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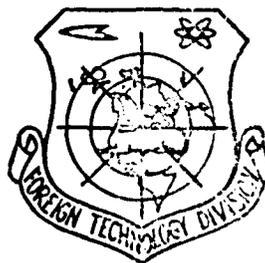
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## EXPERIMENTAL TECHNIQUES AND METHODS

PRINCIPLE OF OPERATION AND MEASUREMENT ACCURACY OF MODEL YYB1-1  
REMOTE STRAIN GAUGE

Chung Ch'ing University      Fan Ching Hung

As the mechanics industry strides forward in the direction of high speed, accuracy and high volume, the problem of strength, vibration and hardness of the rotating parts of all kinds of machines suddenly becomes very important. At home and abroad, current collecting rings are generally used to bring out signals for the purpose of measuring physical parameters of the rotating parts. With such contact-type measurements, the amount of work done is large, and the contact resistance between the current collector and the brush is difficult to stabilize. This is why it has been long conceived that one could employ a signal transmitter installed on the rotating part to transform physical parameters such as strain into electrical signals and use electromagnetic waves to carry these signals onto the receiver, thus fulfilling the requirements of non-contact type measurement. However, such remote strain gauges are usually required to work under unfavorable conditions, especially so in view of the weakness of the signals generated by the strain (on the order of a few  $\mu\text{V}$ ) and the serious electromagnetic interference coming from the electric sparks and operating frequencies in the industrial environment. In addition, space for installing gauges on the rotating parts is usually very limited. This is why for a long period of time no good solution has been found for such problems as interference resistance, reliability, stability, accuracy and micronization of the remote strain gauge. Aiming at solving this problem,

the Strain Remote Measurement Study Group of our university has successfully fabricated the Model YB1-1 Remote Strain Gauge (See Plate I Photo 1) with the help of Ch'ung Ch'ing Steel Company, Tung Fang Electrical and Machinery Factory, Hu Hsin Instruments Company and Ch'eng Tu Academy of Communications Engineering.

I. Characteristics and Principle of Operation of the Remote Gauge

In our design, we have adopted electric field coupling and a lower frequency range instead of the usual ultrashort waves for signal transmission. Thus we were able to avoid the strong absorption by metallic material of ultrashort waves, to simplify the circuit design, and increase the reliability and stability of the system. Also, micronization was facilitated by the elimination of the inductive elements necessary for ultrashort waves. Pulse frequency modulation was used instead of a sinusoidal regime. This increased the system's resistance to interferences. Direct supply bridge was used to circumvent the problem of distributive capacitance (which sometimes becomes more significant in performing measurements on rotating parts<sup>[1]</sup>). Circuit complexity and the necessity for using capacitance balancing potentiometer were also avoided. Figure 1 shows a block diagram of the circuit.

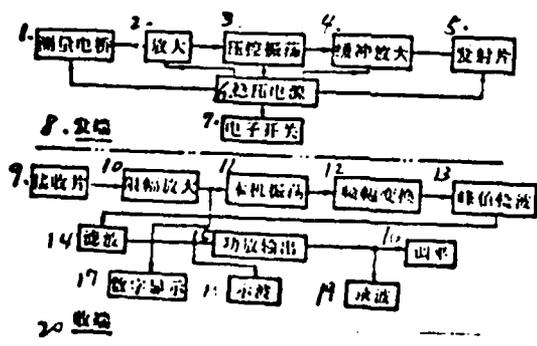


Fig. 1 Block diagram of Model YB1-1 Remote Strain Gauge (Key, next page)

- |                                 |                                   |
|---------------------------------|-----------------------------------|
| 1.Measuring bridge              | 11.Oscillator                     |
| 2.Amplifier                     | 12. Frequency-amplitude converter |
| 3.Pressure-regulated oscillator | 13.Peak amplitude detector        |
| 4.Buffer amplification          | 14.Filter                         |
| 5.Emitting plate                | 15.Power amplifier                |
| 6.Constant voltage source       | 16.Balancing stage                |
| 7.Electronic switch             | 17.Digital display                |
| 8.Emission end                  | 18.Oscilloscope                   |
| 9.Receiving plate               | 19.Recorder                       |
| 10.Amplifier discriminator      | 20.Receiving end                  |

On the signal emission end, variation in strain passes through the measuring bridge and is transformed into voltage variation. This is amplified and used to control a pressure-regulated oscillator. In this manner, variation in strain modulates the frequency of the rectangular pulses, and the train of pulses act as carrier of the desired strain signal. Such signals reach the receiving end via electric field coupling between the emitting plate and the receiving plate, and non-contact transmission of signal is achieved.

On the receiving end, owing to losses and external interference incurred in the process of transmission, the pulse trains transmitted through the electric field are reduced in amplitude, distorted in shape and contain noises. To eliminate these effects, we let the signal pass through an amplifier discriminator so as to have sufficient output amplitude, and to restore the original shape of the pulse train. Then the signals were fed into a digital frequency counter to ascertain their static or average value of strain. The modified pulse train was passed through a frequency-amplitude converter, a rectifier and a filter. The digital signal (pulse frequency) was converted back into analog signal (strain waveform) and either recorded with crystal

oscillator oscilloscope or displayed on cathode ray oscilloscope.

Following is an analysis of the various parts of the remote gauge.

#### The Emission End

Constant voltage source — Variation in source voltage has decided effects on such parts as the bridge circuit. Owing to limitation on volume, the volume of the battery used on the emission end is generally small, and the decrease in voltage with time is significant (Fig. 5). If electricity is supplied externally, then there will be larger variation in the source voltage that is coupled to the axle due to variation in the spacing between the axle and the source. Therefore, to ensure accuracy of the remote gauge measurements, it is very important to design higher-accuracy constant voltage source on the emission end. The difficulty involved lies not only in increased volume of the emission end but, more significantly, also in the higher currents required. This in turn reduces the life of the battery. (A set of high-accuracy constant voltage source uses approximately 10 mA of auxiliary current while the emission end itself only uses up 20-30 mA.) In our design, we made the bridge part of the constant voltage source, and used a series-type constant voltage source that contained feed-back loops so that the remote gauge used only about 3 mA of auxiliary current while it got the benefit of two sets of higher-accuracy constant voltage source. (For the positive source, variation in input voltage was 10%, while that for the output voltage was .1%; for the negative source, variation in input voltage was 10%, while that for the output voltage was 1%.)

The bridge — The sensing element consists of a common resistor-type or foil-type strain plate (Inner bridge uses two 120 $\Omega$  standard resistors) connected according to the general principle of strain measure-

ment into single arm, double arm and complete bridge strain measuring Wheatstone bridge circuits. If the corresponding resistor-type sensor was used, the bridge could be used for measuring speed, acceleration, pressure and other quantities. As only direct current was used, no capacitance balancing loops were required. If a potentiometer was used on the bridge to balance the resistors, then approximately 2% error would result because the resistance of the potentiometer would be on the bridge but would not respond to the variation in the signal. Taking into consideration the fact that the emission end of the remote gauge lies very close to the strain plate, one only needs to make careful choice of the resistance of the strain plate so that there exists only a very small nonlinearity. Therefore, we have adopted unbalanced input on the bridge, and the operating point has been set with the help of the potentiometer in the amplifier.

