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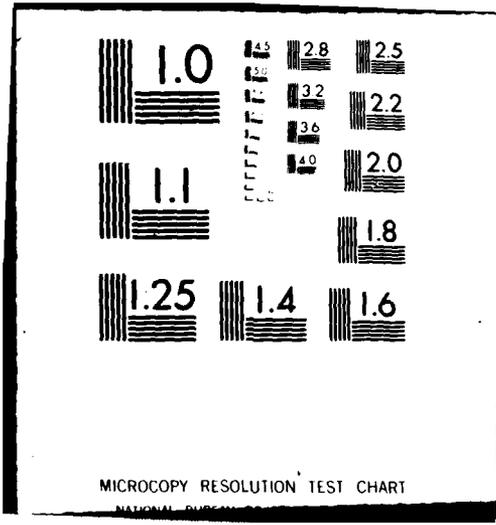
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COMPARISON OF NUMERICAL RESULTS AND MEASURED DATA FOR
SMOOTH AND INDENTED NOSETIPS

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ABSTRACT

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Numerical calculations, using an unsteady implicit numerical algorithm which solves either the inviscid or the thin-layer Navier-Stokes equations, were performed for smooth and severely indented nosetips at hypersonic speed and zero angle of attack. The computed results of inviscid and laminar flowfield are compared to wind tunnel measured data for surface pressure, shock location, heat transfer and density distribution in the shock layer. Good agreements between the calculated and measured flowfield are obtained for smooth nosetip without flow separation. Difficulties in the simulation of severely indented nosetips with large separation bubble or sharp corner are discussed.

NOMENCLATURE

C_p = Pressure coefficient, $p - p_\infty / \frac{1}{2} \rho_\infty U_\infty^2$

J, K = Index of grid in the streamwise and body-to-shock direction respectively

L = Reference length, 2.56 cm (1 in.)

M = Mach number

n = normal direction

p = pressure

Pr = Prandtl number

R = Radius of circular arc

Re = Unit Reynolds number, U_∞ / ν

Rs = Radius of sphere

S = Arc length

St = Stanton number = $\frac{(\mu_l + \mu_t) / \mu_\infty}{Re_\infty (Pr_l + Pr_t)} \left[\frac{T_o}{T_\infty} - \frac{T_w}{T_\infty} \right]^{-1} \frac{\partial (T/T_\infty)}{\partial (n/L)}$

t = time

T = temperature

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U = Velocity
X,Y = Axial and radial coordinates respectively
 ρ = Density
 ϵE = Coefficient for explicit dissipation
 ϵI = Coefficient for implicit dissipation
 ν = Kinematic viscosity of fluid

Subscript

∞ = free stream
 l = laminar flow
 o = stagnation point
 t = turbulent flow
 w = wall

INTRODUCTION

Because of ablation, the nosetip of a spherical body undergoes continuous change during re-entry. The shape of the nosetip has a great influence on the flowfield over the entire body, i.e., the nose region and thus the afterbody. In order to understand the flowfield about indented nosetip shapes that are likely to occur during the re-entry process, considerable effort has been expended both experimentally and theoretically.¹⁻⁴ (also see references listed in Ref. 1 and 2).

Among the experimental work, Refs. 3 and 4 provide a comprehensive set of measurements for a systematic change of nosetip shapes. This includes surface pressures, heating rates, flowfield pictures using electro-optical techniques and velocity mapping using Laser Doppler Velocimeter. These experiments are intended to provide basic information about the important flow features to be expected in the flowfield and also serve as a useful data base for the development of a numerical code to predict the flowfield.

Among the many numerical schemes intended for indented nosetip calculation, a promising and versatile one seems to be due to Kutler et al.¹ As described in Ref. 1, Kutler et al solved the unsteady Navier-Stokes equations with the "thin-layer" approximation for nosetip of arbitrary shapes at zero incidence using the implicit factored numerical algorithm of Warming and Beam.⁵ The steady solution is obtained asymptotically in time and both viscous and inviscid flowfields can be computed using the same computer program, referred to as K-C-L code in this paper. As described in Ref. 2, when the inviscid portion of the K-C-L code was applied to the nosetip shapes reported in Ref. 3 and 4 surmountable difficulties were encountered during the course of calculation because of the presence of small radius expansion corners and a

concave compression turn in these nosetips. It was later found that a special calculation procedure is required in order to obtain reasonable solutions.

For the details of the numerical procedure used in the K-C-L code and the special calculation procedure for the indented nosetip computation, the readers should consult to Ref. 1 and 2 and will not be repeated in this paper. The same calculation procedure has since been applied to compute viscous flow over smooth and indented nosetips and several typical results will be described and compared to measured data.

