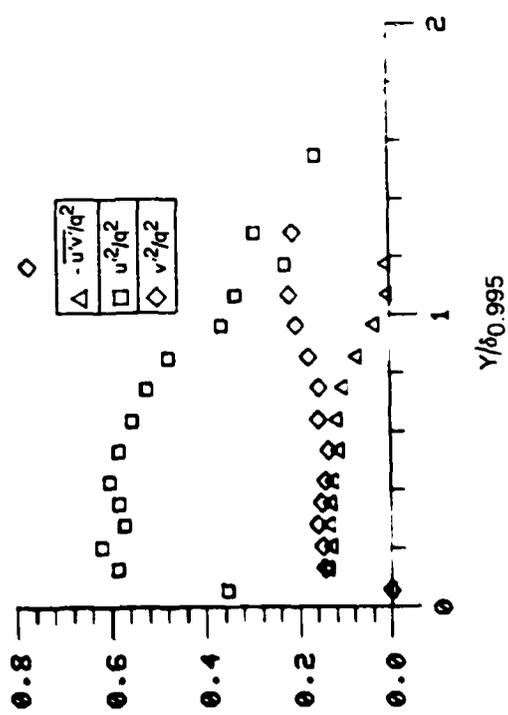
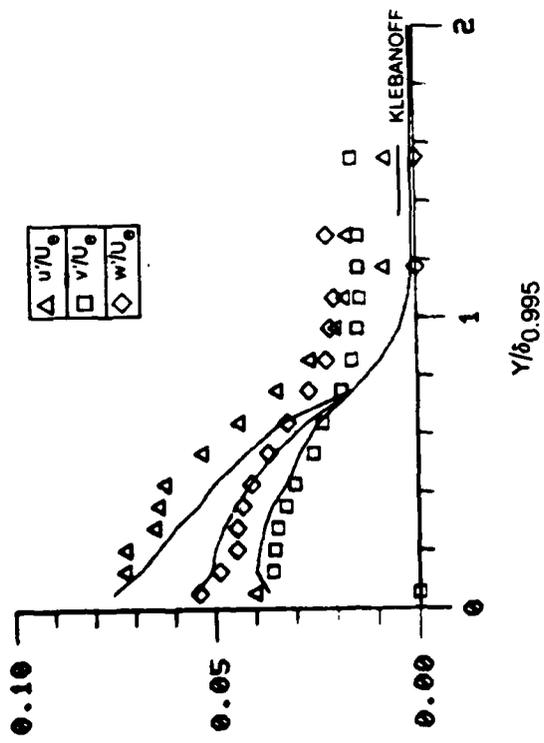
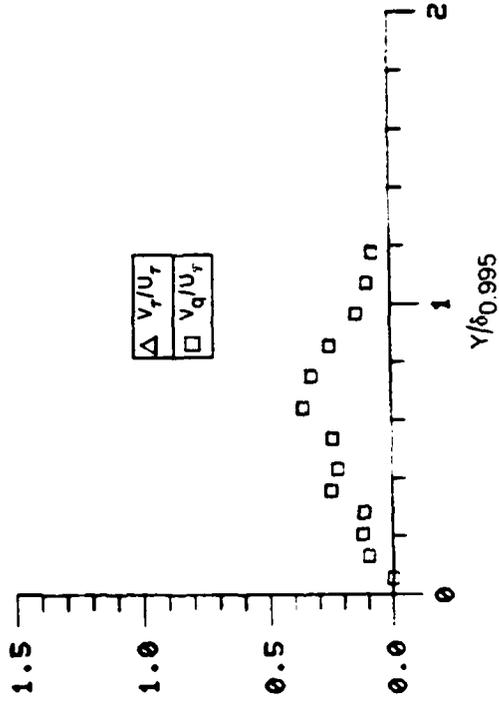
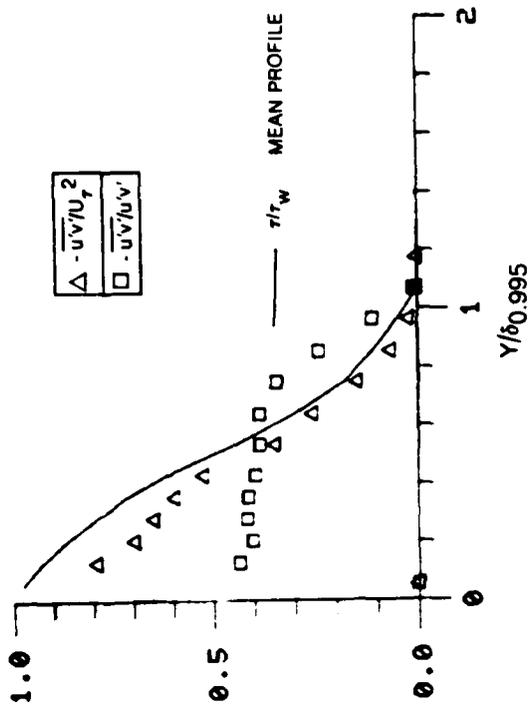


Figure B8A. Boundary Layer Turbulence Quantities $x = 52$ in, $T_e = 1.8\%$

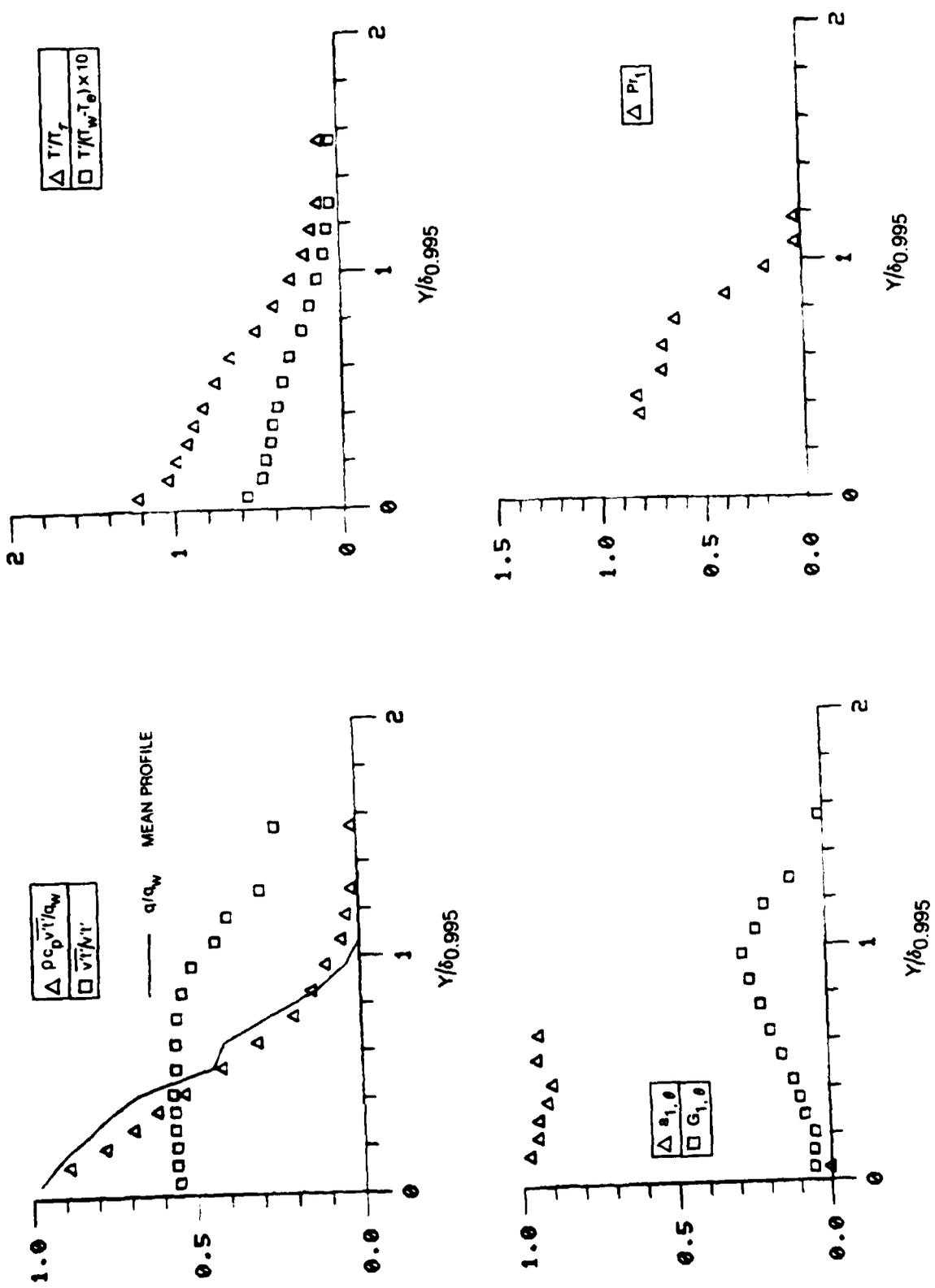


Figure B8B. Boundary Layer Turbulence Quantities $x = 52$ in, $Te = 1.8\%$

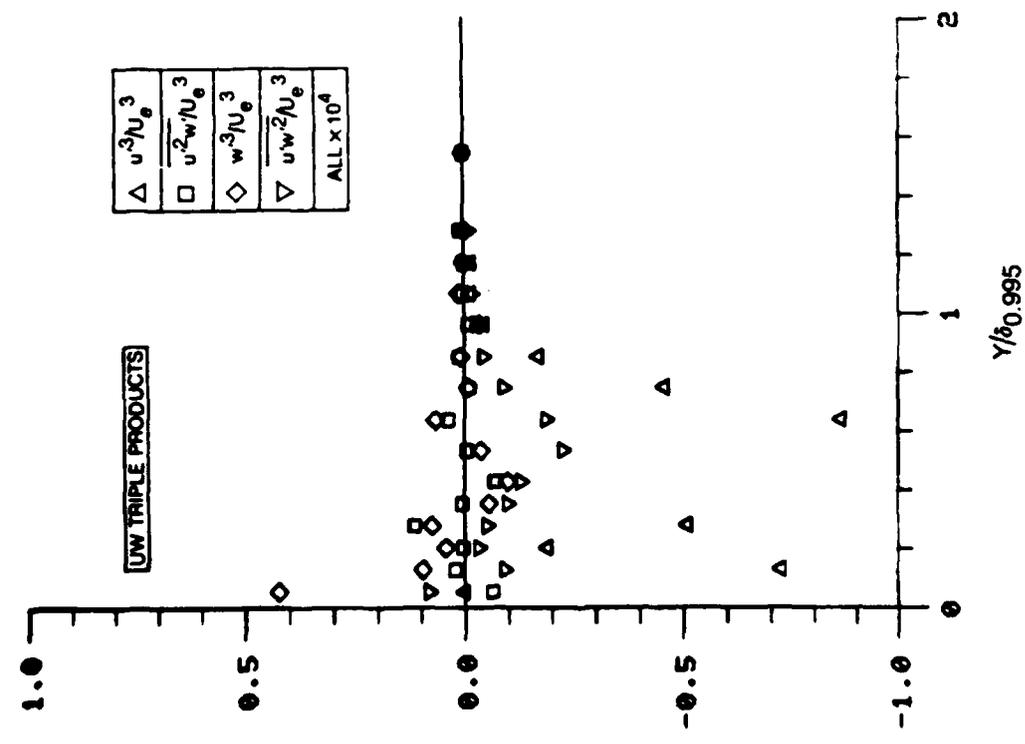
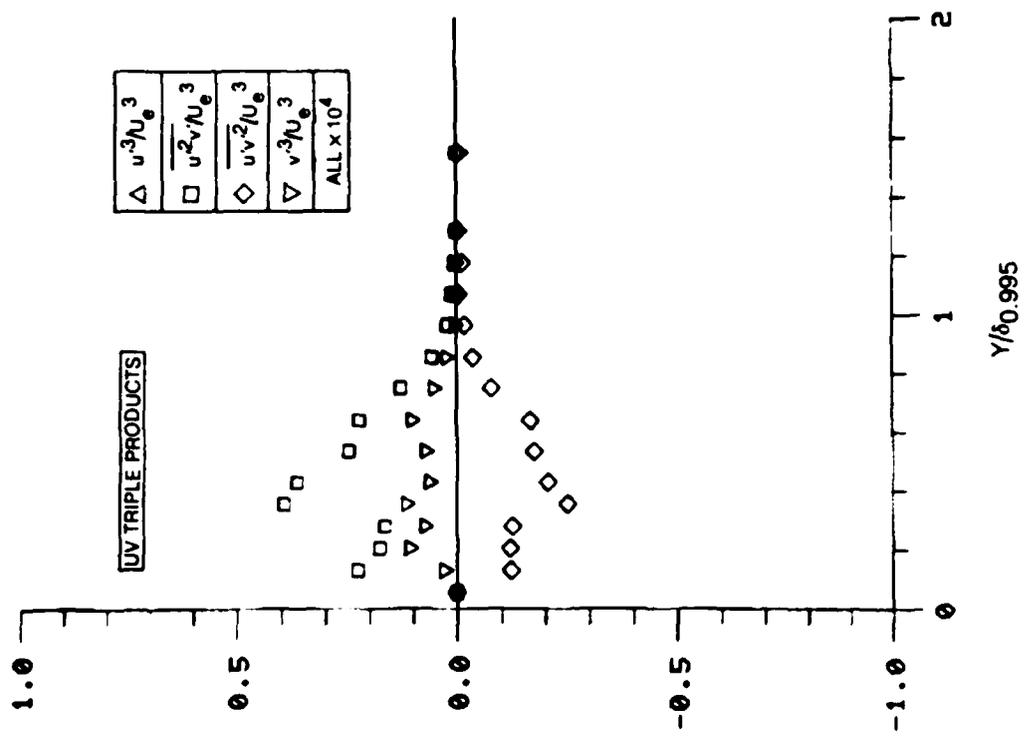


Figure B8C. Boundary Layer Triple Product Distributions $x = 52$ in, $T_e = 1.8\%$

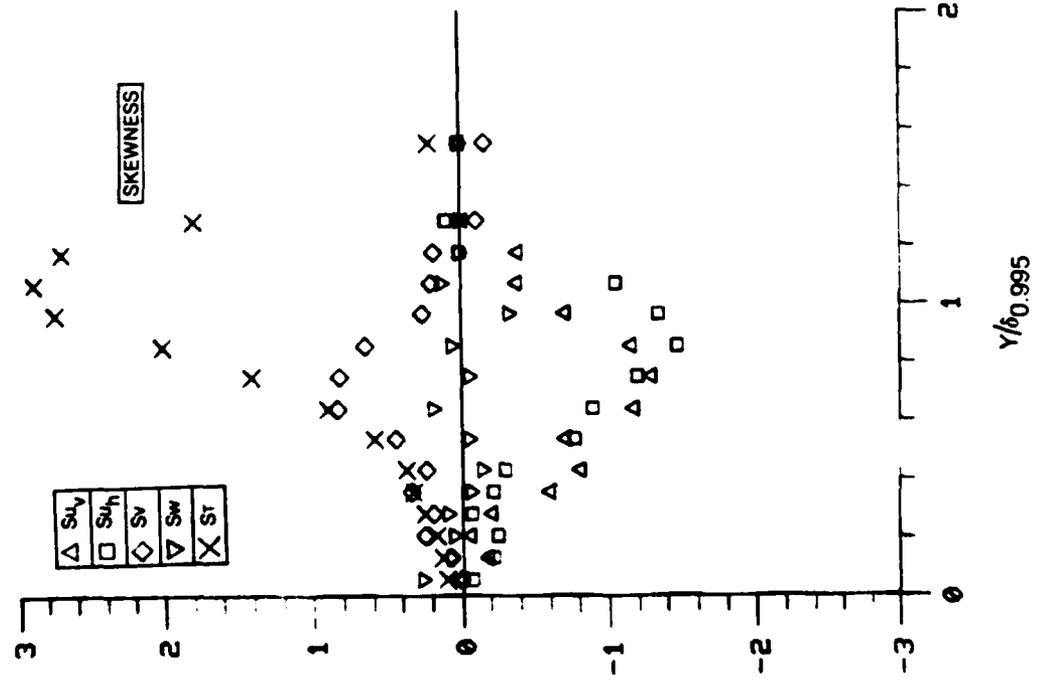
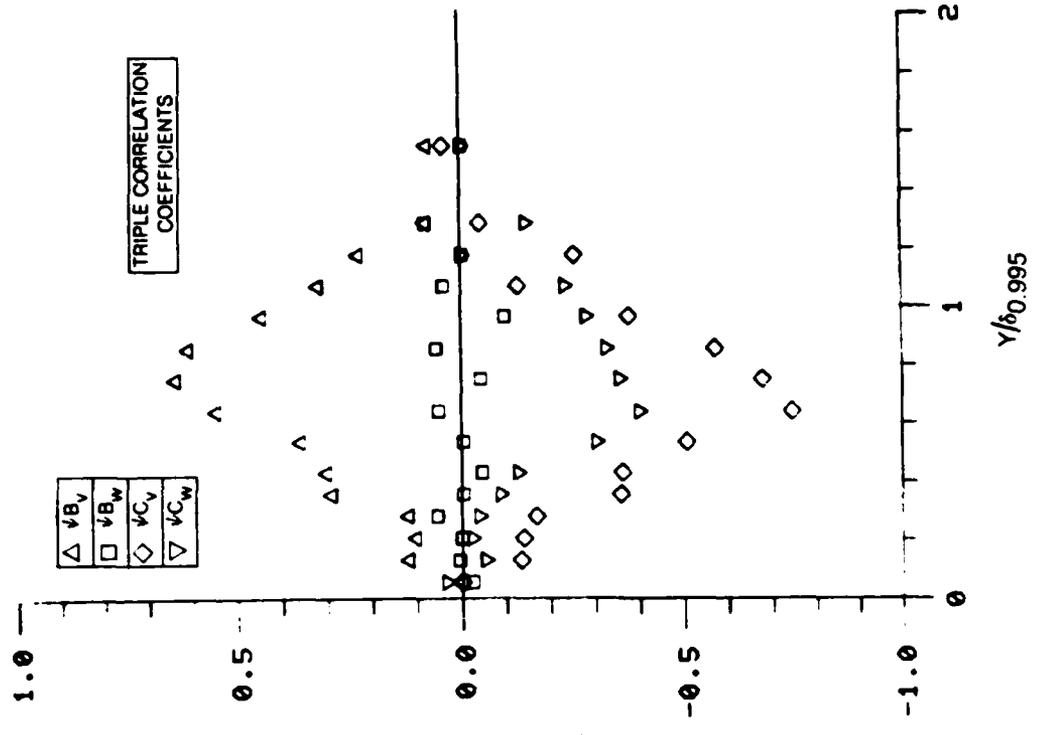


Figure B8D. Boundary Layer Skewness and Triple Product Correlation Coefficient Distributions $x = 52$ in, $T_e = 1.87$

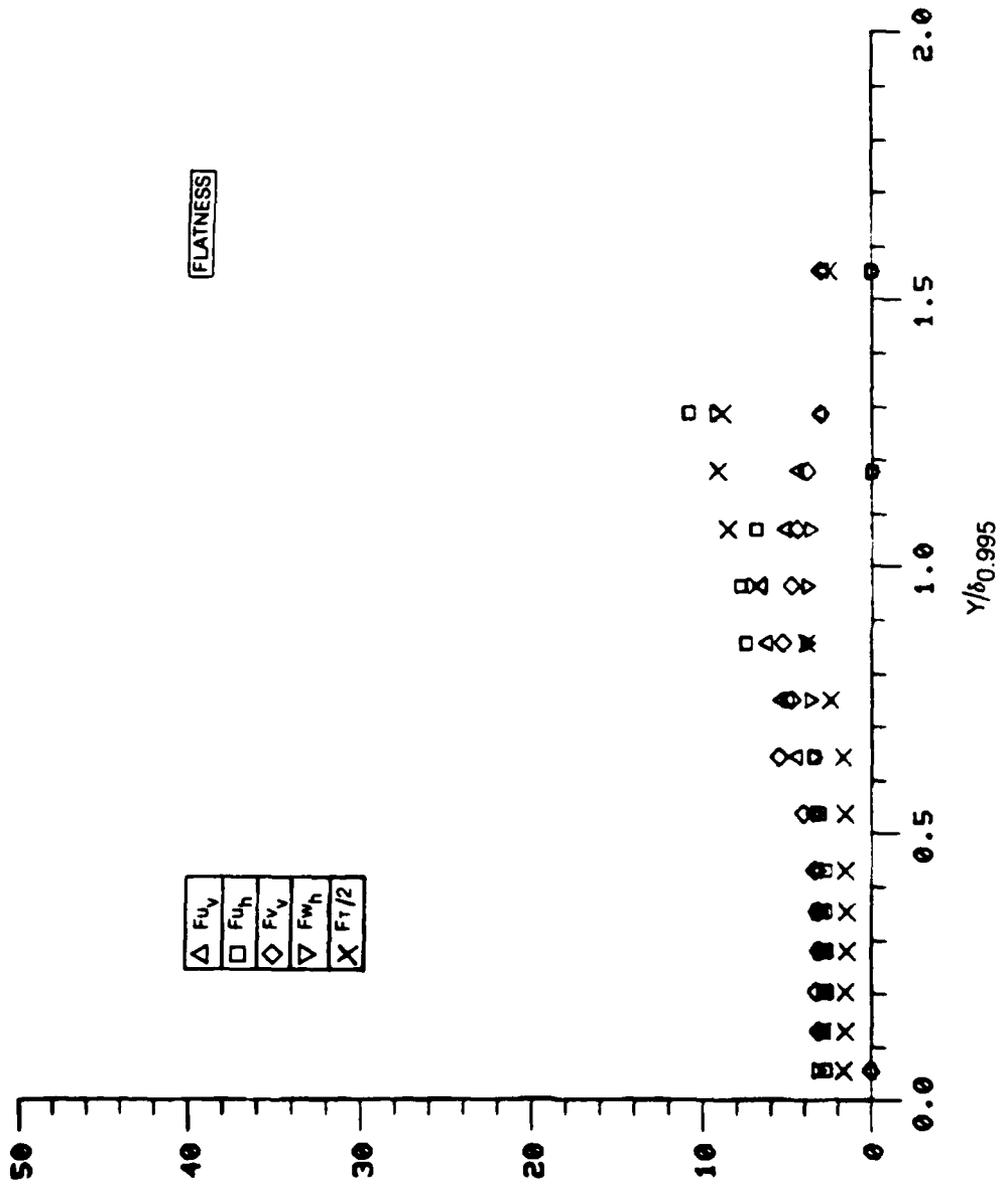


Figure B8E. Boundary Layer Flatness Distributions $x = 52$ in, $T_e = 1.8\%$

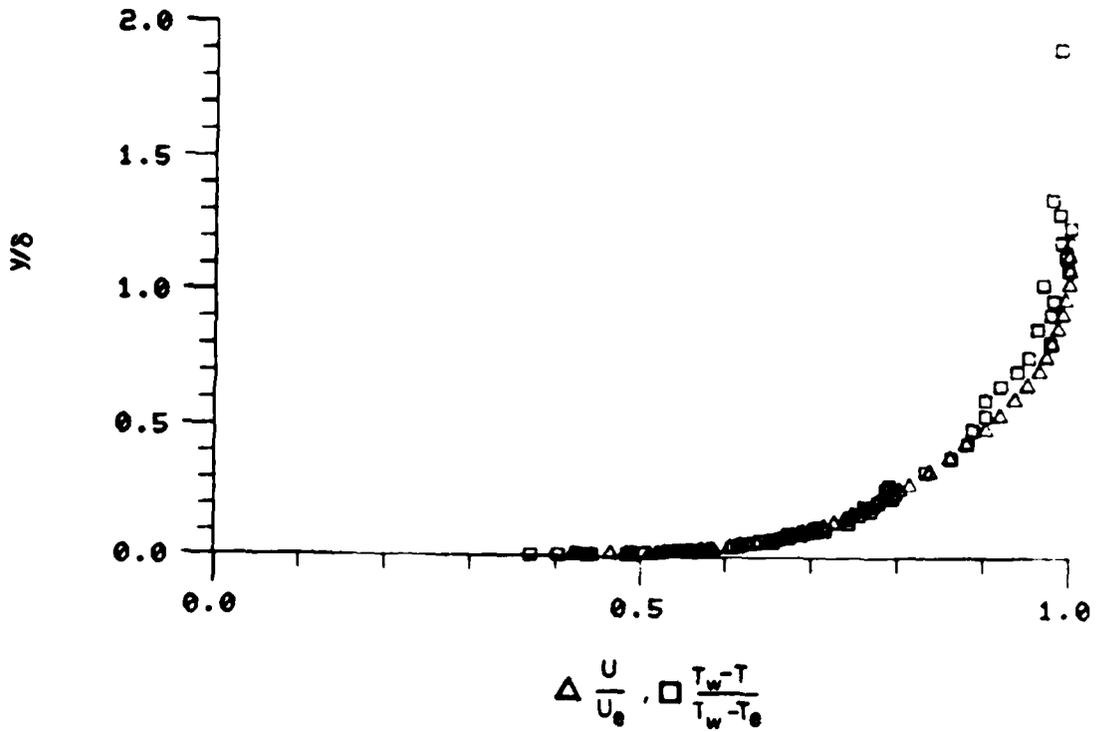
Fluctuating Profile Data

$x = 52 \text{ in.}, Te = 1.8\%$

INCHES	DELTA	V*/UE	V*/UE	V*/UE	V*/UE	V*/UE	V*/UE
0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.05	0.05	0.05	0.05	0.05	0.05	0.05	0.05
0.10	0.10	0.10	0.10	0.10	0.10	0.10	0.10
0.15	0.15	0.15	0.15	0.15	0.15	0.15	0.15
0.20	0.20	0.20	0.20	0.20	0.20	0.20	0.20
0.25	0.25	0.25	0.25	0.25	0.25	0.25	0.25
0.30	0.30	0.30	0.30	0.30	0.30	0.30	0.30
0.35	0.35	0.35	0.35	0.35	0.35	0.35	0.35
0.40	0.40	0.40	0.40	0.40	0.40	0.40	0.40
0.45	0.45	0.45	0.45	0.45	0.45	0.45	0.45
0.50	0.50	0.50	0.50	0.50	0.50	0.50	0.50
0.55	0.55	0.55	0.55	0.55	0.55	0.55	0.55
0.60	0.60	0.60	0.60	0.60	0.60	0.60	0.60
0.65	0.65	0.65	0.65	0.65	0.65	0.65	0.65
0.70	0.70	0.70	0.70	0.70	0.70	0.70	0.70
0.75	0.75	0.75	0.75	0.75	0.75	0.75	0.75
0.80	0.80	0.80	0.80	0.80	0.80	0.80	0.80
0.85	0.85	0.85	0.85	0.85	0.85	0.85	0.85
0.90	0.90	0.90	0.90	0.90	0.90	0.90	0.90
0.95	0.95	0.95	0.95	0.95	0.95	0.95	0.95
1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00