**Amplifier**—The emission end of the remote gauge uses as its amplifier a linear integrated circuit. Different ranges of measurement are obtained by varying its closed-loop feed-back resistance which determines the amplification power.

**Pressure-regulated oscillator**—The function of the pressure-regulated oscillator is to transform pressure variation due to variation in strain into variation in frequency, thus achieving frequency modulation. This type of oscillators arise from multi-harmonic oscillators. They can generate pulse trains continuously. The frequency of the oscilloscope is controlled by the output voltage of the amplifier. Within the range of operation, linearity in pressure and frequency is preserved. This has been predicted by theory and borne out by experiments.

**Buffer amplifier**— Besides amplification of the oscillation, the main

function of the buffer amplifier is to isolate the oscillator from its surroundings so as to reduce parasitic frequency modulation.

Electronic switch— For measurements made on machinery that cannot conveniently be stopped, eg. a moving ship, the remote gauge has a simple electronic switch with the help of which one can control the switching of the power source at the emission end from the receiving end. The switch is connected to the base of the emission end switching crystal. When the switch for emission end power source, located at the receiving end, is in the off position, there is no potential on the base of the switching crystal. The crystal becomes nonconducting, the constant voltage source is cut off and no work is done by the emission end. When the switch is in the on position, current flows through the brush, the switch ring attains ground potential and the switching crystal becomes saturated. The constant voltage circuit is closed, and the emission end starts operating. In order to avoid problems arising from poor contact between the brush and the switch ring, we have designed a RC time-delayed return circuit, so that when the brush is temporarily disengaged from the switch ring, the emission end can still keep on operating normally.

#### The Receiving End

Impedance transformer— We have installed an impedance transformer on the receiving plate. This is an emitter-follower. It has a large input impedance, and can accept weak signals, thus increasing the allowable transmission distance. On the other hand, its output impedance is very low, so that it is allowed to have a very long cable and not suffer from interference.

Amplifier discriminator— To eliminate interference, we designed a two-terminal amplifier discriminator that has high-gain negative

feed-back. The noise waves entering through the receiving antenna are discarded by the discriminator after amplification, and the signal is restored to better defined square waves.

Main oscillator — This is a multi-harmonic oscillator with improved rising edge..

Its natural frequency of oscillation equals the central frequency designed. As the potential of the base is controlled by the amplifier discriminator, when the emission end is operating, the frequency of oscillation is forced to synchronize with the emission end. However, when the emission end is not operating, the frequency of oscillation becomes the central frequency. The oscillator has three functions: When the emission end suddenly has a power outage, the crystal oscillator will be protected; it indicates the regulation of the potentiometer on the emission end; it further improves the waveform of the pulse train.

Frequency-amplitude converter — Its function is to recover the frequency modulated signal. The remote gauge adopts a self-sustaining saw-tooth wave circuit. The rectangular wave is transformed into a saw-tooth wave. The amplitude of the saw-tooth wave is inversely proportional to the frequency so that frequency-amplitude conversion can be achieved.

Peak-amplitude detection and filtering — The purpose of peak-amplitude detector is to obtain the envelope of the saw-tooth wave. The wave is first detected by a diode, and then passed through a RC filter, thus restoring the original strain waveform.

Power amplification — Power amplification is accomplished by using a two-stage cascade emitter-follower. Since the cascaded emitter-follower has a low output impedance, it can take a large load, and put out large power. In order to have zero output of crystal oscillator when there

no strain, we have used a balancing stage, which is also made up of a two-stage cascade emitter-follower. A potentiometer is used to adjust the potential of the output so that the two ends of the crystal oscillator has zero potential difference.

II. Performance Tests And Chief Technical Indices of the Remote Strain Gauge

Linearity

Fig.2 shows the relationship between measured strain and output current. (The 'linear values' in the figure have been obtained by connecting full-scale point A to zero point with a straight line and finding the values on the ordinate corresponding to the given strain values.) In the measurement, the output terminal has a fixed load resistance. (This has been taken to be 15  $\Omega$  in accordance with the FC6-2500 crystal oscillator corresponding to Model SC-16 light ray oscilloscope.) For full-range output (set at 20 mA), error in linearity has been measured to be 1.35% for positive strain and 0.9% for negative strain.

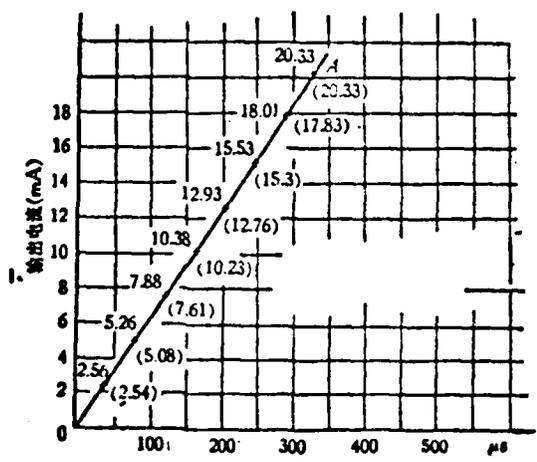


Fig.2 Strain at emission end vs. output current curve  
 Upper figures: average of 3 measurements  
 Lower figures: 'linear values'  
 1. output terminal

### Frequency Response

As a d.c. supply bridge is used, bridge circuit is purely resistive, and has no effect on the frequency response. One can open the bridge circuit to measure the frequency response characteristics of the instrument. A signal generator is used to input a constant amplitude sinusoidal wave at the emission end. After passing through the entire measurement system, the wave reaches the  $15 \Omega$  load. Fig. 3 shows the variation with frequency of the voltage across the load. From this, one obtains an error of less than 3% for frequency response when the frequency is 1000 Hz.

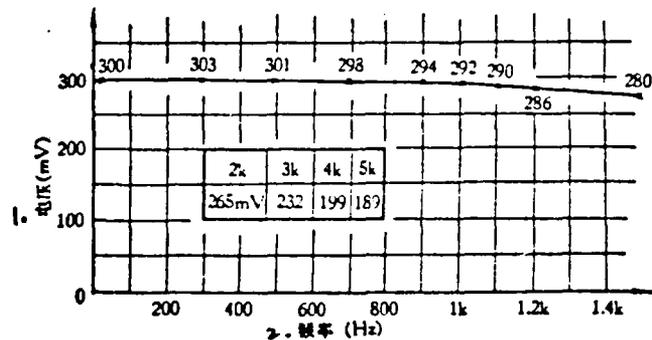


Fig. 3 Frequency response curve

1. voltage
2. frequency

### Stability

Stability with respect to receiving end source voltage fluctuation We examined the effect on output of the input voltage at the emission end when the latter varied from 180 V to 242 V. The result is as shown in Fig. 4. From the figure, it is obvious that the output of the receiver varies very little even for sizable fluctuation of the source voltage of the receiving end.