II. RESULTS AND DISCUSSION

A. Smooth Nosetips

The comparison of inviscid results for surface pressure and density distribution over a sphere is reproduced here from Ref. 2 as shown in Fig. 1. The grid used in the computation is $J = 32$ (along body surface) $\times K = 12$ (across the shock layer) and the total number of time steps used is 600 (with a Courant number of 2.5). This requires about 2 minutes CPU time in CDC-7600 computer. The surface pressure is compared to the work of Inouye and Lomax⁶ and the measured data of Baer⁷ as shown in Fig. 1a. The density distribution in the shock layer is compared to the measured data of Sedney and Kahl⁸ as shown in Fig. 1b. The agreement are seen to be very good.

Viscous flow calculations based on the thin-layer approximation of Navier-Stokes equations, were performed for a hemisphere-cone under the following flow condition: $M_\infty = 5.92$, $Re_\infty = 5.5 \times 10^6/\text{ft}$ and $T_w/T_\infty = 4.78$. A grid of 28×32 was used. For laminar calculation, 400 time steps with a Courant number (CN) of 75 can be considered as the final solution (the nondimensional shock speed is in the order of 10^{-3} and the CPU time is about 5 min). As shown in Fig. 2 the calculated results for heat transfer in term of Stanton number over the surface is compared to the measured data reported in Ref. 7. Also plotted in Fig. 2 is the boundary layer calculation using Cebeci-Smith's boundary layer code as given in Ref. 7. It is seen that the agreement is satisfactory. It should be noted that for hemisphere-cone, the flow is fully attached. The surface pressure agrees well among the inviscid and laminar calculation as well as the measured data. Therefore, the good agreement in heat transfer between the boundary layer and the thin layer N-S calculation is to be expected.

In the numerical procedure, there are two kinds of dissipation used for stability purpose, i.e., the explicit and implicit dissipation terms with coefficients of ϵ^E and ϵ^I respectively. For inviscid and laminar calculation, solution can be obtained without the implicit dissipation. For turbulent calculation, it was found that the implicit dissipation must be added in order to have a converged solution. Since the implicit dissipation terms have no effect on the solution as the steady state is approached, it is therefore used in both laminar and turbulent calculation. The explicit dissipation terms, on the other hand, will always present. In viscous flow calculation, it is desirable that the explicit dissipation coefficient be kept minimum so that the real viscous effects can be simulated. From the experience for indented nosetip calculation, the value of ϵ^E cannot be too small in order to have a smooth bow shock (no wiggling) where the flow can essentially be assumed to be inviscid. Therefore, the ϵ^E value in the η direction is linearly reduced to one tenth of the input

value toward the wall in order to minimize its effects there. This device also consistent with the thin-layer approximation. When ϵE is linearly reduced, it is denoted by $\epsilon E'$ to distinguish from the uniform one. As shown in Fig. 2, the effects of $\epsilon E'$ are seen to be insignificant in the heat transfer results for hemisphere-cone with $\epsilon E'$ varies from 0.1 to 0.02 and $\epsilon I = 3 \epsilon E'$. This is not so for indented nosetip calculation.

In the simulation of turbulent flow, the turbulence model of Baldwin and Lomax⁹ was used. The final laminar solution with $\epsilon E = 0.1$ was used as the initial flowfield for turbulent calculation. A turbulent solution was produced in 200 time steps with the nondimensional shock speed in the order of 10^{-3} . As shown in Fig. 2, the Stanton number increase significantly for turbulent flow as compared to the laminar flow one (no measured data for comparison). It should be pointed out that the surface pressure obtained from the laminar and turbulent solution agree to two digits.

B. Indented Nosetips

The special calculation procedure for inviscid flow over indented nosetips as described in Ref. 2 was applied to calculate viscous flow over indented nosetips. Model 1 and 4 as described in Ref. 2 were chosen for this investigation. The simulation of viscous turbulent flow over indented nosetips has not been very successful as described in the following paragraphs.

Laminar flow over Model 4 was first calculated. The calculation started with a grid of 24×32 ($CN = 150$, $\epsilon E = 0.4$, $\epsilon I = 0$) for 400 time steps to obtain a laminar solution over a sphere at $M_\infty = 5.0$, $Re = 8 \times 10^6/FT$ and $T_w/T_\infty = 5.4$. The sphere was then deformed to the shape of Model 4 in 1800 time steps. The grid was then increased to: (!) 58×32 and (B) 56×48 in another 1600 time steps each with the final values of $CN = 50$, $\epsilon E = 0.1$ for (A) and $\epsilon E' = 0.3$ for (B). The calculated surface pressure and shock locations from these two solutions are close (results not shown). This provides a self verification of the results. Since grid (B) contains more points in the η direction, its solution resolves the flow field better and is used for comparison as shown in Fig. 3 and 4.