Table B12A

VELOCITY AND TEMPERATURE RATIOS



VELOCITY AND TEMPERATURE DISTRIBUTIONS IN UNIVERSAL COORDINATES

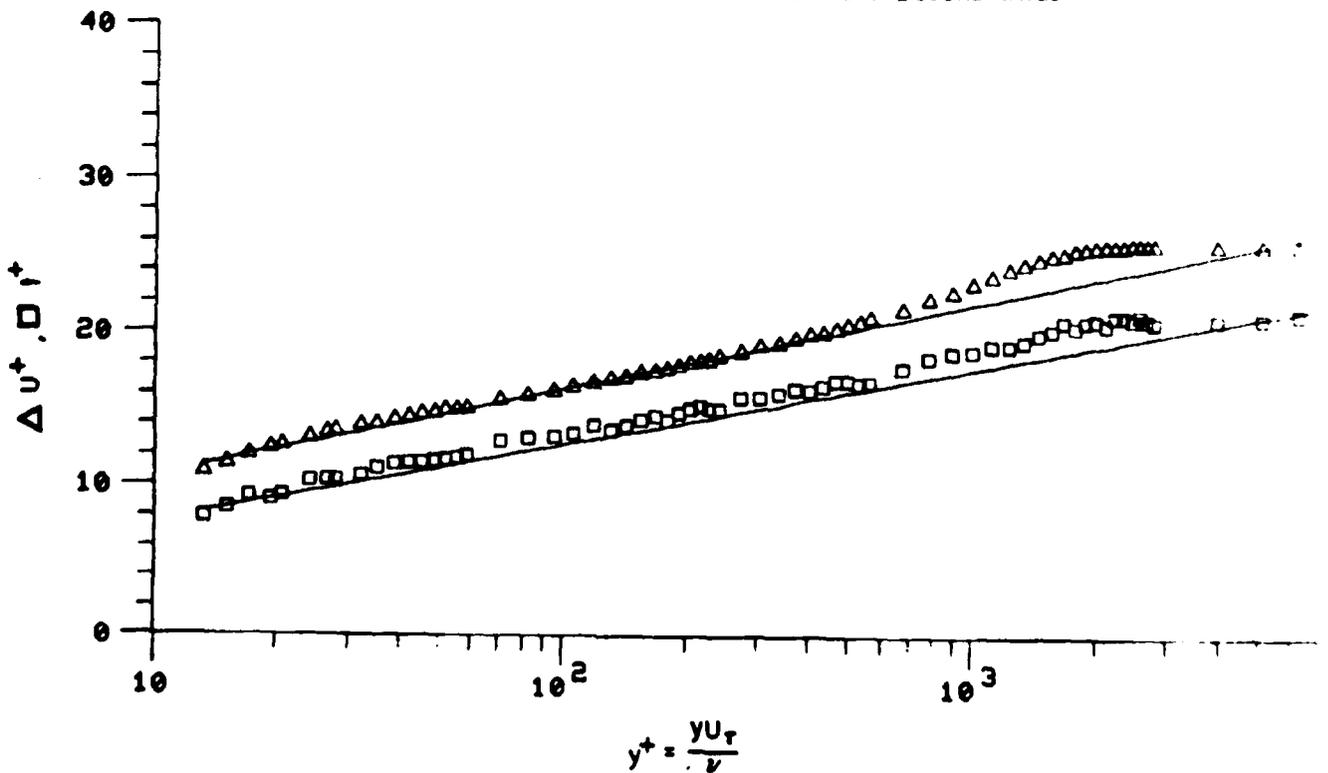


Figure B9. Mean Velocity and Temperature Profiles $x = 68$ in., $Te = 1.6\%$

RUN NO. 3. POINT 2.
 BOUNDARY LAYER PROPERTIES

	LINEAR INTERPOLATION TO WALL	STANDARD SUBLAYER FUNCTION FROM WALL TO $y^+ = 30$
FREE STREAM VELOCITY ==	98.170	98.170
FREE STREAM TEMPERATURE ==	70.180	
WALL TEMPERATURE ==	95.750	
WALL HEAT FLUX ==	.07954	
FREE STREAM DENSITY ==	.07552	
FREE STREAM KINEMATIC VISCOSITY ==	.0001619	
DENSITY OF FLUID AT WALL ==	.07204	
KINEMATIC VISCOSITY OF FLUID AT WALL ==	.0001760	
WALL/FREE STREAM DENSITY RATIO ==	.95396	
LOCATION REYNOLDS NUMBER (REX) ==	3435559.84	
INPUT VALUE OF VELOCITY DELTA ==	1.30000	
INPLT VALUE OF TEMPERATURE DELTA ==	1.30000	
CALCULATED DELTA ==		1.12046
DELTA 99.5% INPUT ==	.00000	
DISPLACEMENT THICKNESS (DELSTAR) ==	.15429	.15443
MOMENTUM THICKNESS (THETA) ==	.10938	.10967
ENERGY-DISSIPATION THICKNESS ==	.19539	.19556
ENTHALPY THICKNESS ==	.00597	.00598
SHAPE FACTOR 12 (DELSTAR/THETA) ==	1.41057	1.40819
SHAPE FACTOR 32 (ENERGY/THETA) ==	1.78634	1.78327
MOMENTUM THICKNESS REYNOLDS NUMBER ==	5526.16	5540.86
DISPLACEMENT THICKNESS REYNOLDS NUMBER ==	7795.01	7802.27
SKIN FRICTION COEFFICIENT ==	.002865	
FRICTION VELOCITY ==	3.81748	
LAW OF THE WALL CONSTANT (K) ==	.41000	
LAW OF THE WALL CONSTANT (C) ==	5.00000	
WAKE STRENGTH ==		.44004
CLAUSERS 'DELTA' INTEGRAL ==	-3.62559	-3.81767
CLAUSERS 'G' INTEGRAL ==	24.97566	24.86843
DISPLACEMENT THICKNESS - CONSTANT DENSITY ==	.14465	.14845
MOMENTUM THICKNESS - CONSTANT DENSITY ==	.11055	.11085
SHAPE FACTOR 12 - CONSTANT DENSITY ==	1.30848	1.33924

LOCATION -X- 68.00000

Te = 1.6%

Table B14

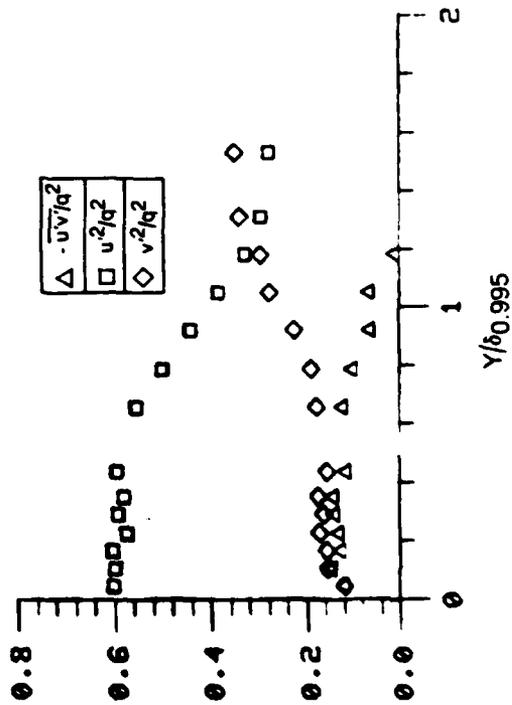
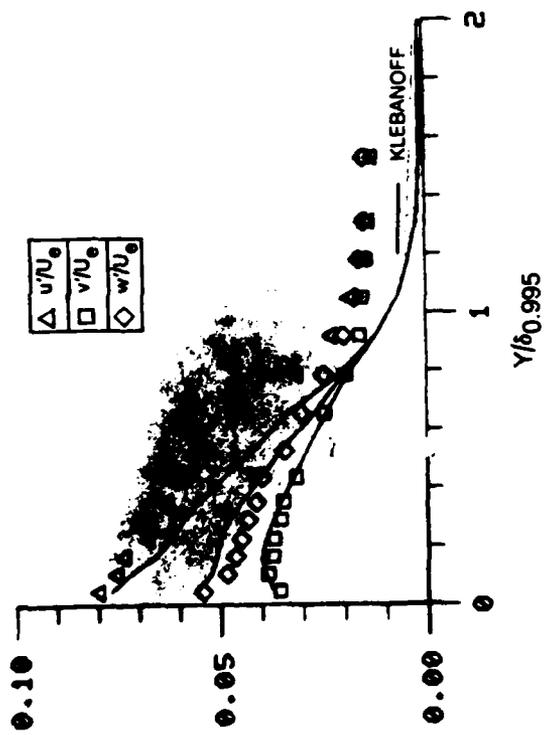
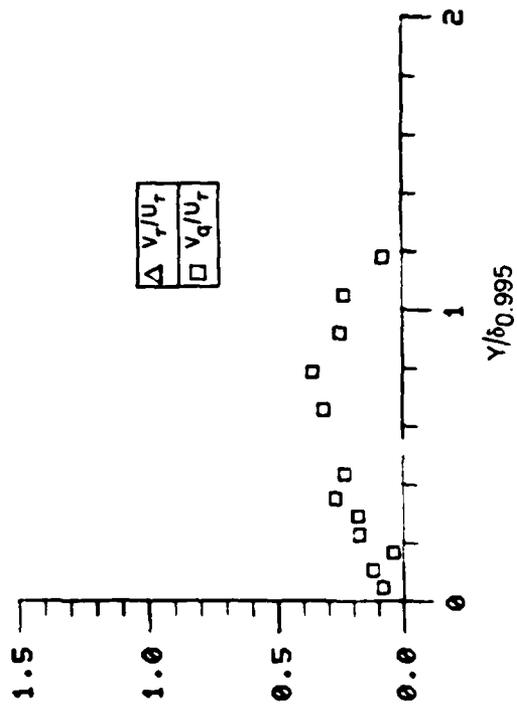
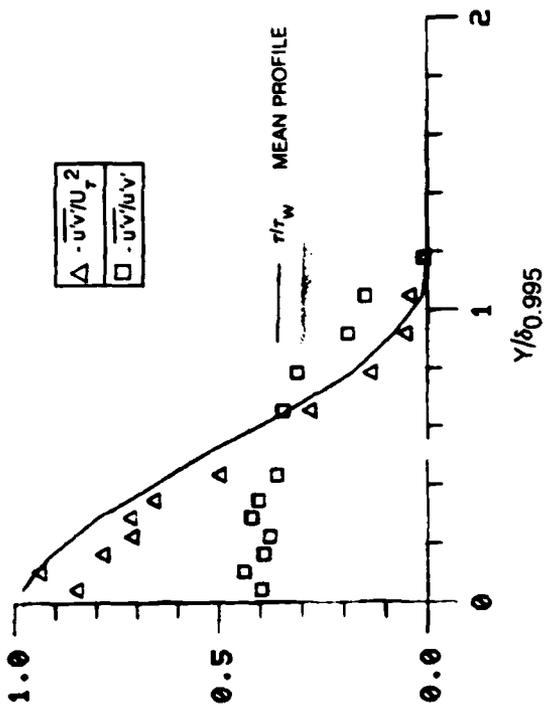


Figure B10A. Boundary Layer Turbulence Quantities $x = 68$ in, $T_e = 1.6\%$

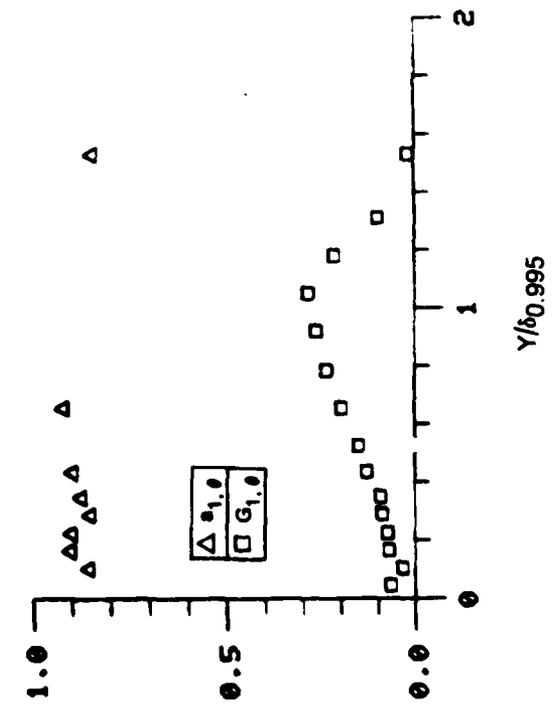
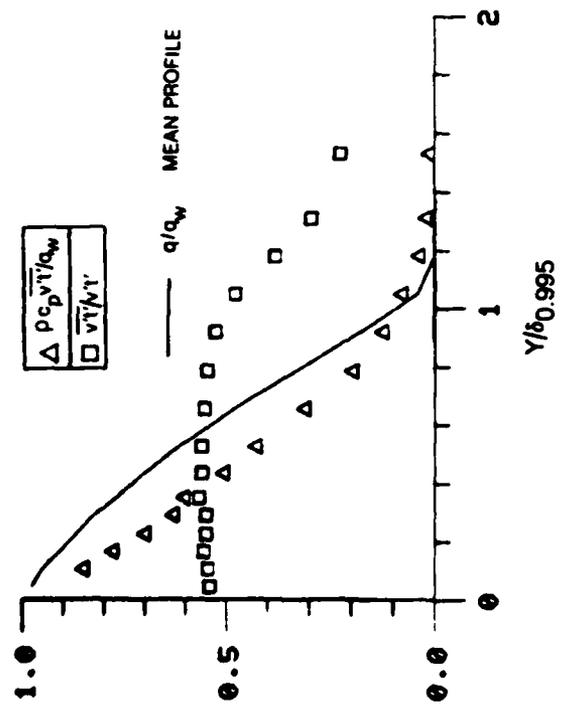
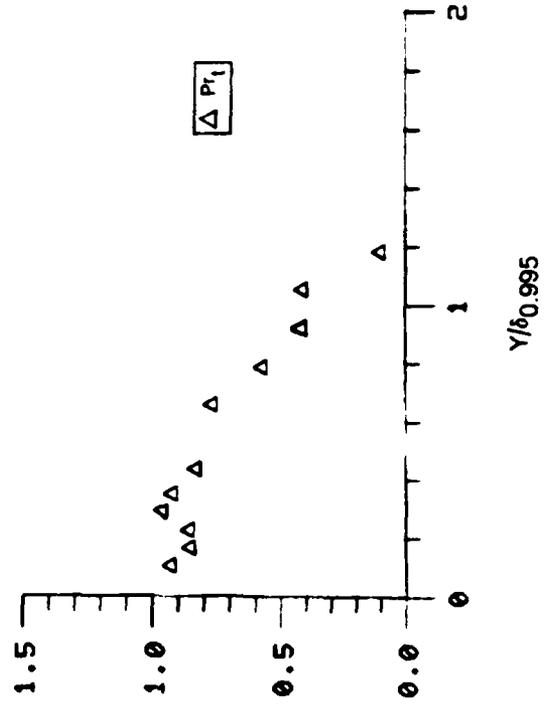
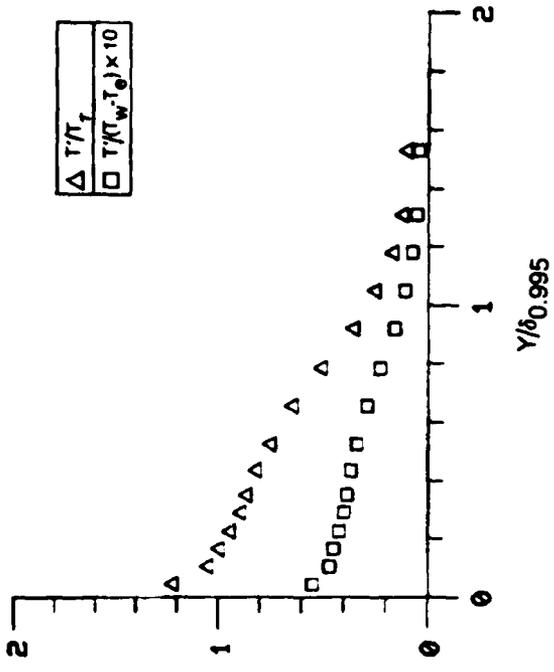


Figure B10B. Boundary Layer Turbulence Quantities $x = 68$ in, $T_e = 1.67$

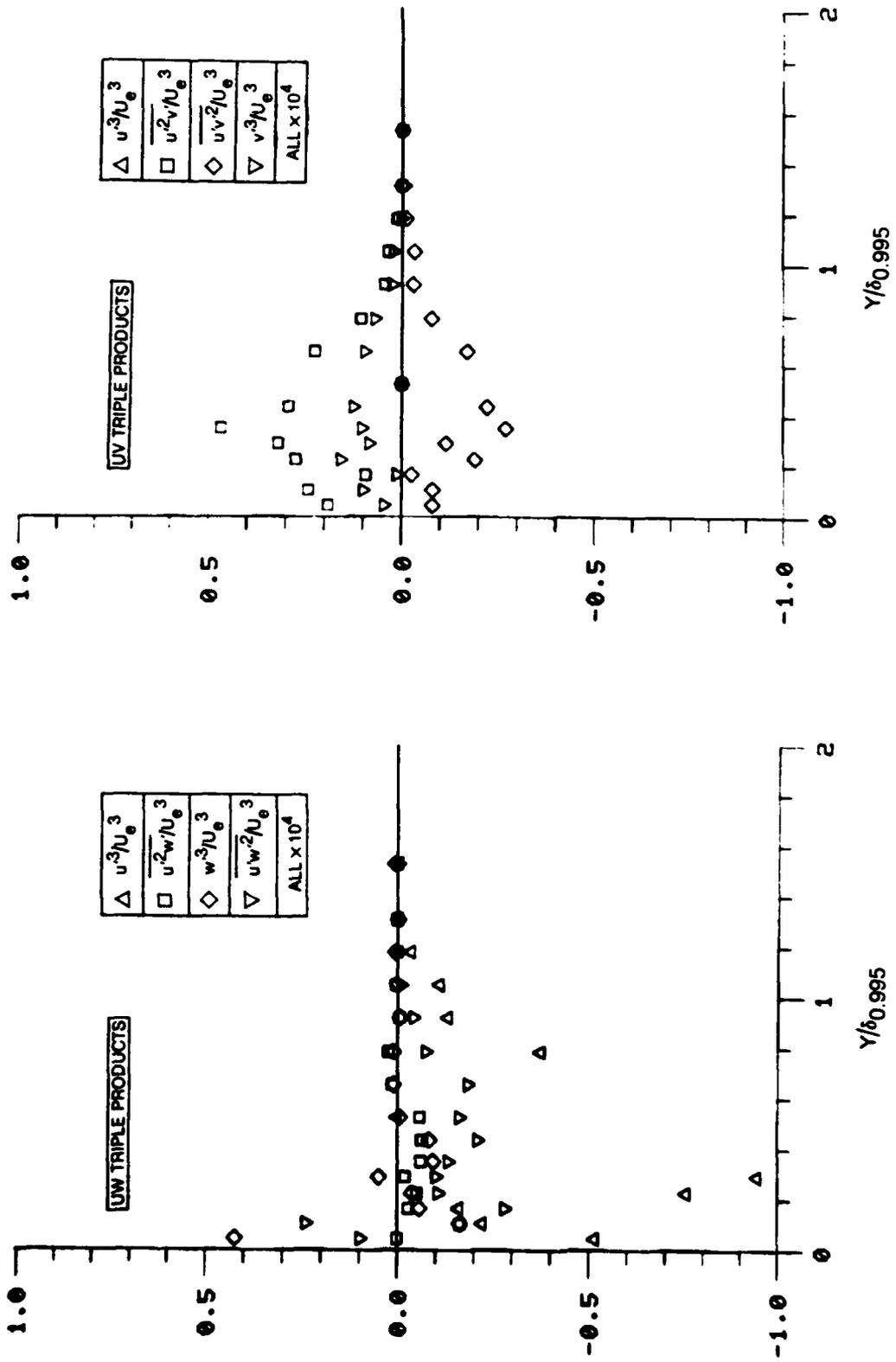


Figure B10C. Boundary Layer Triple Product Distributions $x = 68$ in, $T_e = 1.6\%$

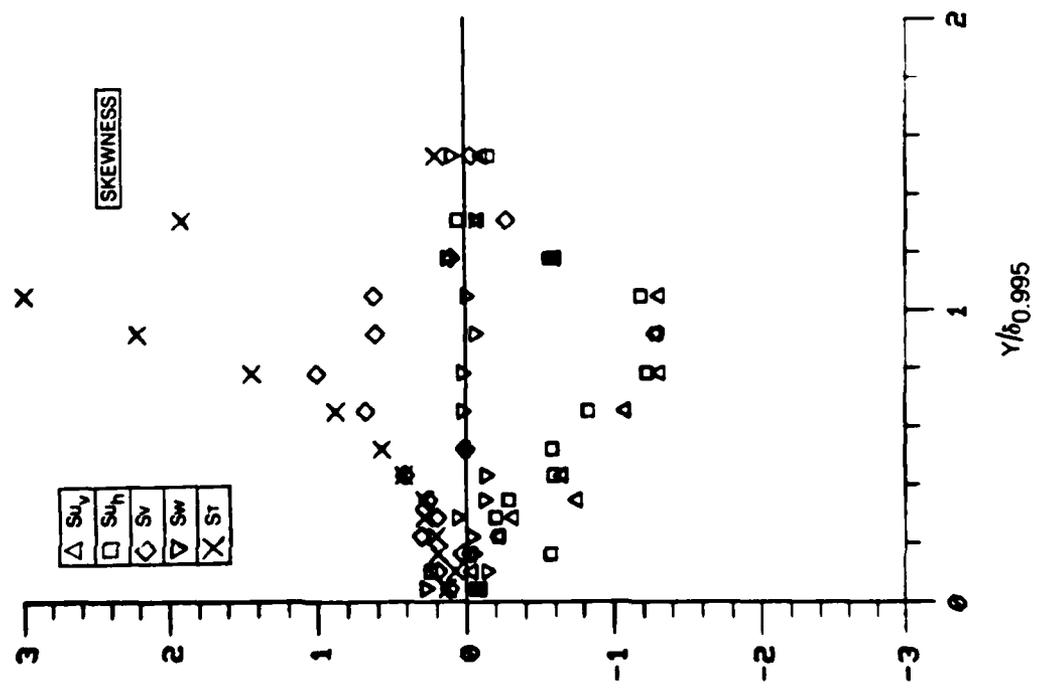
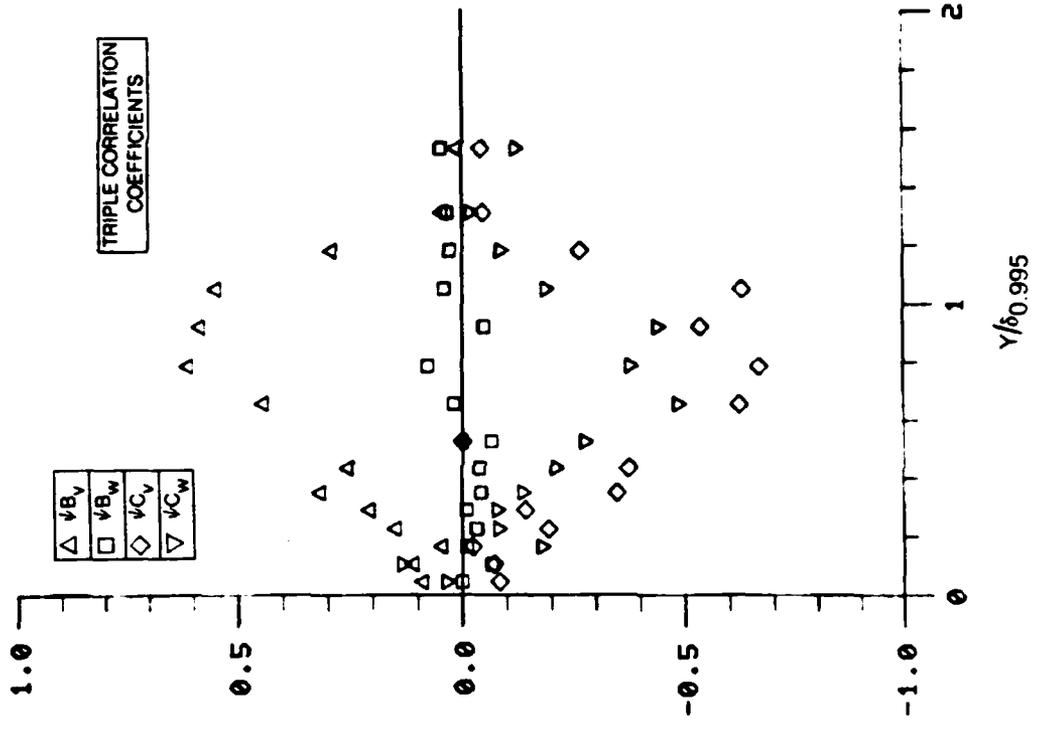


Figure B10D. Boundary Layer Skewness and Triple Product Correlation Coefficient Distributions $x = 68$ in, $Te = 1.6\%$.

