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THE LIFT AND DRAG ON A ROTATING CYLINDER
IN SUPersonic CROSSFLOW

14 January 1960

U. S. NAVAL ORDNANCE LABORATORY
WHITE OAK, MARYLAND

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THE LIFT AND DRAG ON A ROTATING CYLINDER IN SUPersonic CROSSFLOW

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ABSTRACT: Experimental results are presented of the lift (Magnus force) and drag on a rotating cylinder in supersonic crossflow. The data were obtained using smooth and roughened, two-dimensional 3-inch diameter cylinders. The rotating cylinder was tested at Mach numbers 1.7, 2.15, 2.48, and 3.24 over a Reynolds number range from 0.55 x 10^6 to 1.06 x 10^6. Rotational speeds of the model were continuously varied from 0 - 400 cps. The Naval Ordnance Laboratory Supersonic Tunnel No. 1 was used for this investigation.

A discussion of the data is presented in which variations within the three parameters (Mach number, rotational speed, and Reynolds number) are shown along with magnitudes of the lift and drag. A further review of the data presents possible correlations with similar subsonic data.

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R. KENNETH LOBB
By direction
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SYMBOLS

\( a \)  
free-stream speed of sound, \( \text{ft/sec} = \sqrt{\gamma RT} \)

\( a_t \)  
speed of sound at stagnation conditions, \( \text{ft/sec} = \sqrt{\gamma RT_t} \)

\( A_D \)  
drag coefficient reference area, \( \text{ft}^2 \) (3.94)

\( A_L \)  
.lift coefficient reference area, \( \text{ft}^2 \) (3.00)

\( C_D \)  
drag coefficient \( = \frac{F_D}{qA_D} \)

\( C_L \)  
.lift coefficient \( = \frac{F_L}{qA_L} \)

\( D \)  
model diameter, \( \text{ft} \) (0.25)

\( F_D \)  
drag force, \( \text{lbs} \)

\( F_L \)  
lift force (Magnus force), \( \text{lbs} \)

\( M \)  
free-stream Mach number

\( n \)  
rotational speed, \( \text{cps} \)

\( P_S \)  
free-stream static pressure, \( \text{lbs/ft}^2 \)

\( P_t \)  
free-stream stagnation pressure, \( \text{lbs/ft}^2 \)

\( q \)  
free-stream dynamic pressure, \( \text{lbs/ft}^2 = \gamma P_S M^2/2 \)

\( R \)  
gas constant \( = 1716 \text{ ft}^2/\text{sec}^2 \text{ OR} \)

\( Re \)  
free-stream Reynolds number, based on model diameter, \( D \)

\( T_S \)  
free-stream static temperature, \( \text{OR} \)

\( T_t \)  
free-stream stagnation temperature, \( \text{OR} \)

\( u \)  
tangential velocity of point on surface of the cylinder, \( \text{ft/sec} = \pi D n \)

\( V \)  
free-stream air velocity, \( \text{ft/sec} \)

\( \gamma \)  
ratio of specific heats \( = 1.4 \text{ for air} \)

\( \rho \)  
free-stream air density, \( \text{lbs sec}^2/\text{ft}^4 \)
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THE LIFT AND DRAG ON A ROTATING CYLINDER IN SUPERSONIC CROSSFLOW

Introduction

1. The flow about cylinders normal to an airstream has been the subject of considerable analytical and experimental analyses for many years. Many of the concise, mathematical, potential flow analyses have been formulated using a cylinder in a perfect fluid crossflow.

2. From the mathematical concept, when such a cylinder is placed in a two-dimensional, potential crossflow, no net lift force results. This is also true for a viscous, incompressible, homogeneous fluid. Although the boundary layer is considered for this type of fluid, the vortices which are shed (which induce circulation and thus lift) are shed symmetrically and are of opposite sign. Therefore, the net lift is still zero.

3. However, when the cylinder is placed in an airstream and is spun about its axis, a net lift force is produced which is normal to both the airstream and the axis of rotation. This aerodynamic phenomenon is known as the "Magnus effect" after the German scientist who carried out the first experiments which qualitatively proved the existence of such a force.

4. This program was run in order to determine the lift (Magnus force) and drag on a rotating cylinder in supersonic crossflow. The data were obtained at Mach numbers of 1.75, 2.15, 2.48, and 3.24.