Unlike the hemisphere-cone, the turbulent calculations for Model 4 encountered serious difficulties. Large amplitude oscillation of pressure in the flowfield quickly interrupted the computation. The value of CN was gradually reduced and the value of ϵI was increased ($\epsilon E' = 0.3$ was maintained). At $CN = 2$ and $\epsilon I = 6$, it was possible to run for 200 time steps with the non-dimensional shock speed converging to a value of 0.04. The shock speed is then started to increase slowly but steadily. A further increase of ϵI up to 12 would not help to obtain a converged solution. Thus, the solution before the shock speed started to increase is shown in Fig. 3 and 4 for comparison. It is understood that the turbulent solution shown in Fig. 3 and 4 is not the converged solution.

Fig. 3 shows the comparison of shock location between calculations and experiments. The inviscid shock layer is thinner around the indented region as expected. The laminar and turbulent solutions for shock location are almost coincide and fall in between the inviscid solution and the measured data. The primary separation bubble indicated by the laminar solution is

smaller than observed experimentally. The laminar separation point of the primary separation bubble is at the downstream end of the expansion corner, but the flow picture (Fig. 8a of Ref. 2) shows that the flow separates immediately at the beginning of the expansion corner. In general, the effects of turbulence is to move the separation point toward downstream and the separation bubble will be smaller. This fact suggests that the discrepancy is not because of turbulence effects but from some other sources which are not captured in the numerical solution. Also the laminar solution indicates that there is a secondary separation bubble within the primary separation bubble as sketched in Fig. 3. The secondary separation bubble was not reported in Ref. 4.

In Fig. 4, the surface pressure distribution obtained from the inviscid, laminar and turbulent solution are compared to the measured data. It is noted that the viscous solutions compare better with the measured data than the inviscid curve. The region around the expansion corner $S/L \sim 0.26 - 0.7$ where the inviscid and viscous solutions are seen to agree well (i.e., no flow separation) but are lower than the measured data. The dip in the pressure curve in the region $S/L \sim 2.5$ (where the secondary separation bubble starts) is not shown by the measured data. A calculation was made with the complete Navier-Stokes equation for laminar flow with the same grid distribution as used in the thin-layer approximation calculation and the results for surface pressure agree up to two digits.

For Model 1, there is a sharp expansion corner with a radius of 0.062 inch. Three grid points were used to cover the corner as was done for the inviscid calculation reported in Ref. 2 and a total grid points of 33×32 were used for the viscous calculation. Only laminar solution can be obtained. As shown in Fig 5 and 6, the inviscid solution agrees reasonably well with the measured data for both the shock location and the surface pressure because the separation bubble is small as indicated by the flow picture (Fig. 5a of Ref. 2). In contrast to the result of Model 4, however, the laminar solution indicates that the flow separates immediately after the corner and form a large primary separation bubble as shown in Fig. 5. Within the primary separation bubble, there is also a secondary separation bubble around the location of compression turn. As a result of the primary separation bubble, the laminar shock layer becomes thicker near the separation bubble and thinner afterwards as compared to the measured data. Also the surface pressure obtained from the laminar solution looks entirely wrong.

It was not possible to obtain a turbulent solution for Model 1, not even one like that of Model 4. The obvious reason is that the laminar solution is too far off from the measured data, which is assumed to be close to the turbulent solution, therefore the starting flowfield is too poor to carry through the calculation.



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From the above comparison of results, it is concluded that the numerical code works well for smooth nosetips. For those indented nosetips investigated in this paper, the code fail to give a satisfactory solution. Possible reasons for the failure of the code to obtain a solution and for the discrepancy between the viscous calculation and the experimental data are suggested as follows: (1) The turbulence model used is inadequate. (2) The first-order-accurate numerical procedure is not sufficient to simulate the complicated flowfield. (3) The grid points and their distribution are insufficient to resolve the viscous effects. (4) The sharp corner as described in Model 1 requires special treatment to avoid flow separation introduced from sources other than the viscous effects. (5) Because of the prevailing of a large separation bubble, the thin-layer approximation may lead to wrong solution, i.e., for a significant portion of the flowfield, the viscous terms in both the ξ and η directions should be kept.