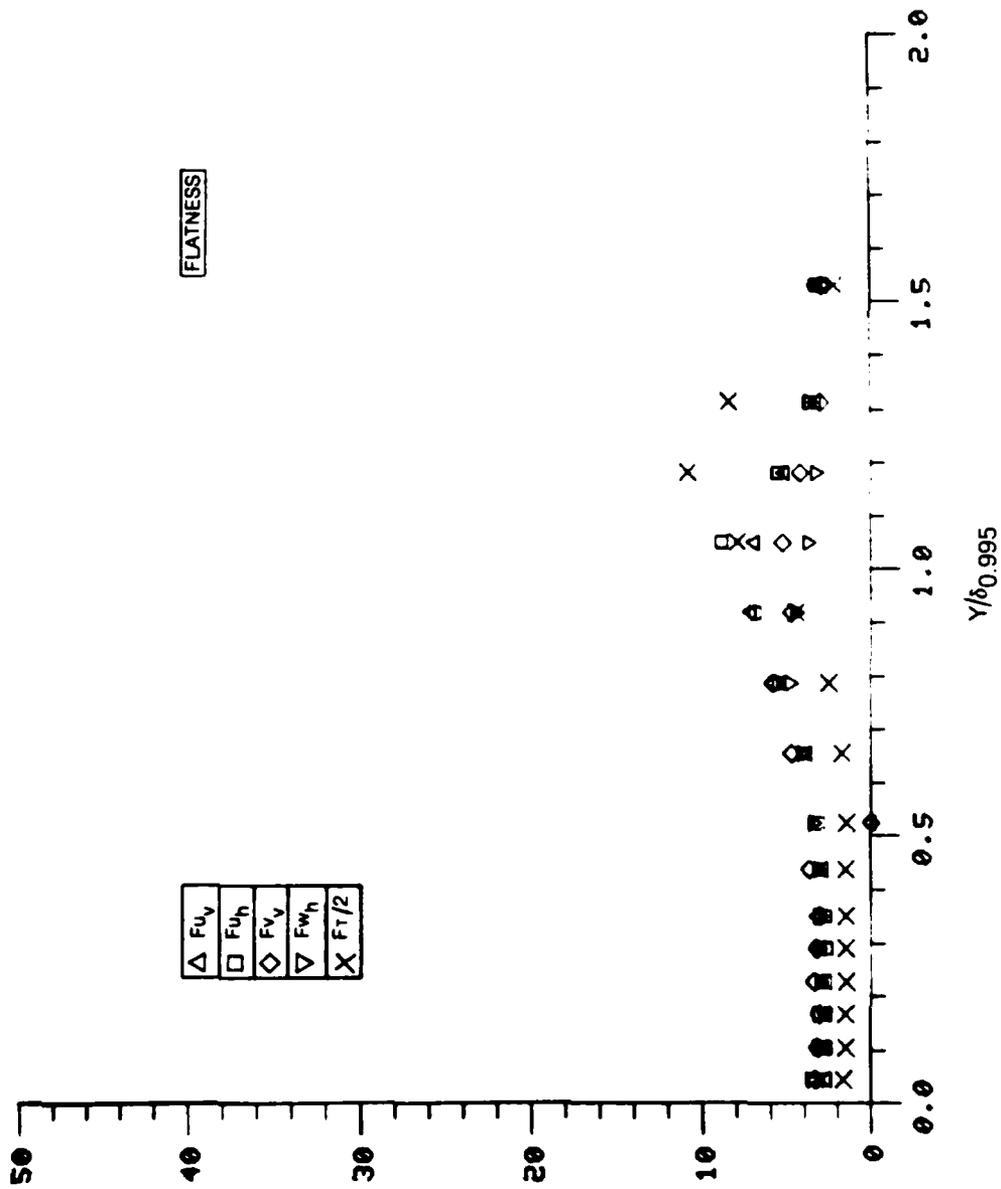
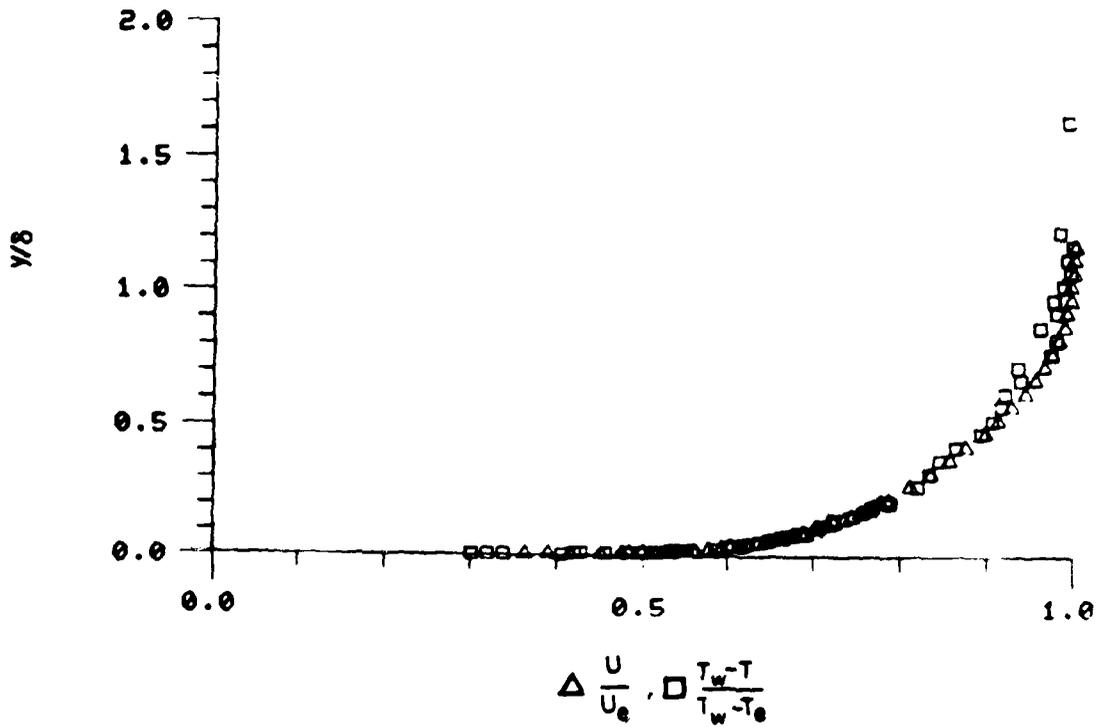


Figure B10E. Boundary Layer Flatness Distributions $x = 68$ in, $T_e = 1.6\%$

VELOCITY AND TEMPERATURE RATIOS



VELOCITY AND TEMPERATURE DISTRIBUTIONS IN UNIVERSAL COORDINATES

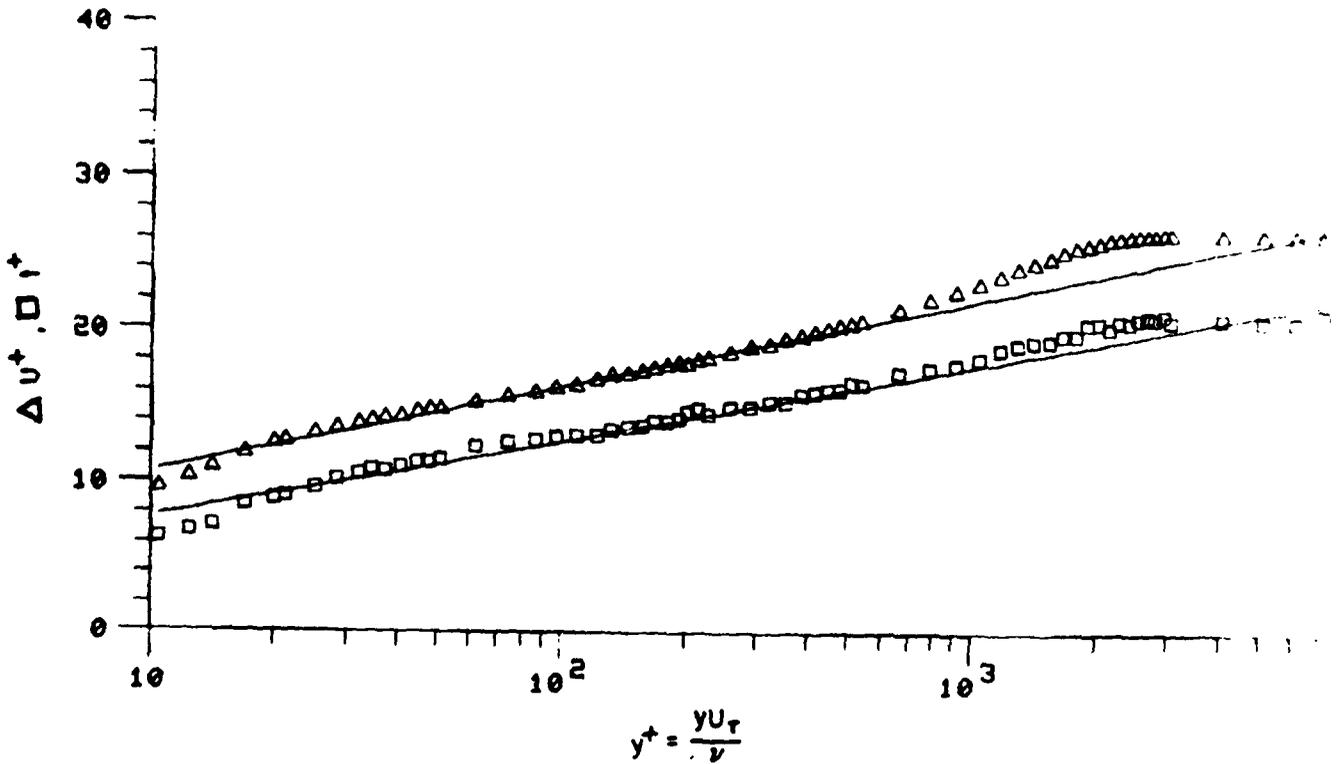


Figure B11. Mean Velocity and Temperature Profiles $x = 84$ in., $Te = 1.4\%$

RUN NO. 3. POINT 1.

BOLNDARY LAYER PROPERTIES

	LINEAR INTERPOLATION TO WALL	STANDARD SUBLAYER FUNCTION FROM WALL TO $Y^+ = 35$
FREE STREAM VELOCITY	97.379	97.379
FREE STREAM TEMPERATURE	69.380	
WALL TEMPERATURE	95.280	
WALL HEAT FLUX	.07879	
FREE STREAM DENSITY	.07563	
FREE STREAM KINEMATIC VISCOSITY	.0001615	
DENSITY OF FLUID AT WALL	.07210	
KINEMATIC VISCOSITY OF FLUID AT WALL	.0001757	
WALL/FREE STREAM DENSITY RATIO	.95333	
LOCATION REYNOLDS NUMBER (REX)	4220991.87	
INPUT VALUE OF VELOCITY DELTA	1.50000	
INPUT VALUE OF TEMPERATURE DELTA	1.80000	
CALCULATED DELTA		1.39524
DELTA 99.5% INPUT	.00000	
DISPLACEMENT THICKNESS (DELSTAR)	.19014	.19039
MOMENTUM THICKNESS (THETA)	.13559	.13572
ENERGY-DISSIPATION THICKNESS	.24207	.24207
ENTHALPY THICKNESS	.00739	.00739
SHAPE FACTOR 12 (DELSTAR/THETA)	1.40237	1.40265
SHAPE FACTOR 32 (ENERGY/THETA)	1.78534	1.78363
MOMENTUM THICKNESS REYNOLDS NUMBER	6813.21	6819.91
DISPLACEMENT THICKNESS REYNOLDS NUMBER	9554.62	9567.30
SKIN FRICTION COEFFICIENT	.002762	
FRICTION VELOCITY	3.70600	
LAW OF THE WALL CONSTANT (K)	.41000	
LAW OF THE WALL CONSTANT (C)	5.00000	
WAKE STRENGTH		.45923
CLAUSEPS 'DELTA' INTEGRAL	-4.64864	-4.81114
CLAUSEPS 'G' INTEGRAL	31.61966	31.69734
DISPLACEMENT THICKNESS - CONSTANT DENSITY	.17988	.18310
MOMENTUM THICKNESS - CONSTANT DENSITY	.13705	.13719
SHAPE FACTOR 12 - CONSTANT DENSITY	1.31253	1.33464

LOCATION -x- 64.00000

Te = 1.4%

Table B17

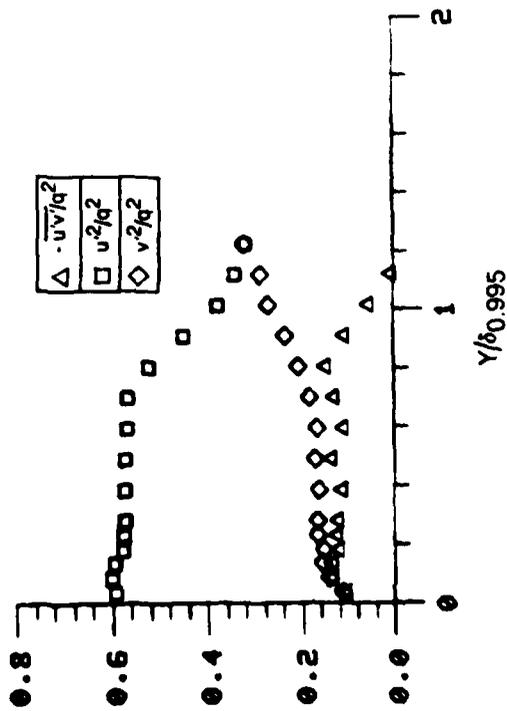
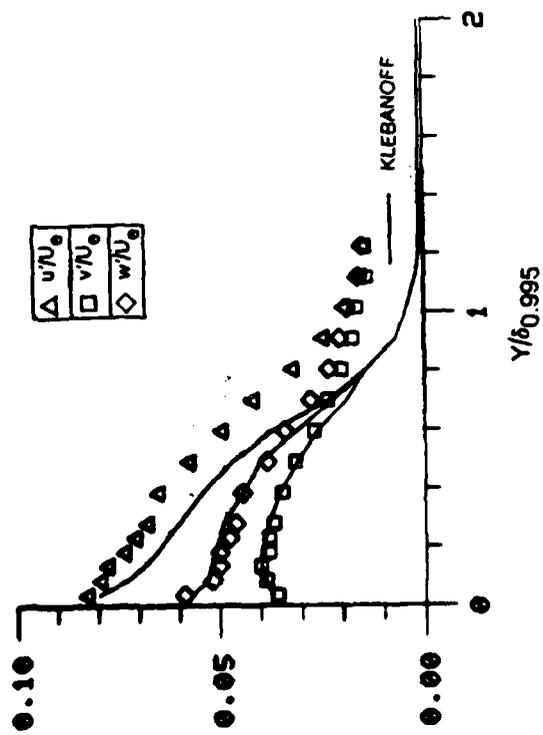
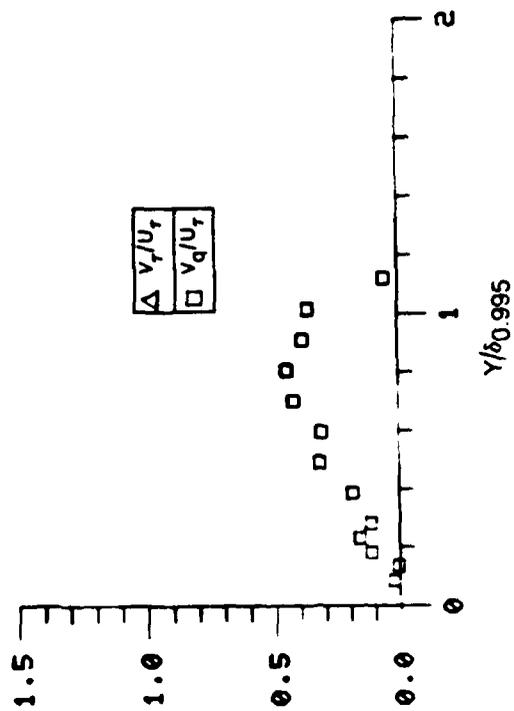
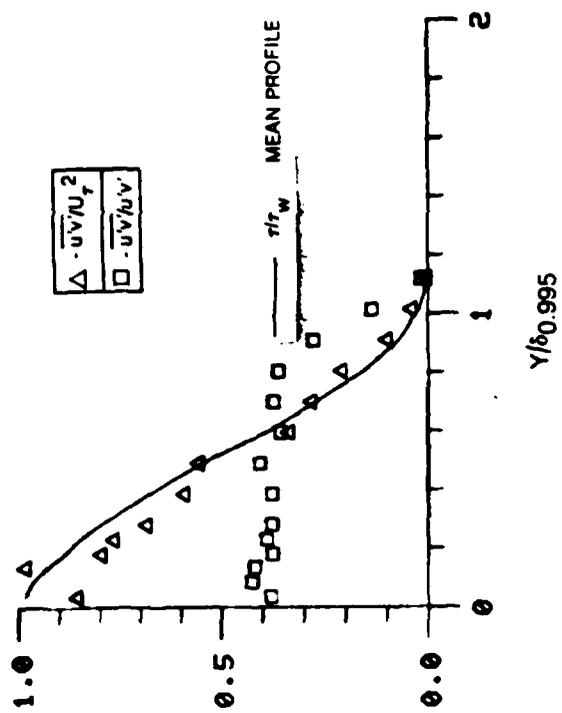


Figure B12A. Boundary Layer Turbulence Quantities $x = 84$ in, $T_e = 1.4\%$.

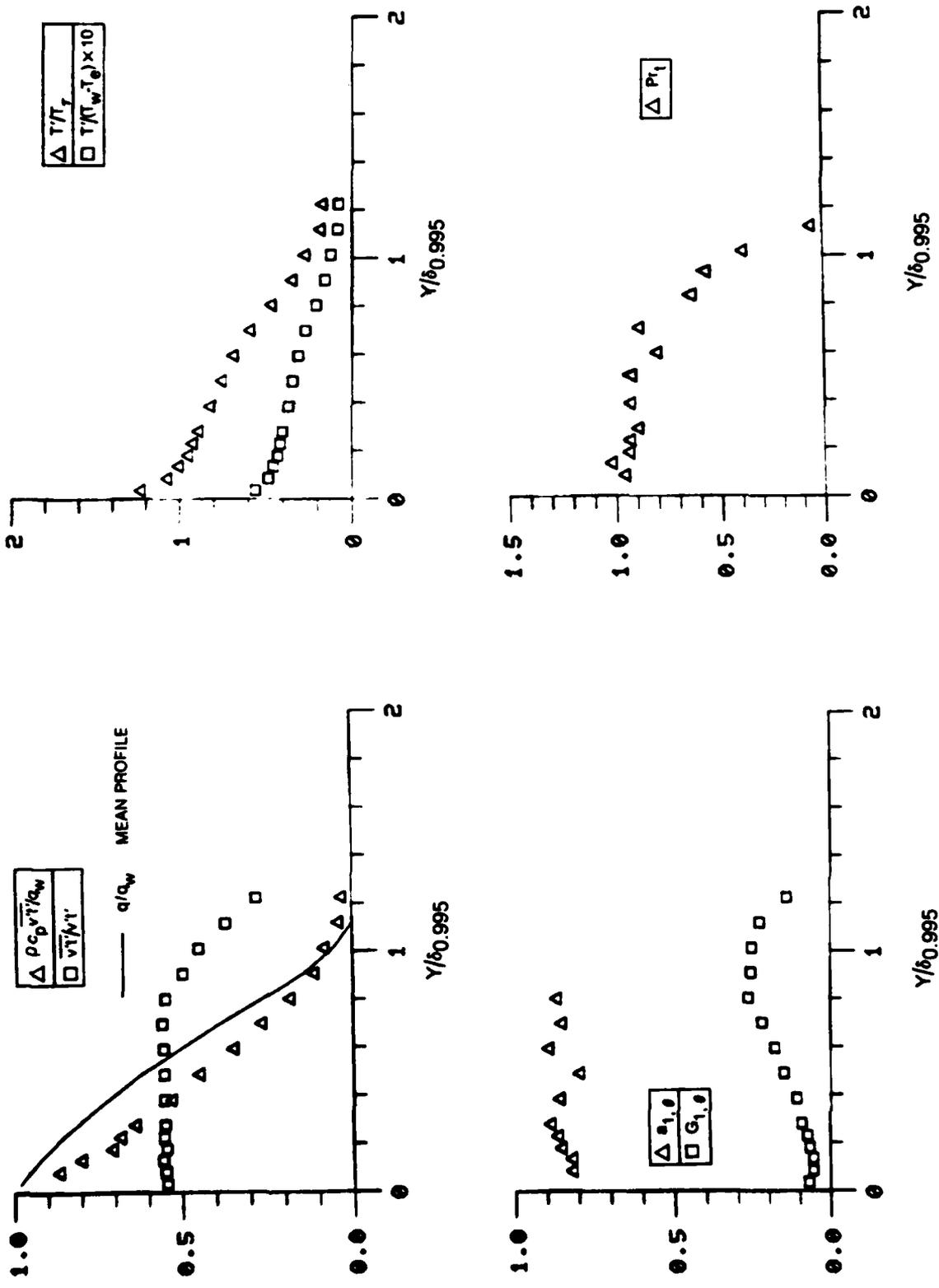


Figure B12B. Boundary Layer Turbulence Quantities $x = 84$ in, $Te = 1.4\%$

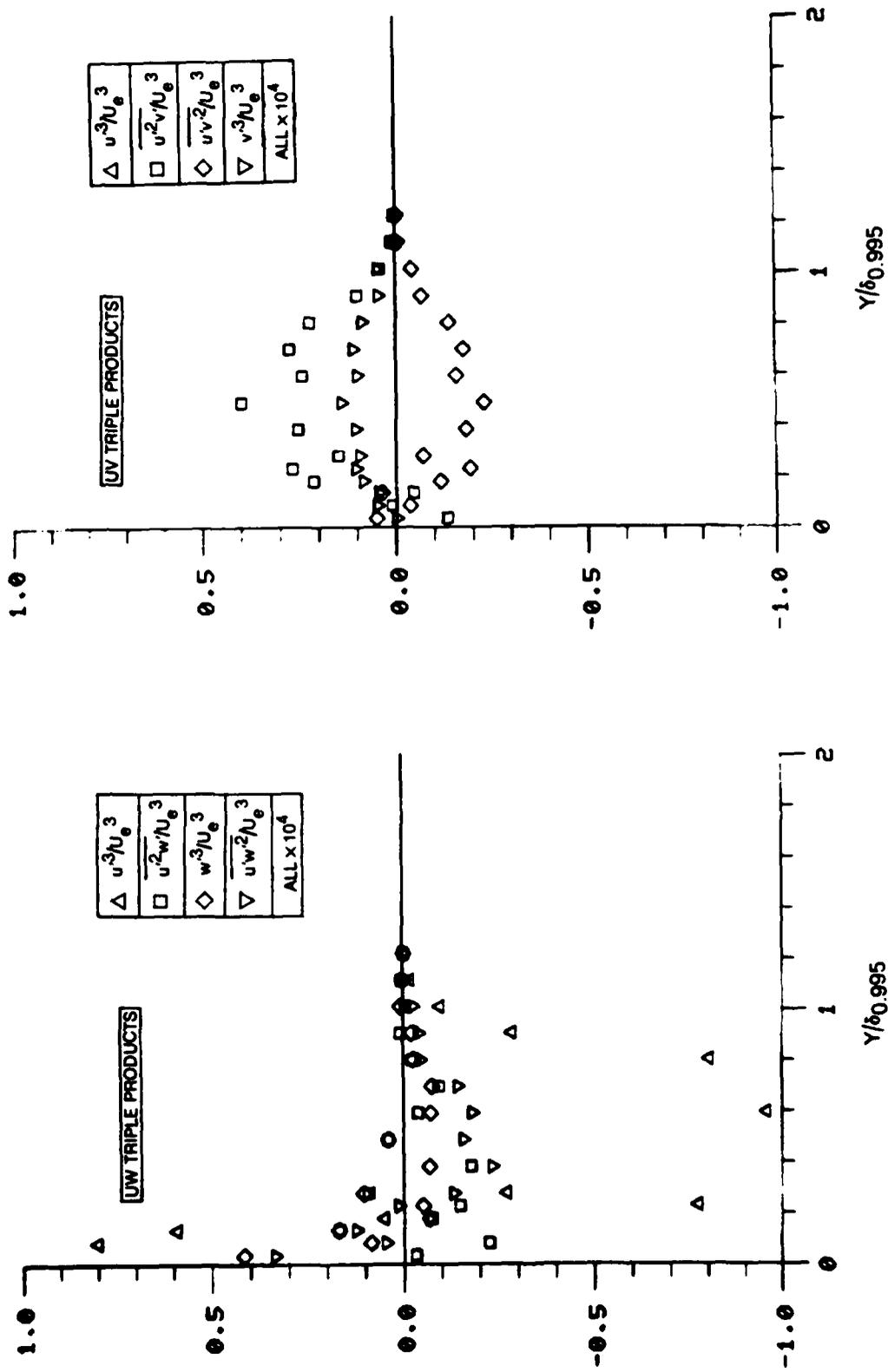


Figure B12C. Boundary Layer Triple Product Distributions $x = 84$ in, $T_e = 1.4\%$