Instrument sensitivity drift. Fig. 5 shows the variation of the measured sensitivity and that of the emission end battery voltage as a function of time. One sees that although the battery voltage drops fairly rapidly, the sensitivity of the gauge varies only slightly with

time.(The batteries used are two sets of 6402-1 button batteries.)

Zero-point drift. Zero-point drift curve is shown in Fig.6. From the figure,one one can see that after 45 minutes warm-up time, the zero-point drift is within 2% of the full scale over a period of 4 hours.

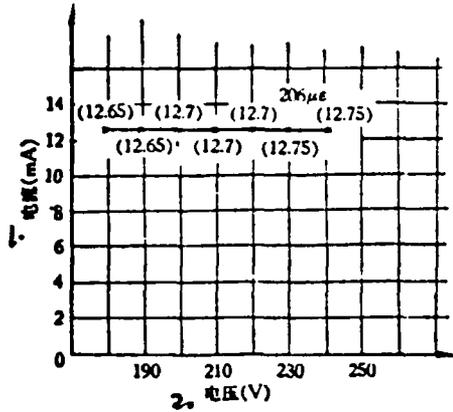


Fig.4 Receiver output current vs. source voltage curve

1. Current
2. Voltage

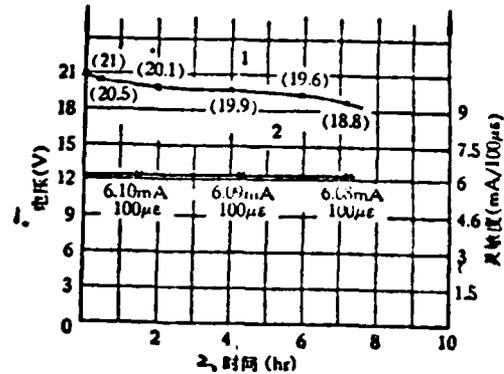


Fig.5 Curve 1:Variation of battery voltage with time

Curve 2:Variation of gauge sensitivity with time

1. Voltage
2. Time

Effect of Distance Between Emitting Plate And Receiving Plate

Fig.7 shows the variation of output current with the distance between the emitting and receiving plates under the condition of constant strain. It is obvious that variation in distance has negligible effect on the accuracy of measurement. (When measuring strain of rotating parts, the distance between the emitting and receiving plates is usually from 20 to 200 mm.)

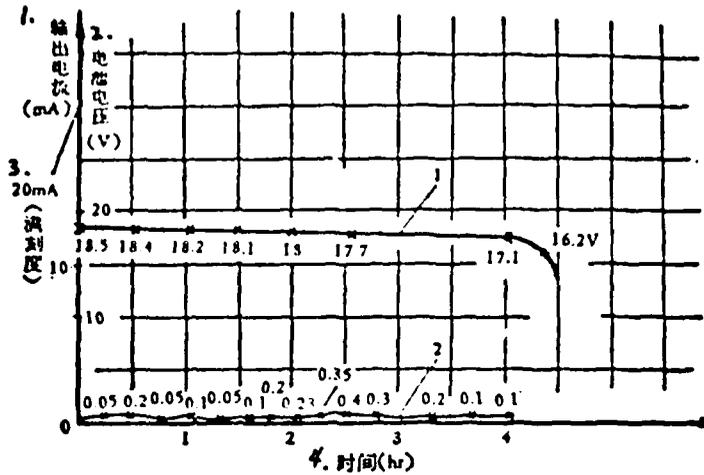


Fig.6 1) Curve showing drop in battery voltage when battery was almost used up

2) Zero-point drift

1. Output current

3. Full scale

2. Battery voltage

4. Time

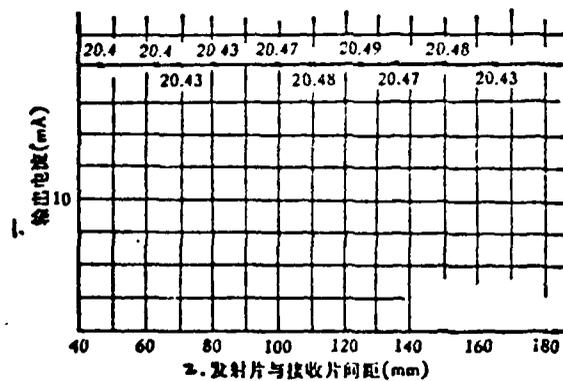


Fig.7 Variation of output current with distance between emitting and receiving plates

1. Output current

2. Distance between emitting and receiving plates

Chief Technical Indices

Operating frequency: 0-1000 Hz; error in frequency characteristics:

< 3%; maximum output current: (with 15  $\Omega$  load)  $\pm$  20 mA; error in linearity: (with 15  $\Omega$  load) < 1.5%; sensitivity: (with 15  $\Omega$  load) 0.05 mA/ $\mu$ e; drift in sensitivity:  $\pm$  2  $\mu$ e/hr; range of measurement: 0-5000  $\mu$ e (6 ranges:  $\pm$  300  $\mu$ e;  $\pm$  600  $\mu$ e;  $\pm$  1200  $\mu$ e;  $\pm$  2000  $\mu$ e;  $\pm$  3000  $\mu$ e;  $\pm$  5000  $\mu$ e); long cable transmission range:  $\leq$  100m.

### III. Rotary- -Bending Method for Testing Accuracy of Model YYB1-1

#### Remote Strain Gauge

To test the accuracy of the remote strain gauge, the usual method is to obtain two sets of signals from the vibrating beam, pass them through the remote strain gauge and a common strain gauge respectively and compare the results of measurement. Such methods do not take into account the effects of rotation. Under rotating conditions, the difficulty of testing the accuracy of the remote strain gauge lies in the choice of a suitable standard strain signal. In order to overcome this difficulty, we performed a rotary-bending measurement. (Plate I Photo 2)

An axle was pushed against the pin on the lathe. <sup>(Fig. 8a)</sup> A cantilever beam was placed on the axle and standard weights were bolted on an end of the beam. When the axle rotates, the strain in the beam consists of a constant strain resulting from centrifugal force, and a dynamic strain caused by the action of weight  $W$  on the beam. The gravitational component  $W_p$  of the beam bending caused by the latter can be expressed as (Fig. 8b)

$$W_p = W \sin \theta$$

Hence, theoretically, the dynamic strain must follow a sinusoidal curve. Since the action of gravitational force on the beam should be the same whether the beam is stationary or rotating at different speeds, its amplitude can be found by using ordinary strain measuring devices under static conditions. This is recorded as  $\epsilon_0$ . As one can accurately

determine the weights of the standard weights and the bolt, the elastic modulus  $E$  of the beam and the location of the strain plate the theoretical value can be obtained by the following equation:

$$\epsilon_t = \frac{M}{W \cdot E}$$

In the above equation,  $M$  is the bending moment at the center of the strain plate caused by the weight of the standard weights and the bolt;  $W$  is section modulus of the beam.