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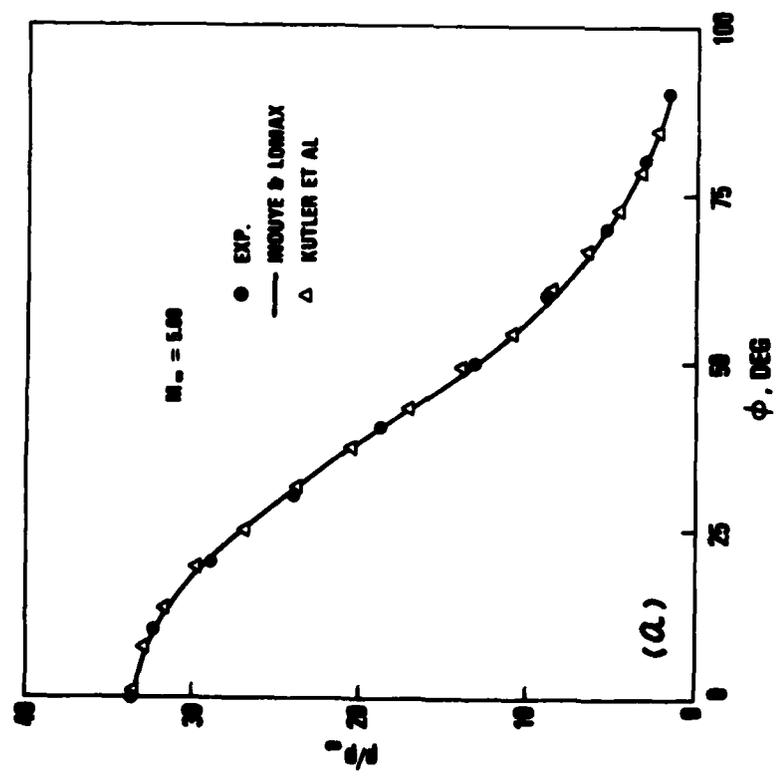
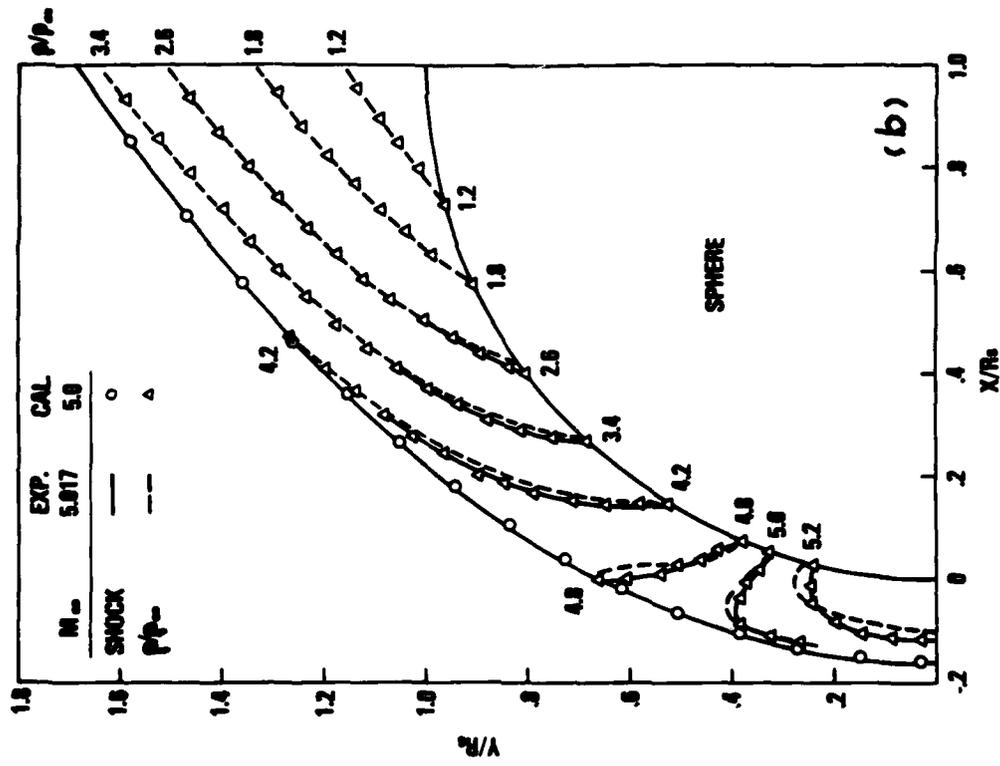


Figure 1 Comparison of Surface Pressure and Density Field Between Calculation and Experiment