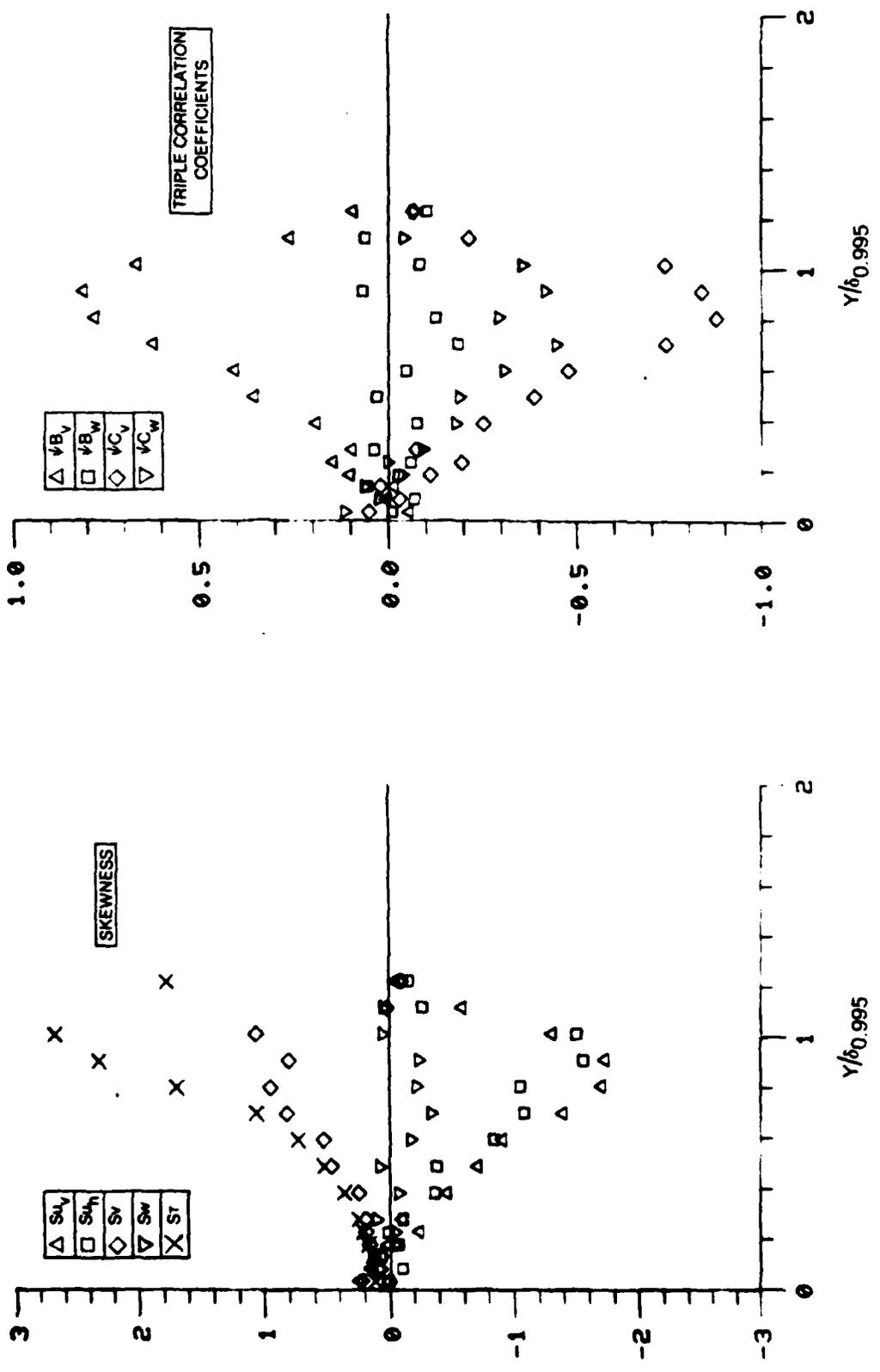


Figure B12D. Boundary Layer Skewness and Triple Product Correlation Coefficient Distributions $x = 84$ in, $T_e = 1.4\%$

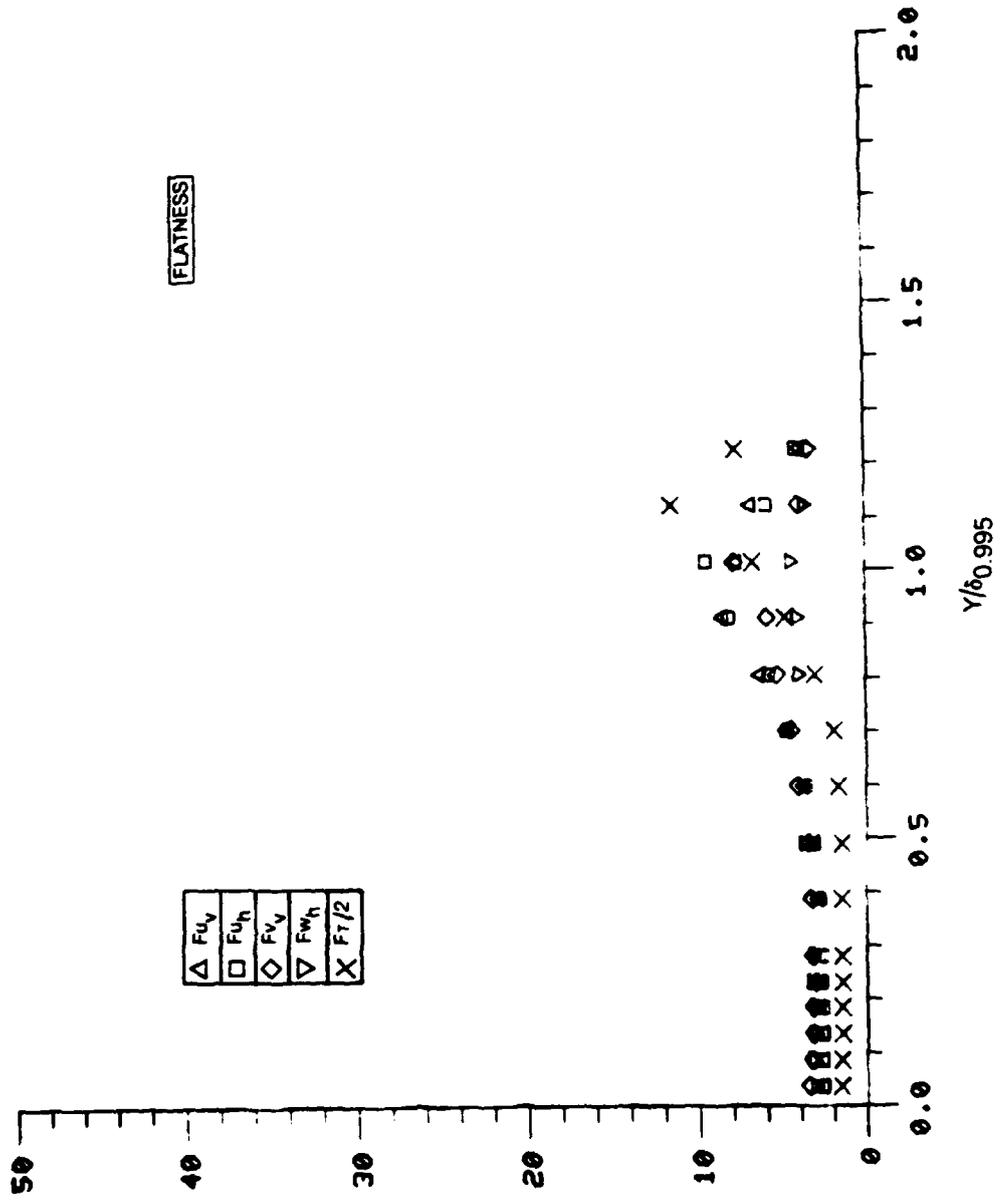


Figure 812E. Boundary Layer Flatness Distributions $x = 84$ in, $Te = 1.4\%$

Fluctuating Profile Data

$x = 84 \text{ in.}, Te = 1.4\%$

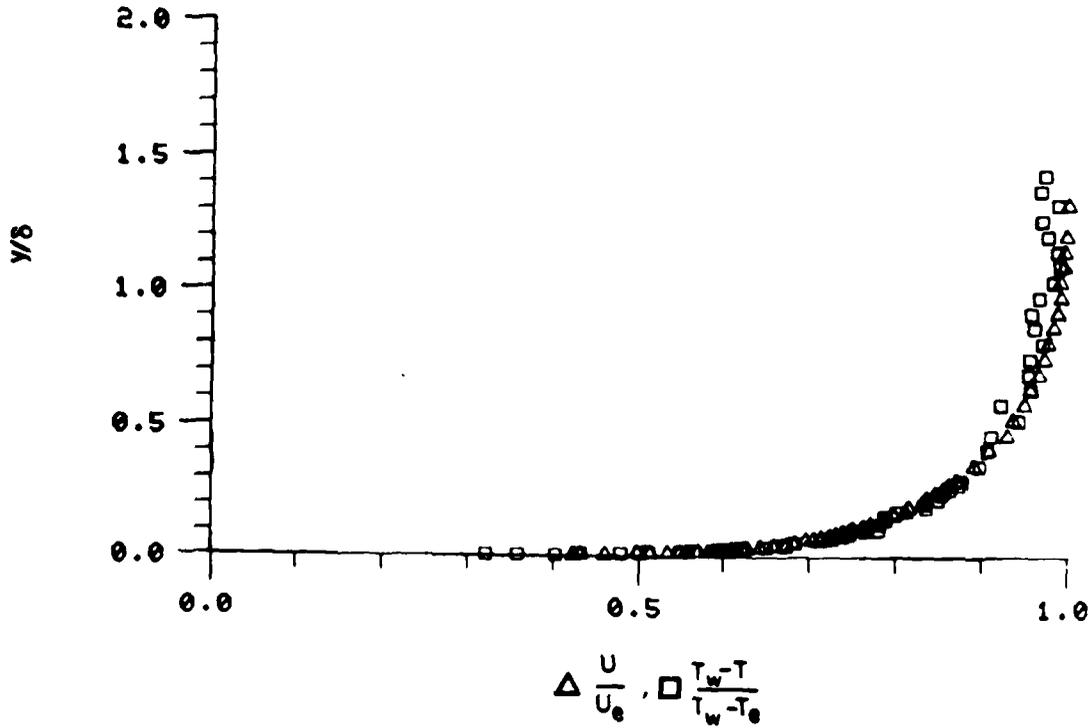
Y INCHES	Y/DELTA	U*/Ue	V*/Ue	W*/Ue	$\sqrt{U^2+V^2+W^2}/Ue$	LTAC	$\sqrt{L^2+T^2+R^2}/L$	CLC
0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.05	0.05	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.10	0.10	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.15	0.15	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.20	0.20	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.25	0.25	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.30	0.30	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.35	0.35	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.40	0.40	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.45	0.45	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.50	0.50	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.55	0.55	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.60	0.60	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.65	0.65	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.70	0.70	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.75	0.75	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.80	0.80	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.85	0.85	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.90	0.90	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.95	0.95	0.00	0.00	0.00	0.00	0.00	0.00	0.00
1.00	1.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00

Y INCHES	Y/DELTA	U*/Ue	V*/Ue	W*/Ue	$\sqrt{U^2+V^2+W^2}/Ue$	LTAC	$\sqrt{L^2+T^2+R^2}/L$	CLC
0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.05	0.05	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.10	0.10	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.15	0.15	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.20	0.20	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.25	0.25	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.30	0.30	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.35	0.35	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.40	0.40	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.45	0.45	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.50	0.50	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.55	0.55	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.60	0.60	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.65	0.65	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.70	0.70	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.75	0.75	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.80	0.80	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.85	0.85	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.90	0.90	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.95	0.95	0.00	0.00	0.00	0.00	0.00	0.00	0.00
1.00	1.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00

Y INCHES	Y/DELTA	$\sqrt{U^2+V^2+W^2}/Ue$	T*/TAU	T*/(TM-TE)	A10	G16	DOT
0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.05	0.05	0.00	0.00	0.00	0.00	0.00	0.00
0.10	0.10	0.00	0.00	0.00	0.00	0.00	0.00
0.15	0.15	0.00	0.00	0.00	0.00	0.00	0.00
0.20	0.20	0.00	0.00	0.00	0.00	0.00	0.00
0.25	0.25	0.00	0.00	0.00	0.00	0.00	0.00
0.30	0.30	0.00	0.00	0.00	0.00	0.00	0.00
0.35	0.35	0.00	0.00	0.00	0.00	0.00	0.00
0.40	0.40	0.00	0.00	0.00	0.00	0.00	0.00
0.45	0.45	0.00	0.00	0.00	0.00	0.00	0.00
0.50	0.50	0.00	0.00	0.00	0.00	0.00	0.00
0.55	0.55	0.00	0.00	0.00	0.00	0.00	0.00
0.60	0.60	0.00	0.00	0.00	0.00	0.00	0.00
0.65	0.65	0.00	0.00	0.00	0.00	0.00	0.00
0.70	0.70	0.00	0.00	0.00	0.00	0.00	0.00
0.75	0.75	0.00	0.00	0.00	0.00	0.00	0.00
0.80	0.80	0.00	0.00	0.00	0.00	0.00	0.00
0.85	0.85	0.00	0.00	0.00	0.00	0.00	0.00
0.90	0.90	0.00	0.00	0.00	0.00	0.00	0.00
0.95	0.95	0.00	0.00	0.00	0.00	0.00	0.00
1.00	1.00	0.00	0.00	0.00	0.00	0.00	0.00

Table B18A

VELOCITY AND TEMPERATURE RATIOS



VELOCITY AND TEMPERATURE DISTRIBUTIONS IN UNIVERSAL COORDINATES

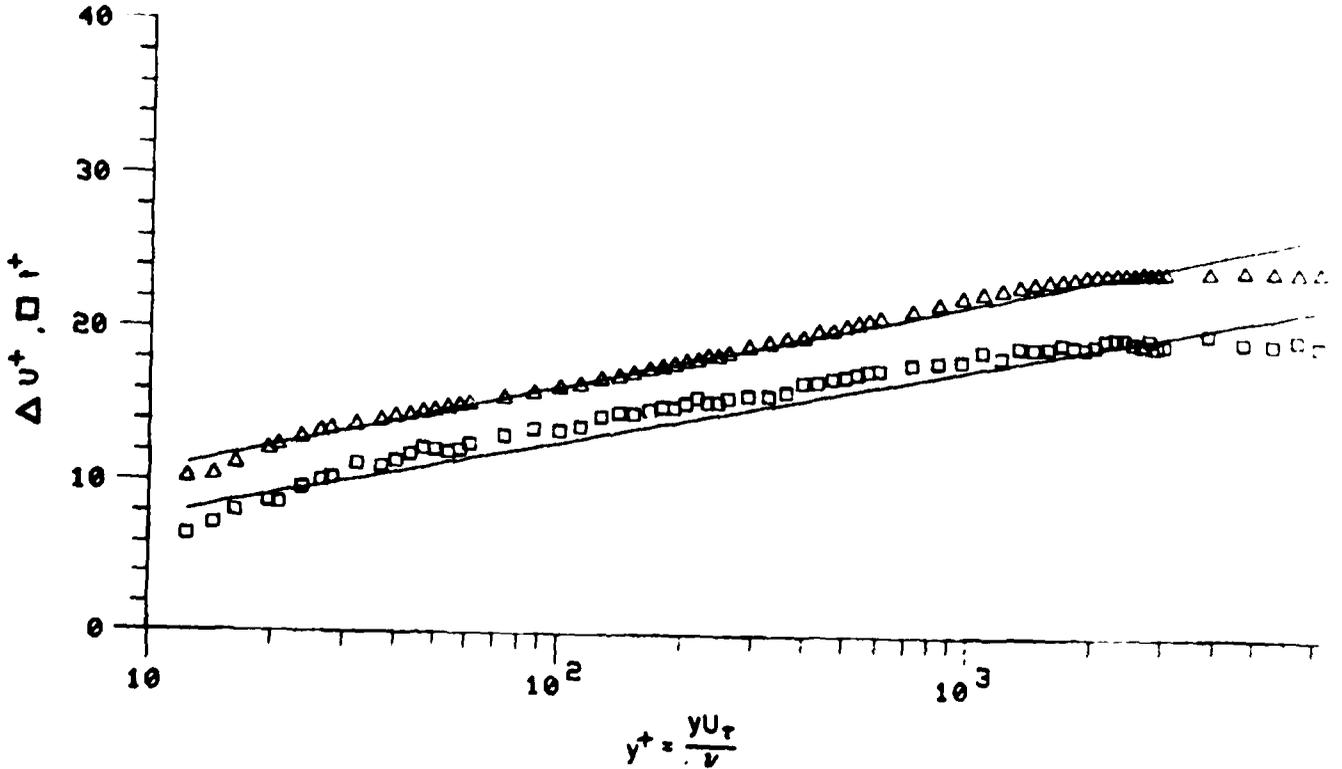


Figure B13. Mean Velocity and Temperature Profiles $x = 52$ in., $Te = 4.7\%$

Mean Profile Data

x = 52 in., Te = 4.7%

Y	DELTA	U	T	U/CF	THETA	U-UE	U(1)	T(1)
1	.0000	42.2	67.6	.422	.321	-13.670	10.223	6.451
2	.0000	42.2	67.6	.422	.359	-13.682	10.410	7.214
3	.0000	42.2	67.6	.422	.403	-12.994	11.097	7.214
4	.0000	42.2	67.6	.422	.432	-12.083	12.010	8.066
5	.0000	42.2	67.6	.422	.426	-11.736	12.357	8.602
6	.0000	42.2	67.6	.422	.480	-11.229	12.864	8.635
7	.0000	42.2	67.6	.422	.501	-10.862	13.211	9.609
8	.0000	42.2	67.6	.422	.512	-10.676	13.417	10.286
9	.0000	42.2	67.6	.422	.558	-10.353	13.740	11.000
10	.0000	42.2	67.6	.422	.549	-10.071	14.022	11.017
11	.0000	42.2	67.6	.422	.567	-9.822	14.271	11.386
12	.0000	42.2	67.6	.422	.597	-9.715	14.376	11.816
13	.0000	42.2	67.6	.422	.606	-9.579	14.514	12.205
14	.0000	42.2	67.6	.422	.602	-9.427	14.666	12.084
15	.0000	42.2	67.6	.422	.614	-9.298	14.799	11.919
16	.0000	42.2	67.6	.422	.604	-9.101	14.992	11.520
17	.0000	42.2	67.6	.422	.623	-8.956	15.137	12.508
18	.0000	42.2	67.6	.422	.649	-8.622	15.471	13.041
19	.0000	42.2	67.6	.422	.673	-8.217	15.676	13.521
20	.0000	42.2	67.6	.422	.670	-7.871	16.222	13.461
21	.0000	42.2	67.6	.422	.681	-7.676	16.417	13.716
22	.0000	42.2	67.6	.422	.695	-7.338	16.755	14.309
23	.0000	42.2	67.6	.422	.704	-7.120	16.973	14.564
24	.0000	42.2	67.6	.422	.712	-6.941	17.152	14.547
25	.0000	42.2	67.6	.422	.723	-6.670	17.423	14.629
26	.0000	42.2	67.6	.422	.729	-6.520	17.573	14.984
27	.0000	42.2	67.6	.422	.738	-6.311	17.782	14.955
28	.0000	42.2	67.6	.422	.746	-6.117	17.976	15.319
29	.0000	42.2	67.6	.422	.750	-5.955	18.074	15.633
30	.0000	42.2	67.6	.422	.766	-5.785	18.300	15.366
31	.0000	42.2	67.6	.422	.765	-5.741	18.352	15.366
32	.0000	42.2	67.6	.422	.777	-5.536	18.555	15.666
33	.0000	42.2	67.6	.422	.788	-5.154	18.939	15.626
34	.0000	42.2	67.6	.422	.796	-4.876	19.217	15.616
35	.0000	42.2	67.6	.422	.811	-4.560	19.533	16.071
36	.0000	42.2	67.6	.422	.834	-4.481	19.612	16.754
37	.0000	42.2	67.6	.422	.836	-4.118	19.975	16.763
38	.0000	42.2	67.6	.422	.849	-3.960	20.133	17.039
39	.0000	42.2	67.6	.422	.853	-3.733	20.360	17.129
40	.0000	42.2	67.6	.422	.863	-3.534	20.559	17.325
41	.0000	42.2	67.6	.422	.863	-3.367	20.726	17.531
42	.0000	42.2	67.6	.422	.869	-3.160	20.933	17.579
43	.0000	42.2	67.6	.422	.869	-2.965	21.142	17.956
44	.0000	42.2	67.6	.422	.896	-2.222	21.871	16.192
45	.0000	42.2	67.6	.422	.906	-1.741	22.352	16.257
46	.0000	42.2	67.6	.422	.936	-1.546	22.547	16.929
47	.0000	42.2	67.6	.422	.943	-1.229	22.866	16.509
48	.0000	42.2	67.6	.422	.957	-1.046	23.047	16.202
49	.0000	42.2	67.6	.422	.964	-0.862	23.231	19.112
50	.0000	42.2	67.6	.422	.972	-0.686	23.406	19.166
51	.0000	42.2	67.6	.422	.977	-0.565	23.526	19.466
52	.0000	42.2	67.6	.422	.981	-0.464	23.629	19.257
53	.0000	42.2	67.6	.422	.986	-0.336	23.757	19.195
54	.0000	42.2	67.6	.422	.990	-0.245	23.852	19.353
55	.0000	42.2	67.6	.422	.994	-0.151	23.942	19.716
56	.0000	42.2	67.6	.422	.996	-0.087	24.086	19.577
57	.0000	42.2	67.6	.422	.996	0.019	24.112	19.441
58	.0000	42.2	67.6	.422	.999	0.029	24.064	19.600
59	.0000	42.2	67.6	.422	1.000	0.009	24.102	19.395
60	.0000	42.2	67.6	.422	1.001	0.020	24.113	19.486
61	.0000	42.2	67.6	.422	1.002	0.050	24.143	20.206
62	.0000	42.2	67.6	.422	1.005	0.123	24.216	19.622
63	.0000	42.2	67.6	.422	1.005	0.131	24.224	19.503
64	.0000	42.2	67.6	.422	1.004	0.102	24.195	19.615
65	.0000	42.2	67.6	.422	1.007	0.170	24.263	19.468
66	.0000	42.2	67.6	.422	1.003	0.073	24.166	19.637
67	.0000	42.2	67.6	.422				
68	.0000	42.2	67.6	.422				

Table B19

RUN NO. 1. POINT 4.