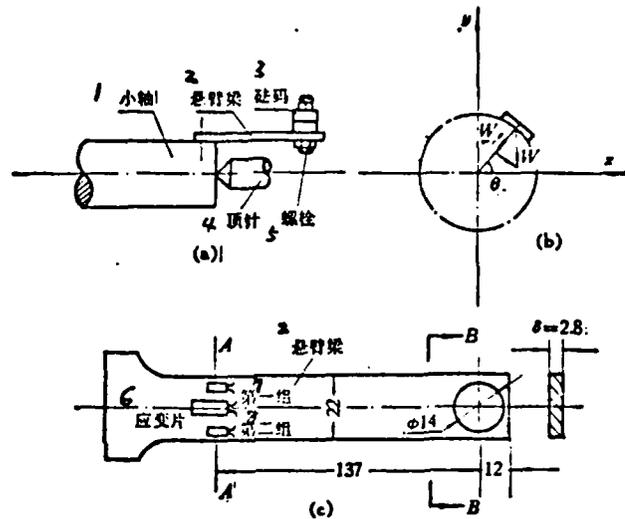


Fig.8 Rotary-bending measurement set-up

- |                    |                |
|--------------------|----------------|
| 1.Small axle       | 5.Bolt         |
| 2.Cantilever beam  | 6.Strain plate |
| 3.Standard weights | 7.First set    |
| 4.Pushing pin      | 8.Second set   |

In order to increase the credibility of the results obtained by the rotary-bending method, we simultaneously used a mercury current collecting ring. For comparison of waveforms, we connected the two sets of strain plates located in plane AA of the beam (Fig. 8c) separately to Model YYB1-1 Strain Gauge and Current Collector-Model YJD1 Strain Measuring System, and took pictures of the waveforms simultaneously. For comparison of strain amplitudes, we used the same set of short

moment strain plates for both the remote gauge and the current collector system.

The results of the comparison are shown on Plate I Photos 3-5. It is obvious that the waveform and phase are basically the same for the two methods. The small ripples superimposed on the sinusoidal wave are due to the additional vibration of the beam caused by vibration of the lathe. (The lathe used in our measurement has low precision, hence the large vibration.) To obtain the strain amplitude  $\epsilon_1$  and  $\epsilon_2$  measured by the remote gauge and the current collector system respectively, one should remove the ripples caused by vibration of the lathe. To do so, first enlarge the curve obtained from the oscilloscope, then connect the mid-points of the rising and falling edges of the ripples to obtain a smooth curve. This is the curve for the strain due to gravity. The vertical distance between the highest and lowest points on this curve was measured separately on the scale ruled for the remote gauge and that for the current-collector system. The values obtained are 4 times the strain amplitude measured. (When the beam rotates from the highest point to the lowest point, the strain due to gravity changes sign, so that the measured value is twice the strain amplitude. The other factor of 2 comes about because the two strain plates on the upper and lower sides of the beam are connected by a half-bridge. Hence the total factor of 4. Similarly for the measured static value  $\epsilon_0$ .)

Values for  $\epsilon_0$ ,  $\epsilon_t$ ,  $\epsilon_1$ ,  $\epsilon_2$  are tabulated in Table 1. The relative errors are with reference to the theoretically calculated value  $\epsilon_t$ .

Table 1

1. 项 目	2. 应变值 $\epsilon$	3. 与计算值的相对误差
4. 静态测试	$\epsilon_0 = 129 \mu\epsilon$	1.6%
5. 理论计算*	$\epsilon_t = 127 \mu\epsilon$	-
6. 遥测应变仪测试	$\epsilon_1 = 125 \mu\epsilon$	-1.6%
7. 集流环测试	$\epsilon_2 = 131 \mu\epsilon$	3.1%

\* 弹性模量  $E$  均取为  $2.12 \times 10^4 \text{ kg/cm}^2$ .

(Key, next page)

1. Item
2. Strain value
3. Errors relative to calculated values
4. Measurement under static conditions
5. Theoretical calculation
6. Measurement using remote gauge
7. Measurement using current collector
8. Elastic modulus E was measured to be

In order to see if the remote gauge can work reliably on a long term basis under realistic unfavorable conditions, and how accurate the measured data are in an industrial environment, we tested the Model YB1-1 Remote Gauge on the Model  $\phi$ 550 Shape Steel Rolling Mill of Ch'ung Ch'ing Steel Mill No.3. Our experiments show that the waveform of torque variation and that of freely decaying torsional vibration as measured by the remote gauge are basically the same as those obtained by the current-collector system. The curve on the oscilloscope is fairly smooth. These waveforms for torque and freely decaying torsional vibration of the axle system agree fully with those expected during shape steel milling. During the 40-hour continuous testing, 40 sets of data were obtained for comparison, which showed that, basically, the values of torque measured by the two methods agree quantitatively also.

Recently, we have successfully built a four-channel remote gauge. It has been applied to the experimental research on machine benches, and better results have been obtained.

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## THIN FILM RADIATIVE HEAT SENSOR

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Wen Kuo

Thin film radiative heat sensor is a radiative heat measuring element that has simple construction, high sensitivity and fast response. It is especially suited for fast measurements on radiation in supersonic equipment. It can also be employed in the calibration of radiated energy. This type of sensor is made by covering the surface of an ordinary thin film resistance thermometer with a layer of carbon black<sup>[1-3]</sup>, so as to increase its absorptivity, broaden the absorption spectrum, and reduce variation of absorptivity with wavelength.

In other countries, thin film radiative heat sensors have usually been constructed by brushing specially prepared liquid metal suspensions onto glass or ceramic substrates<sup>[4]</sup>, or by covering the glass substrate sputtered with platinum film<sup>[5]</sup> with a previously prepared carbon film. Sensors made in this manner have inferior surface quality, and their frequency response curve only covers wavelengths below  $1 \mu$ . We have improved the construction technique by first sputtering a layer of platinum on the glass substrate, and then depositing the carbon directly onto the surface of the platinum film by means of vacuum deposition technique. The carbon film obtained this way has a better uniformity. One has better adhesion between the carbon film and the platinum film. In addition, the frequency response curve is extended to cover wavelengths below  $2 \mu$ , superior to similar sensors made in other countries. (Fig. 3)

### 1. Structure and Principle of Operation

Fig. 1 shows the structure of the thin film radiative

heat sensor. The glass substrate has dimensions  $\phi 20$  mm x 5 mm, the platinum film is generally a few hundred Å thick, and the carbon film is usually from 1000 to 4000 Å thick. Because the carbon film is sufficiently thin, most of the radiative heat flow  $\dot{q}_r$  transferred to it by external hot gases will be passed on to the platinum film [5]. Similarly, as the platinum film is sufficiently thin, its temperature will be representative of that of the glass substrate [3]. If a small current is applied to the platinum film via a silver wire, then when a constant external heat flow  $\dot{q}_{r,s}$  impinges on the platinum film, its temperature and resistance will increase with time. Correspondingly, the voltage drop on the film will also increase. According to the theory of semi-infinite slab heat transfer, the temperature at the surface of the sensor has the following time dependence [6,7]:

$$T(t) - T(0) = \dot{q}_{r,s} \cdot 2\sqrt{t/\pi(\kappa\rho c)}_{\text{glass}} \quad (1)$$