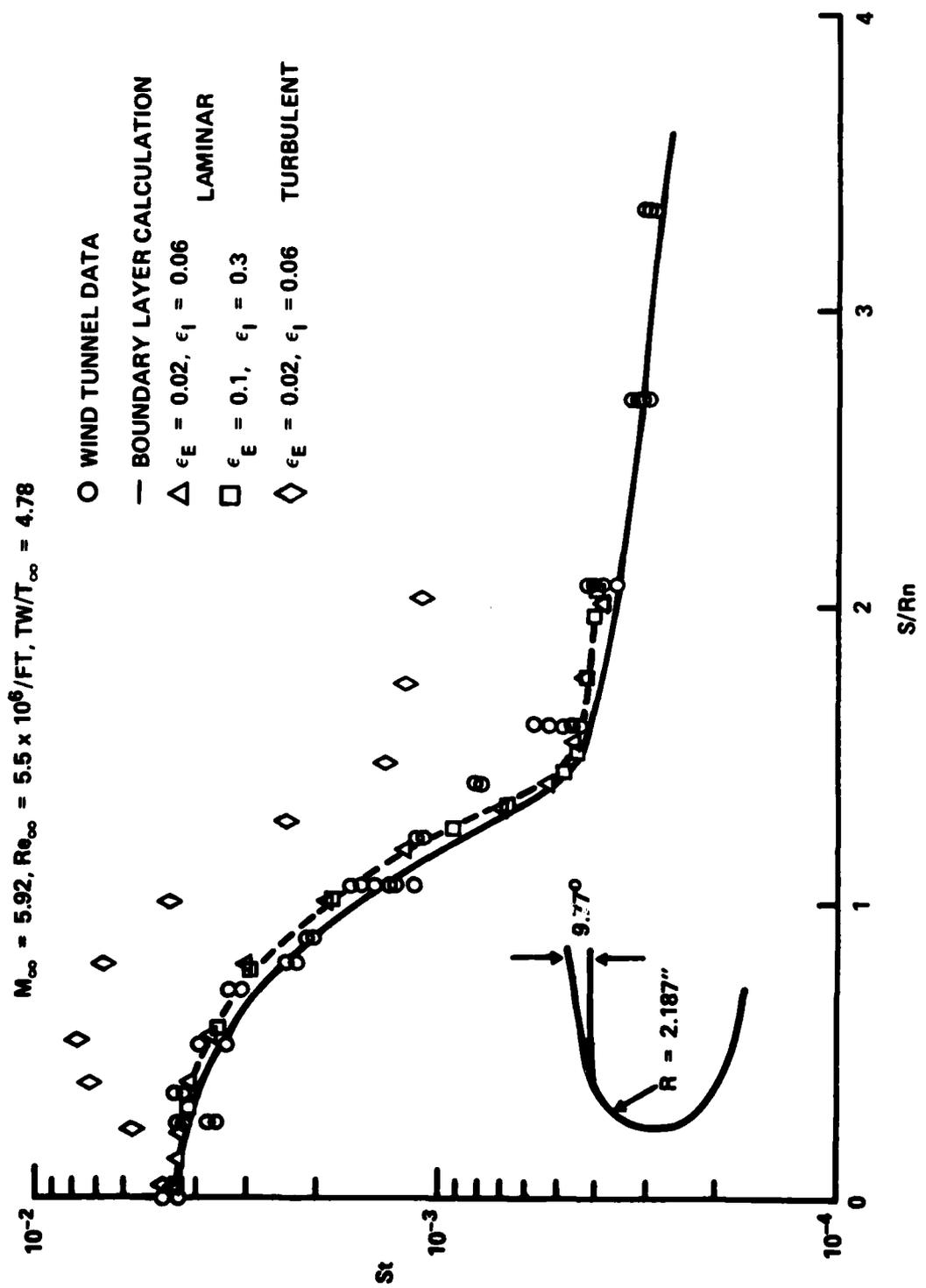


FIGURE 2 COMPARISON OF HEAT TRANSFER MEASUREMENTS ON A HEMISPHERE CONE