BOUNDARY LAYER PROPERTIES

	LINEAR INTERPOLATION TO WALL	STANDARD SUBLAYER FUNCTION FROM WALL TO $Y^+ = 30$
FREE STREAM VELOCITY	99.223	99.223
FREE STREAM TEMPERATURE	72.500	
WALL TEMPERATURE	94.800	
WALL HEAT FLUX	.07890	
FREE STREAM DENSITY	.07489	
FREE STREAM KINEMATIC VISCOSITY	.0001638	
DENSITY OF FLUID AT WALL	.07188	
KINEMATIC VISCOSITY OF FLUID AT WALL	.0001762	
WALL/FREE STREAM DENSITY RATIO	.95978	
LOCATION REYNOLDS NUMBER (REX)	2624417.81	
INPUT VALUE OF VELOCITY DELTA	1.35000	
INPUT VALUE OF TEMPERATURE DELTA	1.45000	
CALCULATED DELTA		1.05346
DELTA 99.5% INPUT	.00000	
DISPLACEMENT THICKNESS (DELSTAR)	.11874	.11877
MOMENTUM THICKNESS (THETA)	.08814	.08841
ENERGY-DISSIPATION THICKNESS	.16040	.16061
ENTHALPY THICKNESS	.00427	.00428
SHAPE FACTOR 12 (DELSTAR/THETA)	1.34716	1.34342
SHAPE FACTOR 72 (ENERGY/THETA)	1.81990	1.81669
MOMENTUM THICKNESS REYNOLDS NUMBER	4448.29	4461.89
DISPLACEMENT THICKNESS REYNOLDS NUMBER	5992.57	5994.21
SKIN FRICTION COEFFICIENT	.003307	
FRICTION VELOCITY	4.11836	
LAW OF THE WALL CONSTANT (K)	.41000	
LAW OF THE WALL CONSTANT (C)	5.00000	
WAKE STRENGTH		.10069
CLAUSERS 'DELTA' INTEGRAL	-2.60818	-2.76130
CLAUSERS 'G' INTEGRAL	14.96252	14.81376
DISPLACEMENT THICKNESS - CONSTANT DENSITY	.11142	.11461
MOMENTUM THICKNESS - CONSTANT DENSITY	.08581	.08909
SHAPE FACTOR 12 - CONSTANT DENSITY	1.25461	1.28645

LOCATION -X- 52.0000

Te = 4.7%

Table B20

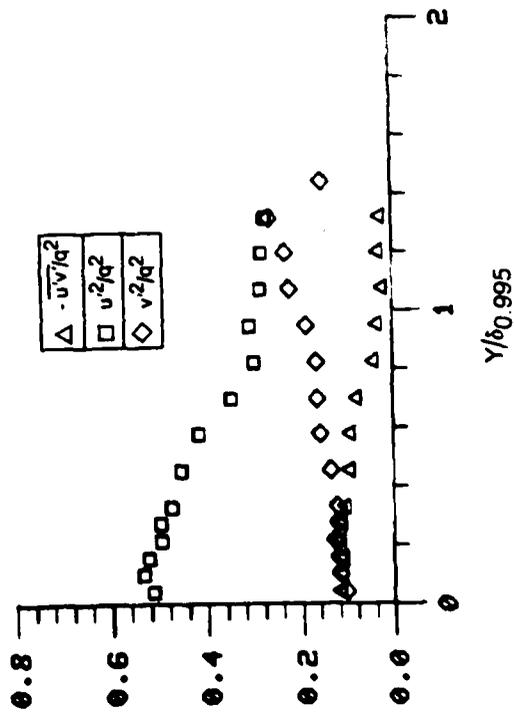
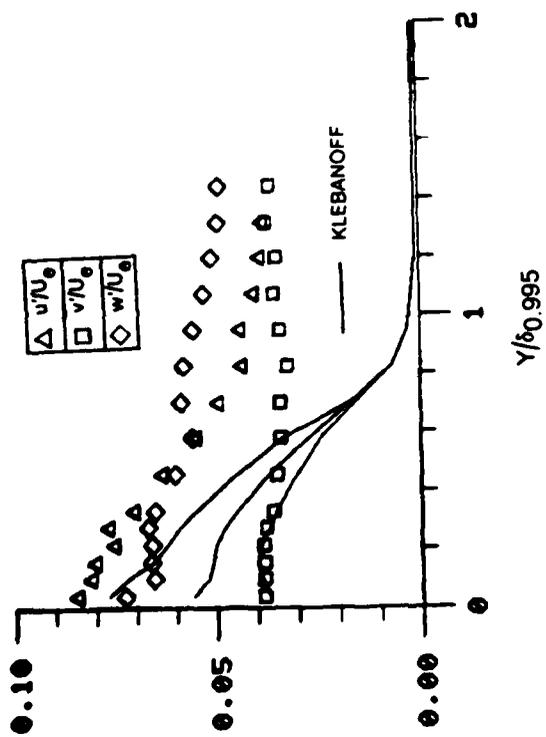
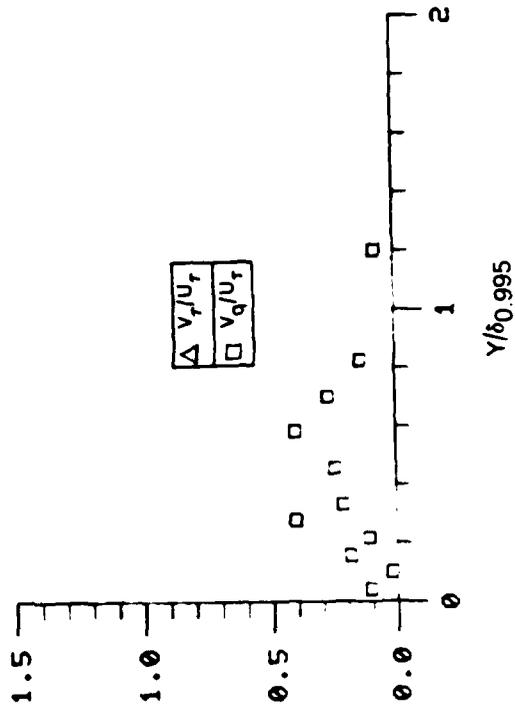
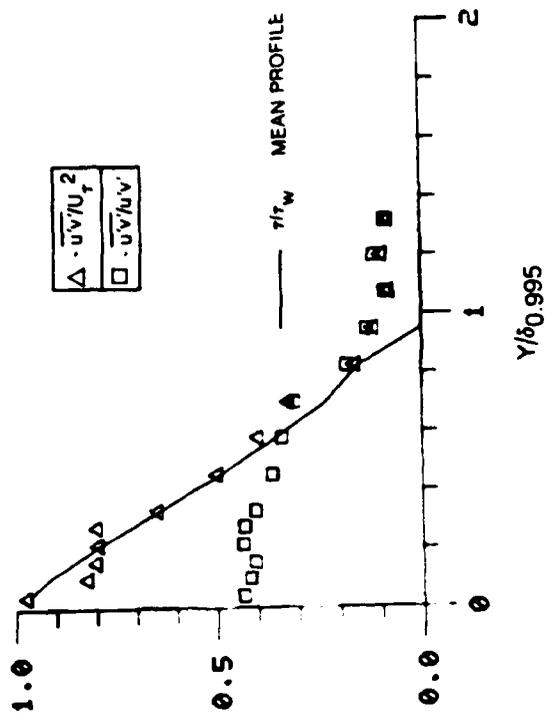


Figure B14A. Boundary Layer Turbulence Quantities, $x = 52$ in, $Te = 4.7\%$

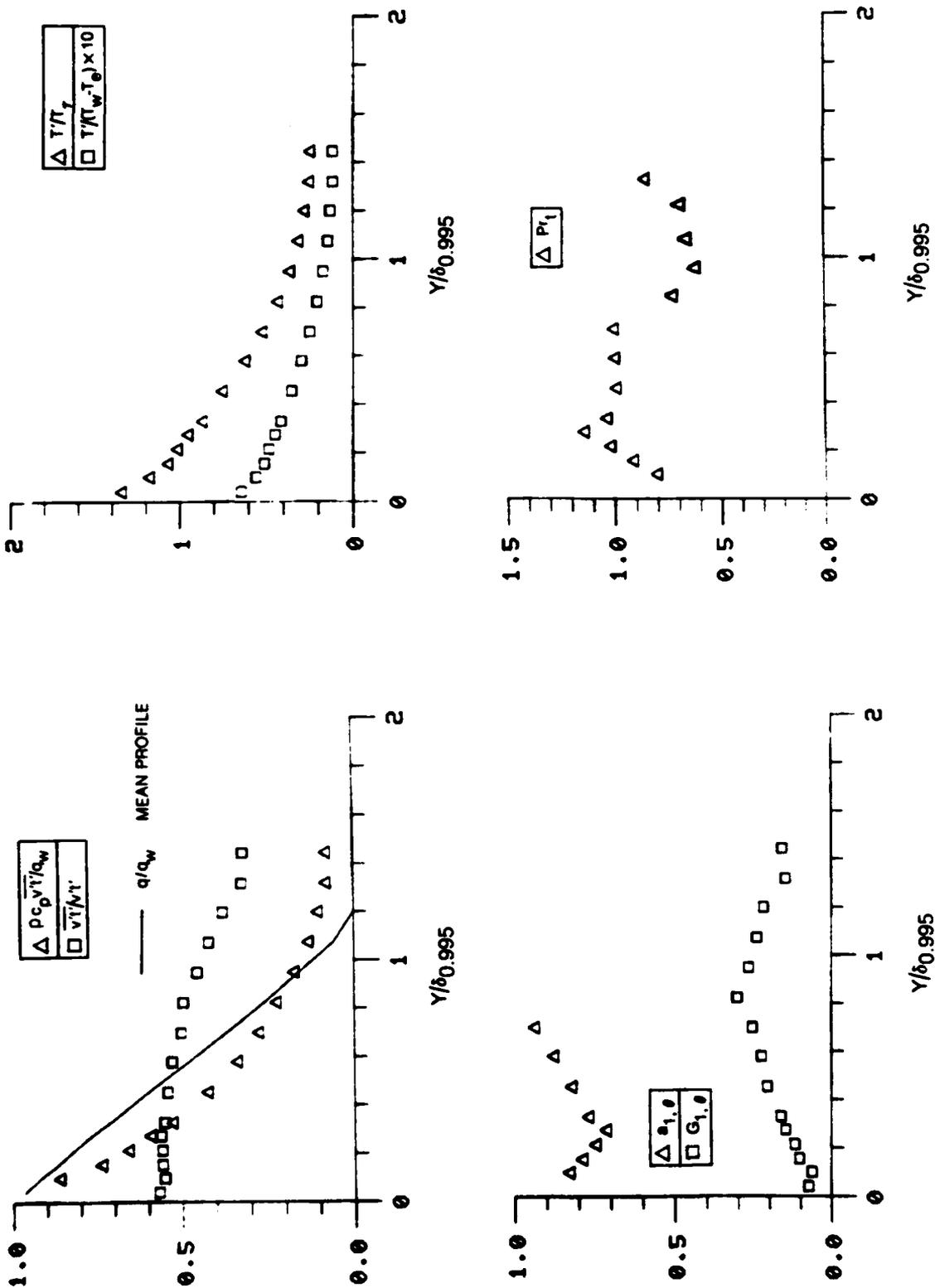


Figure B14B. Boundary Layer Turbulence Quantities, $x = 52$ in, $T_e = 4.7\%$

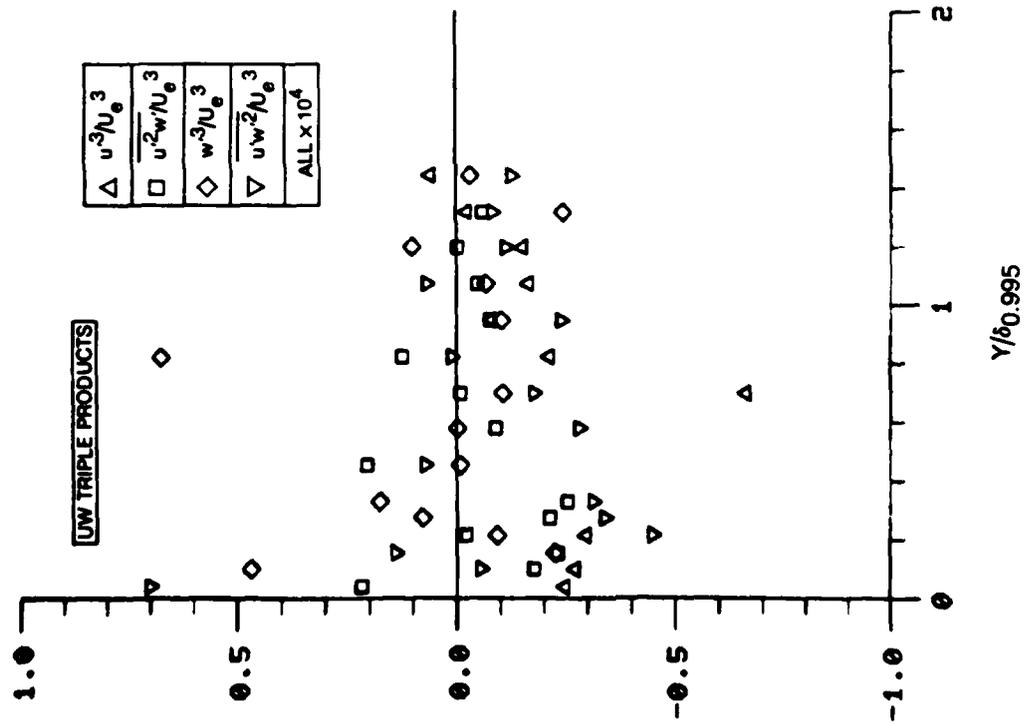
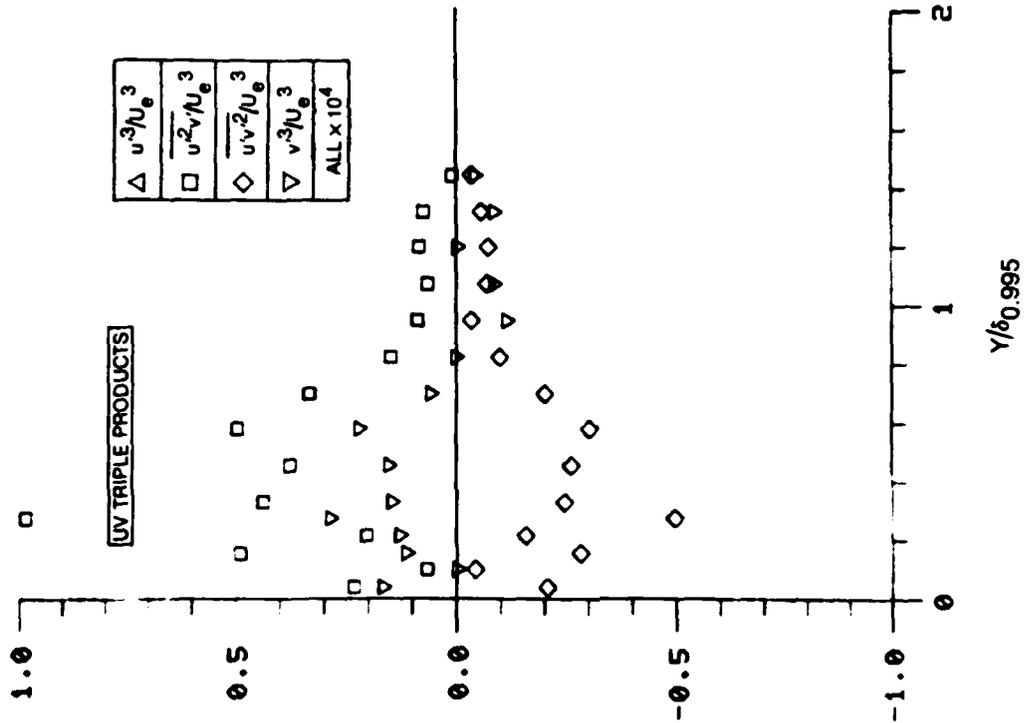


Figure B14C. Boundary Layer Triple Product Distributions $x = 52$ in, $T_e = 4.7\%$

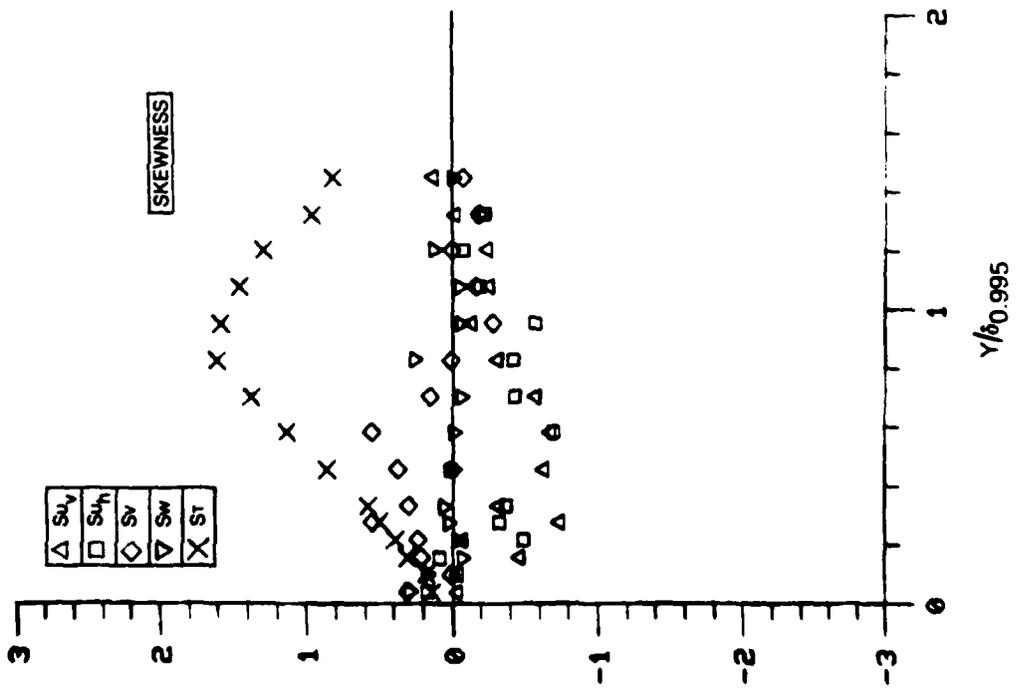
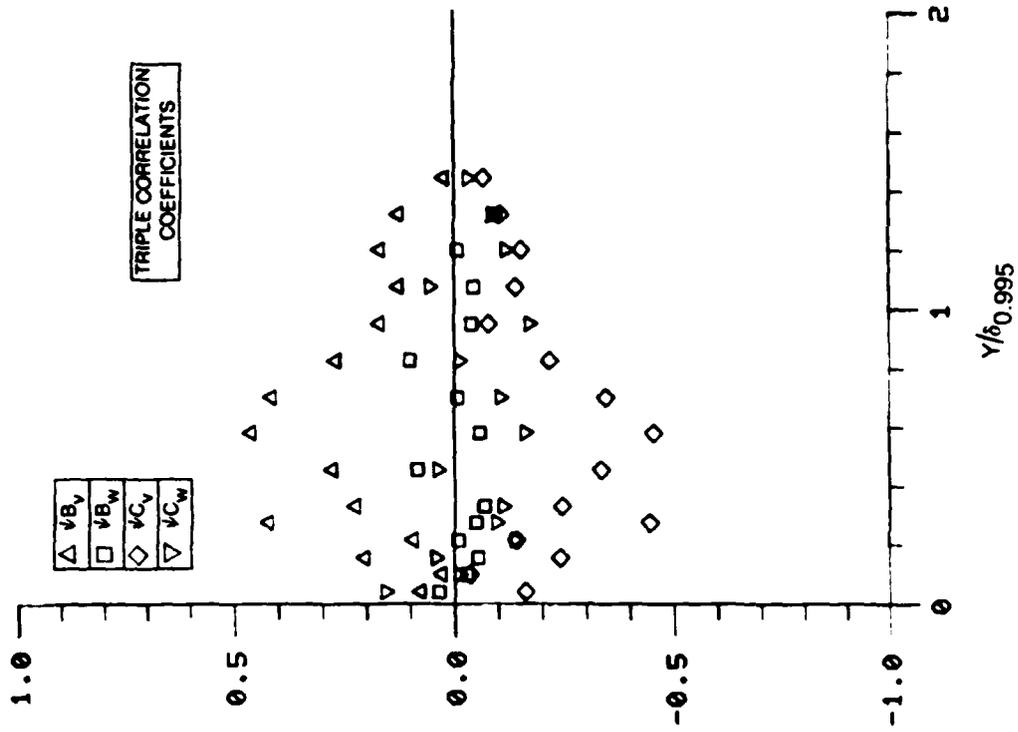


Figure B14D. Boundary Layer Skewness and Triple Product Correlation Coefficient Distributions $x = 52$ in, $T_e = 4.7\%$