When a constant current  $I$  passes through the sensor,

$$T(t) - T(0) = \Delta V/I \cdot R_f \cdot \alpha \quad (2)$$

The relation between the voltage signal output of the thin film radiative heat sensor and heat flow (usually referred to as 'temperature curve') is given by

$$\dot{q}_{r,s} = \frac{\sqrt{\pi(\kappa\rho c)_{\text{glass}}}}{2I \cdot R_f \cdot \alpha} \cdot \frac{\Delta V(t)}{\sqrt{t}} \quad (3)$$

The circuit used in our experiment is as shown in Figure 2. Hence, Equation (3) becomes

$$\dot{q}_{r,s} = \frac{(R_0 + R_f)^2}{R_0 \cdot R_f \cdot E_0} \cdot \frac{\sqrt{\pi(\kappa\rho c)_{\text{glass}}}}{2\alpha} \cdot \frac{\Delta V(t)}{\sqrt{t}} \quad (4)$$

## 2. Fabrication Technique

A layer of platinum was first sputtered onto glass using high vacuum film deposition equipment. The distance between the platinum target and the glass substrate was about 2.5 cm. Pressure inside the bell jar was  $10^{-2}$  mm Hg. D.C. voltage was 1.2-1.5 kV. Current

was 15-20 mA. Sputtering time was about 13 minutes.

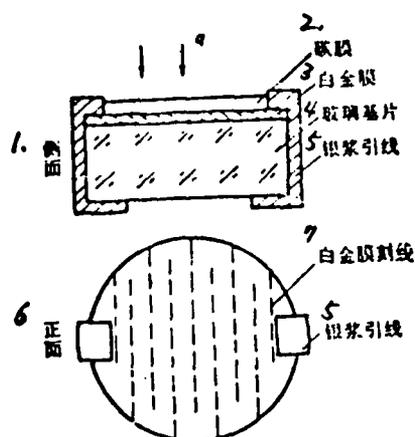


Fig.1 Schematic diagram of thin film radiative heat sensor

1.Side-view

4.Glass substrate

2. Carbon film

5. Silver wire

3. Platinum film

6. Front-view

7. Rulings on platinum film

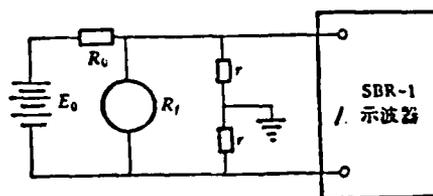


Fig.2 Circuit for measuring 'temperature curve'

1. Oscilloscope

To increase the resistance of the platinum film so as to obtain higher sensor sensitivity, we used a knife blade to make parallel rulings on the platinum film. The total resistance of the platinum film then became approximately  $200-300 \Omega$ . The sensor was then placed in a high temperature oven, heated to  $610-630^\circ \text{C}$ , kept at this temperature for 2 hours, and allowed to cool down gradually by itself. After annealing, the resistance dropped to  $60-100 \Omega$ . A carbon film was deposited on the platinum film by means of vacuum evaporation deposition technique.

Better results were usually obtained when two pairs of carbon electrodes were used. Vacuum had to be better than  $5 \times 10^{-5}$  mm Hg. The current was about 150 A. Evaporation time was about 2 seconds. Finally, aging was achieved by passing a current of several tens of mA through the sensor for 10 to 20 minutes. This was to enhance the stability of the sensor.

### 3. Calibration

Although the transmittance of a single layer of platinum film is generally smaller than 2%, the reflectance is as high as 40-60% and varies significantly with wavelength. A single layer of carbon has a reflectance of 10-20%, but its transmittance can also reach 10-20%. By employing the arrangement of a carbon film on top of a platinum film we have overcome the above difficulties. Fig. 3 gives the variation with wavelength of the total transmittance, the total reflectance and the total absorptivity for the Type II sensor most often used in our experiments. One can see that in the wavelength range between  $0.2 \mu$  and  $2.2 \mu$ . The total absorptivity of the sensor stays within the range  $0.85 \pm 10\%$ . In our experiments, we have approximated the total absorptivity to be 0.85. We measured the temperature dependence of the resistance of the platinum film in the range  $24.4-79.0^\circ\text{C}$ , and the temperature coefficient was found from  $\alpha = (R_{T_2} - R_{T_1}) / R_{T_1} (T_2 - T_1)$ . The two Type II sensors used most often in our experiments both had  $\alpha = 3.06 \times 10^{-3} / \text{degree}$ . We examined the thickness of standard samples of carbon film on various models of sensors with Model 6J interference Microscope. Type II carbon film had a thickness of less than  $1400 \text{ \AA}$ . We also used JSM-U3 Scanning Electron Microscope to examine the surface features of standard carbon film samples. Magnification was increased from 50x to 30000x. Only a few black spots on the order of  $1 \mu$  were seen on the edges of the samples. These were probably gas bubbles or pin holes. The rest of the surface was smooth and flat,

showing good uniformity of the carbon film.

#### 4. Experimental Methods and Set-Up

Eight cylindrical flat-head models with diameter  $r_3 = 29$  mm were installed together in the Model 800<sub>1</sub> Excitation Tube of our laboratory. In the middle of the model head was a 5 mm thick quartz window with radius  $r_2 = 8$  mm or 10 mm. The distance between the surface of the sensor and the outer surface of the window was usually about 6-8 mm. The effective radius of the surface of the sensor was 8 mm or 9.5 mm.

We assume that the gas flow in Region(2) of the tube was uniform during the effective period of the experiment. The radiative energy of the arc excited waves at the model head and that of the high temperature gas between the model windows was transmitted through the window and absorbed by the sensor. The voltage signal generated in the platinum film was displayed on the SBR-1 Oscilloscope. One can use Equation (4) to calculate the flow rate  $q_{r,i}$  of radiative heat incident on the sensor.

Under our present experimental conditions, the high temperature gas at the model head is optically dilute. Let us further assume that the temperature and density are constant throughout the arc excited wave. Then, according to geometric allocation, under nodal-point conditions, the relation between the total radiative energy  $E$  of the high temperature gas and the flow rate  $q_{r,i}$  of the radiative heat received by the sensor will be given by

$$q_{r,i} = F_g \cdot T_w \cdot \rho \cdot \delta \cdot F_A \cdot \frac{E}{2} \quad (5)$$

$$F_A = \frac{1}{2\pi \cdot s_1} \iint d s_1 \cdot \iint \frac{\cos \beta}{r^2} d s_2 \quad (6)$$

In our work, we have chosen  $F_{arc} = 1$ ,  $T_{window} = 0.9$ ,  $\rho = 0.85$ ,  $\delta/r_3 = 0.55$ , approximately. The geometric factors of the Type II sensors used most often are  $F_g = 0.38$  (#1) and 0.24 (#6).