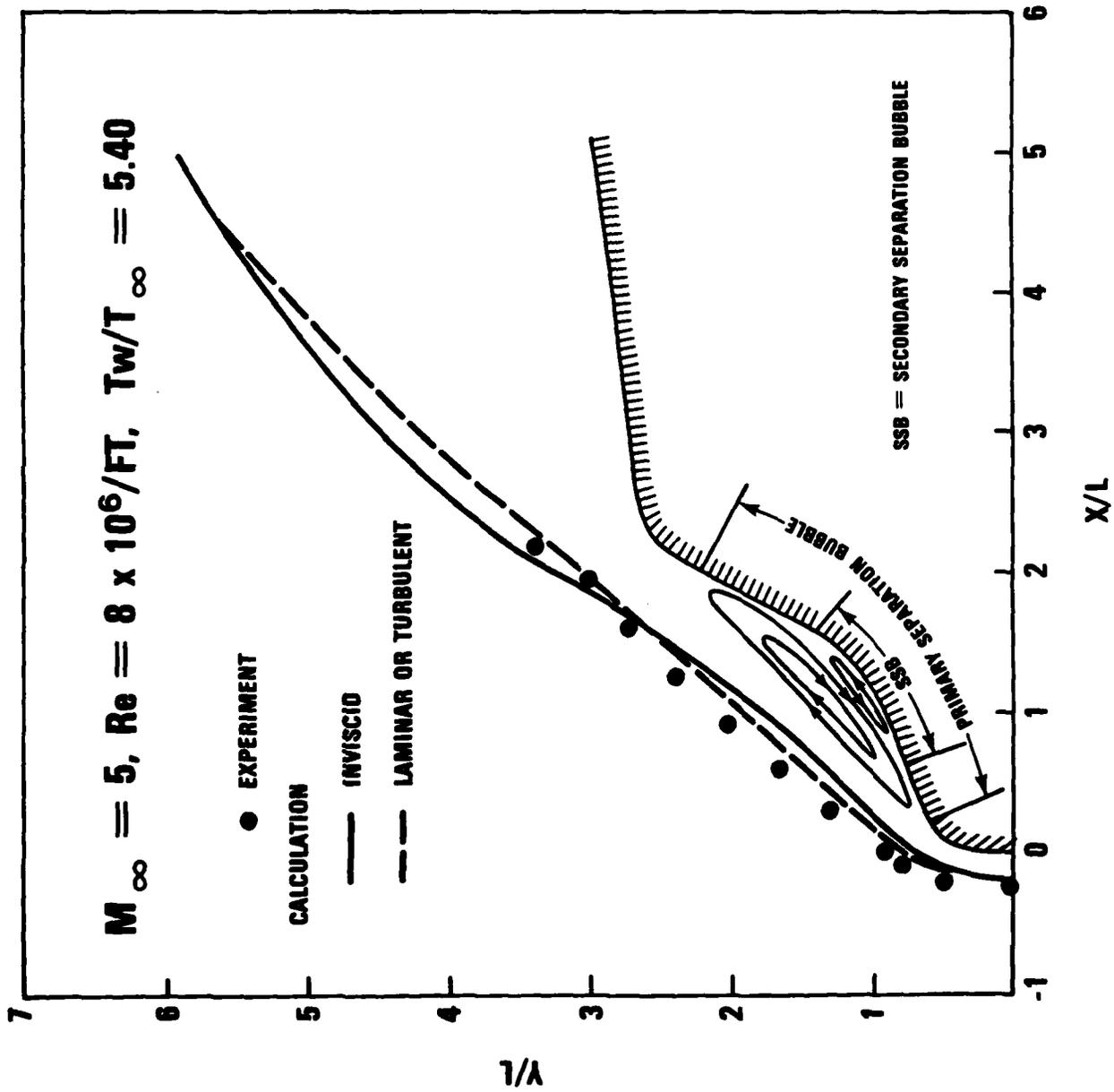


Figure 3 Comparison of Shock Location Between Calculation and Experiment for Model 4