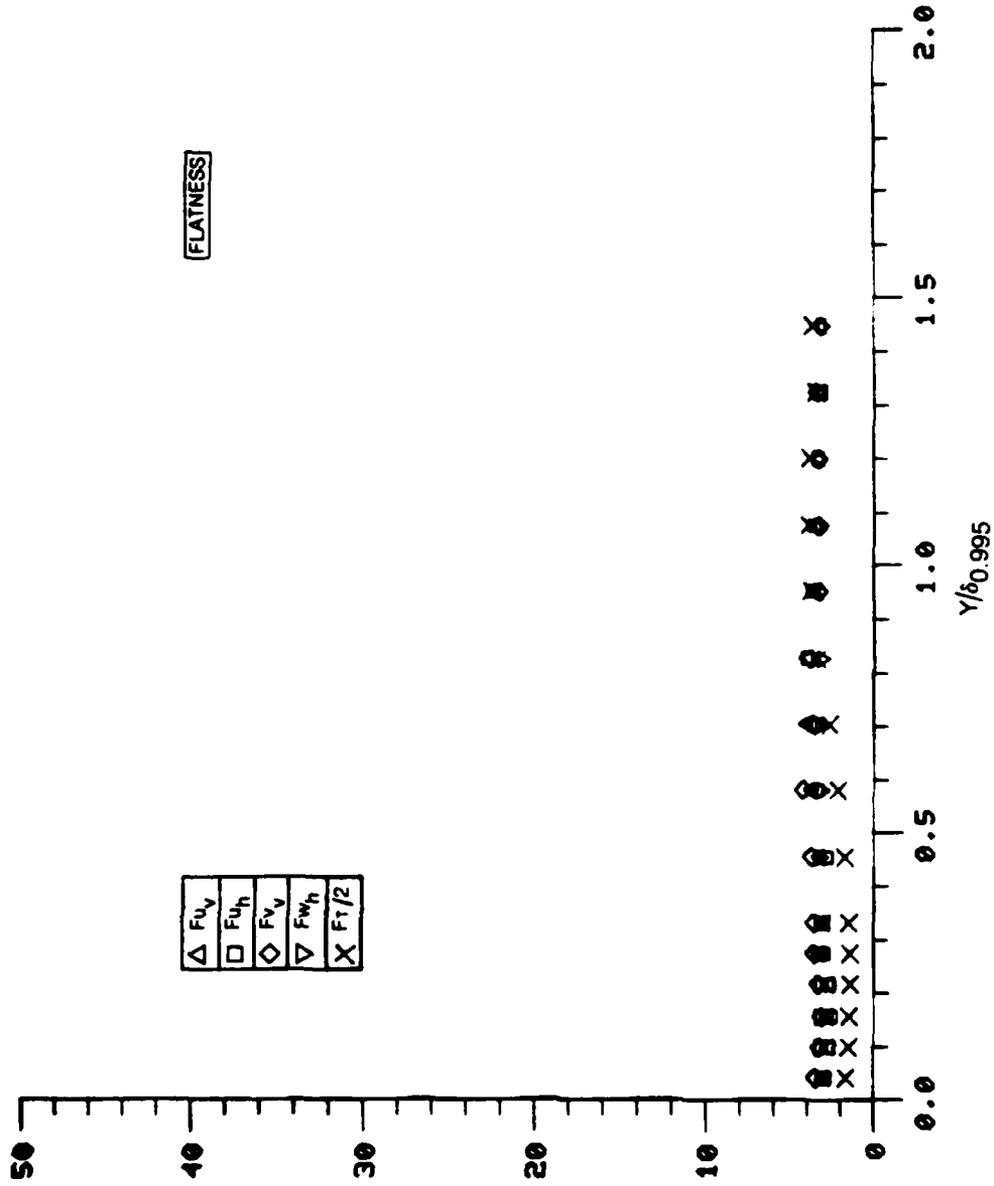


Figure B14E. Boundary Layer Flatness Distributions $x = 52$ in, $T_e = 4.7\%$

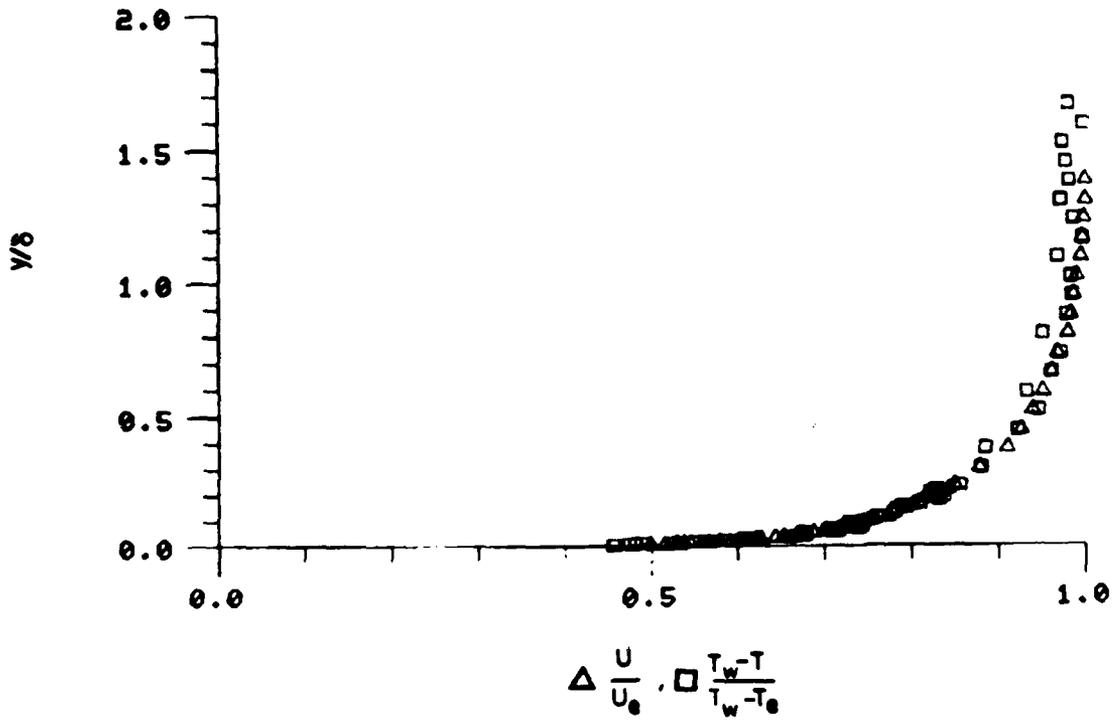
Fluctuating Profile Data

$x = 52 \text{ in.}, Te = 4.7\%$

N	Y/ INCHES	Y/ DELTA	U*/UE	V*/UE	W*/UE	√(V* ² +W* ²)/UE	LTAC	T*/(T* ⁰ -T _e)	C/UE
1	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
2	0.05	0.05	0.00	0.00	0.00	0.00	0.00	0.00	0.00
3	0.10	0.10	0.00	0.00	0.00	0.00	0.00	0.00	0.00
4	0.15	0.15	0.00	0.00	0.00	0.00	0.00	0.00	0.00
5	0.20	0.20	0.00	0.00	0.00	0.00	0.00	0.00	0.00
6	0.25	0.25	0.00	0.00	0.00	0.00	0.00	0.00	0.00
7	0.30	0.30	0.00	0.00	0.00	0.00	0.00	0.00	0.00
8	0.35	0.35	0.00	0.00	0.00	0.00	0.00	0.00	0.00
9	0.40	0.40	0.00	0.00	0.00	0.00	0.00	0.00	0.00
10	0.45	0.45	0.00	0.00	0.00	0.00	0.00	0.00	0.00
11	0.50	0.50	0.00	0.00	0.00	0.00	0.00	0.00	0.00
12	0.55	0.55	0.00	0.00	0.00	0.00	0.00	0.00	0.00
13	0.60	0.60	0.00	0.00	0.00	0.00	0.00	0.00	0.00
14	0.65	0.65	0.00	0.00	0.00	0.00	0.00	0.00	0.00
15	0.70	0.70	0.00	0.00	0.00	0.00	0.00	0.00	0.00
16	0.75	0.75	0.00	0.00	0.00	0.00	0.00	0.00	0.00
17	0.80	0.80	0.00	0.00	0.00	0.00	0.00	0.00	0.00
18	0.85	0.85	0.00	0.00	0.00	0.00	0.00	0.00	0.00
19	0.90	0.90	0.00	0.00	0.00	0.00	0.00	0.00	0.00
20	0.95	0.95	0.00	0.00	0.00	0.00	0.00	0.00	0.00
21	1.00	1.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
22	1.05	1.05	0.00	0.00	0.00	0.00	0.00	0.00	0.00
23	1.10	1.10	0.00	0.00	0.00	0.00	0.00	0.00	0.00
24	1.15	1.15	0.00	0.00	0.00	0.00	0.00	0.00	0.00
25	1.20	1.20	0.00	0.00	0.00	0.00	0.00	0.00	0.00
26	1.25	1.25	0.00	0.00	0.00	0.00	0.00	0.00	0.00
27	1.30	1.30	0.00	0.00	0.00	0.00	0.00	0.00	0.00
28	1.35	1.35	0.00	0.00	0.00	0.00	0.00	0.00	0.00
29	1.40	1.40	0.00	0.00	0.00	0.00	0.00	0.00	0.00
30	1.45	1.45	0.00	0.00	0.00	0.00	0.00	0.00	0.00
31	1.50	1.50	0.00	0.00	0.00	0.00	0.00	0.00	0.00
32	1.55	1.55	0.00	0.00	0.00	0.00	0.00	0.00	0.00
33	1.60	1.60	0.00	0.00	0.00	0.00	0.00	0.00	0.00
34	1.65	1.65	0.00	0.00	0.00	0.00	0.00	0.00	0.00
35	1.70	1.70	0.00	0.00	0.00	0.00	0.00	0.00	0.00
36	1.75	1.75	0.00	0.00	0.00	0.00	0.00	0.00	0.00
37	1.80	1.80	0.00	0.00	0.00	0.00	0.00	0.00	0.00
38	1.85	1.85	0.00	0.00	0.00	0.00	0.00	0.00	0.00
39	1.90	1.90	0.00	0.00	0.00	0.00	0.00	0.00	0.00
40	1.95	1.95	0.00	0.00	0.00	0.00	0.00	0.00	0.00
41	2.00	2.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
42	2.05	2.05	0.00	0.00	0.00	0.00	0.00	0.00	0.00
43	2.10	2.10	0.00	0.00	0.00	0.00	0.00	0.00	0.00
44	2.15	2.15	0.00	0.00	0.00	0.00	0.00	0.00	0.00
45	2.20	2.20	0.00	0.00	0.00	0.00	0.00	0.00	0.00
46	2.25	2.25	0.00	0.00	0.00	0.00	0.00	0.00	0.00
47	2.30	2.30	0.00	0.00	0.00	0.00	0.00	0.00	0.00
48	2.35	2.35	0.00	0.00	0.00	0.00	0.00	0.00	0.00
49	2.40	2.40	0.00	0.00	0.00	0.00	0.00	0.00	0.00
50	2.45	2.45	0.00	0.00	0.00	0.00	0.00	0.00	0.00
51	2.50	2.50	0.00	0.00	0.00	0.00	0.00	0.00	0.00
52	2.55	2.55	0.00	0.00	0.00	0.00	0.00	0.00	0.00
53	2.60	2.60	0.00	0.00	0.00	0.00	0.00	0.00	0.00
54	2.65	2.65	0.00	0.00	0.00	0.00	0.00	0.00	0.00
55	2.70	2.70	0.00	0.00	0.00	0.00	0.00	0.00	0.00
56	2.75	2.75	0.00	0.00	0.00	0.00	0.00	0.00	0.00
57	2.80	2.80	0.00	0.00	0.00	0.00	0.00	0.00	0.00
58	2.85	2.85	0.00	0.00	0.00	0.00	0.00	0.00	0.00
59	2.90	2.90	0.00	0.00	0.00	0.00	0.00	0.00	0.00
60	2.95	2.95	0.00	0.00	0.00	0.00	0.00	0.00	0.00
61	3.00	3.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
62	3.05	3.05	0.00	0.00	0.00	0.00	0.00	0.00	0.00
63	3.10	3.10	0.00	0.00	0.00	0.00	0.00	0.00	0.00
64	3.15	3.15	0.00	0.00	0.00	0.00	0.00	0.00	0.00
65	3.20	3.20	0.00	0.00	0.00	0.00	0.00	0.00	0.00
66	3.25	3.25	0.00	0.00	0.00	0.00	0.00	0.00	0.00
67	3.30	3.30	0.00	0.00	0.00	0.00	0.00	0.00	0.00
68	3.35	3.35	0.00	0.00	0.00	0.00	0.00	0.00	0.00
69	3.40	3.40	0.00	0.00	0.00	0.00	0.00	0.00	0.00
70	3.45	3.45	0.00	0.00	0.00	0.00	0.00	0.00	0.00
71	3.50	3.50	0.00	0.00	0.00	0.00	0.00	0.00	0.00
72	3.55	3.55	0.00	0.00	0.00	0.00	0.00	0.00	0.00
73	3.60	3.60	0.00	0.00	0.00	0.00	0.00	0.00	0.00
74	3.65	3.65	0.00	0.00	0.00	0.00	0.00	0.00	0.00
75	3.70	3.70	0.00	0.00	0.00	0.00	0.00	0.00	0.00
76	3.75	3.75	0.00	0.00	0.00	0.00	0.00	0.00	0.00
77	3.80	3.80	0.00	0.00	0.00	0.00	0.00	0.00	0.00
78	3.85	3.85	0.00	0.00	0.00	0.00	0.00	0.00	0.00
79	3.90	3.90	0.00	0.00	0.00	0.00	0.00	0.00	0.00
80	3.95	3.95	0.00	0.00	0.00	0.00	0.00	0.00	0.00
81	4.00	4.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
82	4.05	4.05	0.00	0.00	0.00	0.00	0.00	0.00	0.00
83	4.10	4.10	0.00	0.00	0.00	0.00	0.00	0.00	0.00
84	4.15	4.15	0.00	0.00	0.00	0.00	0.00	0.00	0.00
85	4.20	4.20	0.00	0.00	0.00	0.00	0.00	0.00	0.00
86	4.25	4.25	0.00	0.00	0.00	0.00	0.00	0.00	0.00
87	4.30	4.30	0.00	0.00	0.00	0.00	0.00	0.00	0.00
88	4.35	4.35	0.00	0.00	0.00	0.00	0.00	0.00	0.00
89	4.40	4.40	0.00	0.00	0.00	0.00	0.00	0.00	0.00
90	4.45	4.45	0.00	0.00	0.00	0.00	0.00	0.00	0.00
91	4.50	4.50	0.00	0.00	0.00	0.00	0.00	0.00	0.00
92	4.55	4.55	0.00	0.00	0.00	0.00	0.00	0.00	0.00
93	4.60	4.60	0.00	0.00	0.00	0.00	0.00	0.00	0.00
94	4.65	4.65	0.00	0.00	0.00	0.00	0.00	0.00	0.00
95	4.70	4.70	0.00	0.00	0.00	0.00	0.00	0.00	0.00
96	4.75	4.75	0.00	0.00	0.00	0.00	0.00	0.00	0.00
97	4.80	4.80	0.00	0.00	0.00	0.00	0.00	0.00	0.00
98	4.85	4.85	0.00	0.00	0.00	0.00	0.00	0.00	0.00
99	4.90	4.90	0.00	0.00	0.00	0.00	0.00	0.00	0.00
100	4.95	4.95	0.00	0.00	0.00	0.00	0.00	0.00	0.00

Table B21A

VELOCITY AND TEMPERATURE RATIOS



VELOCITY AND TEMPERATURE DISTRIBUTIONS IN UNIVERSAL COORDINATES

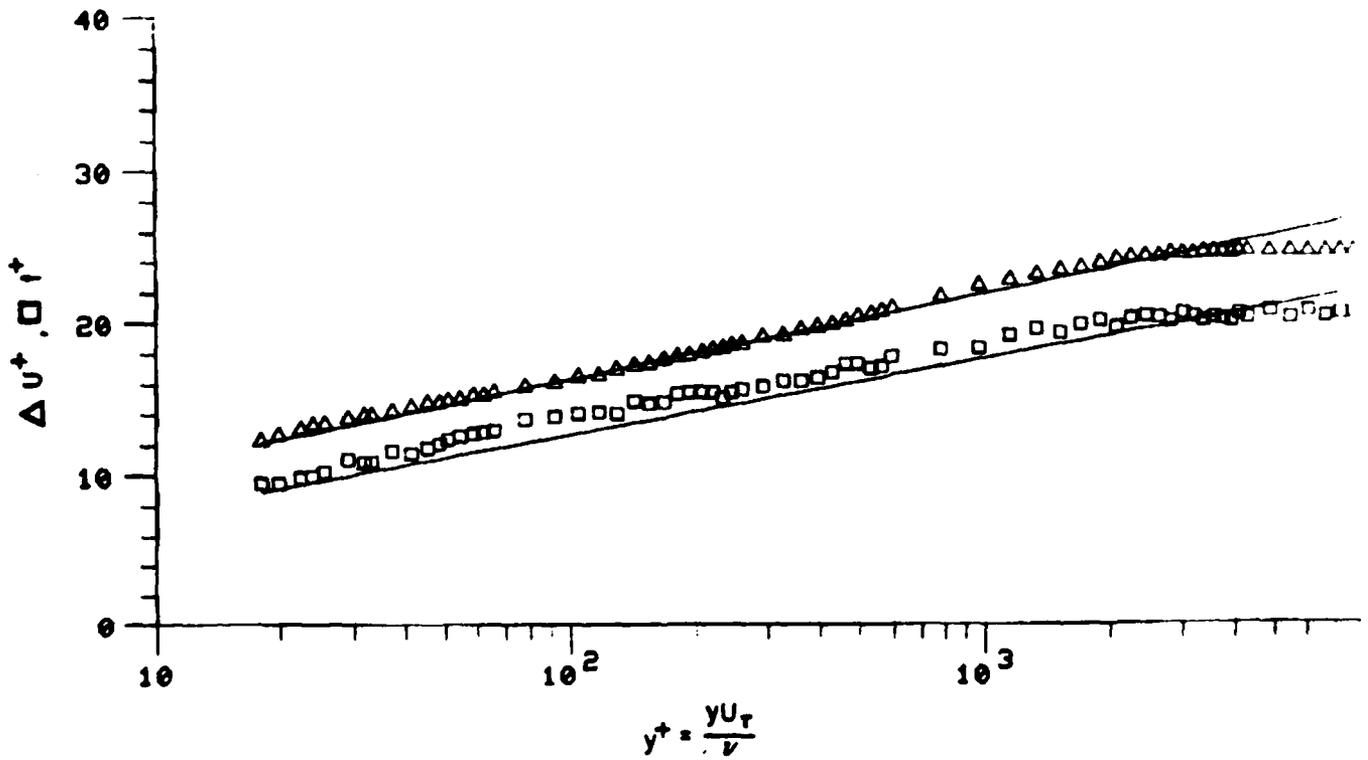


Figure B15. Mean Velocity and Temperature Profiles $x = 68$ in., $Te = 4.2\%$

Mean Profile Data

x = 68 in., Te = 4.2%

1	1	Y	Y/	U	T	L/UE	THETA	L-UE	U	Y	Y
2	2	1	1	1	1	1	1	1	1	1	1
3	3	1	1	1	1	1	1	1	1	1	1
4	4	1	1	1	1	1	1	1	1	1	1
5	5	1	1	1	1	1	1	1	1	1	1
6	6	1	1	1	1	1	1	1	1	1	1
7	7	1	1	1	1	1	1	1	1	1	1
8	8	1	1	1	1	1	1	1	1	1	1
9	9	1	1	1	1	1	1	1	1	1	1
10	10	1	1	1	1	1	1	1	1	1	1
11	11	1	1	1	1	1	1	1	1	1	1
12	12	1	1	1	1	1	1	1	1	1	1
13	13	1	1	1	1	1	1	1	1	1	1
14	14	1	1	1	1	1	1	1	1	1	1
15	15	1	1	1	1	1	1	1	1	1	1
16	16	1	1	1	1	1	1	1	1	1	1
17	17	1	1	1	1	1	1	1	1	1	1
18	18	1	1	1	1	1	1	1	1	1	1
19	19	1	1	1	1	1	1	1	1	1	1
20	20	1	1	1	1	1	1	1	1	1	1
21	21	1	1	1	1	1	1	1	1	1	1
22	22	1	1	1	1	1	1	1	1	1	1
23	23	1	1	1	1	1	1	1	1	1	1
24	24	1	1	1	1	1	1	1	1	1	1
25	25	1	1	1	1	1	1	1	1	1	1
26	26	1	1	1	1	1	1	1	1	1	1
27	27	1	1	1	1	1	1	1	1	1	1
28	28	1	1	1	1	1	1	1	1	1	1
29	29	1	1	1	1	1	1	1	1	1	1
30	30	1	1	1	1	1	1	1	1	1	1
31	31	1	1	1	1	1	1	1	1	1	1
32	32	1	1	1	1	1	1	1	1	1	1
33	33	1	1	1	1	1	1	1	1	1	1
34	34	1	1	1	1	1	1	1	1	1	1
35	35	1	1	1	1	1	1	1	1	1	1
36	36	1	1	1	1	1	1	1	1	1	1
37	37	1	1	1	1	1	1	1	1	1	1
38	38	1	1	1	1	1	1	1	1	1	1
39	39	1	1	1	1	1	1	1	1	1	1
40	40	1	1	1	1	1	1	1	1	1	1
41	41	1	1	1	1	1	1	1	1	1	1
42	42	1	1	1	1	1	1	1	1	1	1
43	43	1	1	1	1	1	1	1	1	1	1
44	44	1	1	1	1	1	1	1	1	1	1
45	45	1	1	1	1	1	1	1	1	1	1
46	46	1	1	1	1	1	1	1	1	1	1
47	47	1	1	1	1	1	1	1	1	1	1
48	48	1	1	1	1	1	1	1	1	1	1
49	49	1	1	1	1	1	1	1	1	1	1
50	50	1	1	1	1	1	1	1	1	1	1
51	51	1	1	1	1	1	1	1	1	1	1
52	52	1	1	1	1	1	1	1	1	1	1
53	53	1	1	1	1	1	1	1	1	1	1
54	54	1	1	1	1	1	1	1	1	1	1
55	55	1	1	1	1	1	1	1	1	1	1
56	56	1	1	1	1	1	1	1	1	1	1
57	57	1	1	1	1	1	1	1	1	1	1
58	58	1	1	1	1	1	1	1	1	1	1
59	59	1	1	1	1	1	1	1	1	1	1
60	60	1	1	1	1	1	1	1	1	1	1
61	61	1	1	1	1	1	1	1	1	1	1
62	62	1	1	1	1	1	1	1	1	1	1
63	63	1	1	1	1	1	1	1	1	1	1
64	64	1	1	1	1	1	1	1	1	1	1
65	65	1	1	1	1	1	1	1	1	1	1
66	66	1	1	1	1	1	1	1	1	1	1
67	67	1	1	1	1	1	1	1	1	1	1
68	68	1	1	1	1	1	1	1	1	1	1

Table B22

RUN NO. 1. POINT 3.

BOLNDARY LAYER PROPERTIES

	LINEAR INTERPOLATION TO WALL	STANDARD SUBLAYER FUNCTION FROM WALL TO $y^+ = 35$
FREE STREAM VELOCITY	99.365	99.365
FREE STREAM TEMPERATURE	72.200	
WALL TEMPERATURE	95.630	
WALL HEAT FLUX	.07913	
FREE STREAM DENSITY	.07493	
FREE STREAM KINEMATIC VISCOSITY	.0001637	
DENSITY OF FLUID AT WALL	.07177	
KINEMATIC VISCOSITY OF FLUID AT WALL	.0001766	
WALL/FREE STREAM DENSITY RATIO	.95781	
LOCATION REYNOLDS NUMBER (REX)	3440281.97	
INPUT VALUE OF VELOCITY DELTA	1.80000	
INPLT VALUE OF TEMPERATURE DELTA	2.15000	
CALCULATED DELTA		1.33179
DELTA 99.5% INPUT	.00000	
DISPLACEMENT THICKNESS (DELSTAR)	.14511	.14498
MOMENTUM THICKNESS (THETA)	.10836	.10894
ENERGY-DISSIPATION THICKNESS	.19802	.19856
ENTHALPY THICKNESS	.00565	.00567
SHAPE FACTOR 12 (DELSTAR/THETA)	1.33910	1.33073
SHAPE FACTOR 32 (ENERGY/THETA)	1.82738	1.82262
MOMENTUM THICKNESS REYNOLDS NUMBER	5482.26	5511.75
DISPLACEMENT THICKNESS REYNOLDS NUMBER	7341.33	7334.64
SKIN FRICTION COEFFICIENT	.003181	
FRICTION VELOCITY	4.04924	
LAW OF THE WALL CONSTANT (K)	.41000	
LAW OF THE WALL CONSTANT (C)	5.00000	
WAKE STRENGTH		.08475
CLAUSERS 'DELTA' INTEGRAL	-3.20155	-3.42684
CLAUSERS 'G' INTEGRAL	16.42084	17.96570
DISPLACEMENT THICKNESS - CONSTANT DENSITY	.13513	.13965
MOMENTUM THICKNESS - CONSTANT DENSITY	.10921	.10961
SHAPE FACTOR 12 - CONSTANT DENSITY	1.23740	1.27169

LOCATION -x- 68.00000

Te = 4.2%

Table B23

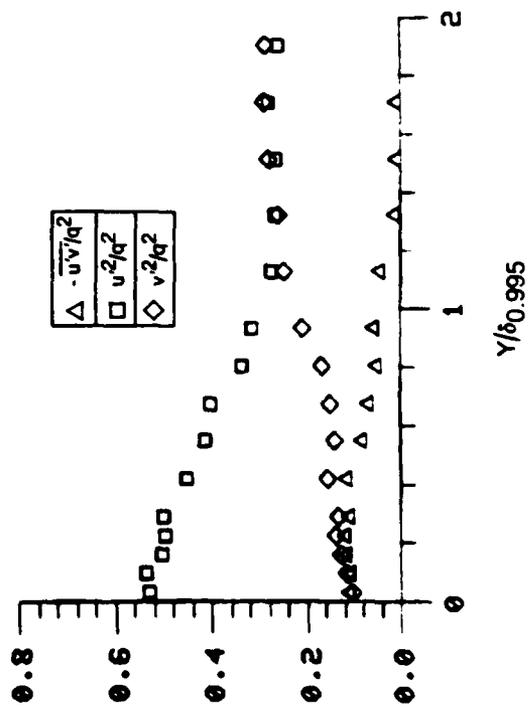
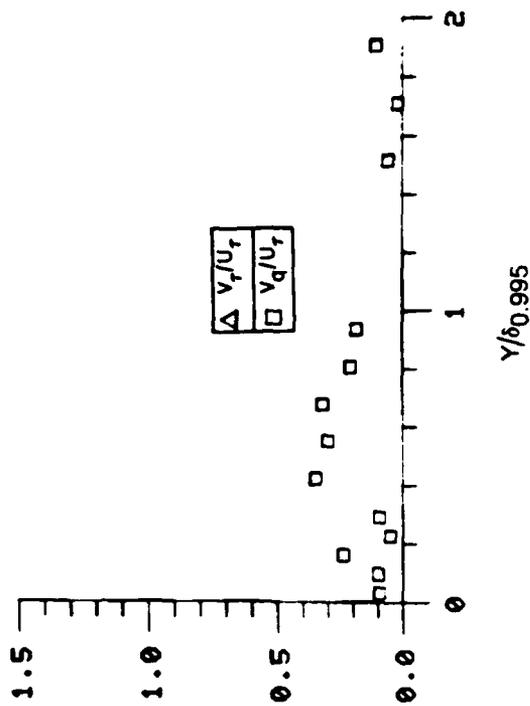
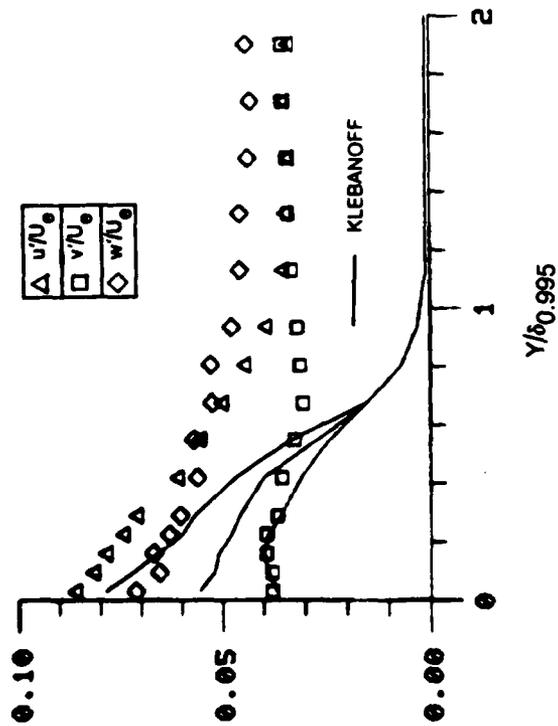
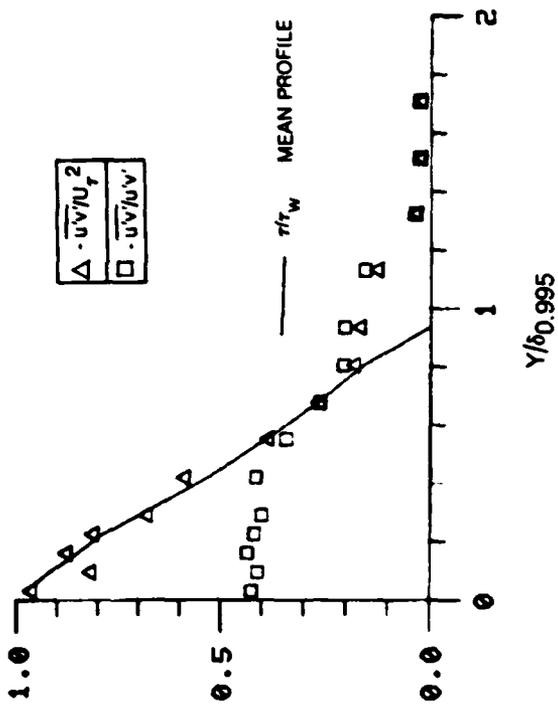


Figure B16A. Boundary Layer Turbulence Quantities, $x = 68$ in, $T_e = 4.2\%$

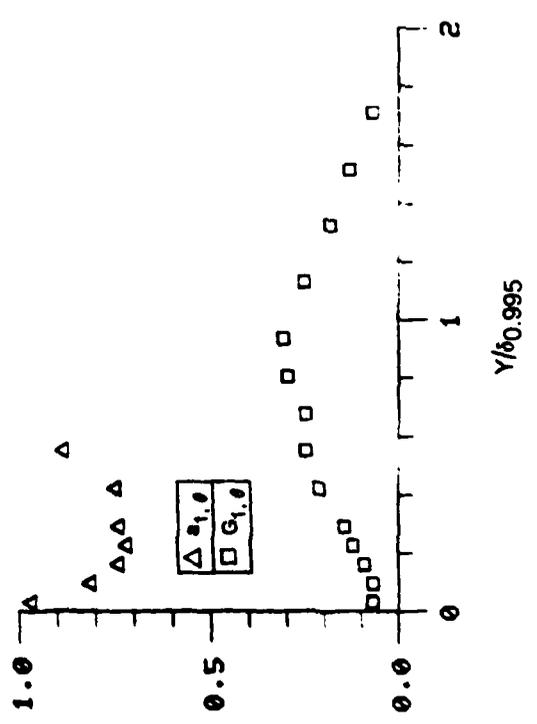
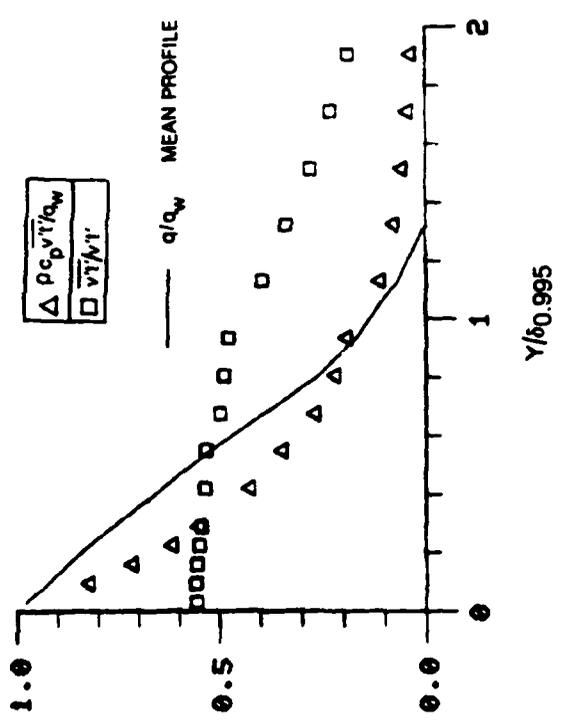
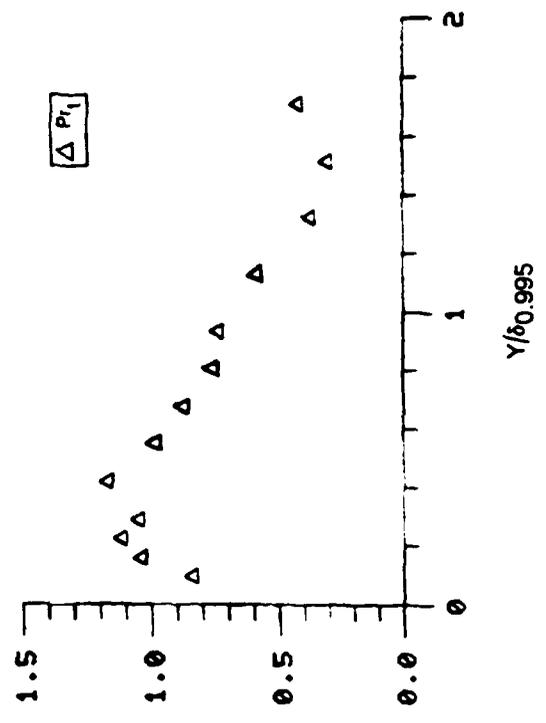
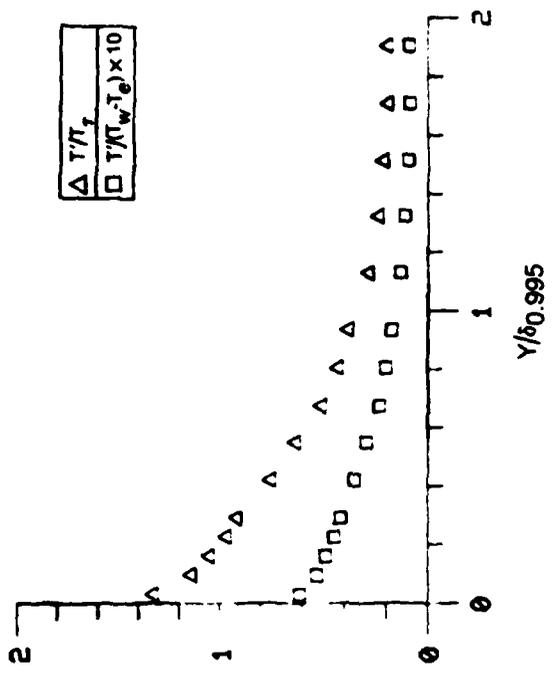