Historical Sketch

5. The effect now known as the "Magnus effect" was first observed sometime in the eighteenth century. During that time it was noted that cannon balls exhibited a dispersion which could not be explained (references (1) and (2)). In 1852, Professor Magnus (hence the term Magnus) was assigned the task of finding if a force due to rotation did exist (reference (3)). The proof of such a force was established, but no measurements of magnitude were made. It was not until the 1870's that a plausible theory was advanced. By using the superposition of two different potential flow fields, Lord Rayleigh presented a mathematical picture concerning the irregular flight of a tennis ball (reference (4)). This classic formulation appears in today's theoretical textbooks (references (5-9)). From then until now the "Magnus effect" has received only sporadic attention from both experimental and theoretical fields (references (10-23)).
6. In the 1940's, ballisticians began placing more attention to the aerodynamics involved in the dynamic stability of spinning shells (reference 24). As a result of this, the "Magnus effect" has received much more organized and consistent consideration in the past 15 years (references (25-32)).

7. However, even after many years of work, no fully satisfactory method has been found for predicting Magnus forces.

Objectives of the Test

8. The prime purpose of the program was to extend, experimentally, the two-dimensional subsonic Magnus force on a rotating cylinder in crossflow into the supersonic flow regions.

9. Qualitative and quantitative results were desired for aerodynamic design and to provide experimental data for Magnus force theories which are being developed at the present time.

10. Finally, it was desired to determine if any correlation or parallelism exists between the subsonic and supersonic Magnus force phenomenon.

Some Design Considerations

11. Although, to the author's knowledge, no analytical or experimental data on spinning cylinders in compressible crossflow have been published, previous work from ballistic ranges, wind-tunnels, and theory have dictated two important criteria for any undertaking of experimental Magnus work. These criteria are as follows:

a. A force-sensitive system strong enough to withstand ordinary lift and drag forces, yet sensitive enough to detect accurately the small Magnus forces.

b. A system for spinning the model, small enough to fit inside the model, but powerful enough to rotate the model at very high spin rates.

12. Fortunately, the rapid advances and refinements in Magnus instrumentation at the Naval Ordnance Laboratory (references (33) and (34)) within the past five years have solved the above requirements by the development of:

a. The internally mounted strain-gage balance.

b. Small Air turbines.
13. In connection with this program, a critical problem arose during the early design work for the models. In order to optimize the model system it was necessary to know what loads were to be expected. Due to the lack of experimental data and analytical approaches for predicting the aerodynamic loads, certain assumptions were made:

   a. That the lift (Magnus force) would be at least one order of magnitude lower (and very conceivably two orders of magnitude lower) than the drag on the cylinder in crossflow.

   b. That the drag would not change appreciably with spin (since, for a blunt body in a supersonic stream the greater portion of its drag is wave drag), and thus the limited drag data available for non-spinning cylinders (reference (35)) in supersonic crossflow could be used in estimating the drag.

14. From the drag estimation, the magnitude of the lift could then be estimated using assumption (a) above.

Model and Balances

15. The model, (Figures 1 and 2) used in this test may be divided into three main sections:

   a. The rotating section

   b. The stationary section

   c. The air-coaster turbine

16. The rotating section is a hollow thin-walled aluminum tube, twelve inches long and three inches in diameter. The model had two interchangeable rotating sections. One rotating section was smooth, (number 32 machine finish) while the other rotating section was roughened by knurling ten inches of the cylinder length. The knurls were approximately 0.007 inches deep. The cylinder was built to a roundness tolerance of 0.001 inch. A steel shaft runs through the center of the cylinder and extends into the stationary sections.

17. The stationary sections, one at each end of the cylinder, are also constructed of aluminum and are three inches in diameter. These sections house the bearings, the air-coaster turbines, and the tachometers.
18. The air-coaster turbine derives its name from the manner in which it is used. The turbine is designed to "power up" the model to some high rotational speed. Then the air to the turbine is shut off, the tunnel started, and the model is allowed to coast to a stop during the blow. Data are taken during the "coasting" period. The turbine has two main parts, the nozzle and the turbine wheel. The high pressure (150 psi) air is piped into the annular ring formed by the bearing support and the nozzle, both of which remain stationary during operation. The air goes through the nozzle and then through the turbine wheel (which is fixed to the shaft and thus the rotating cylinder) and is exhausted through the cylinder to the airstream. Figure 3 shows the model mounted in the wind tunnel.