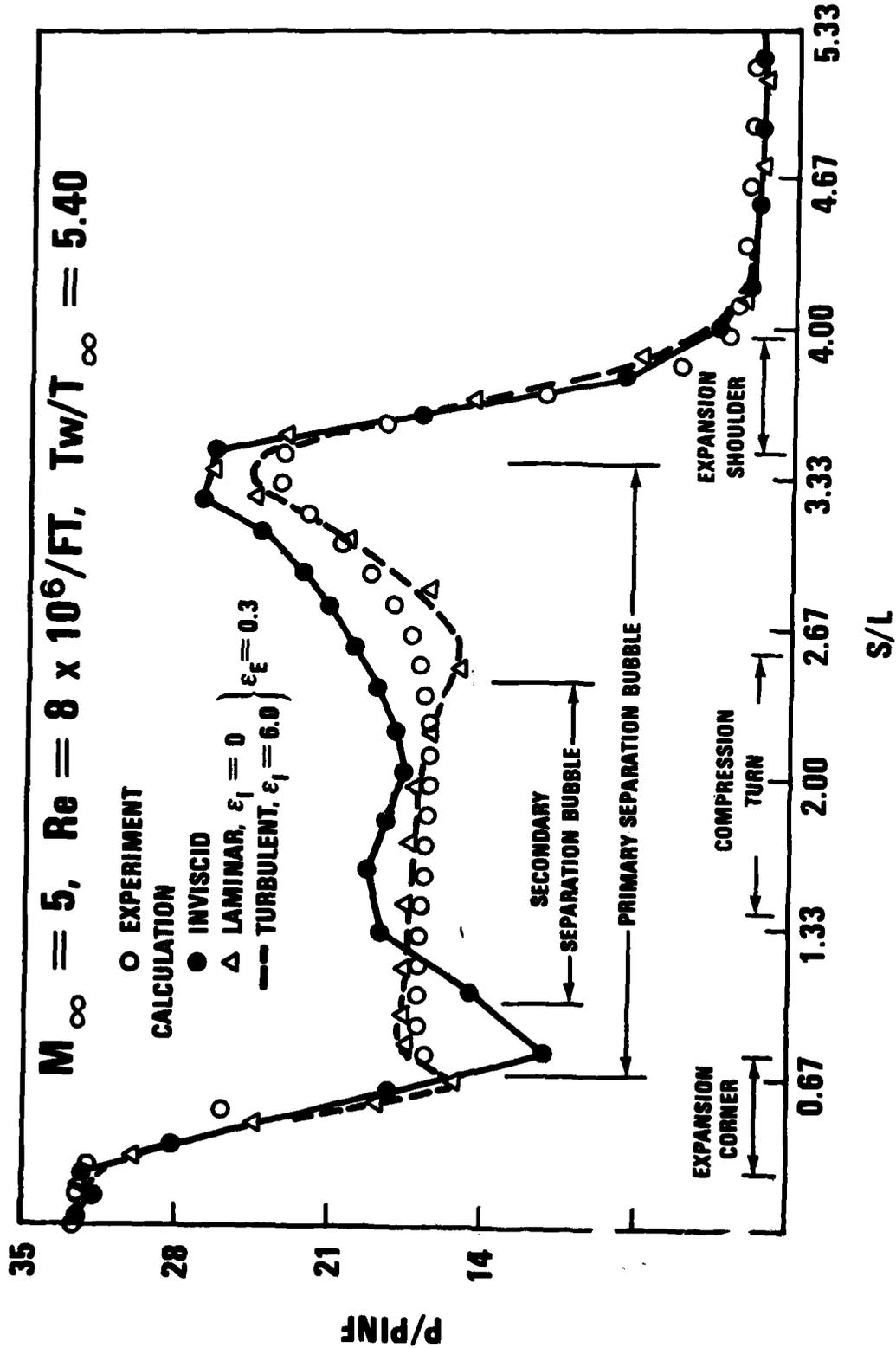


Figure 4 Comparison of Surface Pressure Between Calculation and Experiment for Model 4