Figure B16B. Boundary Layer Turbulence Quantities, $x = 68$ in, $T_e = 4.2\%$

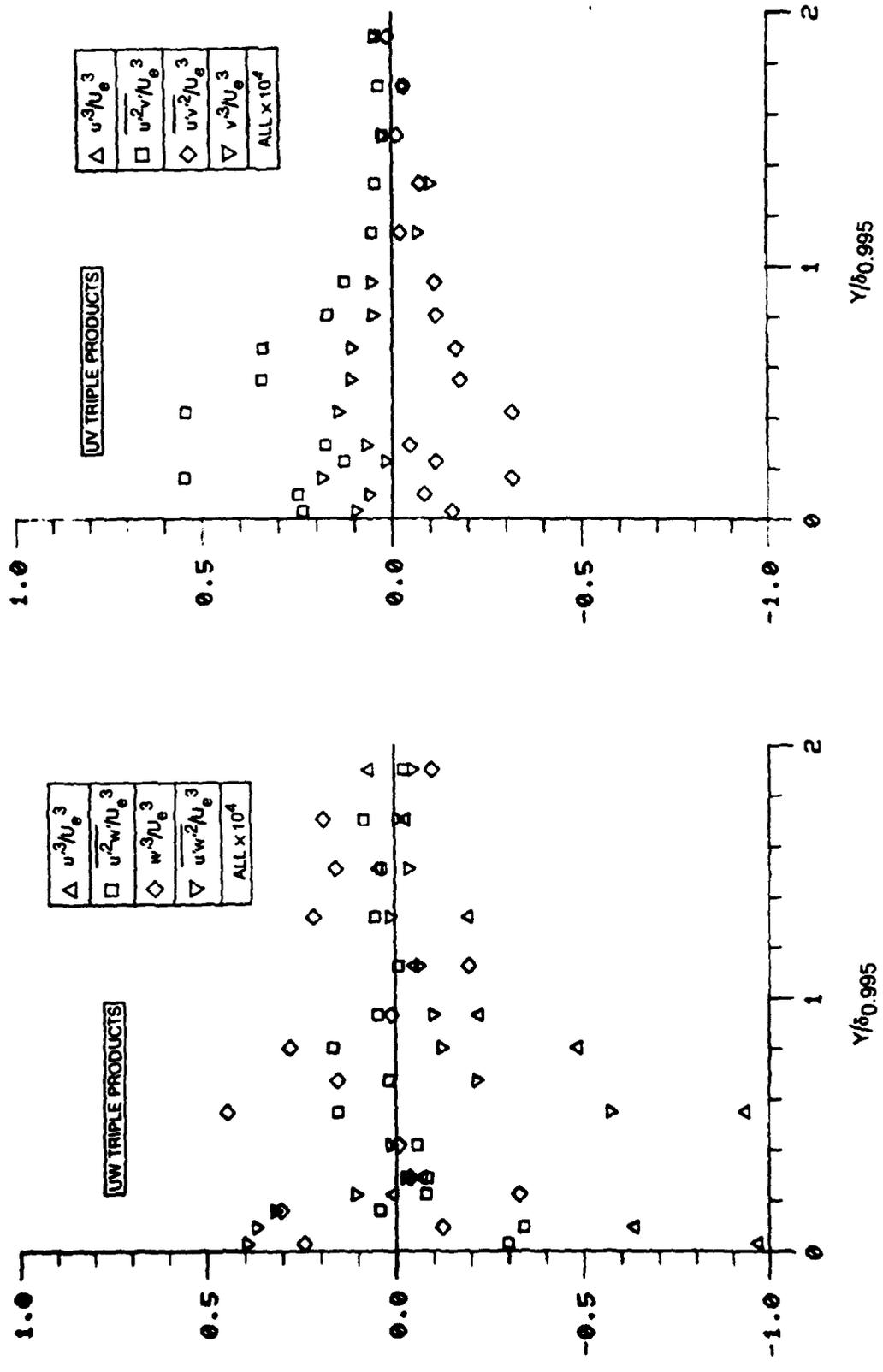


Figure B16C. Boundary Layer Triple Product Distributions $x = 68$ in, $Te = 4.2\%$

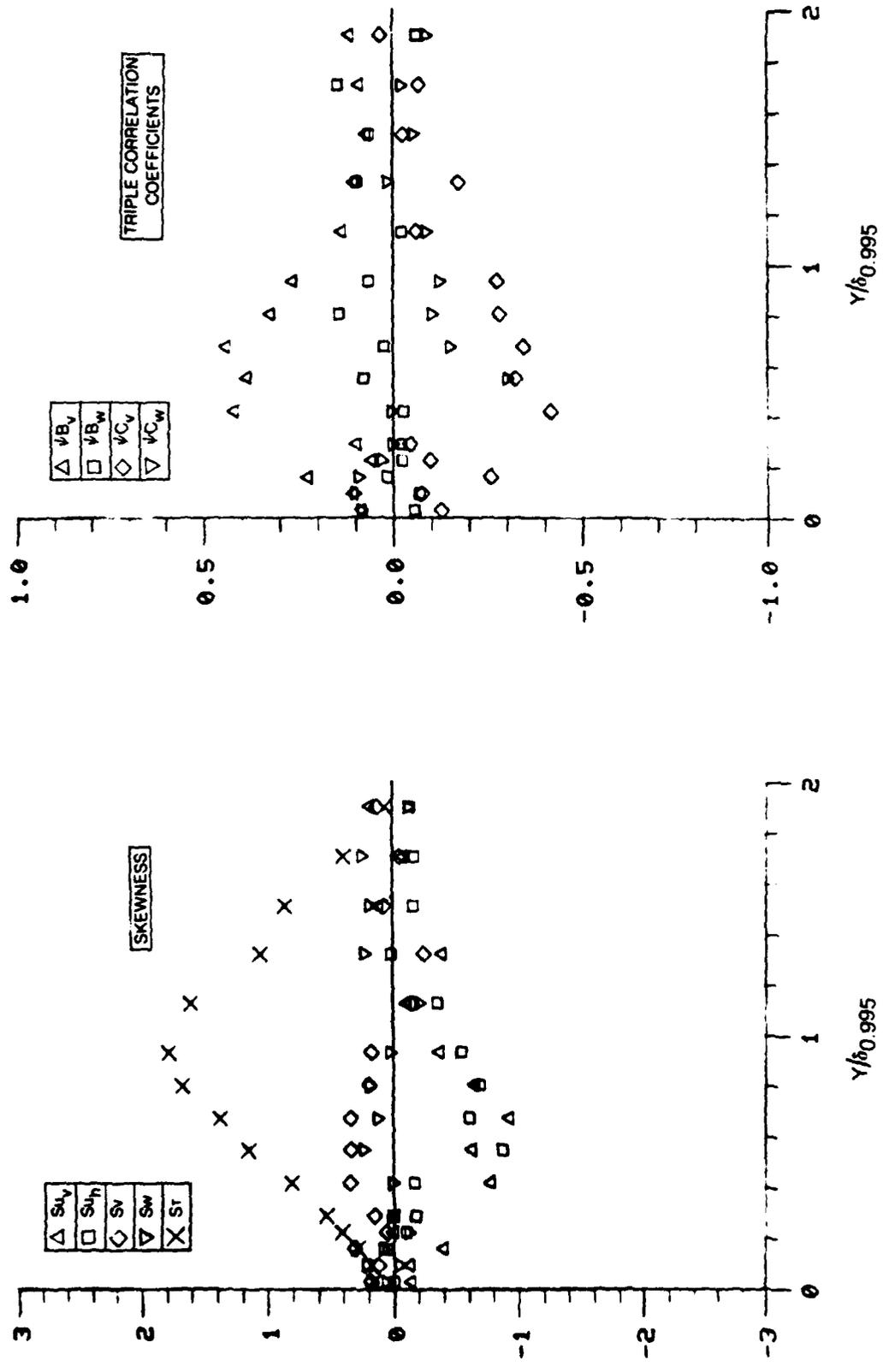


Figure B16D. Boundary Layer Skewness and Triple Product Correlation Coefficient Distributions $x = 68$ in, $T_e = 4.2\%$

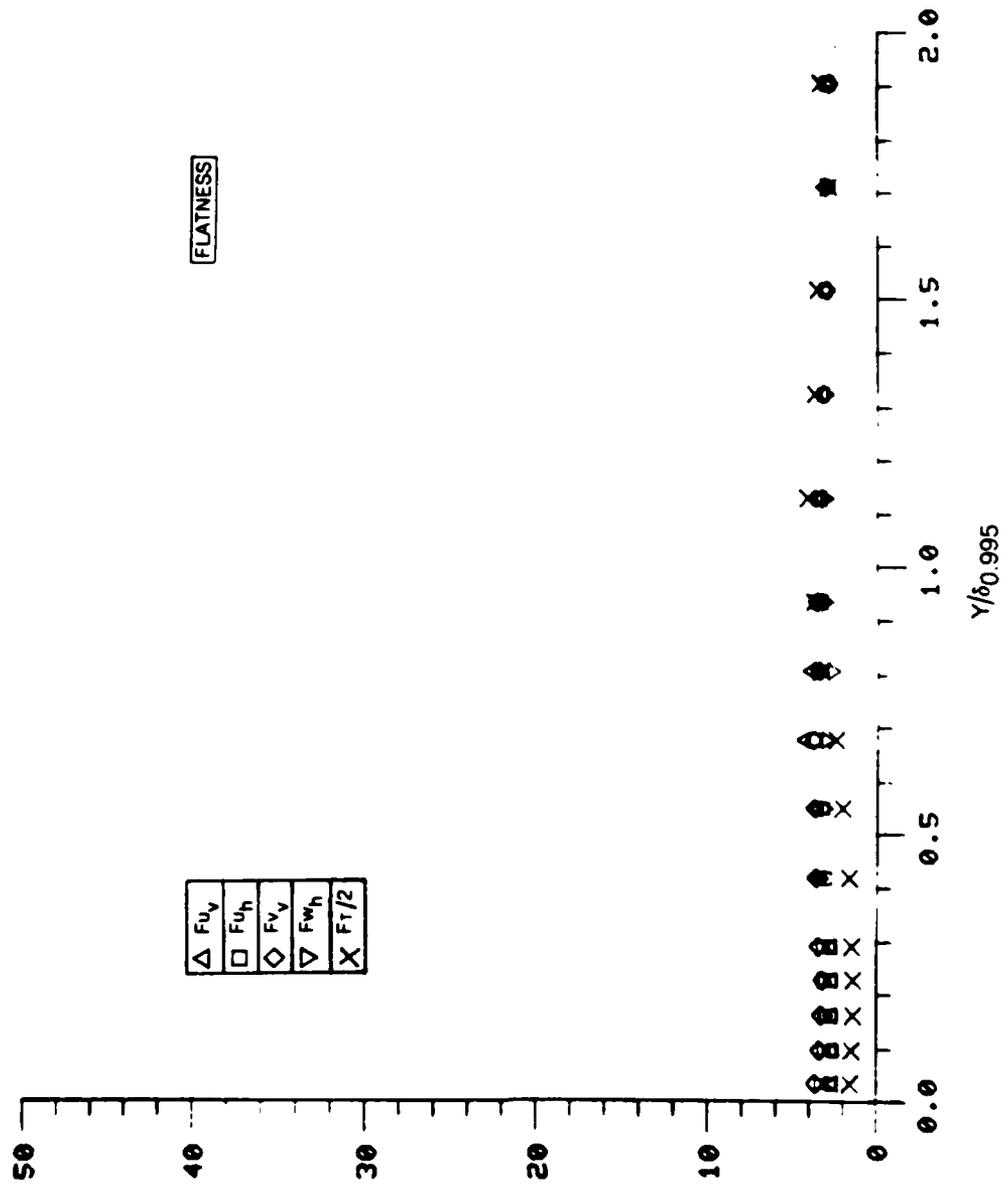


Figure B16E. Boundary Layer Flatness Distributions $x = 68$ in, $T_e = 4.2\%$

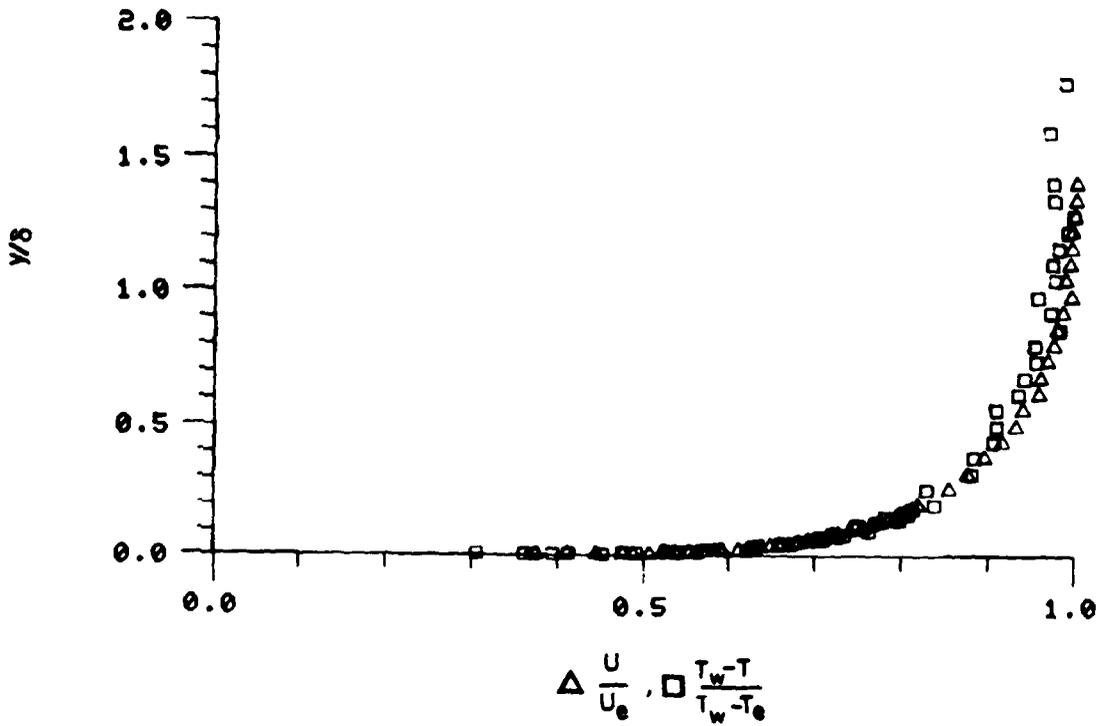
Fluctuating Profile Data

x = 68 in., Te = 4.2%

INCHES	DELTA	PSICV	PSICH	FU V	FU H	FL V	FL H	PSI V	PSI H	SV	SU V	DELTA	INCHES
1.0	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	1.0
1.1	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	1.1
1.2	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	1.2
1.3	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	1.3
1.4	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	1.4
1.5	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	1.5
1.6	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	1.6
1.7	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	1.7
1.8	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	1.8
1.9	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	1.9
2.0	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	2.0
2.1	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	2.1
2.2	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	2.2
2.3	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	2.3
2.4	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	2.4
2.5	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	2.5
2.6	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	2.6
2.7	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	2.7
2.8	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	2.8
2.9	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	2.9
3.0	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	3.0

Table B24B

VELOCITY AND TEMPERATURE RATIOS



VELOCITY AND TEMPERATURE DISTRIBUTIONS IN UNIVERSAL COORDINATES

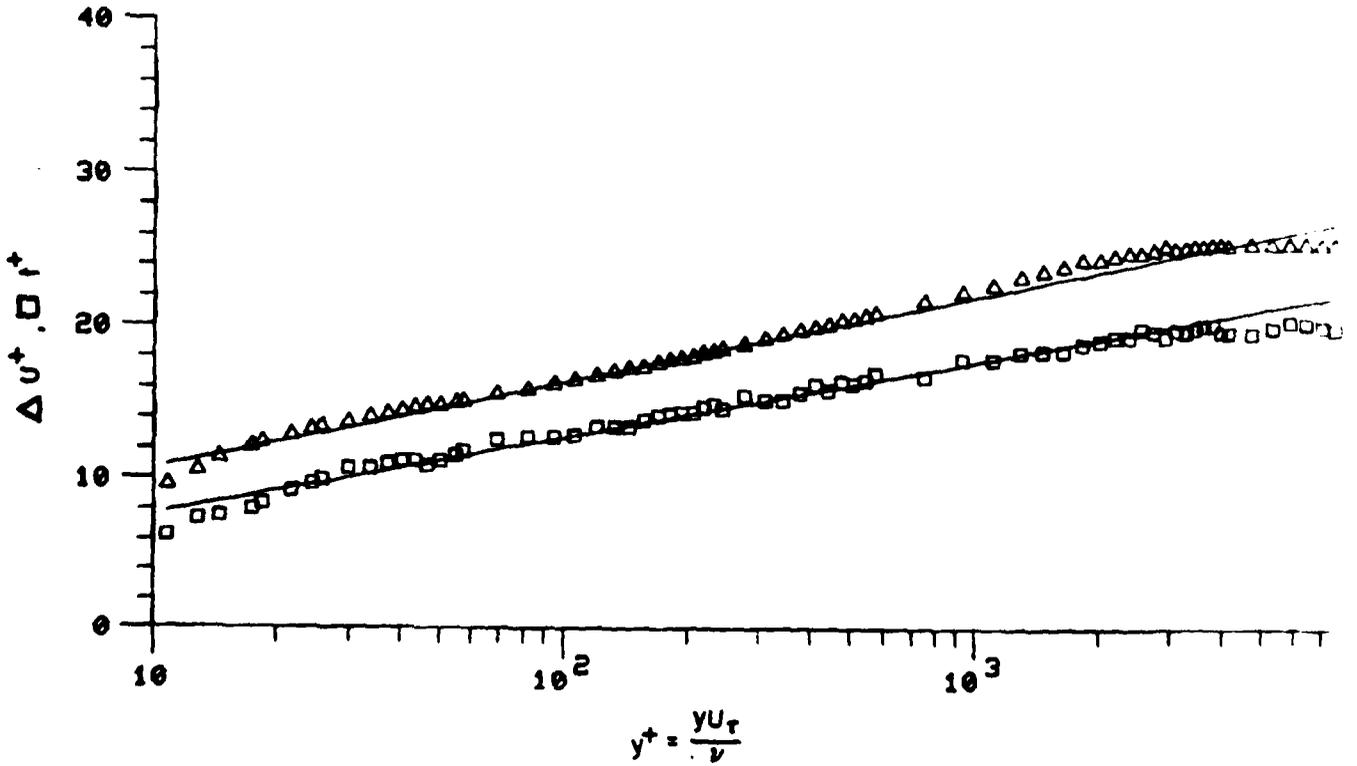


Figure B17. Mean Velocity and Temperature Profiles $x = 84$ in., $Te = 3.9\%$

Mean Profile Data

x = 85 in., Te = 3.9%

INCH	Y	DELTA	U	T	U/UE	THETA	U-TAU	T	Y
INCH	INCH	INCH	FT/SEC	DEG.F		DEG	INCH	DEG	INCH
1	1	1	37.25	87.96	.376	-15.764	9.523	6.152	10.222
2	2	2	43.81	86.62	.412	-14.873	10.434	7.152	12.426
3	3	3	47.11	86.39	.446	-14.029	11.276	7.522	14.466
4	4	4	47.12	85.87	.476	-13.261	12.034	7.963	16.238
5	5	5	48.24	85.41	.467	-12.573	12.634	8.353	17.346
6	6	6	50.10	84.47	.506	-12.498	13.034	8.746	18.238
7	7	7	51.60	83.93	.521	-12.114	13.299	9.146	19.238
8	8	8	52.99	83.57	.525	-12.015	13.522	9.546	20.238
9	9	9	55.55	82.69	.538	-11.702	13.605	9.946	21.238
10	10	10	57.76	82.73	.553	-11.301	14.006	10.346	22.238
11	11	11	59.22	82.42	.561	-11.100	14.200	10.666	23.238
12	12	12	60.15	82.13	.567	-11.000	14.400	11.000	24.238
13	13	13	61.81	81.46	.574	-10.783	14.555	11.100	25.238
14	14	14	62.67	81.46	.583	-10.564	14.755	11.100	26.238
15	15	15	63.72	81.69	.586	-10.551	14.966	11.100	27.238
16	16	16	65.55	81.41	.592	-10.338	15.177	11.100	28.238
17	17	17	67.43	81.47	.610	-9.859	15.388	11.100	29.238
18	18	18	69.91	80.29	.622	-9.558	15.599	11.100	30.238
19	19	19	71.91	80.34	.636	-9.222	16.084	12.000	31.238
20	20	20	74.14	80.18	.646	-8.909	16.398	12.784	32.238
21	21	21	76.49	79.49	.656	-8.655	16.655	13.374	33.238
22	22	22	78.97	79.56	.669	-8.364	16.942	13.784	34.238
23	23	23	81.39	79.55	.677	-8.185	17.121	13.322	35.238
24	24	24	83.75	79.04	.681	-8.077	17.229	13.756	36.238
25	25	25	86.21	78.70	.690	-7.853	17.454	14.046	37.238
26	26	26	88.69	78.33	.699	-7.613	17.694	14.346	38.238
27	27	27	91.19	78.39	.706	-7.504	17.900	14.555	39.238
28	28	28	93.70	78.42	.707	-7.402	18.089	14.666	40.238
29	29	29	96.22	78.01	.720	-7.114	18.191	14.800	41.238
30	30	30	98.75	77.79	.726	-6.924	18.333	14.855	42.238
31	31	31	101.27	77.11	.740	-6.574	18.533	15.388	43.238
32	32	32	103.74	77.43	.755	-6.200	19.107	15.117	44.238
33	33	33	106.16	77.49	.763	-5.991	19.316	15.066	45.238
34	34	34	108.55	76.93	.776	-5.664	19.642	15.567	46.238
35	35	35	110.91	76.28	.785	-5.446	19.659	16.094	47.238
36	36	36	113.27	76.69	.792	-5.267	20.040	15.746	48.238
37	37	37	115.64	76.07	.806	-5.022	20.285	16.277	49.238
38	38	38	118.01	75.16	.806	-4.920	20.387	16.197	50.238
39	39	39	120.38	75.93	.811	-4.780	20.526	16.387	51.238
40	40	40	122.74	75.36	.816	-4.605	20.702	16.676	52.238
41	41	41	125.11	74.99	.827	-4.370	21.600	16.703	53.238
42	42	42	127.48	74.56	.836	-4.134	22.133	17.754	54.238
43	43	43	129.85	74.14	.854	-3.905	22.666	17.816	55.238
44	44	44	132.22	73.71	.894	-3.677	23.200	18.277	56.238
45	45	45	134.59	73.28	.914	-3.446	23.733	18.277	57.238
46	46	46	136.96	72.85	.931	-3.217	24.266	18.333	58.238
47	47	47	139.33	72.42	.939	-2.986	24.800	18.333	59.238
48	48	48	141.70	72.03	.955	-2.755	25.333	18.333	60.238
49	49	49	144.07	71.64	.955	-2.524	25.866	18.333	61.238
50	50	50	146.44	71.25	.975	-2.293	26.400	18.333	62.238
51	51	51	148.81	70.86	.978	-2.062	26.933	18.333	63.238
52	52	52	151.18	70.47	.978	-1.831	27.466	18.333	64.238
53	53	53	153.55	70.08	.984	-1.600	28.000	19.222	65.238
54	54	54	155.92	69.69	.984	-1.369	28.533	19.222	66.238
55	55	55	158.29	69.30	.993	-1.138	29.066	19.222	67.238
56	56	56	160.66	68.91	.994	-0.907	29.600	19.222	68.238
57	57	57	163.03	68.52	.994	-0.676	30.133	19.222	69.238
58	58	58	165.40	68.13	.993	-0.445	30.666	19.222	70.238
59	59	59	167.77	67.74	.994	-0.214	31.200	19.222	71.238
60	60	60	170.14	67.35	.994	0.017	31.733	19.222	72.238
61	61	61	172.51	66.96	.999	0.248	32.266	19.222	73.238
62	62	62	174.88	66.57	.999	0.479	32.800	19.222	74.238
63	63	63	177.25	66.18	1.001	0.710	33.333	19.222	75.238
64	64	64	179.62	65.79	1.001	0.941	33.866	19.222	76.238
65	65	65	181.99	65.40	1.001	1.172	34.400	19.222	77.238
66	66	66	184.36	65.01	1.001	1.403	34.933	19.222	78.238
67	67	67	186.73	64.62	1.001	1.634	35.466	19.222	79.238
68	68	68	189.10	64.23	1.001	1.865	36.000	19.222	80.238

Table B25

RUN NO. 1. POINT 2.