19. The model, when assembled, spans the wind tunnel in the horizontal plane. Its physical location is approximately the center of the test region formed by the open-jet test section. A model which spans the wind-tunnel nozzle is used in order to approach, as close as possible, two-dimensional flow.

20. The balances are typical strain-gage balances with slight modifications in design. There are two pitch sections four inches apart, which straddle the diameter of the cylinder. Further aft, in a plane normal to the axis of the pitch balances, are the drag sections. The balances were designed using the load estimations previously described. The estimates made, for a Mach number of 2, were $F_D = 310$ lbs., $F_L$ (Magnus force) = 5 lbs. The drag coefficient is based on frontal area.

Test Instrumentation

21. Two main readout systems are necessary for recording the data. These systems are as follows:

   a. Strain-gage readout system.

   b. Rotational speed readout system.

22. The strain-gage readout system was composed of dry cell batteries to supply d.c. power to the gages; a nulling unit to balance the bridges and for calibration purposes; a Leeds and Northrup amplifier system for signal amplification and visual observation; an analog computer for automatically computing the desired coefficients; and a Leeds and Northrup $X_1, X_2, Y$ recorder, where the data, in coefficient form were recorded.
23. The rotational speed readout system was composed of a
tachometer, mounted inside of the model to detect rotation;
an audio oscillator for speed calibration of the recorder
chart; an amplifier for signal amplification; and a frequency
meter to convert the variable frequency from the tachometer
and oscillator to a d.c. voltage which varies proportionally
with the frequencies. The output of the frequency meter was
used for the input to the Y component of the Leeds and
Northrup $X_1$, $X_2$, Y recorder. A Berkeley EPUT meter and a
three-inch oscilloscope was used for visual monitoring of
the rotational speed.

24. The two systems were "brought together" at the Leeds and
Northrup $X_1$, $X_2$, Y recorder. Here the strain-gage system out-
put was used for $X_1$ and $X_2$ inputs, and the speed recording
system output was used as the Y input. Thus the final data
were displayed as a continuous trace of force (abscissa)
versus rotational speed (ordinate).

Test Technique, Data Reduction, and Results

25. A limited wind-tunnel program was planned to investigate
the Magnus forces on rotating cylinders at supersonic speeds.
The following possible test variables were considered important.
Reynolds number, Mach number, rotational speed, and surface
roughness. The major parameter was Mach number. The
rotational speed was varied from 0 to 400 cps.

26. The data, as presented in this paper, were obtained
during several test periods. It was determined that the
repeatability of lift forces was within 5 percent.

27. The data obtained were in the form of continuous traces
of rotational speed (cps) versus forces (lbs). These traces
exhibited some oscillations. Traces having identical
variables are grouped together, and data are then averaged
for the various runs. The average values are shown plotted
as a solid curve through the test points.

28. The Mach numbers and associated Reynolds number are
based on free-stream conditions. The Reynolds number is
based on the diameter of the cylinder.
29. The lift coefficient is based on the frontal area of the rotating portion of the cylinder since the stationary portion of the cylinder in the airstream does not contribute to the lift.

30. The velocity ratio, $u/V$, for each Mach number was obtained from the supply temperature, test section calibrations, and the compressible flow tables (reference 36). The tangential velocity, $u$, of the cylinder was determined by the expression $u = \pi Dn$ for various $n$'s.

31. The drag coefficient is calculated using a frontal area equal to the model diameter times the nozzle width. Corrections for the areas of the model enveloped by the nozzle boundary layer and the free jet mixing layer were not made because the good repeatability and the good agreement with data of reference (35) indicate that the effect is negligible.

32. Several checks were made to determine if the velocity was two-dimensional. These checks were made several ways:

   a. A pressure tube was mounted about 3/4-inch behind the model and readings were taken for both the no-spin and spinning conditions.

   b. Temporary end plates were mounted on the model and the lift readings were compared with the data for no-end plates present.

   c. The drag coefficients obtained during the tests were compared with two-dimensional drag data on cylinders in cross-flow as found in the literature.

33. Figure 4 is a drawing showing the sign convention used throughout the test.

34. Figures 5 through 8 are plots of the rotational speed (cps) vs. the lift coefficient $C_L$ at Mach numbers of 2.15, 2.48, and 3.24 for the smooth cylinder.

35. Figure 9 is a plot of $C_L$ vs. Mach number for the smooth cylinder. In this plot the rotational speed, cps, is used as a parameter.