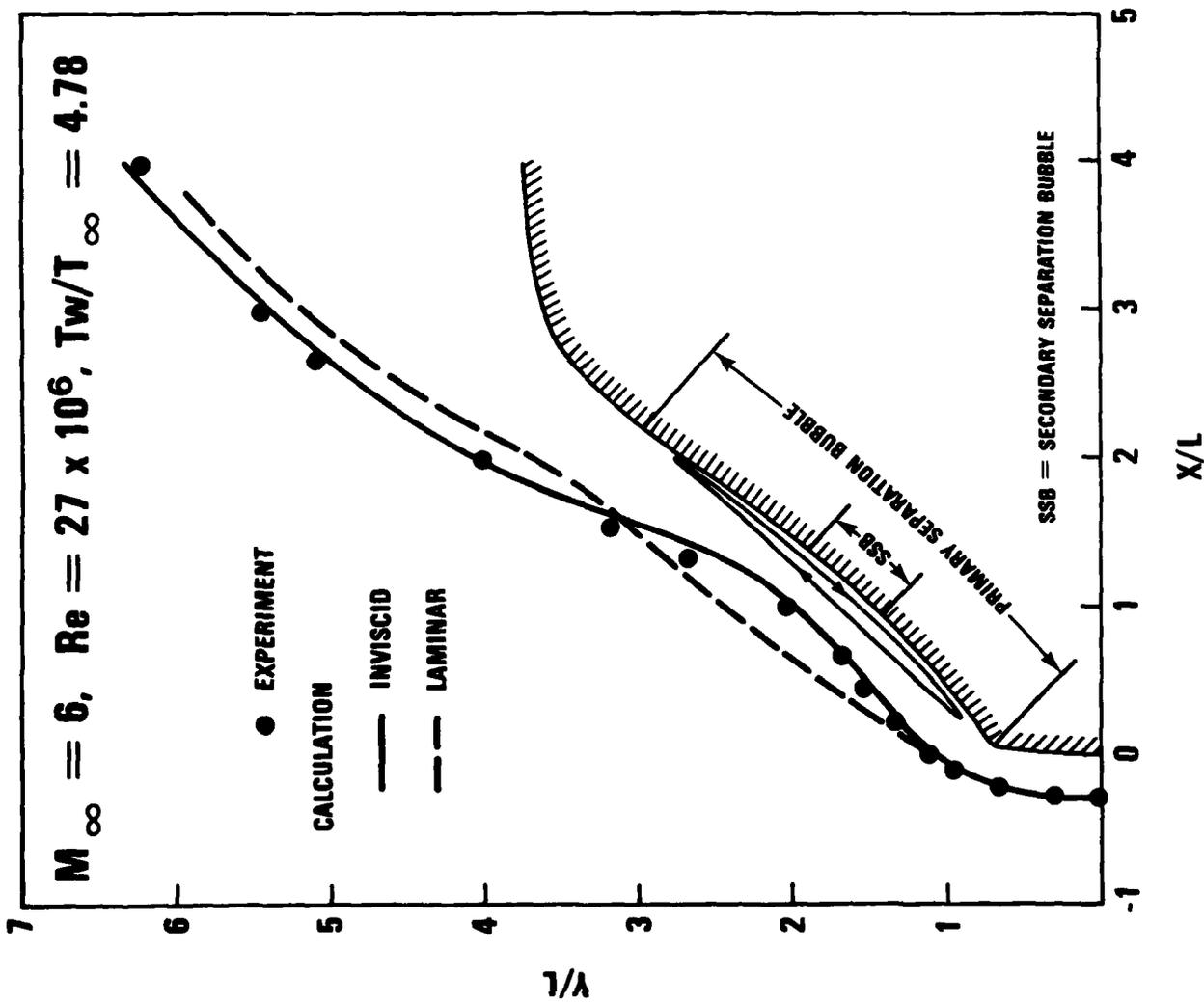


Figure 5 Comparison of Shock Location Between Calculation and Experiment for Model 1

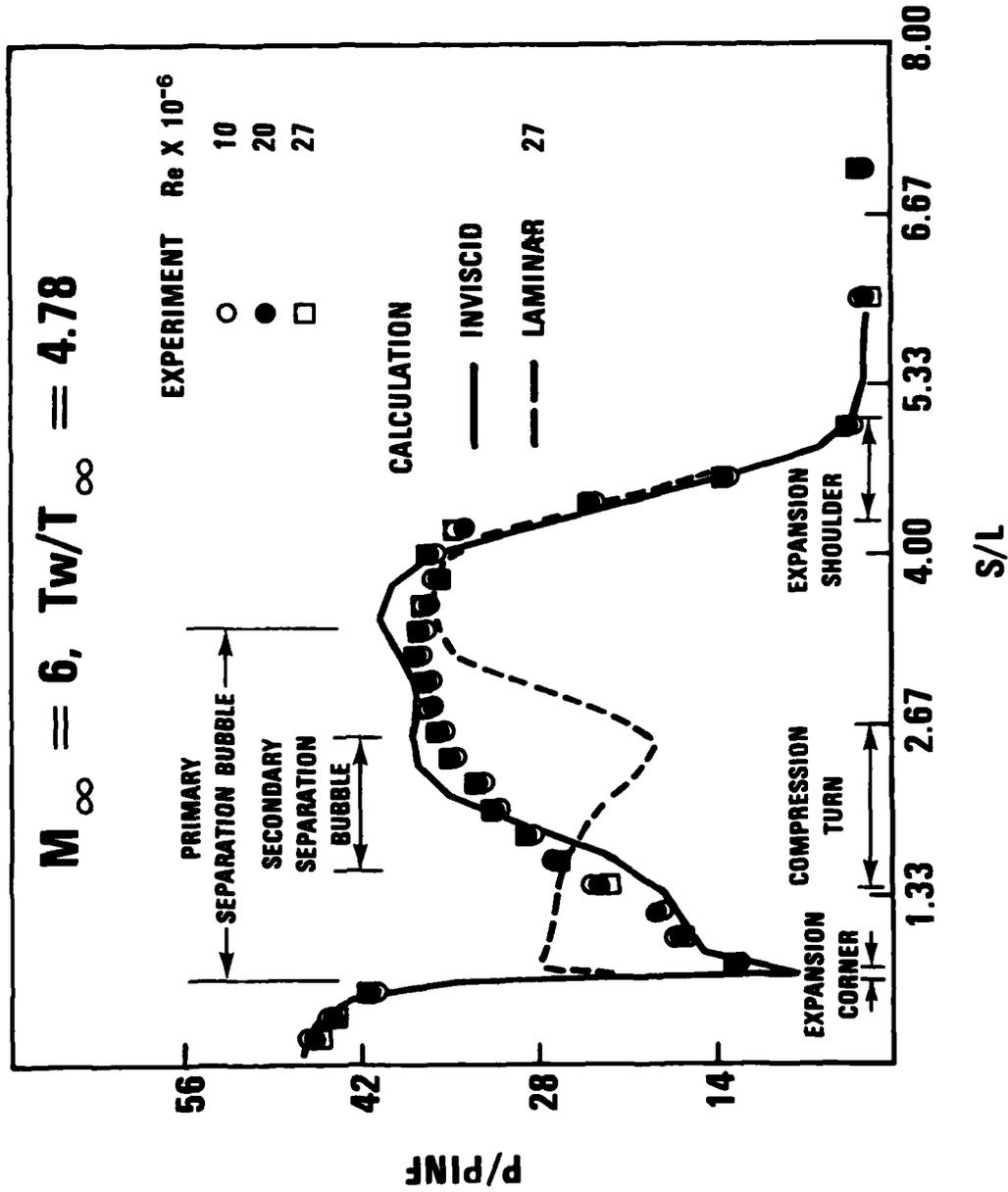


Figure 6 Comparison of Surface Pressure Between Calculation and Experiment for Model 1