At the same time, we measured the radiative heat transfer in Region(5) under the conditions of reflected excitation wave. Namely, a reflecting cylinder with 9.2 cm diameter and 8 cm depth was added to the front of the model head. If one considers the rate of increase in

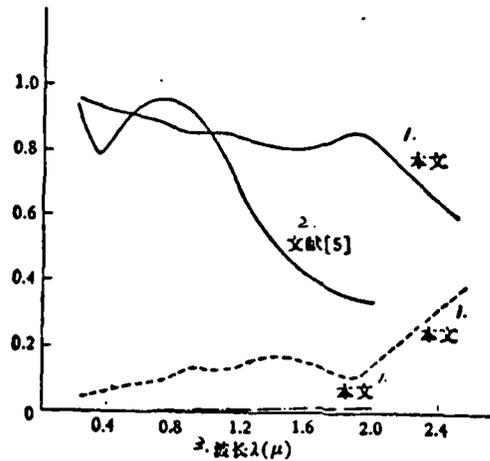


Fig.3 Optical properties of Type II Thin Film Radiative

Heat Sensor

— absorptivity

--- reflectance(MPS-5000 Monochromator Spectrometer)

-.- transmittance(Beckman Monochromator Spectrometer)

1. Present article

2. Ref. [5]

3. Wavelength

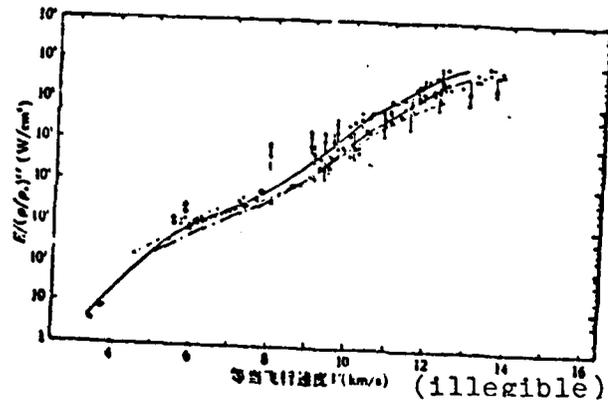


Fig.4 Comparison of theory and experiment for radiative energy of high temperature air under equilibrium conditions

1. Equivalent speed of flight 2. Experimental values

3. Theoretical values 4. Reference 5. Present article

the thickness of Region(5) to be approximately uniform, and the temperature and density of this region to be constant, then because the heat flow rate at the window and at the surface of the sensor is directly proportional to the thickness of Region(5), it will also be directly proportional to time:

$$q_{r,f}(t) = A \cdot t \quad (7)$$

After integration, the voltage signal generated at the surface of the sensor is found to be

$$\Delta V(t) = B \cdot t^{3/2} \quad (8)$$

Under constant voltage  $E_0$  conditions, Eq.(4) becomes

$$q_{r,f}(t) = \frac{3\pi (R_0 + R_f)^2}{4 R_0 \cdot R_f \cdot E_0} \cdot \frac{\sqrt{\pi(\kappa\rho c)} \text{ glass}}{2\alpha} \cdot \frac{\Delta V(t)}{\sqrt{t}} \quad (9)$$

Correspondingly, Eq.(5) becomes

$$\dot{q}_{r,f}(t) = F_2 \cdot T_{\text{in}} \cdot a_f \cdot \delta(t) F_A \cdot \frac{E}{2} \quad (10)$$

where  $\dot{q}_{r,f}(t)$  is the heat flow rate at time  $t$ ;  $\delta(t)$  is the thickness of Region(5) at time  $t$ ; the other coefficients are the same as in Eq.(5).

## 5. Experimental Results

We obtained radiative heat flow rate repeatedly at the nodal point and in Region(5) under the three conditions  $p_1=10, 1$  and  $10^{-1}$  mm Hg. The scatter of the testing points was generally within  $\pm 20\%$ .

A comparison of our experimental results with experimental results and theoretical calculations done abroad is given in Fig.4. The deviation of the average value we obtained from those obtained by others is generally within a few tens percent. This is on the same order of magnitude as the deviation of values among the results obtained by the other authors. The trend of variation with different conditions also agrees with that observed by other authors. This means that these data should be useful in practical engineering applications.

From the preliminary experimental results, it can be seen that our sensor meets the requirements on durability, sensitivity and

response time. Reproducibility and stability in repeated experiments are also good. Of course, fabrication is still crude and calibration is not yet complete. Experimental errors arose mainly from irreproducibility of experimental conditions and inaccuracy in the determination of the speed of the excitation wave. Other factors include calibration, non-uniformity of gas flow and presence of impurities. Improvements need to be made in the future.

We would like to extend our thanks to Comrade Shan Tse Chen who gave us a great deal of help with the platinum film sputtering.

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## RESEARCH REPORT

DETERMINATION OF PARAMETERS OF MODEL  $\phi$ 800 EXCITATION TUBE

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## 1. Experimental Set-Up and Principle of Operation

In our experiment, a cylindrical stainless steel wave excitation tube was used. High voltage section has length 1.6 m and inner diameter 223 mm; low voltage and experimental region has inner diameter 800 mm. Hydrogen-oxygen combustion was used for driving. Driving pressure was about 80 ata. The effective distance between the measuring stand and the sensing film was about 14 m.

We used a thin-film-resistance-thermometer type <sup>[1]</sup> gold film heat sensor in the excitation tube to measure the length of Region(2) and the shape of the wavefront. The interface between the Intermediate Region and Region(3) was determined using an electric probe. These signals were fed directly or via a thermoelectric simulation circuit into SBR-1 Oscilloscope.

When the excitation wave first arrives at the gold film, the heat flow carried by the high temperature gas causes the film temperature to rise abruptly. Hence, arrival of the wavefront is indicated by a sudden appearance of signal on the temperature curve or heat flow curve (after passing through the thermoelectric simulation circuit) recorded by the oscilloscope. After this comes Region(2) of uniform flow. Basically, the speed of heat flow from the high temperature gas to the gold film does not vary, and one obtains on the oscilloscope a smooth parabolic temperature curve or a horizontal heat flow curve. When the interface between the Intermediate Region and Region(2) arrives, because of the change in state of flow before and after the interface, one should see an obvious point of inflection on both the

temperature curve and the heat flow curve recorded by the oscilloscope. Thus the appearance of signal and the point of inflection mark the endpoints of the range of homogeneous flow of Region(2).

In order to measure the shape of the wavefront, we placed more than two sensors on the same horizontal plane perpendicular to the axis of the excitation tube, at various distances from the axis. To avoid the problem of the two oscilloscope scanning beams being unsymmetrical, we fed the signal from two sensors into the same scanning beam, one with positive polarity and the other with negative polarity. The signal thus recorded is a superposition of two gold film temperature curves. When the wavefront reaches gold film #1 first, the total signal rises rapidly and when the wavefront reaches gold film #2 the total signal drops abruptly; conversely, if the wavefront reaches gold film #2 first, the total signal first drops, then suddenly rises when the wavefront reaches gold film #1. In this manner, one can accurately determine order in which the wavefront reaches the gold films. The distance between the two points of change on the total signal is simply the time difference between the arrival of the wavefront at the two gold films. This, multiplied by the speed of the excitation wave, gives the degree of bending of the wavefront between the two positions.