36. Figure 10 is a plot of the lift coefficient $C_L$ vs. velocity ratio $u/V$ for the smooth cylinder. This figure is a summary plot of the average lift coefficients vs. the velocity ratio with Mach number as a parameter.
37. Figure 11 is a summary of the drag data obtained with the smooth cylinder. This figure is a plot of rotational speed, cps, vs. drag coefficient, $C_D$. The no-spin drag coefficients are shown as points on the $n = 0$ line. The curves drawn from these points represent the drag coefficient as a function of spin rate for each of the four Mach numbers tested. Here it may be seen that the earlier assumption that the drag would be insensitive to spin is essentially substantiated.

38. The curve of the drag coefficient $C_D$ (no spin) vs. Mach number is presented in Figure 12. Also shown on this plot, for calibration purposes, are the data from reference (35). Included in this figure are the data obtained with a roughened cylinder. The roughened cylinder was tested at Mach numbers 1.75, 2.15, and 2.48. Only data obtained at Mach number 1.75 were usable due to model and instrumentation difficulties.

39. Figure 13 is a plot of the lift coefficient $C_L$ vs. the velocity ratio, $u/V$, and Mach number 1.75 for the roughened and the smooth cylinder. The data from the roughened cylinder show an appreciable difference in comparison with data from the smooth cylinder. The roughened cylinder shows an average of 20 percent more lift than does the smooth cylinder.

Conclusions

40. From the preceding discussions the following conclusions may be drawn:

a. The magnitudes of forces and slopes are small when compared to subsonic data. The maximum lift coefficient obtained in the test program was $C_L = 0.009$, which occurred at a Mach number of 2.15. Up to about 200 cps the lift coefficient is linear with increasing rotational speed, and the initial slope values $dC_L/d(u/V)$, were approximately 0.086 with the exception of the data obtained at $M = 1.75$.

b. All of the curves show (or tend to show) that a maximum lift coefficient is reached at spin ratios of $(1.0 \leq u/V \leq 2.0)$. Beyond the point of maximum lift, a loss of lift is experienced for increasing spin up to 400 cps.

c. In spite of smaller magnitudes of lift and slopes, the data show that a qualitative parallelism may exist in a comparison of the Magnus phenomenon in incompressible (reference 31 and 32) and compressible flow.
d. The drag is shown to be very nearly independent of the spin rate; the maximum variation is approximately 3 percent up to 400 cps.

e. Although an appreciable difference in aerodynamic coefficients is shown between the roughened and the smooth cylinder, it is felt that the result at only one Mach number does not justify applying the results over the entire supersonic Mach number range.
REFERENCES


1. CYLINDER (rotating section)
2. CYLINDER (stationary section)
3. TURBINE RETAINING NUT & RING MAGNET
4. TACHOMETER COIL
5. TURBINE WHEEL
6. NOZZLE
7. BEARING
8. END PLATE & BEARING HOLDER
9. END PLATE (rotating section)
10. STEEL SHAFT
11. BALANCE & MODEL HOLDER

FIG. 1
PHOTOGRAPH OF ROTATING CYLINDER
MOUNTED IN WIND TUNNEL

FIG. 3
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SIGN CONVENTION

FIG. 4
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LIFT COEFFICIENT versus ROTATIONAL SPEED

Re = $1.06 \times 10^6$

$M = 1.75$

FIG. 5
LIFT COEFFICIENT versus ROTATIONAL SPEED

Re = 0.91 x 10

M = 2.15

FIG. 6
LIFT COEFFICIENT versus ROTATIONAL SPEED

\[ Re = 0.78 \times 10^6 \]
\[ M = 2.48 \]

FIG. 7
LIFT COEFFICIENT versus ROTATIONAL SPEED

Re = 0.55 x 10^6
M = 3.24

FIG. 8
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LIFT COEFFICIENT versus MACH NUMBER

![Graph showing lift coefficient versus Mach number for different frequencies. The graph includes data points at Mach numbers from 1.5 to 3.0, and lift coefficients at various frequencies up to 400 cps.](fig_9)
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LIFT COEFFICIENT versus VELOCITY RATIO

FIG. 10
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DRAG COEFFICIENT versus ROTATIONAL SPEED

FIG. II
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DRAG COEFFICIENT (NO-SPIN) versus MACH NUMBER
LIFT COEFFICIENT versus VELOCITY RATIO

FIG. 13