BOLNDARY LAYER PROPERTIES

	LINEAR INTERPOLATION TO WALL	STANDARD SUBLAYER FUNCTION FROM WALL TO $Y^+ = 35$
FREF STREAM VELOCITY =	98.987	98.987
FREE STREAM TEMPERATURE =	71.450	
WALL TEMPERATURE =	95.270	
WALL HEAT FLUX =	.07956	
FREE STREAM DENSITY =	.07504	
FREE STREAM KINEMATIC VISCOSITY =	.0001633	
DENSITY OF FLUID AT WALL =	.07182	
KINEMATIC VISCOSITY OF FLUID AT WALL =	.0001764	
WALL/FREE STREAM DENSITY RATIO =	.95708	
LOCATION REYNOLDS NUMBER (REX) =	4244178.31	
INPUT VALUE OF VELOCITY DELTA =	2.05000	
INPLT VALUE OF TEMPERATURE DELTA =	2.10000	
CALCULATED DELTA =		1.58079
DELTA 99.5% INPUT =	.00000	
DISPLACEMENT THICKNESS (DELSTAR) =	.17986	.16012
MOMENTUM THICKNESS (THETA) =	.13446	.13465
ENERGY-DISSIPATION THICKNESS =	.24493	.24498
ENTHALPY THICKNESS =	.00715	.00715
SHAPE FACTOR 12 (DELSTAR/THETA) =	1.33749	1.33772
SHAPE FACTOR 32 (ENERGY/THETA) =	1.82137	1.81943
MOMENTUM THICKNESS REYNOLDS NUMBER =	6794.61	6803.22
DISPLACEMENT THICKNESS REYNOLDS NUMBER =	9087.75	9100.78
SKIN FRICTION COEFFICIENT =	.002989	
FRICTION VELOCITY =	3.91148	
LAW OF THE WALL CONSTANT (K) =	.41000	
LAW OF THE WALL CONSTANT (C) =	5.00000	
WAKE STRENGTH =		.17313
CLAUSERS 'DELTA' INTEGRAL =	-4.22327	-4.37735
CLAUSERS 'G' INTEGRAL =	23.76288	23.81295
DISPLACEMENT THICKNESS - CONSTANT DENSITY =	.16980	.17297
MOMENTUM THICKNESS - CONSTANT DENSITY =	.13561	.13579
SHAPE FACTOR 12 - CONSTANT DENSITY =	1.25212	1.27363

LOCATION -X- 84.00000

Te = 3.9%

Table B26

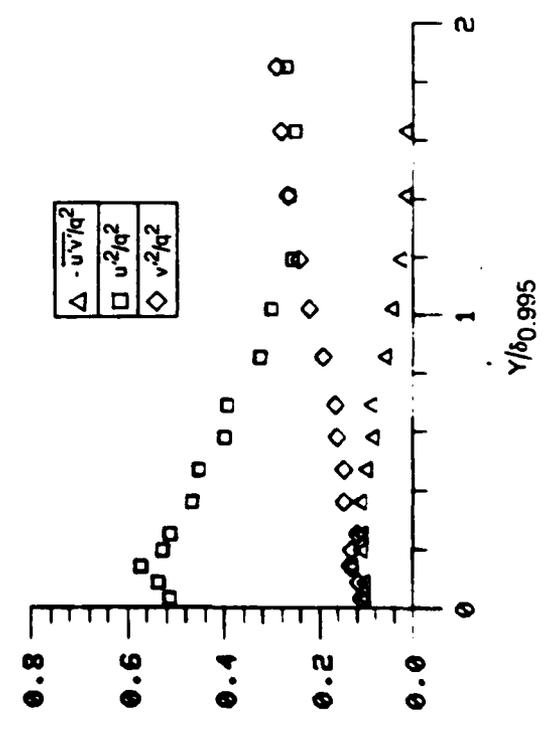
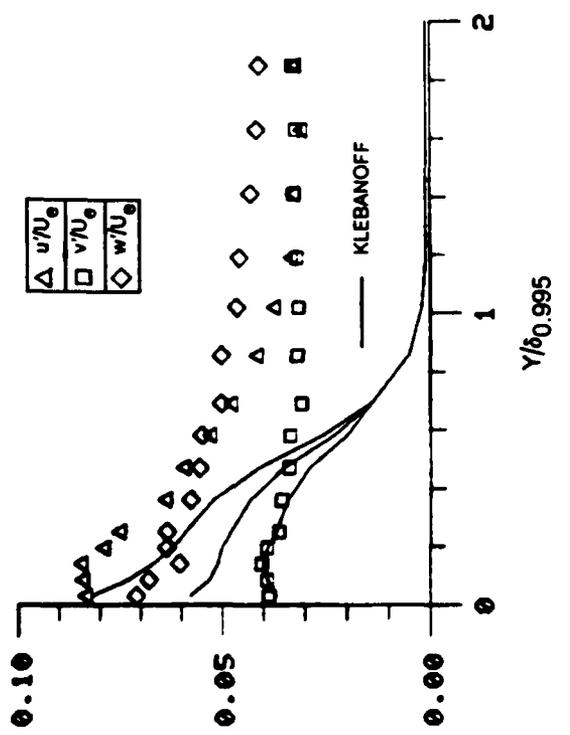
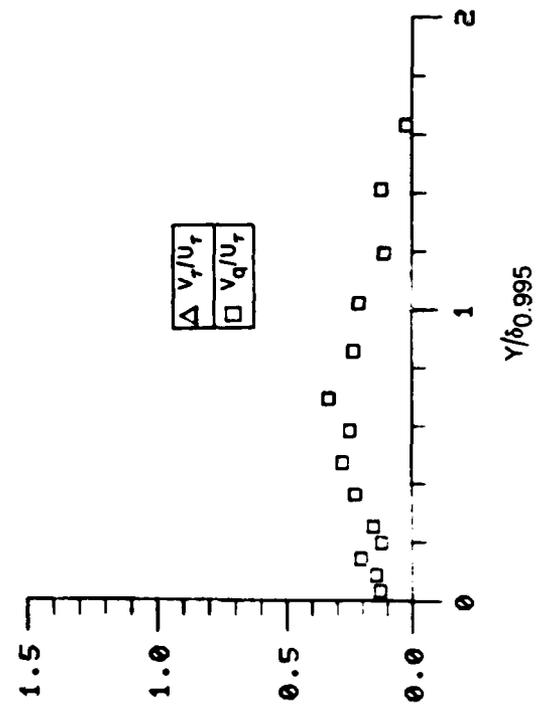
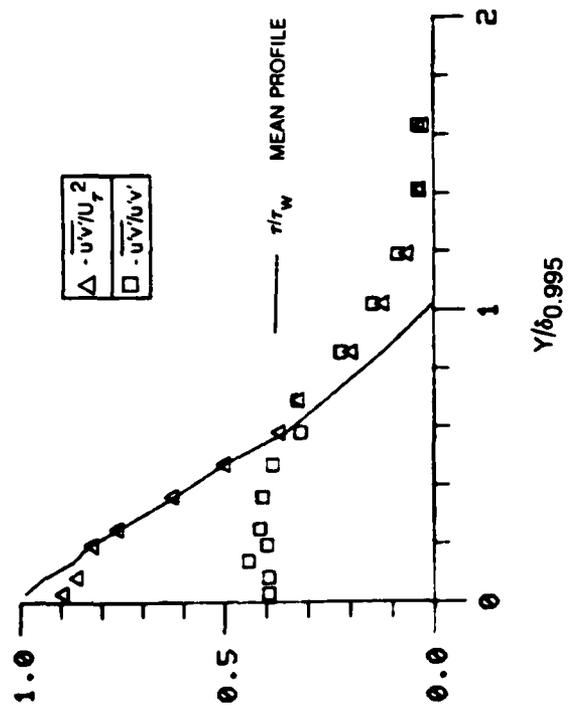


Figure B18A. Boundary Layer Turbulence Quantities, $x = 84$ in, $T_e = 3.9\%$

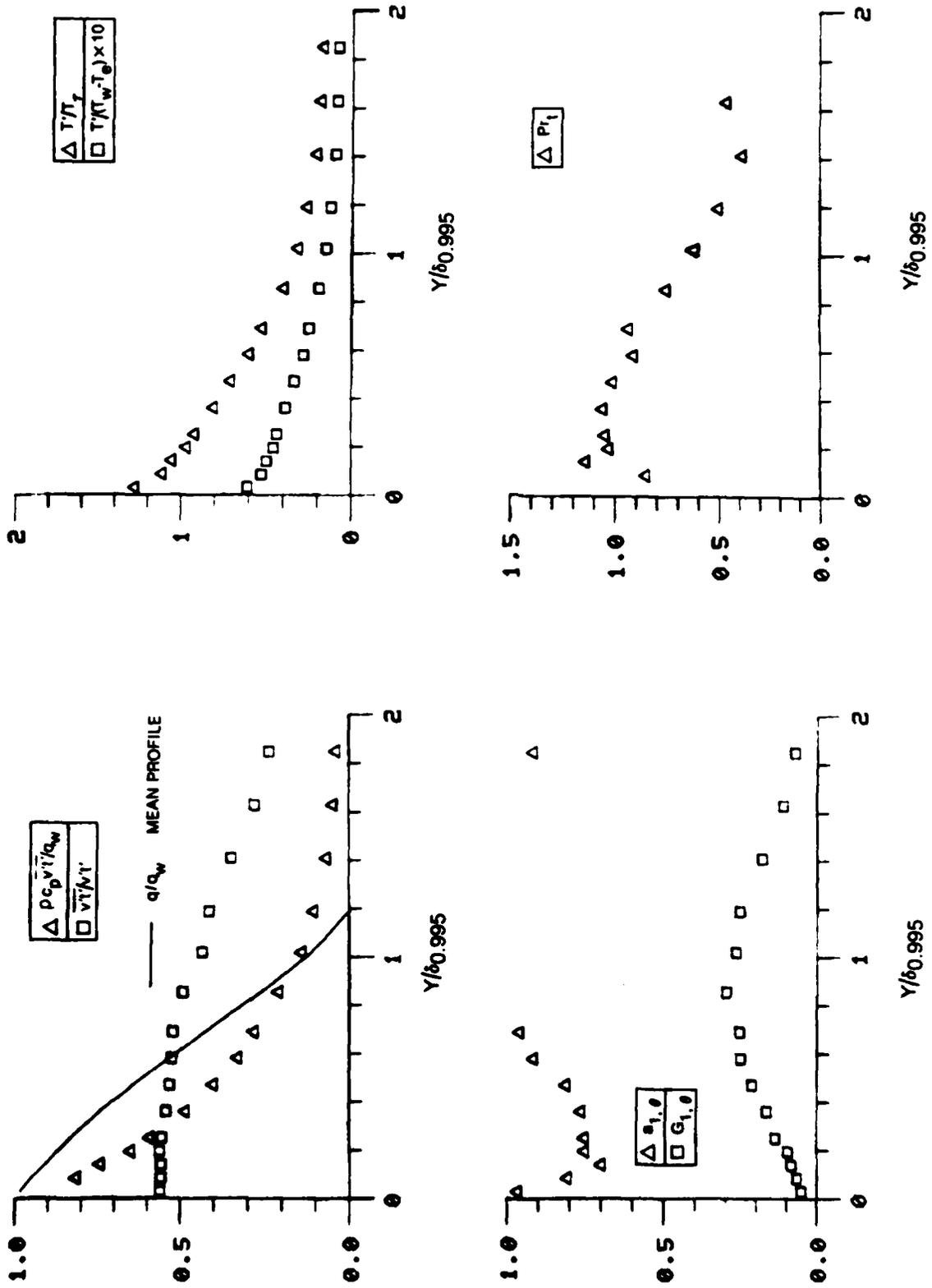


Figure B18B. Boundary Layer Turbulence Quantities $x = 84$ in, $T_e = 3.97$

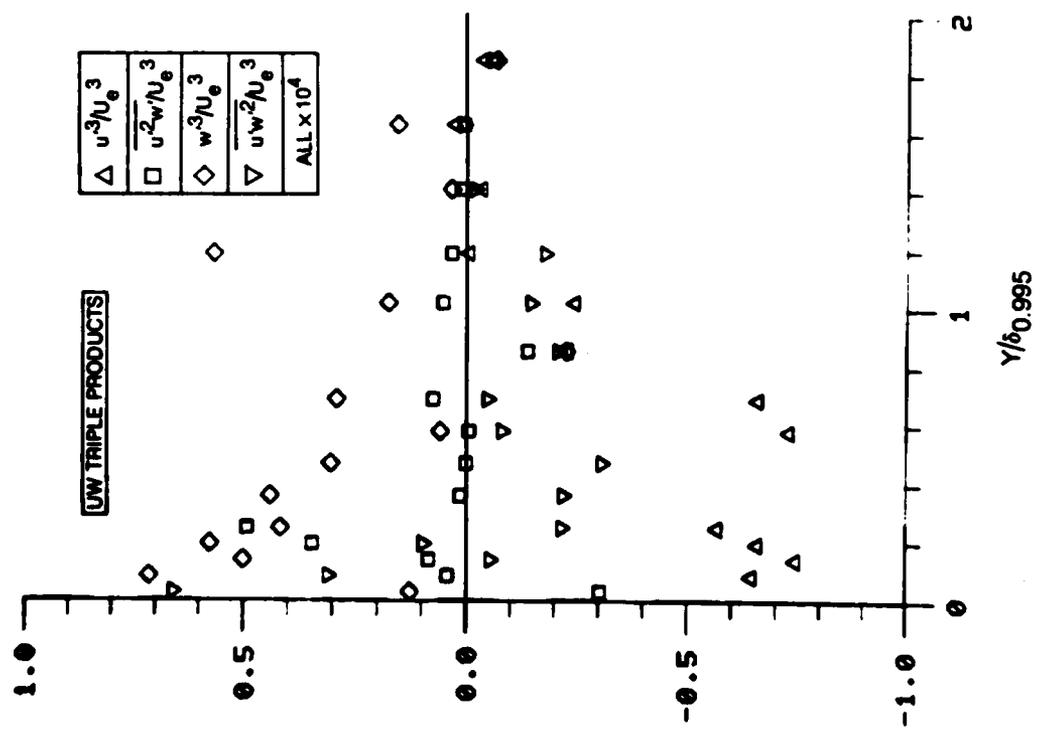
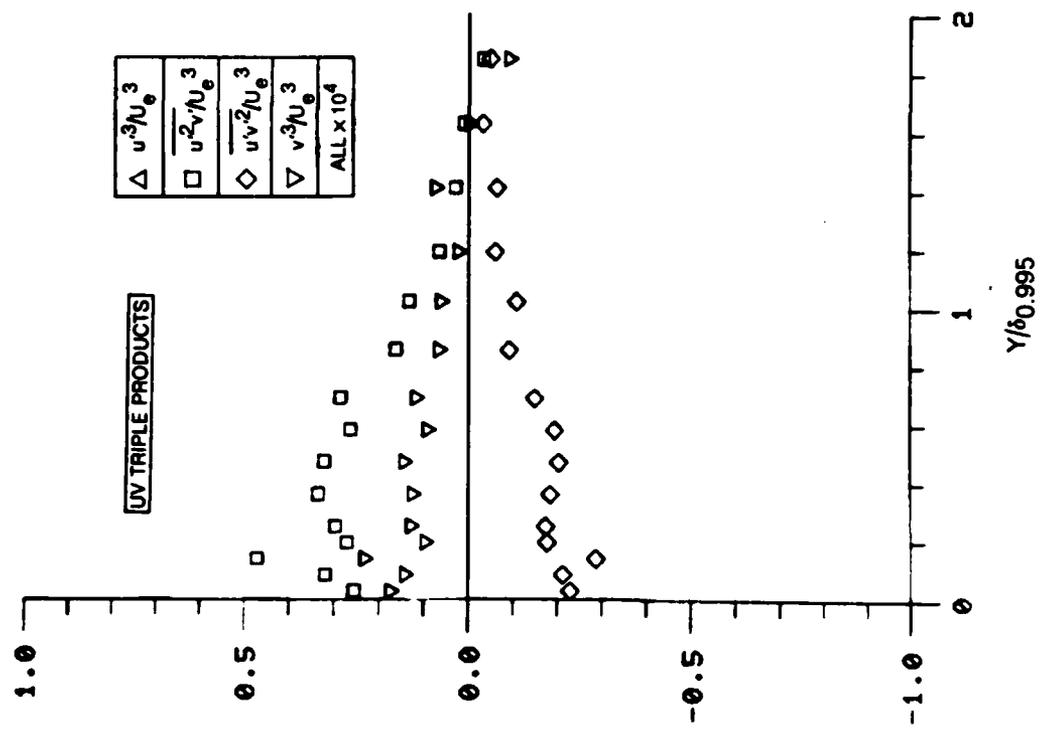


Figure B18C. Boundary Layer Triple Product Distributions $x = 84$ in, $T_e = 3.9\%$

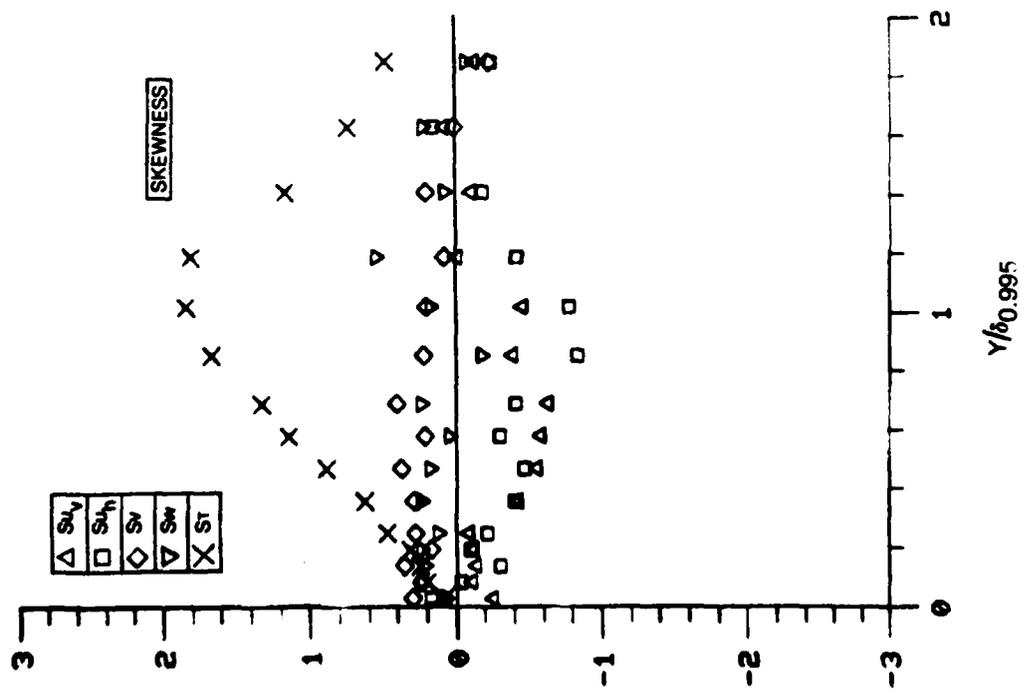
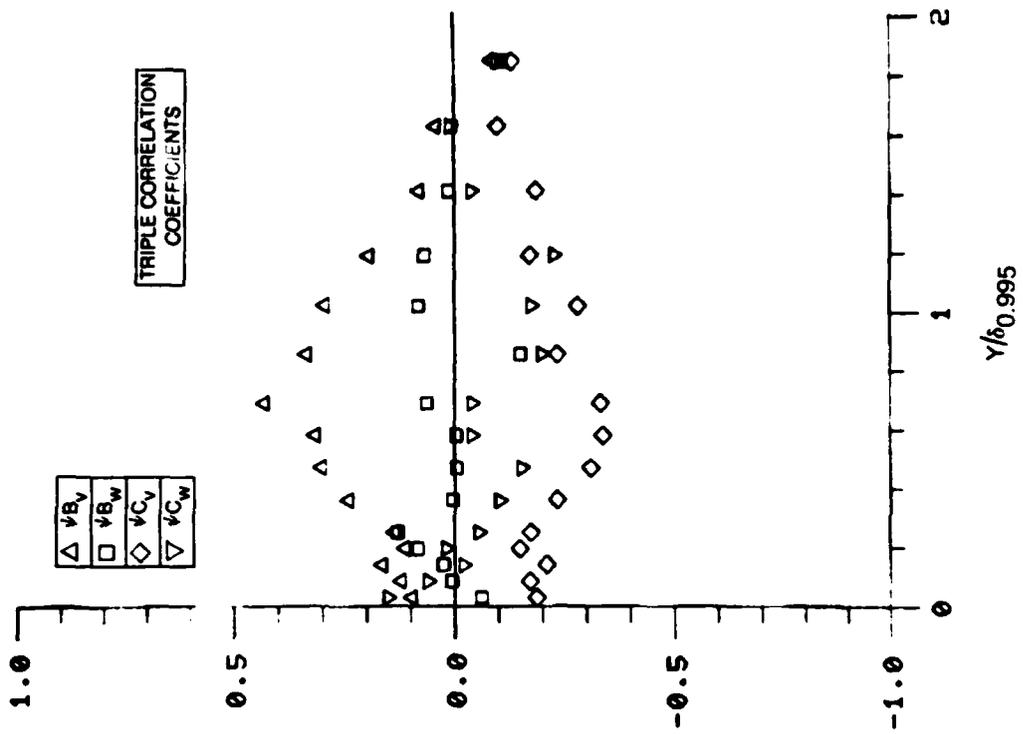


Figure B18D. Boundary Layer Skewness and Triple Product Correlation Coefficient Distributions $x = 84$ in, $Te = 3.9\%$

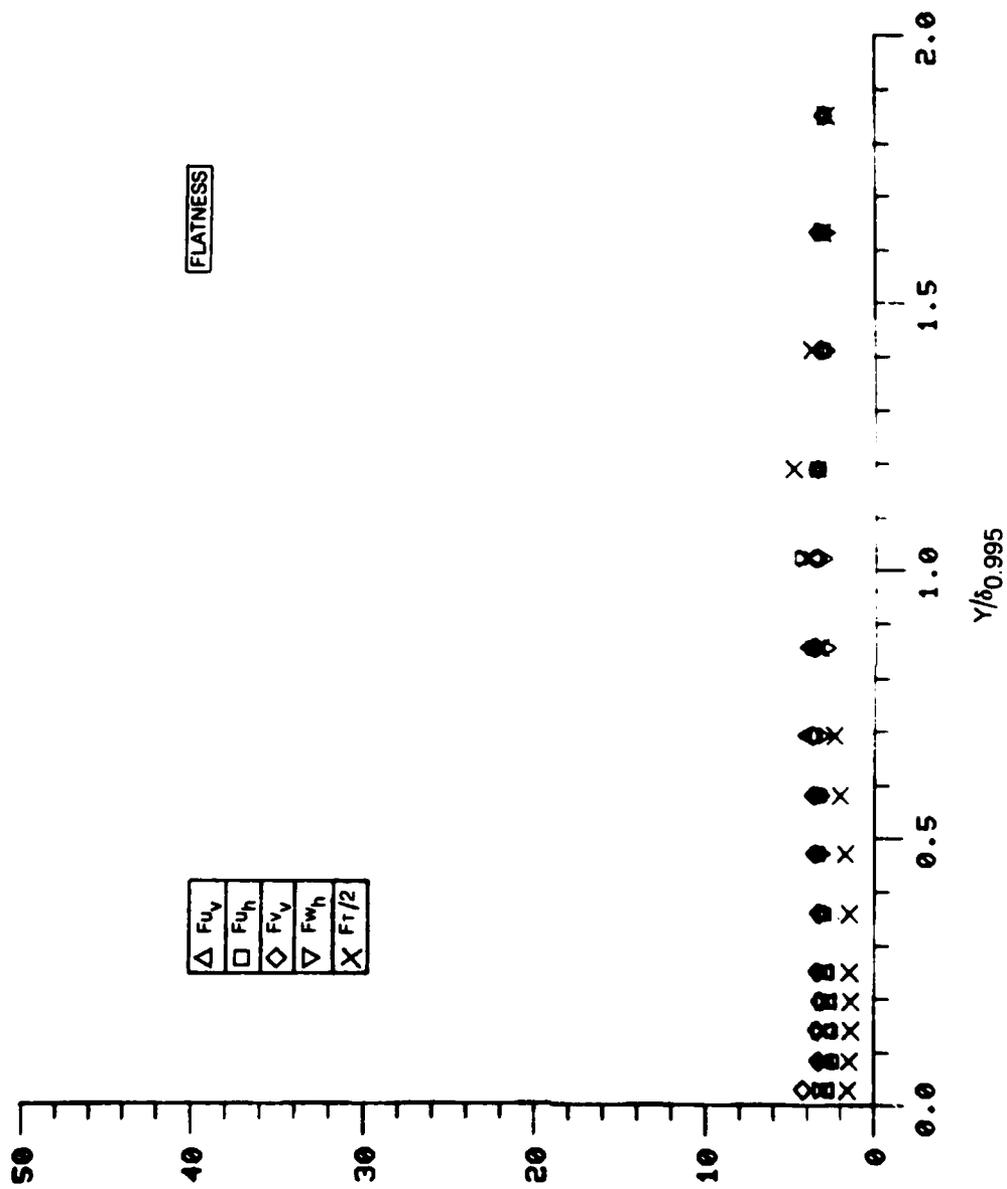


Figure B18E. Boundary Layer Flatness Distributions $x = 84$ in, $T_e = 3.9\%$

Fluctuating Profile Data

$x = 84 \text{ in.}, Te = 3.9\%$

Y/ INCHES	Y/ DELTA	U ² /UE	V ² /UE	W ² /UE	$\sqrt{U^2+V^2+W^2}$ /UE	U ² /V ²	V ² /W ²	U ² /W ²
0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.05	0.05	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.10	0.10	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.15	0.15	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.20	0.20	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.25	0.25	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.30	0.30	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.35	0.35	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.40	0.40	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.45	0.45	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.50	0.50	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.55	0.55	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.60	0.60	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.65	0.65	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.70	0.70	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.75	0.75	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.80	0.80	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.85	0.85	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.90	0.90	0.00	0.00	0.00	0.00	0.00	0.00	0.00
0.95	0.95	0.00	0.00	0.00	0.00	0.00	0.00	0.00
1.00	1.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00

Table B27A

DATE
ILME