In actual excitation tubes, the mass loss due to the presence of the boundary layer and the turbulent mixing of the gases between Region(2) and Region(3) result in a fairly long Intermediate Region. As the gas in Region(3) is cold and contains no charged particles, while the Intermediate Region contains some charged particles due to mixing with the gas of Region(2), we were able to use an electric probe to determine the location of the interface between the Intermediate Region and Region(3).

## 2. Experimental Results

Based on the experimental results, the state of flow in the excitation tube can be qualitatively divided into several regions as shown in Fig.1.

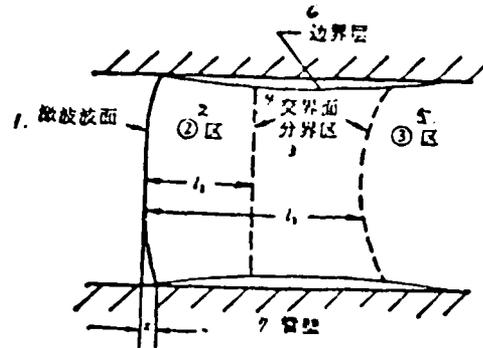


Fig.1 Schematic diagram of state of flow in the excitation tube

1. Wavefront

5. Region(3)

2. Region(2)

6. Boundary layer

3. Intermediate Region

7. Tube wall

4. Interfaces

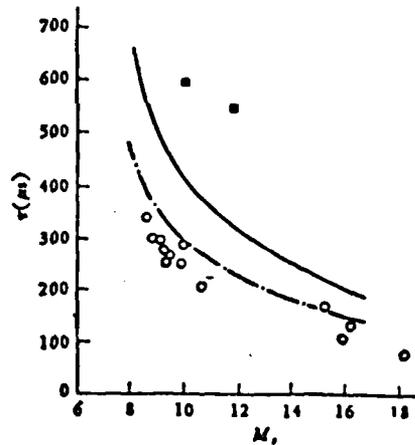


Fig.2 Determination of the location of the Intermediate Region

$$p_1 = 1 \text{ mm Hg}, x_s = 14 \text{ m}$$

○ Interface between Region(2) and Intermediate Region as determined with heat sensor

■ Interface between Region(3) and Intermediate Region as determined with the electric probe

— Period of uniform flow in Region(2) of an ideal tube

--- Period of uniform flow after Mirels correction for turbulent flow<sup>[4]</sup>

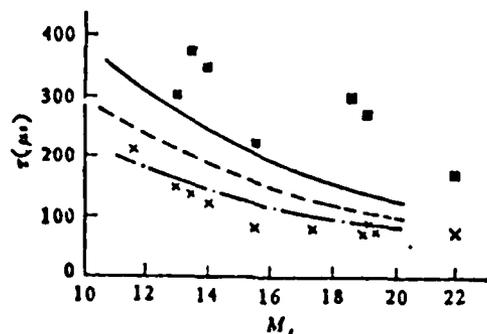


Fig.3 Determination of the location of the Intermediate Region

$$p_1 = 10^{-1} \text{ mm Hg}, x_s = 14 \text{ m}$$

- x Interface between Region(2) and the Intermediate Region as determined with heat sensor
- Interface between Region(3) and Intermediate Region as determined with electric probe
- Period of uniform flow in Region(2) of an ideal tube
- Period of uniform flow after Mirels correction for laminar flow [3]
- .- Period of uniform flow after Mirels correction for turbular flow [4]

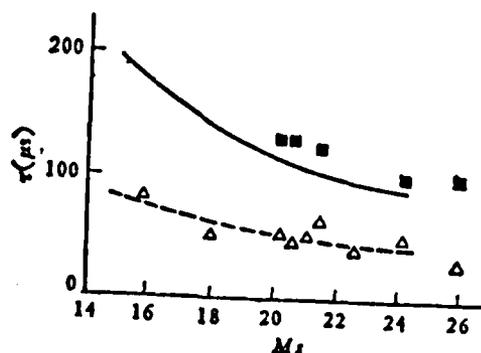


Fig.4 Determination of the location of the Intermediate Region

$$p_1 = 10^{-2} \text{ mm Hg}, x_s = 14 \text{ m}$$

- Δ Interface between Region(2) and the Intermediate Region as determined with heat sensor
- Interface between Region(3) and the Intermediate Region as determined with the electric probe
- Period of uniform flow in Region(2) of an ideal tube
- Period of uniform flow after Mirels correction for laminar flow [3]

1) Length of Region(2)  $p_1 = 1, 10^{-1}, 10^{-2}$  mmHg Figs. 2-4 give the length of time of uniform flow in Region(2). The trend of variation of  $\tau_2$  with  $M_s$  agrees with theoretical analysis. The value of  $\tau_2$  actually measured is usually only 1/2 to 2/3 that of the ideal excitation tube. This shows that loss at the boundary layer and mixing at the first interface result in a substantial reduction of the effective period of uniform flow in Region(2) of the tube.

Quite a few authors have studied and discussed the problem of length reduction of Region(2) due to the presence of the boundary layer in the excitation tube. Mirels' theory<sup>[3,4]</sup> is generally regarded as being more reliable. Comparing our experimental results with the theoretical values obtained after Mirels correction, we found that at  $p_1 = 1$  mm Hg, the experimental points are a little lower than the curve corrected for boundary layer turbulent flow; at  $p_1 = 10^{-1}$  mm Hg, for high and low  $M_s$ , the experimental points fall between the curve corrected for laminar flow and that corrected for turbulent flow, while for intermediate  $M_s$  values, the experimental points lie a little lower than the corrected curves; at  $p_1 = 10^{-2}$  mm Hg, the experimental points agree fairly well with the curve corrected for laminar flow for the entire range of values of  $M_s$ . Our experimental results agree basically with those obtained by Lin Shau Chi et al<sup>[5]</sup> for their 24-inch excitation tube at  $p_1 = 10^{-1}$  and  $10^{-2}$  mm Hg. An observation worth mentioning is the tendency for the experimental results to agree better with the theoretical curve corrected for boundary layer laminar flow at lower  $p_1$ , and to agree better with the curve corrected for boundary layer turbulent flow at higher  $p_1$ ; a state of transition is exhibited for intermediate  $p_1$ .

In order to further study the variation of  $\tau_2$  with  $p_1$ , we measure the value of  $\tau_2$  corresponding to different  $p_1$ , holding  $M_s$  constant. The results for  $M_s = 10$  and 15 are shown in Figs. 5 and 6. We see that

for  $p_1 = 10^{-2}$  mm Hg or lower,  $\tau_2$  agrees with the value predicted by laminar flow theory, i.e. under this condition, the state of flow in the boundary layer of the tube is mainly laminar. For  $p_1 = 1$  mm Hg or higher,  $\tau_2$  agrees with the value predicted by turbulent flow theory, and is much lower than that predicted by the laminar flow theory. Under this condition, the state of flow in the boundary layer is mainly turbulent. In both of these regions,  $\tau_2$  increases with  $p_1$ .

In the Intermediate Region, the contributions from neither the laminar flow nor the turbulent flow should be neglected. As  $p_1$  increases, the increase in  $\tau_2$  becomes smaller and there is even a drop in  $\tau_2$ .

This transition occurs

approximately between  $10^{-1}$  and 1 mm Hg for our excitation tube.

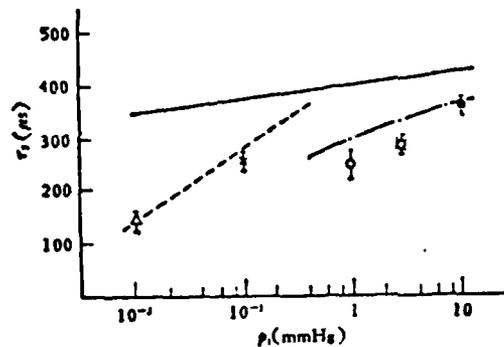


Fig.5 Variation of period of uniform flow in Region(2) with  $p_1$   
 $M_s = 10$ ,  $x_s = 14m$

— Period of uniform flow in an ideal tube

— Period of uniform flow after Mirels correction for laminar flow

-.- Period of uniform flow after Mirels correction for turbulent flow

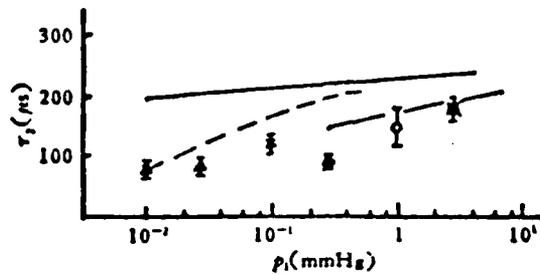


Fig.6 Variation of period of uniform flow in Region(2) with  $p_1$   
 $M_s = 15$ ,  $x_s = 14m$

- Period of uniform flow in an ideal tube
- - - Period of uniform flow after Mirels correction for laminar flow
- · - · Period of uniform flow after Mirels correction for turbulent flow

To examine the validity of the above viewpoint, we used gold-film heat sensors installed on the side wall of the excitation tube to determine the position  $l_t$  of the turning point in the boundary layer, and compared it with length  $l_2 = \tau_2 \cdot u_2$  of Region(2) measured inside the tube. At  $p_1 = 1$  mm Hg,  $l_t \approx l_2/3$ . (Under this condition, the Reynold number  $Re_t = (W - 1)^2 u_{c,0} \cdot \rho_{c,0} \cdot l_t / \mu_{c,0} \approx 2 \times 10^4$ .)

This shows that the turning point lies far in front of the first interface. The flow in that part of the boundary layer corresponding to Region(2) is mainly in a turbulent flow state. Hence, basically, we should use turbulent theory to estimate the effect of loss on length reduction of Region(2). At  $p_1 = 3 \times 10^{-2}$  mm Hg,  $l_t > l_2$ , the turning point is located behind the first interface, i.e. the flow in that part of the boundary layer corresponding to Region(2) is entirely in a laminar flow state. Therefore, one should use the laminar flow theory to estimate the effect of loss in the boundary layer on the reduction in length of Region(2).

## 2) Shape of the Wavefront

Fig 7 gives the results of measurement of wavefront under the conditions  $p_1 = 1 \times 10^{-2} - 3 \times 10^{-2}$  mm Hg and  $M_s = 20$ .  $r$  denotes the

distance between the point of measurement and the axis of the tube;  $x$  denotes the difference in distance on the wavefront between the point of measurement and the center of the wavefront. Taking the center of the wavefront as the origin, we assign a positive value for lagging points on the wavefront and a negative value for leading points on the wavefront. The waveform distortion as measured by our experiment is on the average about 5 times as large as that calculated from existing boundary layer theories [6,7], and 2-3 times as large as the results obtained by Lin Shau Chi et al. [5]. Near the wall, the average value of  $x/R$  is around 7%. This shows that the degree of wavefront distortion is quite large. However, in the region within a radius of 10 cm from the center, the wavefront is basically flat. When  $p_1$  increases, unevenness of the wavefront decreases. In our set-up, the serious wavefront distortion may have been due to the larger inner diameter and the not sufficiently stable hydrogen-oxygen combustion drive.

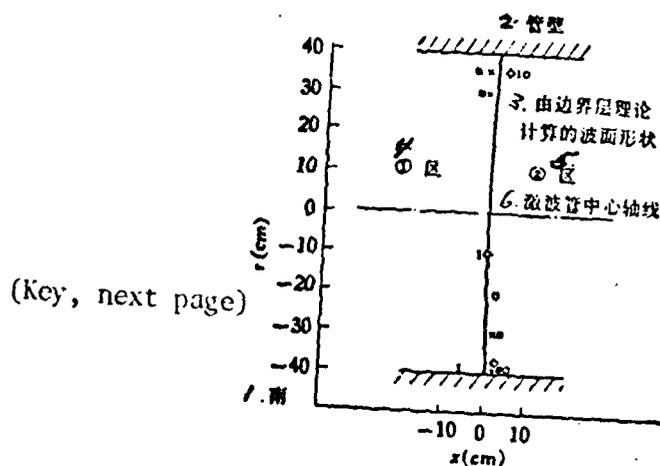


Fig.7 Experimental results of measurement of shape of wavefront

$$M_s = 20, p_1 = (1-3) \times 10^{-2} \text{ mm Hg}$$

Center of wavefront is taken as the origin.

■, ×, ✕, J, ○

represent 5 typical results of measurement

1. South
2. Wall
3. Shape of wavefront as calculated from boundary layer theory
4. Region(1)
5. Region(2)
6. Axis of tube

3) The Intermediate Region between Region(2) and Region(3) Figs. 2-4 give the position  $l_2$  or  $\tau_2$  of the interface between Region(2) and the Intermediate Region as well as the position  $l_3$  or  $\tau_3$  of the interface between the Intermediate Region and Region(3). Roughly speaking,  $\tau_3$  is about twice as large as  $\tau_2$ , i.e.  $l_3$  is twice as large as  $l_2$ . In other words, the lengths of the Intermediate Region and Region(2) are on the same order of magnitude. We compared the value of  $\tau_3$  we obtained with an electric probe inside the tube with that obtained previously by Comrades Tsui Chi P'ing et al who used an electric probe to make measurements in the boundary layer at the tube wall. Under the conditions  $p_1 = 10^{-2}$  mmHg,  $M_g = 20$ ,  $\tau_3 \cong 130 \mu s$  inside the tube while  $\tau_3$  is approximately 400-500  $\mu s$  along the tube wall. This shows that the Intermediate Region is longer at the wall than inside the tube, as is evident from Fig. 1.

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